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## THE PROCESS REACTION CURVE

- All **analytical** tuning relations assume that an open-loop model of the process ( $K_p; \tau_p; \theta_p$ ) is available, from which the tuning ( $K_C; \tau_I; \tau_D$ ) is obtained
- The process **reaction curve** is a way to obtain this model (**system identification**)
- In principles, a **single step** test is sufficient (with the controller in manual mode)
  - ▷ The system TF is identified  $\rightarrow (K_p; \tau_p; \theta_p; \dots)$
  - ▷ From the model parameters, the controller tuning is obtained using one set of analytical relations
  - ▷ Step amplitude: depends on signal-to-noise ratio
- **Drawbacks**
  - ▷ The test is carried out at **open-loop**
    - ❖ If a disturbance enters the process while the test is being done, the process dynamics is altered  $\rightarrow$  the TF identified does not represent the true process dynamics
  - ▷ For a **nonlinear process**, the identification result depends both on the amplitude and on the direction of the step
    - ❖ It is convenient to step the manipulated input both above and below its nominal value
  - ▷ The reaction curve cannot be obtained for **open-loop unstable** processes
  - ▷ The open-loop response may be very slow

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## ANALYTICAL TUNING RELATIONS

- From a general viewpoint, **analytical PID tuning relations** can be obtained by following different rationales
  - ▷ Considering an IMC approach (or a DS one, typically not employed in industry)
    - ❖ The analytical relations are «automatically» derived from IMC controller synthesis
  - ▷ Requiring that, under PID control, some performance indices (CL response) are optimized → minimization of IAE, ISE or ITAE
    - ❖ The analytical relations are those allowing these indices to be minimized
  - ▷ Requiring that, under PID control, a given performance criterion is obtained
    - ❖ A given decay ratio → Ziegler-Nichols; Cohen-Coon
    - ❖ A good tradeoff between performance and robustness → Hägglund-Åström; Skogestad
- Analytical relations abound in the technical literature
  - ▷ Either directly or indirectly, they all rely on the fact that an open-loop model of the process to be controlled is available
  - ▷ Finding a tuning relation that is «universally optimal» is impossible
- **Caution!**
  - ▷ Always check for which implementation of the PID controller the analytical relation is valid
    - ❖ Parallel or serial? Interacting or non-interacting?
    - ❖ See SEMD, Table 12.2, p.208
  - ▷ **The parallel form is always consider here**

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## ANALYTICAL RELATIONS FROM PERFORMANCE INDICES

- The «quality» of a controlled system response can be quantified using performance indices
  - ▷ IAE, integral of the absolute value of the error  $IAE = \int_0^{\infty} |e(t)| dt$
  - ▷ ISE, integral of the squared error:  $ISE = \int_0^{\infty} [e(t)]^2 dt$ 
    - ❖ Penalizes large errors; very aggressive
  - ▷ ITAE, integral of the time-weighted absolute error  $ITAE = \int_0^{\infty} t|e(t)| dt$ 
    - ❖ Penalizes persisting errors; conservative and usually preferable
- There exist analytical tuning relations that minimize the performance indices
  - ▷ The tuning for regulatory control and for servo control are different
  - ▷ Usually, the tuning for servo control is more conservative

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## TUNING BASED ON CLOSED-LOOP TESTS

- The tuning parameters are determined from the tests performed **at closed-loop** on the real system (i.e., on field)
- The tests are aimed at extracting information on the process **critical behavior** ( $\omega_u; P_u; K_{cu}$ )
  - ▷ This information is related to the **onset of instability** (marginal stability)
  - ▷ From this information, reasonable values for the PID controller settings can be obtained (using analytical relations)
- Afterwards, the response may be improved by fine-tuning the controller on field

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## CLOSED-LOOP TESTS

- The values of  $\omega_u$ ,  $P_u$  and  $K_{cu}$  may be obtained:
  - ▷ by actually bringing the system on the verge of instability, i.e. at the marginal stability condition
    - ▷ **Continuous cycling method** (i.e., method of Ziegler-Nichols or «metodo della perturbazione continua»)
  - ▷ Or by testing the system away from the marginal stability
    - ▷ **Relay method**
- Once the values of  $\omega_u$ ,  $P_u$  and  $K_{cu}$  are known, the controller settings are obtained from **analytical relations** according to a desired objective
  - ▷ 1/4 (QAD; Quarter-Amplitude Decay) → Ziegler-Nichols (1942), Z-N
  - ▷ Performance + robustness → Tyreus-Luyben (1997), T-L
- The Ziegler-Nichols tuning relations are **not recommended**
  - ▷ A QAD ressesponds to 50% overshoot (see SEMD, Chapter 5, p.77)
  - ▷ But for most **process** control applications, a QAD response is too oscillatory
  - ▷ For loops that behave «normally», decay ratios of **1/6 – 1/8** are more reasonable
    - ▷ Overshoots between 35% and 40%

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## TUNING ACCORDING TO ZIEGLER-NICHOLS AND LUYBEN

### Ziegler-Nichols (not recommended)

- ▷ When the I action is switched on in addition to the P one, the Z-N gain must be decreased (~10%)
- ▷ If the D action is then included,  $K_c$  can be increased to a value larger than that used when the P-only action is used
  - ❖  $K_c$  increases by ~33% with respect to PI
  - ❖ Additionally, the integral time can be decreased

### Tyres-Luyben

- ▷ It exists only for PI and PID controllers
- ▷ It is more conservative than Z-N
- ▷ When the D action is switched on,  $K_c$  increases by ~45% with respect to PI
  - ❖ The integral time does not change

Ziegler-Nichols	$K_c$	$\tau_I$	$\tau_D$
P	$0.5K_{cu}$	-	-
PI	$0.45K_{cu}$	$P_u/1.2$	-
PID	$0.6K_{cu}$	$P_u/2$	$P_u/8$
Tyres-Luyben	$K_c$	$\tau_I$	$\tau_D$
PI	$0.31K_{cu}$	$2.2P_u$	-
PID	$0.45K_{cu}$	$2.2P_u$	$P_{cu}/6.3$

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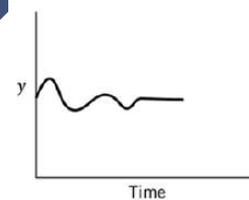
## FINDING THE CRITICAL PARAMETERS – CONTINUOUS CYCLING METHOD

### Proposed by Ziegler and Nichols, it is based on a trial-and-error procedure

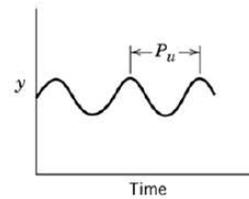
1. Manually bring the process at **steady-state**
2. Activate **only the P action** in the controller, and assign  $K_c$  a «small» value; set the controller to the automatic mode
3. Introduce a «small» set-point change (step). **Gradually** increase  $K_c$  until a sustained oscillation (constant amplitude) occurs in the controlled variable → **marginal stability**
4. The value of  $K_c$  that produces **continuous cycling** is equal to  $K_{cu}$  (ultimate gain). The period of oscillation is equal to  $P_u$  (ultimate period)
5. Calculate the PID controller settings using the Z-N of the **T-L relations**
6. Evaluate the loop **performance** by introducing a small set-point change
7. Fine-tune if necessary

### Notice.

- ▷ If by accidents,  $K_c > K_{cu}$  is selected, valve saturation makes the cycling appear as continuous, even if it is not in reality (undamped response)



(a)  $K_c < K_{cu}$



(b)  $K_c = K_{cu}$

(Seborg et al., 2011)

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## DRAWBACKS OF THE CONTINUOUS CYCLING METHOD

- **Several trials** are often needed before continuous cycling is attained
  - ▷ If the process dynamics is slow, the process operation is perturbed and the product quality remains poor for a long time
- The process is **pushed to its stability limits**
  - ▷ If an external disturbance enters the process, or if the process undergoes internal change, unstable operation may result
- The tuning procedure cannot be applied to **integrating** or **open-loop unstable** systems
  - ▷ Their control loops are unstable both high and low values of  $K_c$ , whereas they are stable for intermediate values
- Purely FO or SO systems (no dead time) **cannot be made unstable** (hence marginally stable) for any choice of  $K_c$  (provided that the sign of  $K_c$  is correct)
  - ▷ However, this is more a theoretical issue than a practical one

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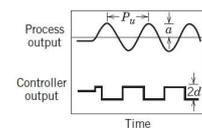
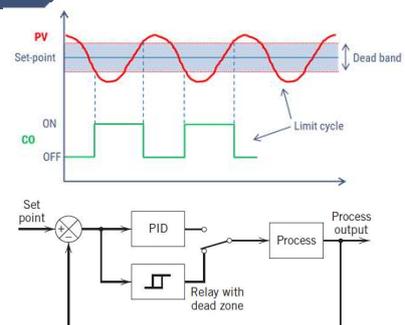
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## FINDING THE CRITICAL PARAMETERS – RELAY METHOD (ATV-Auto-Tune Variation)

- Proposed by Hägglund and Åström (1984)
- A **single test** is sufficient, from which information is extracted that enables one to determine the values of  $K_{cu}$  and  $P_u$ , **without bringing the system on the verge of instability**
- It uses an **on-off controller** (relay) with hysteresis (dead band)
  - ▷ After closing the control loop, a limit cycle appears, and the oscillation period is equal to  $P_u$
  - ▷ The ultimate gain is calculated as: 
$$K_{cu} = \frac{4d}{\pi a}$$
  - ▷ 2 to 4 complete cycles are required
    - ❖ For slow processes this may not be acceptable
- **Advantages with respect to continuous cycling**
  - ▷ The process is not pushed to the instability limit
  - ▷ A single test is enough, no trial-and-error procedure
  - ▷ The response amplitude  $a$  can be adjusted by adjusting the relay amplitude  $d$
  - ▷ The experimental test can be easily automated
    - ❖ Most commercial electronic controllers include a built-in «autotuning» or «autotaratura» feature



Seborg et al. (2016)

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## GENERAL TUNING RECOMMENDATIONS

- If an open-loop process model is available, it is convenient to **tune** the PID controller using that **model** (analytical relations)
  - The model may be obtained by step testing (at open-loop)
  - **The IMC relations** are usually a very good starting point
  - Other good relations: Hägglund-Åström (2002) and Skogestad (2003)
- Tuning based on **performance indices** (error integrals) provide a good quantitative term of comparison, but usually are not enough robust
  - When different tunings exist for regulatory control and servo control, the regulatory control tuning is usually preferable
  - However, the resulting tuning may be too aggressive → to reduce the oscillations for set-point changes, the set-point profile should be filtered
- QAD criteria (Ziegler-Nichols; Cohen-Coon) are **not** recommended
- For closed-loop tuning, use the **relay method** whenever possible

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## GENERAL REMARKS ON TUNING

- The controller **gain**  $K_c$  should be **inversely related** to the product of the other loop gains
  - Not strange at all: recall the stability condition
- $\tau_D$  is usually **smaller** than  $\tau_I$ 
  - In most well-tuned controllers,  $\tau_D/\tau_I = 0.1 - 0.3$
  - A good starting point is setting  $\tau_D/\tau_I = 0.25$
- The larger the **ratio**  $\theta_p/\tau_p$ , the smaller should  $K_c$  be
  - Usually, the control performance worsens when  $\theta_p/\tau_p$  increases
    - Both the settling time and the overshoot increase
- The larger the ratio  $\theta_p/\tau_p$ , the larger should  $\tau_I$  and  $\tau_D$  be
- When the **I action** is activated on a P-only controller,  $K_c$  **should be decreased**
  - A subsequent activation of the D action allows one to increase  $K_c$  to a value larger than that used with P-only control

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## TYPICAL PID CONTROLLER SETTING IN COMMON LOOPS

Control variable	PB [%]	$K_c$ [%/%]	$\tau_I$ [s]	$\tau_D$ [s]
Flow	100-500	0.2-1.0	0.2-2.0	0
Pressure (gas)	1-15	7-100	5-100	0
Pressure (liquid)	100-500	0.2-1.0	0.2-2.0	0
Level	5-50	2.0-20	5-60	0
Temperature	10-50	2.0-10	40-4000	30-2000*
Composition	100-1000	0.1-1.0	100-5000	30-4000*

(Riggs and Karim, 2008)

\*  $\tau_D$  should be always smaller than  $\tau_I$

Ch.12 independent study

■ SEMD, Example 12.1 p.203

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