

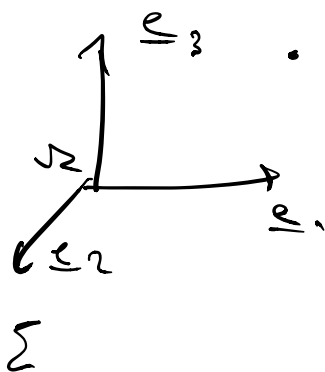
MECCANICA RAZIONALE

Ingegneria Civile
Navale

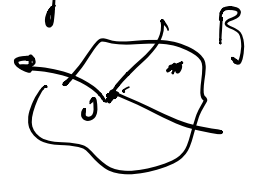
Lezione 3 marzo 2021

Corpo rigido

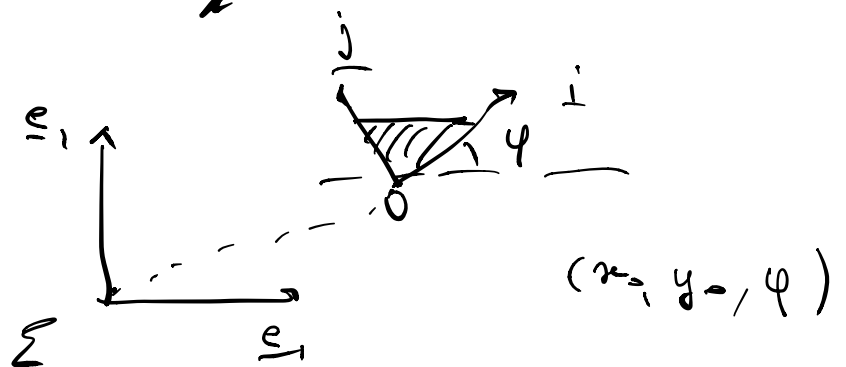
Studio del
punto materiale



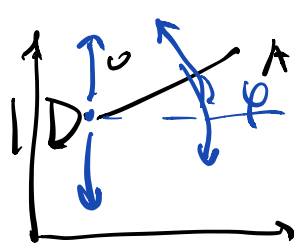
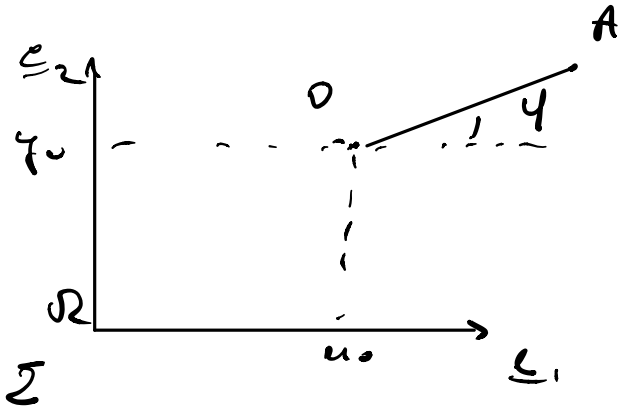
Studio del corpo
rigido



$$3 + 3 = 6$$



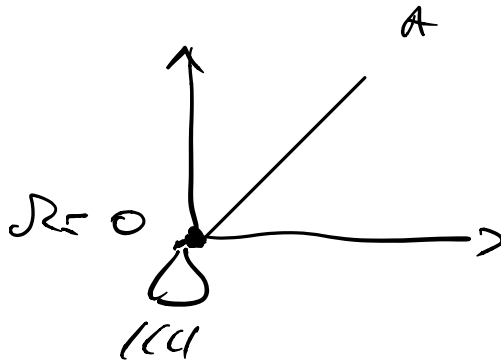
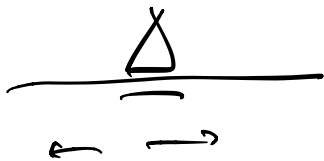
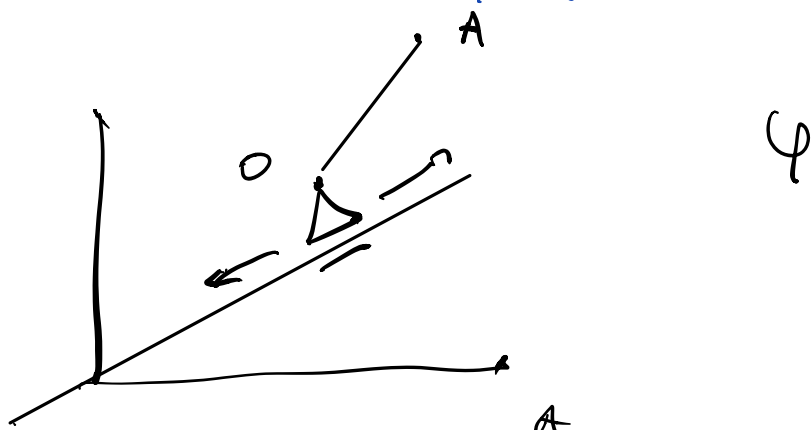
$$2 + 1 = 3$$



$$x_0 = 0$$

$$x_0$$

$$(x_0, y_0, \varphi) \rightarrow (\cancel{x_0}, y_0, \varphi)$$



$$\begin{cases} x_0 = 0 \\ y_0 = 0 \\ \varphi = \alpha \end{cases}$$

Vincoli semplici : Tolpono & grado
 di libert  → principio di
 sovrapposizione dei vincoli

Σ sistema materiale

m gradi di libertà
 m vincoli

$m < n$

IPERSTATICO

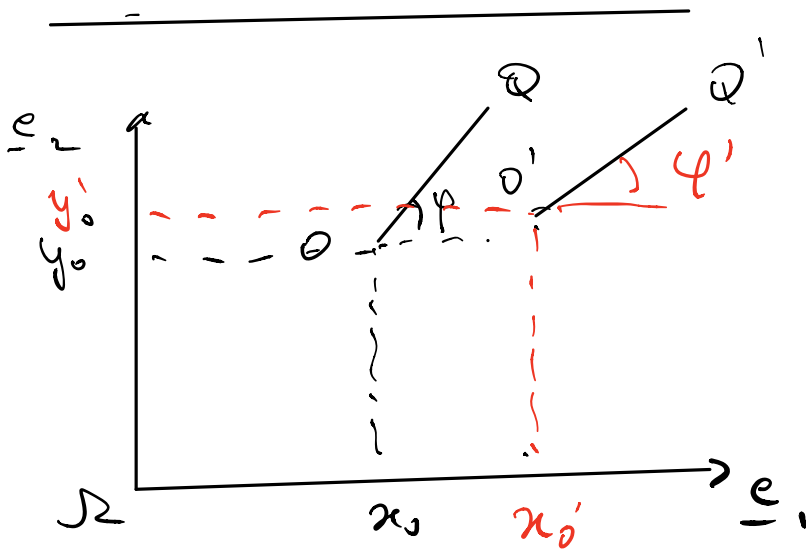
$m = n$

IPOSTATICO

$m > n$

IPERSTATICO

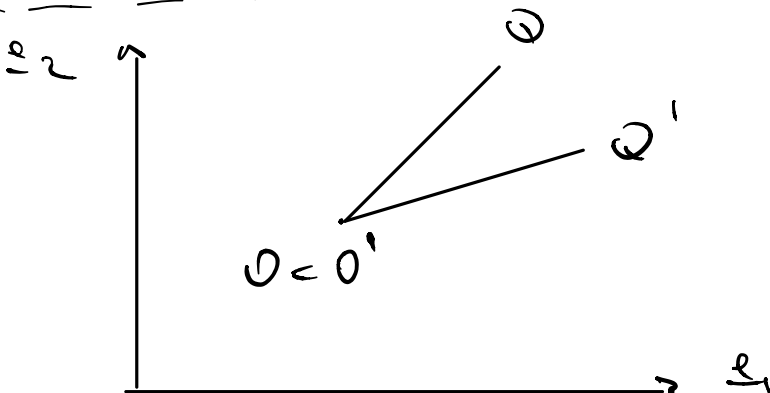
Sistemi articolati



OQ
 (x_0, y_0, φ)
 $O'Q'$
 (x'_0, y'_0, φ')

Coordinate indipendenti (libere)

$$\overline{OQ} = \overline{O'Q'} = l$$



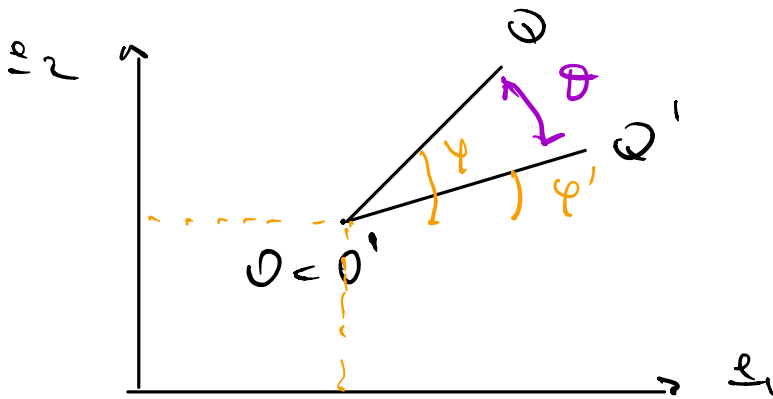
cerniera mobile
 invariante

$$\begin{cases} x_0 = x'_0 \\ y_0 = y'_0 \end{cases}$$

son opposition est
due simplement
à la différence

$$(x_0, x'_0, y_0, y'_0, \varphi, \varphi')$$

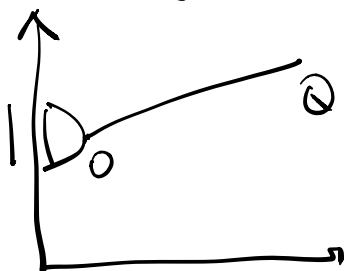
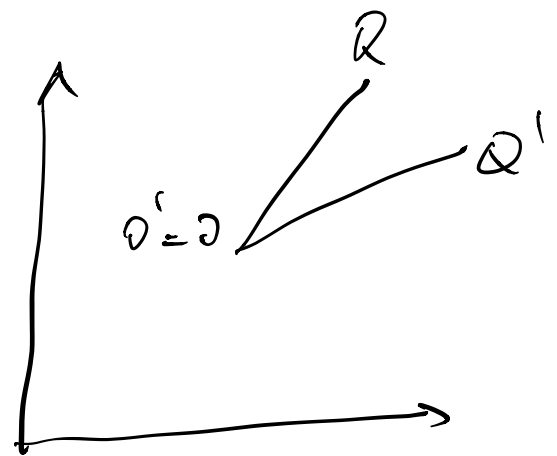
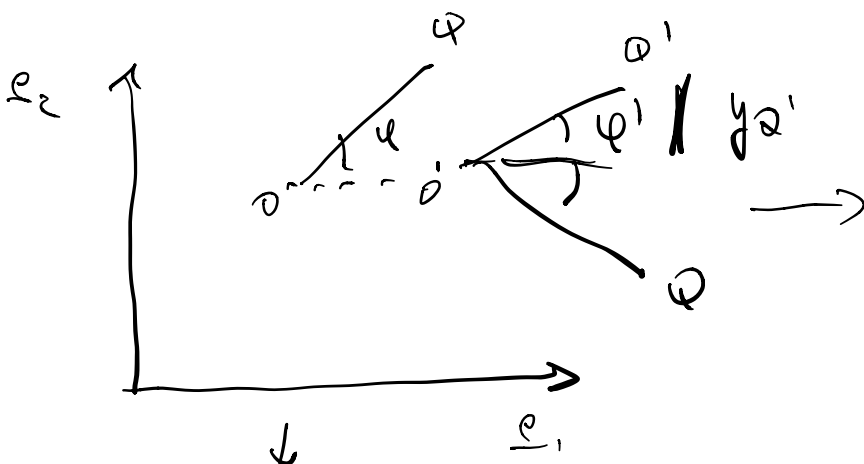
$$6 \rightarrow 4$$



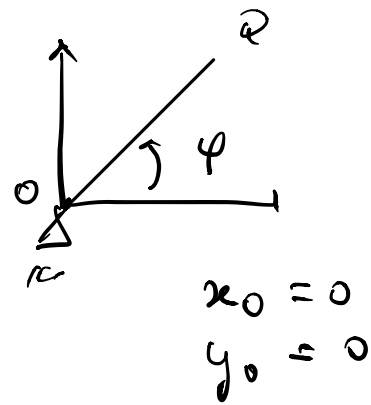
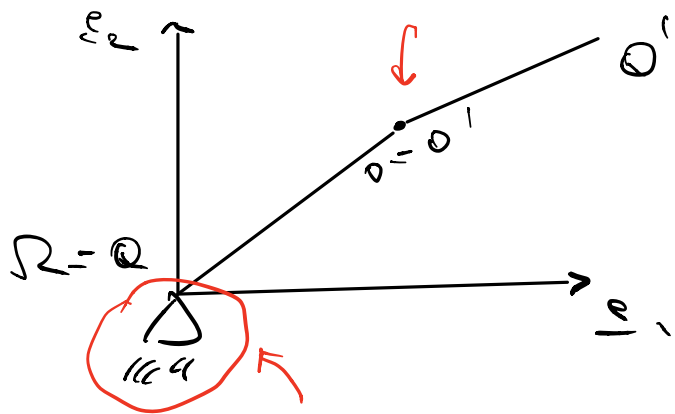
$$(x_0, y_0, \varphi, \varphi')$$

$$(x_0, y_0, \varphi, \theta)$$

Seconda parte



Ad esempio

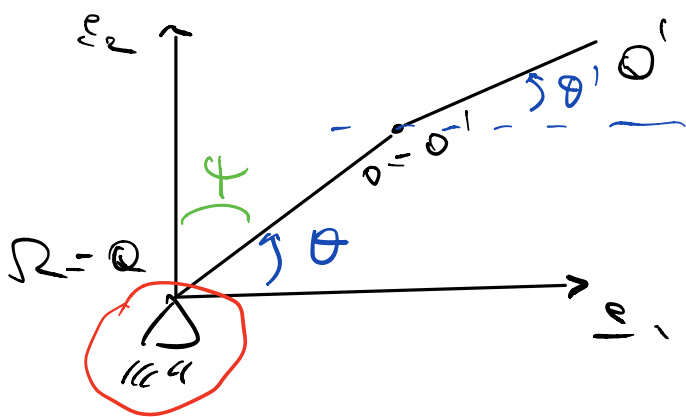


Partiamo da $(x_0, y_0, \varphi, x'_0, y'_0, \varphi')$ 6 gradi di libertà
cervice esterna

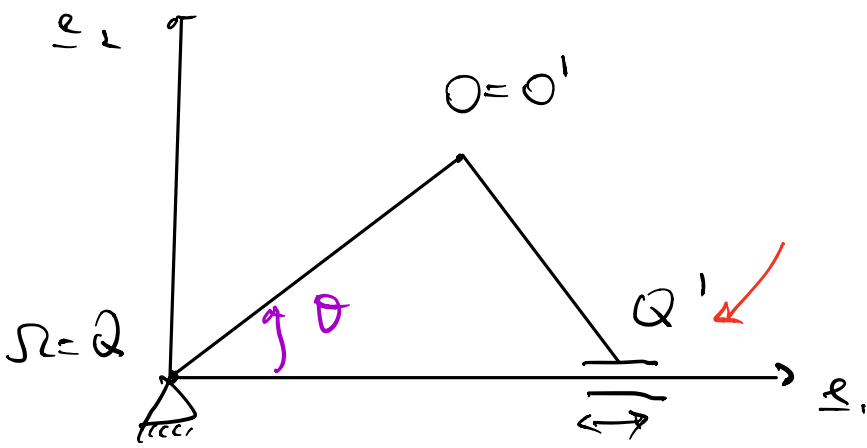
cervice interna

$$\begin{cases} x_0 = x'_0 \\ y_0 = y'_0 \end{cases}$$

$$\begin{cases} x_Q = 0 = \underline{x_0} + l \cos \underline{\varphi} \\ y_Q = 0 = \underline{y_0} + l \sin \underline{\varphi} \end{cases}$$



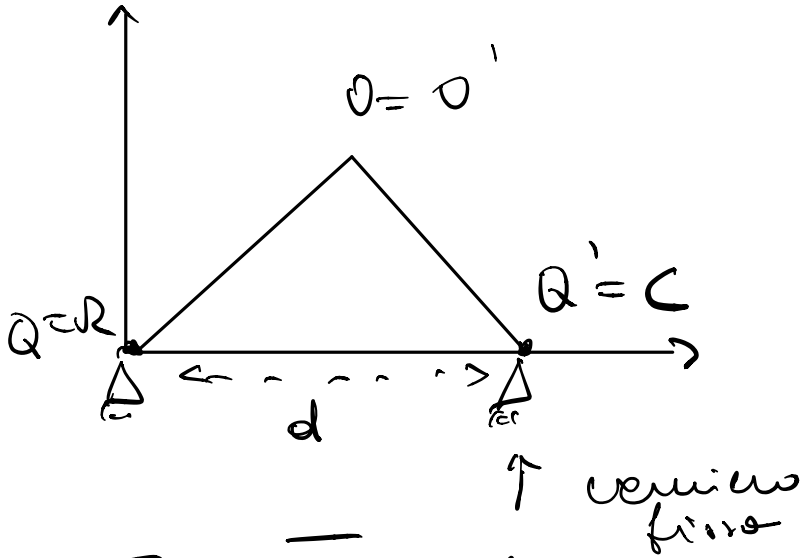
$6 - 4 = 2$
gradi di libertà



$$\begin{aligned} \rightarrow y_{Q'} &= 0 \\ &= y'_0 + l \sin \varphi' \end{aligned}$$

$6 - 5 = 1$ grado di libertà

$$6 - 4 - 1 = 6 - 5 = 1 \quad \theta$$



$6 - 6 = 0$ sistema
isostatico

$$x_{O'} = d$$

$$= x_O' + l \cos \varphi'$$

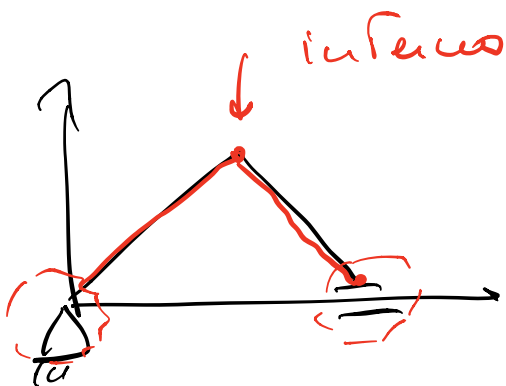
$$\overline{OQ} = \overline{O'Q'} = l$$

→ principio di sovrapposizione dei
vincoli



Spazio delle configurazioni
 $e = (x_0, y_0, \varphi)$

$$y_0 = 0$$



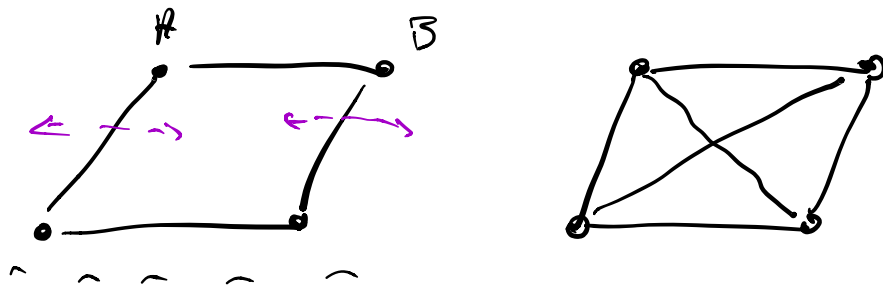
Commento

collegate

Trovature :

aste

da cerniere



Fondamento simile a quello sviluppato per i vincoli

Nodi \rightarrow punti materiali

Asse \rightarrow vincoli

Tetto pardo

Classificazione delle forze agenti su un sistema

1. Forze

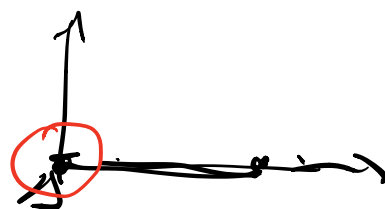
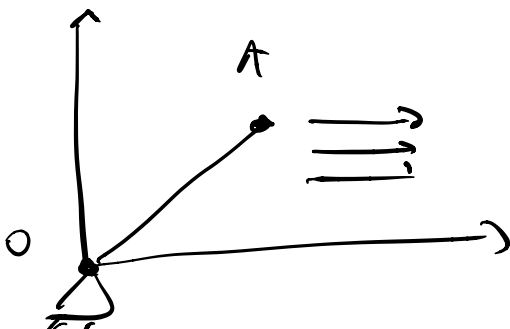
attive

forze peso
forze elastiche



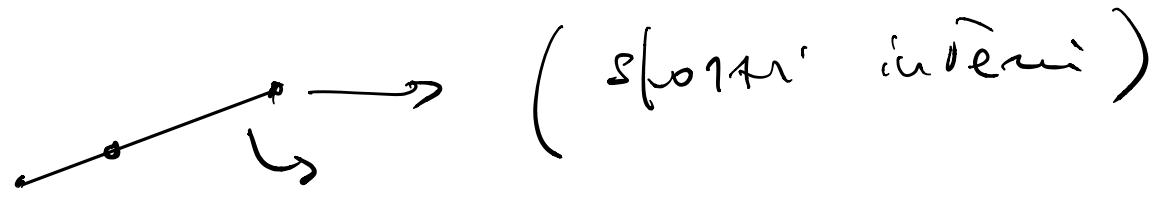
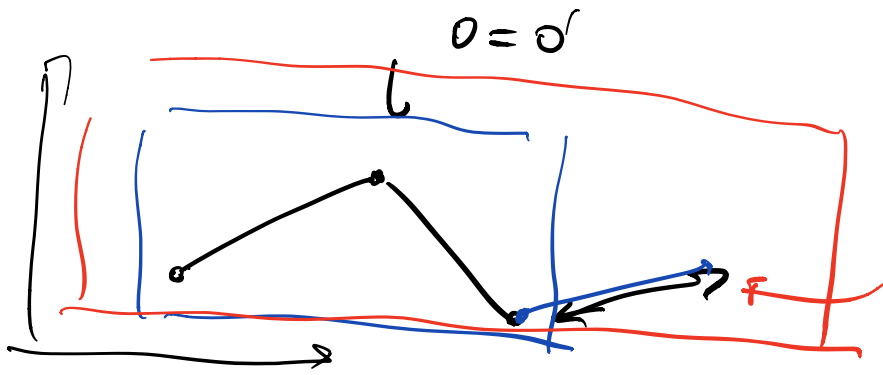
reattive

: forze dovute ai vincoli e di risposta alle forze attive



2. forze esterne : esercitate da
oggetti esterni

interne : esercitate da
una parte del
sistema sull'altra



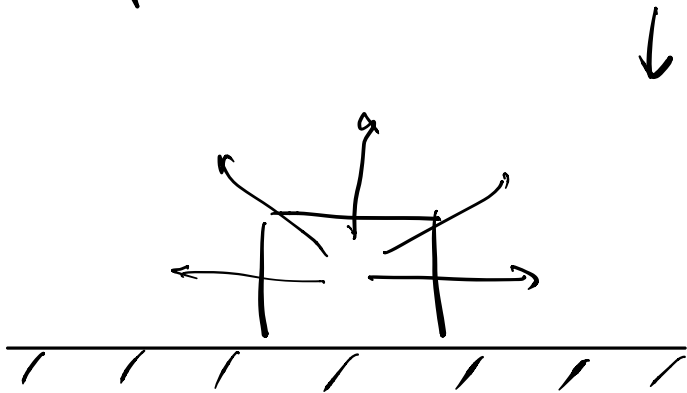
3. forze concentrate
distribuite

PRINCIPIO DEI LAVORI VIRTUALI

Forze attive

(vvedi → lemmosioni pesometriche)

Esempio



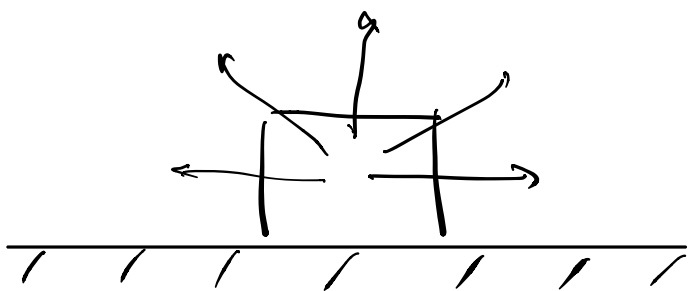
↓ q

Forza peso
 mg

↳ forze di reazione

Idea: considerare le forze attive
→ lavoro svolto da queste forze
se pensiamo di spostare il sistema
materiale in modo compatibile con i
vincoli.

↓ q

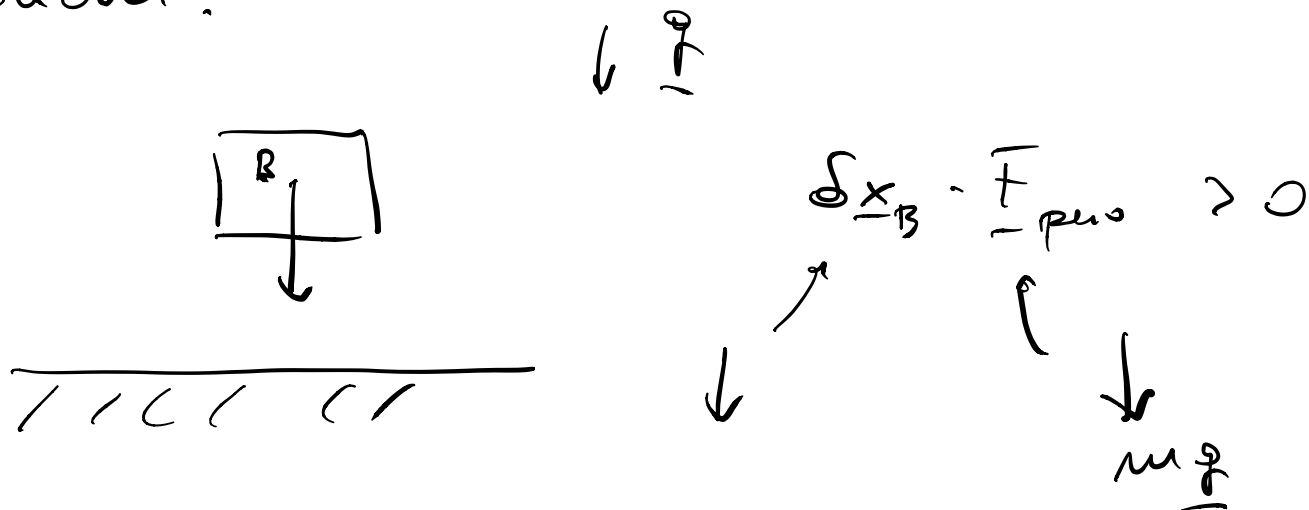


Spostamenti
virtuali

δx_B

Vedremo: All'equilibrio il lavoro
virtuale delle forze attive è
minore o uguale a zero per ogni
spostamento virtuale consentito dei

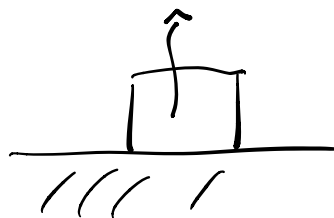
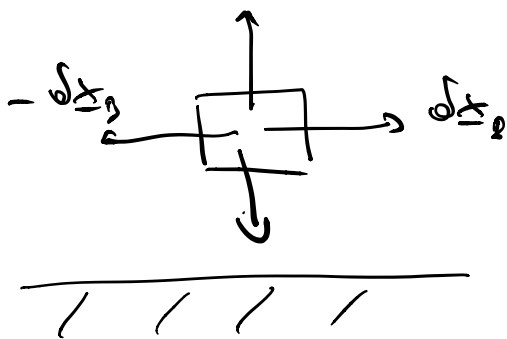
vincoli:



Spostamenti virtuali:

• invertibile se:

δx_B è un spostamento virtuale $\Leftrightarrow -\delta x_B$ è



Lavoro virtuale $L.V. = \bar{F}_B \cdot \delta x_B$

forze attive nel punto B

spostamento virtuale del punto B

All'eq. $\bar{F}_B \cdot \delta x_B \leq 0$

Se δx_B è invertibile, allora $-\bar{F}_B$

e' spontaneo virtuale e quindi

all' eq. $\underline{F}_B \cdot (-\delta x_B) \leq 0$

Se δx_B invertibile, all' eq.

$$\left. \begin{array}{l} \underline{F}_B \cdot \delta x_B \leq 0 \\ \underline{F}_B \cdot (-\delta x_B) \leq 0 \end{array} \right\} \Rightarrow \underline{F}_B \cdot \delta x_B = 0$$

↑