

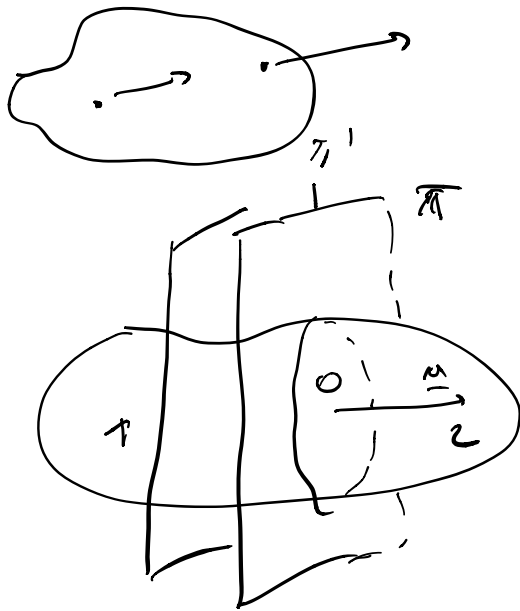
MECCANICA RAZIONALE

— LEZIONE DEL 25 MARZO 2020

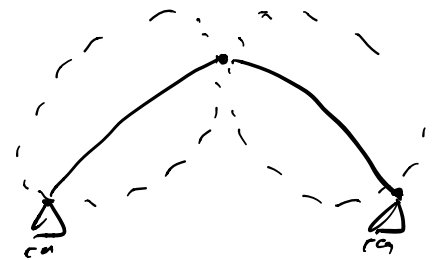
— PRIMA PARTE

Sforzi interni ad un corpo rigido

→ sollecitazioni interne al sistema
che lo mantengono rigido



ECS



$$\begin{cases} \underline{R}_1^{(e)} + \underline{R}_1^{(i)} = \underline{0} \\ \underline{M}_1^{(e)} + \underline{M}_1^{(i)} = \underline{0} \end{cases}$$

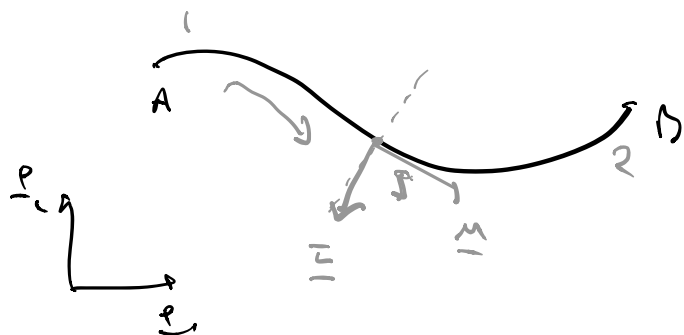
$$\underline{N} = \underline{R}^{(i)} \cdot \underline{u}$$

$$\underline{I} = \underline{R}^{(i)} - \underline{N} \underline{u}$$

$$\underline{M}_T(0) = \underline{M}^{(i)}(0) \cdot \underline{u}$$

$$\underline{M}_f(0) = \underline{M}_1(0) - \underline{M}_T \underline{u}$$

Caso piano



3 funzioni scalari

$$\underline{I}(s) = \underline{I}(s) \underline{z}$$

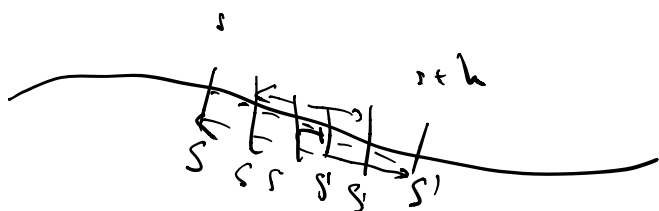
$$\underline{M}_f(s) = M_f \underline{e}_3$$

$$\begin{cases} \underline{R}^{(e)}_{-k} + \underline{N} \underline{u} + \underline{T} \underline{z} = \underline{0} \\ \underline{M}^{e,1}_{(s)} + \underline{M}_f(s) \underline{e}_3 = \underline{0} \end{cases}$$

Relazioni differenziali

$$\begin{cases} \frac{d}{ds} (\underline{N} \underline{u} + \underline{T}) = - \underline{f}(s) \\ \frac{d}{ds} \underline{M}_f(s) = \underline{I}(s) \wedge \underline{u}(s) \end{cases}$$

$$h \rightarrow 0$$



$$\underline{x} = \underline{x}(s)$$

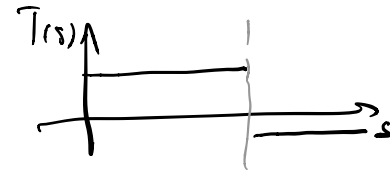
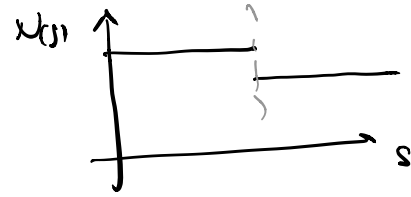
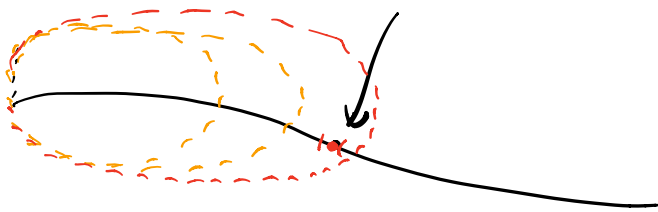
$$\underline{u} = \frac{d}{ds} \underline{x}(s)$$

$$\underline{z} = \underline{u} \wedge \underline{e}_3$$

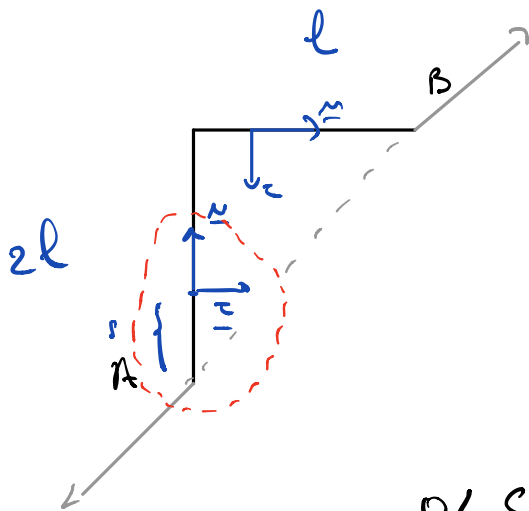
$$\underline{N}(s) \quad \text{lungo } \underline{u}$$

$$\underline{T}(s) \quad \text{lungo } \underline{z}$$

$$\underline{M}_f(s) \quad \text{lungo } \underline{e}_3$$



Archi scordati



$$0 < s < 2l$$

$\underline{n}$ verticale fino $s < 2l$ ($\underline{e}_2$)
 $\underline{n}$ orizzontale da $s > 2l$ ($\underline{e}_1$)

$$0 < s < 2l$$

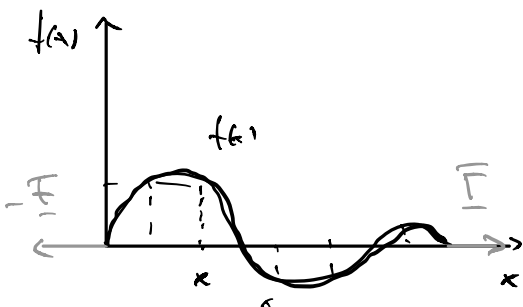
$$2l < s < 3l$$

$$-\underline{F} + N \underline{e}_2 + T \underline{e}_1 = \underline{0}$$

$$-\underline{F} + N \underline{e}_1 + T (-\underline{e}_2) = \underline{0}$$

$\uparrow$ $\uparrow$
 $\underline{n}$ $\underline{t}$

Esercizio



arco rigido determinato
da una funzione $f(x)$
calcoliamo gli sforzi
interni al variare di x

$$\underline{F} = F \underline{e}_1$$

L lunghezza dell'arco



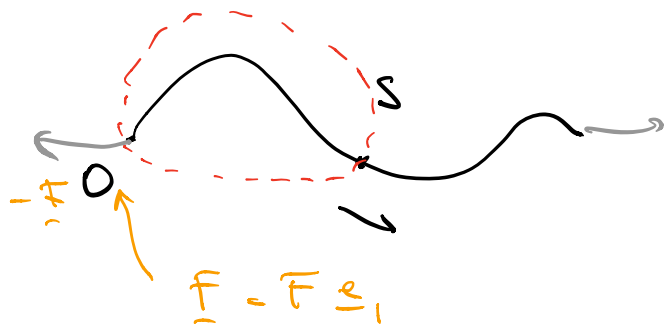
l'arco $\bar{e}$ dato

$$\{x, y : x \in [0, L], y = f(x)\}$$

Calcoliamo $\underline{u}(x)$: vettore tangente alla curva nel punto $(x, f(x))$

$$\underline{u} = \frac{\underline{e}_1 + f' \underline{e}_2}{\sqrt{1 + (f')^2}}$$

$$\underline{z} = \underline{u} \wedge \underline{e}_3 = \frac{\underline{e}_1 + f' \underline{e}_2}{\sqrt{1 + (f')^2}} \wedge \underline{e}_3 = \frac{f' \underline{e}_1 - \underline{e}_2}{\sqrt{1 + (f')^2}}$$



consideriamo OS
e scriviamo le eq
di bilancio per OS

$$1) -F + N \underline{u} + T \underline{z} = \underline{0}$$

$$2) M_f(x) + (x_0 - x_s) \wedge (-F) \cdot \underline{e}_3 = 0$$

1) $\bar{e}$ un'equazione vettoriale

Per risolvere : moltiplichiamo scalare

per $\underline{u}$ e per $\underline{z}$, usando $\underline{u} \cdot \underline{z} = 0$

$$(-\underline{F} + N\underline{u} + T\underline{z}) \cdot \underline{u} = 0$$

$$\rightarrow -\underline{F} \cdot \underline{u} + N = 0$$

$$\Rightarrow N = \underline{F} \cdot \underline{u} = \frac{F}{\sqrt{1 + (f')^2}}$$

$$(-\underline{F} + N\underline{u} + T\underline{z}) \cdot \underline{z} = 0$$

$$-\underline{F} \cdot \underline{z} + T = 0$$

$$\Rightarrow T = \underline{F} \cdot \underline{z} = \frac{F f'}{\sqrt{1 + (f')^2}}$$

$$\begin{aligned} 2) \quad \underline{M}_f(x) &= (\underline{x}_0 - \underline{x}_s) \wedge \underline{F} \cdot \underline{e}_3 \\ &= - (x \underline{e}_1 + f(x) \underline{e}_2) \wedge \underline{F} \underline{e}_1 \cdot \underline{e}_3 \\ &= \underline{f(x) F} \end{aligned}$$

$$\frac{d}{ds} \underline{M}_f(x(s)) = \underline{T}(x(s))$$

$$\frac{d}{ds} \underline{M}_f(x(s)) = \frac{d}{dx} \underline{M}_f(x) \frac{dx}{ds} = \frac{1}{\sqrt{1 + (f')^2}} \frac{d\underline{M}_f}{dx}$$

$$= f' F \cdot \frac{1}{\sqrt{1 + (f')^2}} = T(x(s))$$

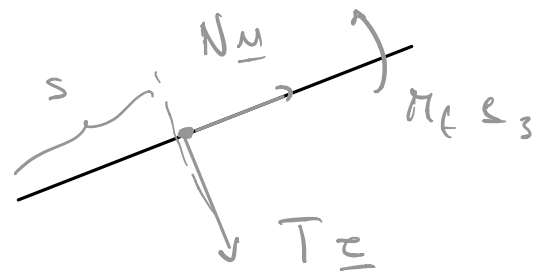
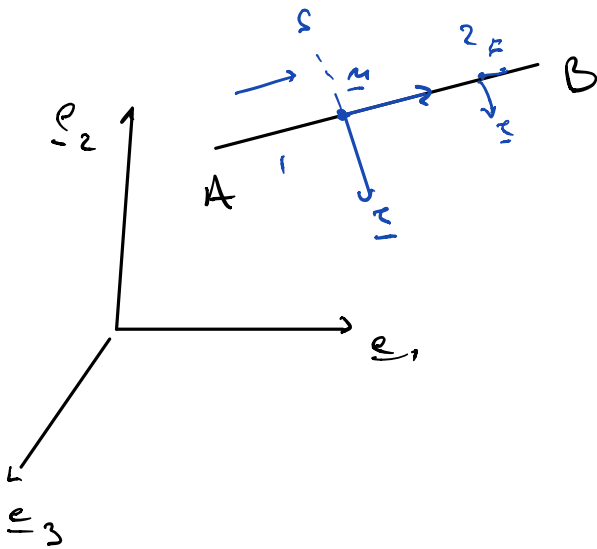
MECCANICA RAZIONALE

- LEZIONE DEL 25 MARZO 2020

- SECONDA PARTE

Aste

AB un'asta rettilinea, sottoposta a carichi continui e a carichi concentrati



Consideriamo la parte I

(sezione s non coincide con i punti di

applicazione dei carichi concentrati)

La parte 1 è in equilibrio sotto
l'azione delle seguenti forze

1) Tutti i carichi concentrati agenti
sulla parte 1

- risultante $\underline{R}_1^c$

- momento risultante
rispetto al polo S : $\underline{M}_1^c(S)$

2) Carichi distribuiti su 1

$\underline{f}(s)$ la forza specifica

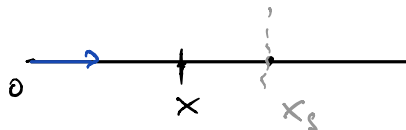
$$\underline{R}_1^d = \int_0^s \underline{f}(x) dx$$

$$\underline{M}_1^d(S) = \int_0^s (x-s) \underline{u} \wedge \underline{f}(x) dx$$

$\underline{f}(x)$ è applicato nel punto x

$$\underline{x}_r = s \underline{u}$$

$$\underline{x} = x \underline{u}$$



$$\underline{x} - \underline{x}_s = \underline{x}_M - \underline{s}_M = (\underline{x} - \underline{s}) \underline{M}$$

$$(\underline{x} - \underline{x}_s) \wedge \underline{f}(\underline{x}) = (\underline{x} - \underline{s}) \underline{M} \wedge \underline{f}(\underline{x})$$

3) Sforzi interni

applicabile in S $\left\{ \begin{array}{l} N(s) \underline{M} + T(s) \underline{\varepsilon} \\ M_f(s) \underline{\varepsilon}_3 \end{array} \right.$

In generale: bilanciamento $1) + 2) + 3)$

- ricavare N & T dall'equazione vettoriale delle risultanti

- M_f dall'eq. dei momenti: (dove lo sforzo s è scelto come polo)

Equivalentemente: usiamo le relazioni differenziali

$$\frac{d}{ds} (N \underline{M} + T \underline{\varepsilon}) = \frac{dN}{ds} \underline{M} + \frac{dT}{ds} \underline{\varepsilon} = - \underline{f}(s)$$

perché per $\underline{M}$ e $\underline{\varepsilon}$ sono costanti:

→ due eq. scalari

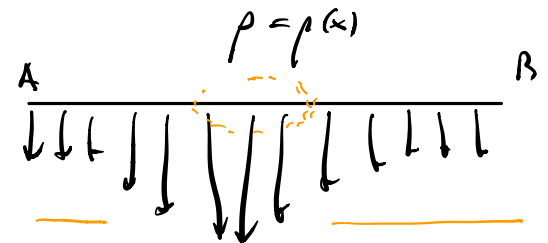
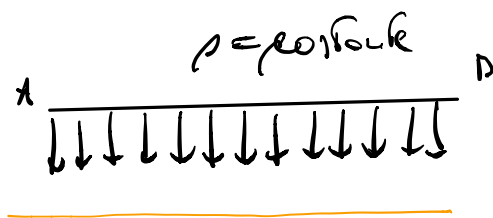
$$\left\{ \begin{array}{l} \frac{dN}{ds} = - \underline{f}(s) \cdot \underline{M} \\ \frac{dT}{ds} = - \underline{f}(s) \cdot \underline{\varepsilon} \end{array} \right.$$

$$\frac{d}{ds} M_f(s) = T(s) \quad \rightarrow \quad M_f(s) - M_f(0) = \int_0^s T(x) dx$$

$\uparrow$
 condizioni iniziali

Convergenza

-) Non bisogna fissare una sezione, ma calcolare gli sforzi al variare della sezione
-) non confondere forze specifiche e forze concentrate
 $\rightarrow$ forza peso non è carico concentrato nel centro di massa



-) carichi distribuiti non sono distribuiti in modo uniforme ($\neq$ costante)

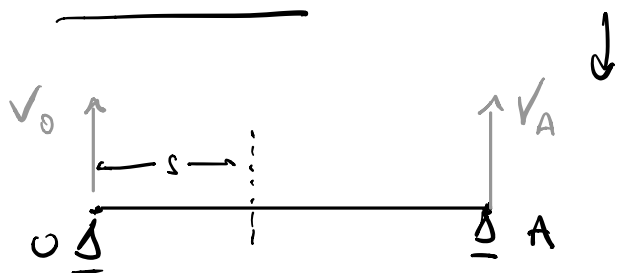
•) $\frac{d}{ds} M_{f(s)} = T(s)$ eq. differenziale

MECCANICA RAZIONALE

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- TERZA PARTE

Esercizio



$$\overline{OA} = L$$

$$\underline{f}(s) = \underline{\rho}(s) \underline{g}$$

Bilanciare gli sforzi interni in funzione di s , con le forze esterne

Primo: calcolare le forze di reazione

Dopo: calcolare gli sforzi interni

1. Forze di reazione

Forza peso: applicata nel centro di massa

$$G \quad s_G = \frac{1}{M} \int_0^L \underset{\uparrow}{s} \underset{\uparrow}{\rho(s)} ds \quad \left(s_G = \frac{\sum \underline{x}_p \cdot m_p}{M} \right)$$

$$M = \int_0^L \rho(s) ds$$

ECS $\rightarrow$ forze di reazione

$$\underline{R}^e \equiv 0 : V_0 + V_A - Mg = 0$$

$$\underline{V_A} = Mg - V_0 \leftarrow$$

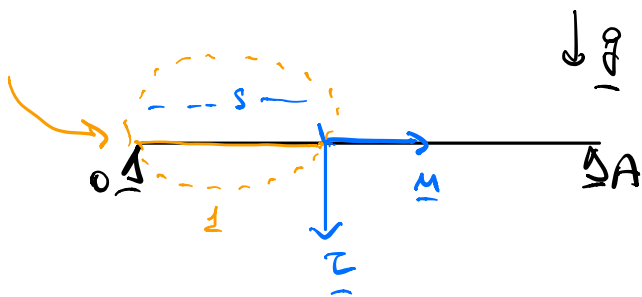
$$\underline{M}^e(A) = 0$$

$$\Rightarrow V_0 L = Mg (L - s_G)$$

$$\underline{V_0} = Mg \left(1 - \frac{s_G}{L} \right) \leftarrow$$

2. Stati interni

$$0 < s < L$$



$$\underline{N} = \text{vers } (x_A - x_0)$$

$$\underline{T} = \text{vers } \underline{g}$$

$$\text{Allora : } \underline{N} + \underline{T} - \underline{V_0} + m(s)g = 0$$

$$\underline{m(s)} = \int_0^s \rho(x) dx$$

masse del
tratto OS

$$N = 0$$

$$T = V_0 - m(s)g \leftarrow$$

$$\text{momento flettente : } \frac{dM}{ds} = T \quad \text{sapendo}$$

$$\text{due } M_f \rightarrow 0 \\ s \rightarrow 0$$

$$M_f(s) = \int_0^s T(x) dx$$

Esempio di $\rho = \rho(s)$

- densità costante

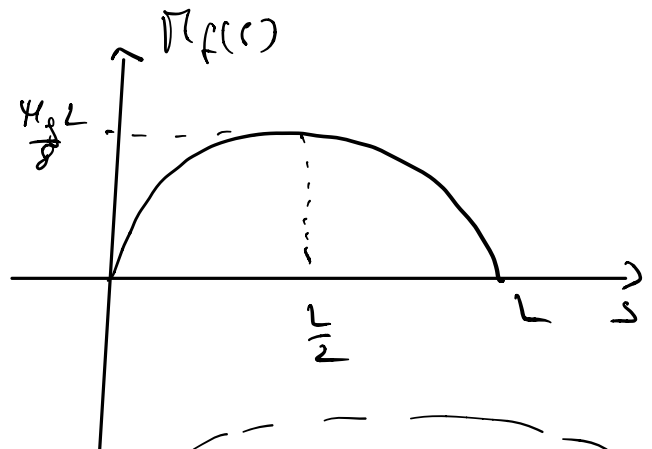
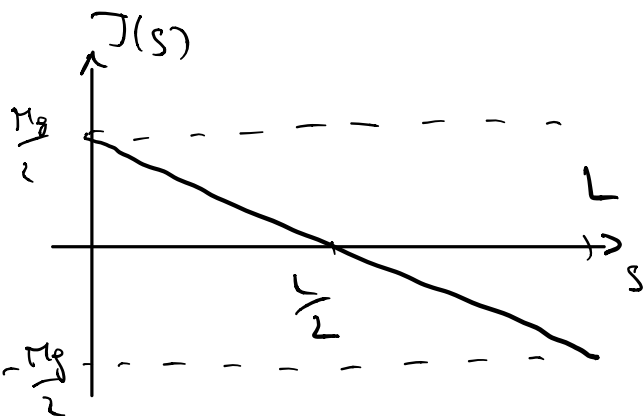
$$\rho = \frac{M}{L}$$

$$V_0 = \frac{Mg}{2} = V_A$$

$$T_{1/2} = Mg \left(\frac{1}{2} - \frac{s}{L} \right)$$

$$\int_0^s \frac{M}{L} = \frac{M}{L} s$$

$$M_f = \frac{Mg}{2} s \left(1 - \frac{s}{L} \right) \\ = \frac{Mg}{2} \left(s - \frac{s^2}{L} \right)$$



$$M_f = \frac{Mg}{2} s - \frac{Mg}{2} \frac{s^2}{L}$$

$$\frac{d}{ds} M_f = \frac{Mg}{2} - Mg \frac{s}{L} = 0$$

- dunque

max $\rightarrow \left(\frac{Mg}{2} - Mg \frac{s}{L} = 0 \right)$
 $s = \frac{L}{2}, \quad M_f = \frac{MgL}{8}$

$$M_f(s = \frac{L}{2}) = \frac{Mg}{2} \frac{L}{2} - \frac{Mg}{2} \frac{L^2}{L} \frac{1}{4} = \frac{MgL}{8}$$

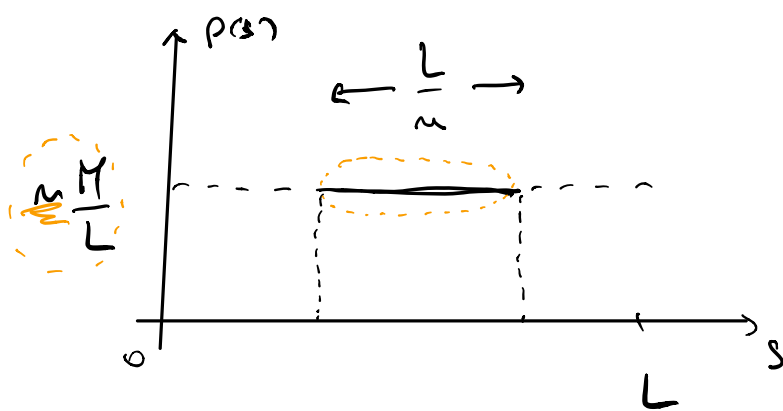
$$0 < s < \frac{L}{2} \left(1 - \frac{1}{n} \right)$$

$$\frac{L}{2} \left(1 - \frac{1}{n} \right) < s < \frac{L}{2} \left(1 + \frac{1}{n} \right)$$

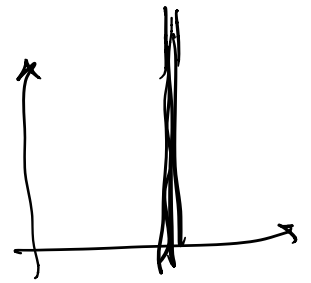
$$\frac{L}{2} \left(1 + \frac{1}{n} \right) < s < L$$

$$\rho(s) = \begin{cases} 0 \\ n \frac{M}{L} \\ 0 \end{cases}$$

$$n > 1$$



quindi
 $n \rightarrow \infty$



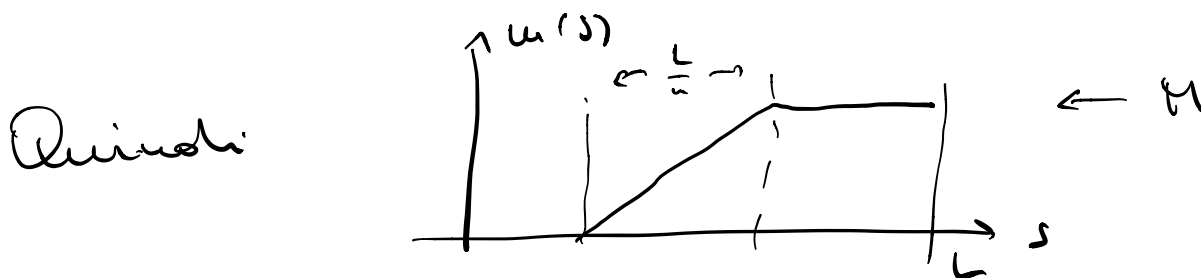
→ massa totale, centro di massa, velocità
 rimangono uguali al caso precedente

$$m(s) = \begin{cases} 0 & 0 \leq s < \frac{L}{2} \left(1 - \frac{1}{n}\right) \\ n \frac{M}{L} \left[s - \frac{L}{2} \left(1 - \frac{1}{n}\right) \right] & \frac{L}{2} \left(1 - \frac{1}{n}\right) \leq s < \frac{L}{2} \left(1 + \frac{1}{n}\right) \\ M & \frac{L}{2} \left(1 + \frac{1}{n}\right) \leq s < L \end{cases}$$

$\int_0^s \rho(x) dx$

$$\int_0^s \rho(x) dx$$

— $0 - \frac{L}{2} \left(1 - \frac{1}{n}\right), \frac{L}{2} \left(1 - \frac{1}{n}\right) - s$



Stare interni

→ $T = V_0 - \underline{m(s)} g$

$$V_0 = \frac{Mg}{2}$$

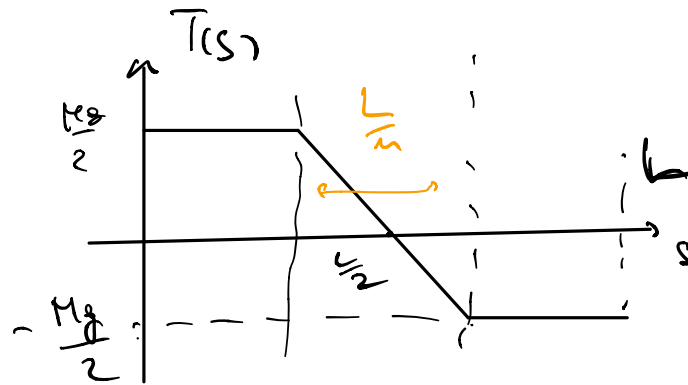
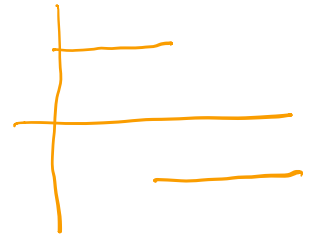
$$M_f(s) = \int_0^s T(x) dx$$

$$0 < s < \frac{L}{2} \left(1 - \frac{1}{u} \right)$$

$$\frac{L}{2} \left(1 - \frac{1}{n} \right) < s < \frac{L}{2} \left(1 + \frac{1}{n} \right)$$

$$\frac{L}{2} \left(1 + \frac{1}{n} \right) < s < L$$

(Verificábelo)


$$u \sim \infty$$


Moments fléchissants (vérifiables !)

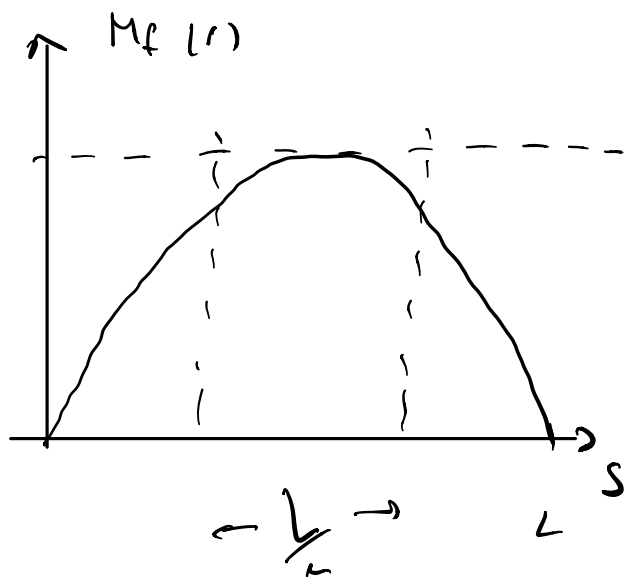
$$M_f(s) = \begin{cases} \frac{Mg}{2} s & 0 < s < \frac{L}{2} \left(1 - \frac{1}{u}\right) \\ Mg \omega \left(\frac{s}{2} - \frac{r^2}{2L} \right) + q & \frac{L}{2} \left(1 - \frac{1}{u}\right) < s < \frac{L}{2} \left(1 + \frac{1}{u}\right) \\ - \frac{Mg}{2} (L - s) & \frac{L}{2} \left(1 + \frac{1}{u}\right) < s < L \end{cases}$$

$$M_f = \int_0^S T(x) dx$$

Costante di integrazione

$$a = \frac{MgL}{4} \left(1 - \frac{M}{2} - \frac{1}{2m} \right)$$

imponendo che $M_f(s)$
sia una funzione continua



$$M_0 \frac{L}{4} \left(1 - \frac{1}{2\mu} \right)$$

$$\mu \rightarrow \infty$$

→

