Cyber-Physical Systems

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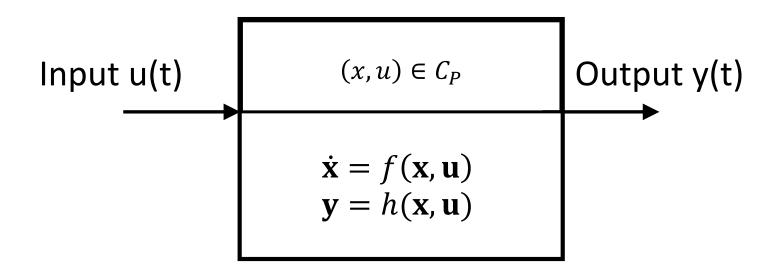
Lecture 2: Dynamical Systems

[Many Slides due to J. Deshmukh, Toyota]

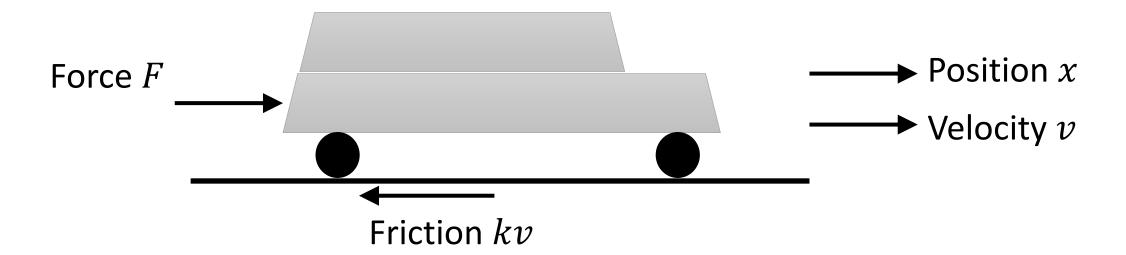
Dynamical Systems

- Most natural model for describing most physical systems
- Systems that continuously evolve over time
- It is represents by equations that involve the rates of change of quantities that describe the state of the phenomena
- Quantities describe the state of the phenomena, modeled as state variables
 - Pressure, Temperature, Velocity, Acceleration, Current, Voltage, etc.
- Could include algebraic relations between state variables

Dynamical Systems



Order Differential Equation



Newton's law of motion:
$$F = m \frac{d^2 x}{dt^2} + kv$$
; $v = \frac{dx}{dt}$

State-Space representation

$$\dot{\mathbf{x}} = f(\mathbf{x}, \mathbf{u})$$

 $\mathbf{y} = h(\mathbf{x}, \mathbf{u})$

Example:

Convert

$$\dot{x} = v(t)$$
$$\dot{v} = \frac{F(t) - kv(t)}{m}$$

> It is numerically efficient to solve

- It can handle complex systems
- > It allows for a more geometric understanding of dynamic systems
- > It forms the basis for much of modern control theory

Order Differential Equation

All derivatives are with respect to single independent variable, often representing time.

Order of ODE is determined by highest-order derivative of state variable function appearing in ODE

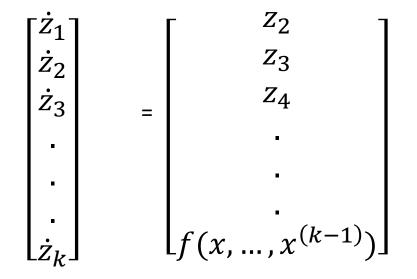
ODE with higher-order derivatives can be transformed into equivalent first-order system.

$$x^{(k)} = f(x, ..., x^{(k-1)})$$

 $z_1 = x, z_2 = \dot{x}, ..., z_k = x^{(k-1)}$

(k-1)

(k)



Executions of Car

Let \mathbb{T} represent a set representing time instants, i.e. $\mathbb{T} \subseteq \mathbb{R}^{\geq 0}$

Input Signal: Function F from $\mathbb{T} \to \mathbb{R}$

Input signal is assumed to be continuous or piecewise-continuous

Given an initial state (x_0, v_0) and an input signal F(t), the execution of the system is defined by **state-trajectories** x(t) and v(t) (from \mathbb{T} to \mathbb{R}) that satisfy the **initial-value problem**:

Sample Execution of Car

Suppose $\forall t: F(t) = 0, x_0 = 5 \text{ m}, v_0 = 20 \text{ m/s}, m = 1000 \text{kg}, k = 50 \text{Ns/m}$

- Then, we need to solve:
 - x(0) = 5; v(0) = 20• $\dot{x} = v; \dot{v} = -\frac{kv}{m}$

Solution to above differential equation (solve for v first, then x):

Sample Execution of Car with constant force

Compute solution using Simulink/Matlab/Breach

Differential Equation

The state of the system is characterized by state variables, which describe the system. The rate of change is (usually) expressed with respect to time

Simple Example: Temperature equations

$$\frac{dT}{dt} = -aT + T_{ext} + K_H u$$

Continuous-Time Component Definition

- Set I of real-valued input variables
- Set O or real-valued output variables
- Set X of real-valued (continuous) state variables
- Initialization Init specifying a set X₀ of initial values for states
- Dynamics: for each state variable, x, a real valued expression f over I and X
- Output Function: for each output variable, y, a real valued expression h over I and X.

Execution Definition

• Convention:
$$\mathbf{x} = (x_1, x_2, ..., x_n), \mathbf{y} = (y_1, y_2, ..., y_m)$$

- Given an input signal $u: \mathbb{T} \to \mathbb{R}$, an execution consists of a *differentiable* state signal $\mathbf{x}(t)$, and an output signal $\mathbf{y}(t)$, such that: 1. $\mathbf{x}(0) \in X_0$
 - 2. For each output variable y and time t, y(t) = h(u(t), x(t))

3. For each state variable
$$x$$
, $\frac{d}{dt}x(t) = f(u(t), x(t))$

Input u(t)
$$x(0) = x_0$$
 Output y(t)
 $\dot{\mathbf{x}} = f(\mathbf{x}, \mathbf{u})$
 $\mathbf{y} = h(\mathbf{x}, \mathbf{u})$

Order Differential Equation

Existence and Uniqueness of Solutions

- Given an input signal u(t), when are we guaranteed that the system has at least one execution? Is there nondeterminism in continuous-time components?
- Input signal should be piecewise-continuous, and additional conditions need to be imposed on the RHS of dynamics (f) and output functions (h)
- Related to solutions for the initial value problem in the classical theory of ODEs

$$\dot{\mathbf{x}} = f(\mathbf{x}, \mathbf{u})$$
$$\mathbf{y} = h(\mathbf{x}, \mathbf{u})$$

Existence

For the exists at least one solution $\mathbf{x}(t)$ if the function f is continuous

▶ Definition of continuity uses notion of distance between points
 ▶ Euclidean distance: d(x, y) = ||x - y||₂ = √(x₁ - y₁)² + ··· + (x_n - y_n)²

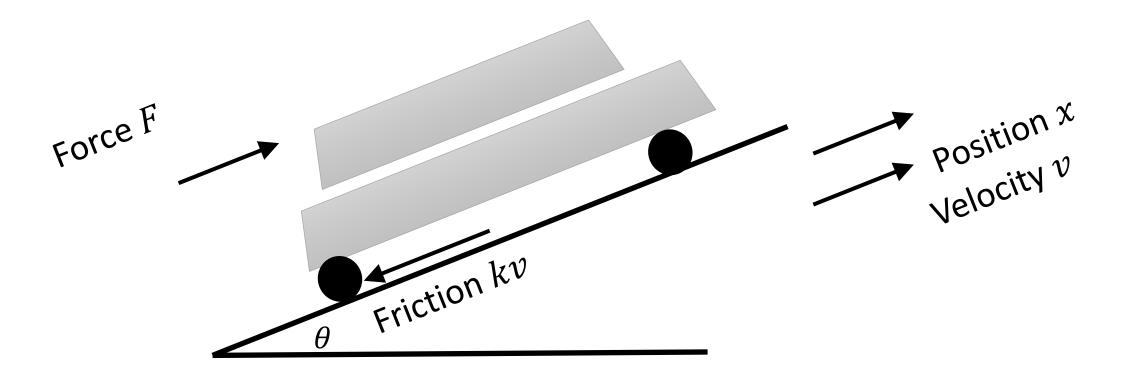
► f is continuous if for all $\mathbf{x} \in \mathbb{R}^n$, for all $\epsilon > 0$, there exists a $\delta > 0$, such that for all $\mathbf{y} \in \mathbb{R}^n$, if $\|\mathbf{x} - \mathbf{y}\|_2 < \delta$, then $\|f(\mathbf{x}) - f(\mathbf{y})\|_2 < \epsilon$.

Example when solution does not globally exist: $\frac{dx}{dt} = 1/t$

Uniqueness

- Solution to initial value problem is unique if *f* is Lipschitz continuous
- Lipschitz-continuity is a stronger version of continuity: upper bounds how fast a function can change
- Function f is **Lipschitz-continuous** if there exists a constant L (called the Lipschitz constant) such that: $\forall \mathbf{x}, \mathbf{y} \in \mathbb{R}^n : ||f(\mathbf{x}) - f(\mathbf{y})|| \le L ||\mathbf{x} - \mathbf{y}||$
- Examples:
 - Linear functions (e.g. $x_1 3x_2$) are Lipschitz continuous
 - Functions: x^2 , \sqrt{x} are not Lipschitz continuous over \mathbb{R}^n
- Can restrict T and X to some bounded and closed set such that f is piecewise-continuous and Lipschitz to get unique solutions over such compact domains
- Rely on numerical integration schemes/solvers to obtain solutions
 ode45, ode23, ode15, etc.

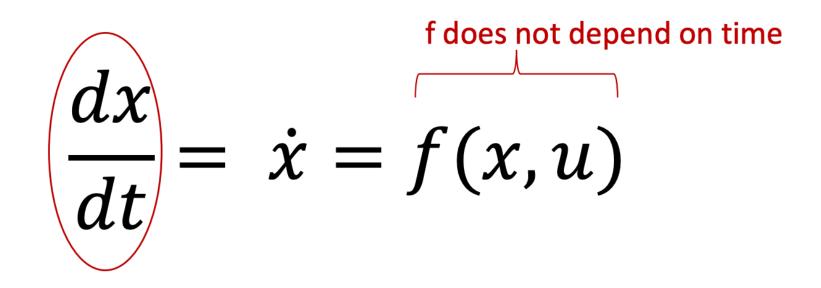
Model with disturbance



Newton's law of motion:
$$F = m \frac{d^2 x}{dt^2} + kv + mg \sin(\theta)$$

Time Invariant System

The system is time invariant because the output does not depend on the particular time the input is applied.



The underlying physical laws themselves do not typically depend on time.

Linear Systems

Equation of simple car dynamics can be written compactly as:

$$\begin{bmatrix} \dot{x} \\ \dot{v} \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ 0 & -k/m \end{bmatrix} \begin{bmatrix} x \\ v \end{bmatrix} + \begin{bmatrix} 0 \\ 1 \end{bmatrix} [F]$$

• Letting
$$A = \begin{bmatrix} 0 & 1 \\ 0 & -k/m \end{bmatrix}$$
, $B = \begin{bmatrix} 0 \\ 1 \end{bmatrix}$, we can re-write above equation in the form:

$$\dot{\mathbf{x}} = A\mathbf{x} + B\mathbf{u}$$
, where $\mathbf{x} = \begin{bmatrix} x & v \end{bmatrix}$, and $\mathbf{u} = \begin{bmatrix} F \end{bmatrix}$

Linear Components

Linear components model linear systems

- ► f is of the form $a_1x_1 + a_2x_2 + \cdots + a_nx_n$ or compactly, $f = A\mathbf{x}$
- ▶ h is of the form $b_1u_1 + b_2u_2 + \cdots + b_mu_m$ or compactly, $h = B\mathbf{u}$

- Linear systems have many nice properties:
 - Many analysis methods in the frequency domain (using Fourier/Laplace transform methods)
 - Superposition principle (net response to two or more stimuli is the sum of responses to each stimulus)

Solutions to Linear Systems

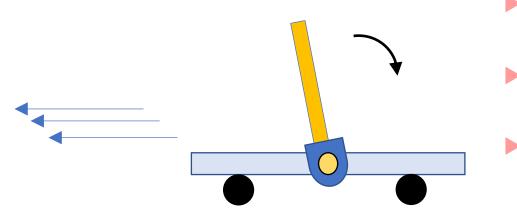
• **Autonomous** linear system has no inputs: $\dot{\mathbf{x}} = A\mathbf{x}$

- Solution of autonomous linear system can be fully characterized:
 - $\mathbf{k}(t) = e^{At} \mathbf{x}_0$
 - Computing e^A is easy if A is a diagonal matrix (non-zero elements are only on the diagonal)
- For a linear system with *exogenous* inputs? $x(t) = e^{At}x_0 + \int_0^t e^{A(t-\tau)}Bu(\tau)d\tau$

In practice, numerical integration methods outperform matrix exponential

Stability of Systems

- Property capturing the ability of a system to return to a quiescent state after perturbation
 - Stable systems recover after disturbances, unstable systems may not
 - Almost always a desirable property for a system design
- Fundamental problem in control: design controllers to stabilize a system



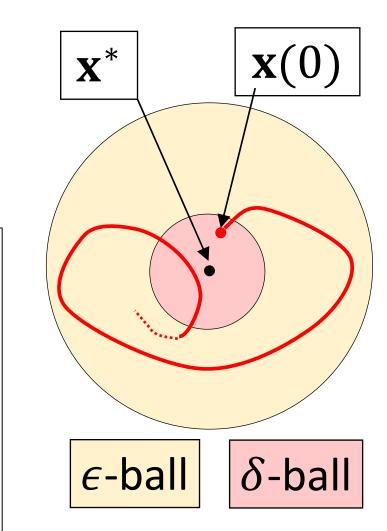
- Problem: Inverted Pendulum on a moving cart is inherently unstable, aim: keep it upright
- Solution Strategy: Move cart in direction in the same direction as the pendulum's falling direction
- Design problem: Design a controller to stabilize the system by computing velocity and direction for cart travel

Lyapunov stability

Solutions starting δ close from equilibrium point must remain close (within ϵ) forever

System x = f(x) with f Lipschitz continuous
Equilibrium point when f(x) is zero (say x*)
Equilibrium point x* is Lyapunov-stable if:
For every ε > 0,
There exists a δ > 0, such that

- if $\|\mathbf{x}(0) \mathbf{x}^*\| < \delta$, then,
- for every $t \ge 0$, we have $\|\mathbf{x}(t) \mathbf{x}^*\| < \epsilon$



Asymptotic Stability

Solutions not only remain close, but also converge to the equilibrium

System
$$\dot{\mathbf{x}} = f(\mathbf{x})$$

• Equilibrium point \mathbf{x}^* is asymptotically-stable if:

x* is Lyapunov-stable +

► There exists $\delta > 0$ s.t. if $\|\mathbf{x}(0) - \mathbf{x}^*\| < \delta$, then $\lim_{t \to \infty} \|\mathbf{x}(t) - \mathbf{x}^*\| = 0$

Exponential Stability

Solutions not only converge to the equilibrium, but in fact converge at least as fast as a known exponential rate

- All stable linear systems are exponentially stable
- This need not be true for nonlinear systems!

System $\dot{\mathbf{x}} = f(\mathbf{x})$

• Equilibrium point \mathbf{x}^* is exponentially-stable if:

x*is asymptotically stable +

► There exist $\alpha > 0$, $\beta > 0$ s.t. if $||\mathbf{x}(0) - \mathbf{x}^*|| < \delta$, then for all $t \ge 0$:

 $\|\mathbf{x}(t) - \mathbf{x}^*\| \le \alpha \|\mathbf{x}(0) - \mathbf{x}^*\| e^{-\beta t}$

Bounded-Input-Bounded-Output (BIBO) stability

If the output signal is bounded for all input signals that are bounded.

Example:
•
$$x(0) = x_0; v(0) = v_0$$

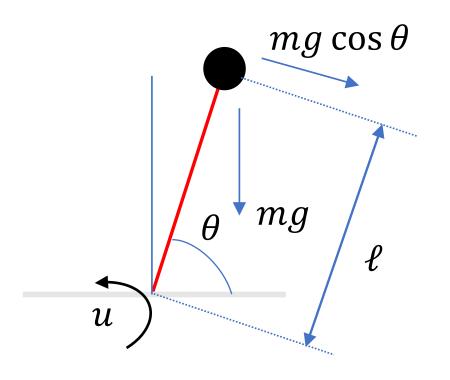
• $\dot{x} = v(t); \dot{v} = \frac{F(t) - kv(t)}{m}$

Feedback Linearization

- Equations of motion for inverted pendulum: $m\ell^2\ddot{\theta} + d\dot{\theta} + m\ell g\cos\theta = u$
- Control Input: Torque u
- Rewriting, with $x_1 = \theta$, $x_2 = \dot{\theta}$:

$$\dot{x_1} = x_2$$

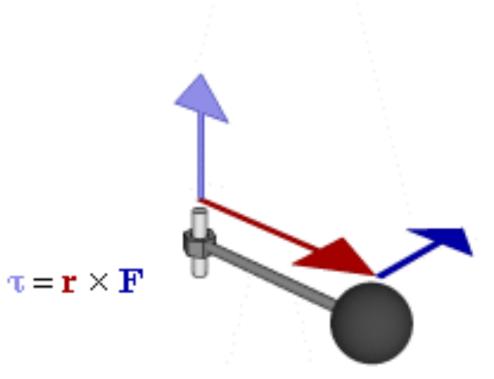
$$\dot{x_2} = \left(-\frac{d}{m\ell^2}x_2 - \frac{g}{\ell}\cos x_1\right) + \left(\frac{1}{m\ell^2}u\right)$$



Torque is the angular version of force

A force applied at a right angle to a lever multiplied by its distance from the lever's fulcrurs (the length of the lever arm) is its torque

r radius arm
F force orthogonal to arm
M mass of the object



$$\tau = I\ddot{\theta} = Mr^2\ddot{\theta} = r F \sin\theta$$