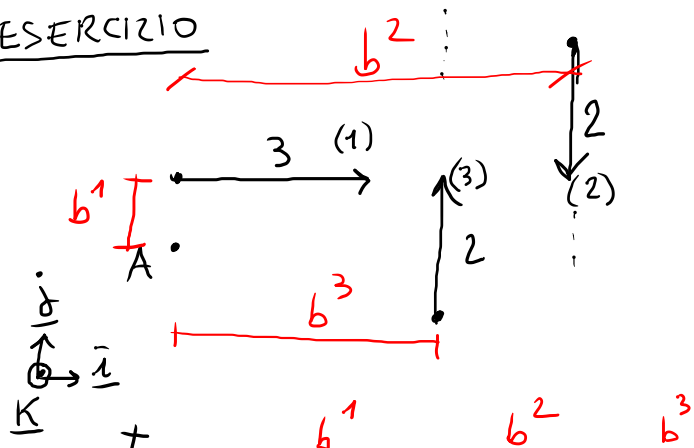


ESERCIZIO

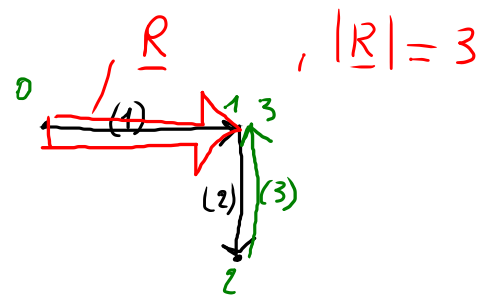


SIST. PIANO DI VETTORI

3/03/22

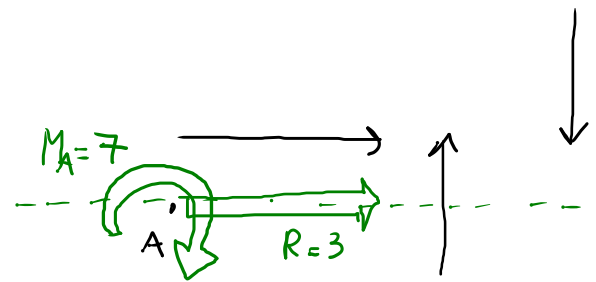
\underline{R} ? M_A ?

\underline{R} : metodo grafico
poligono dei
vettori.



$$M_A: -3 \cdot 1 - 2 \cdot 6 + 2 \cdot 4 = -7$$

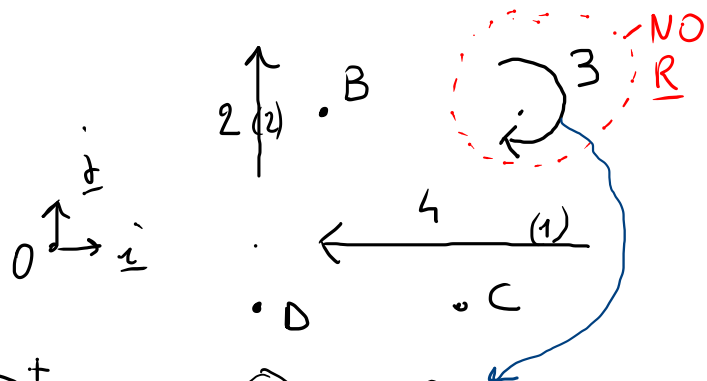
Se i vettori sono
FORZE (N), i
MOMENTI \rightarrow Nm



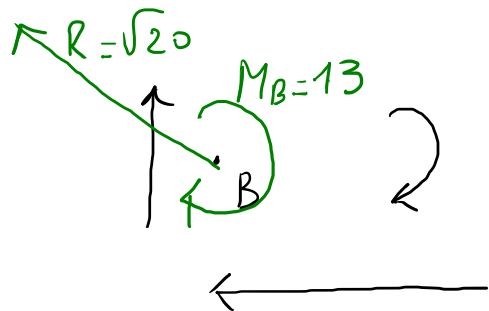
NON È
L'ASSE CENTRALE

SISTEMA PIANO $\Rightarrow R, M_A$

ES: SIST. PIANO DI VETTORI CON MOMENTO CONCENTRATO

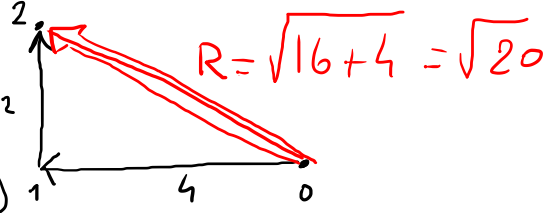


$$\overset{+}{M}_B : -4 \cdot (2) - 2 \cdot (1) - 3 = -13$$



R? M_B

R:



$$\underline{R} = -4\underline{i} + 2\underline{j} \quad ; \quad |\underline{R}| = \sqrt{16 + 4}$$

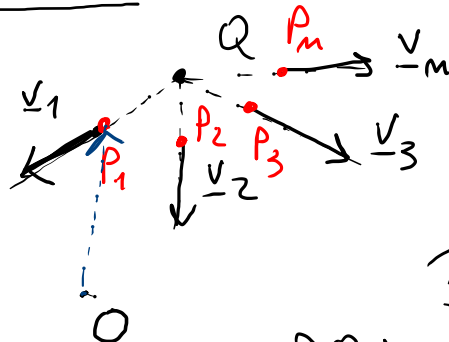
Cosa succede se cambio il POLO?

R non cambia!

M_B cambia in funzione del POLO STESSO! ($M_B \neq M_C \neq M_D$)

TEOREMA DI VARIGNON

nello spazio:



$M_{-O}?$

$$\boxed{M_{-O} = \sum_{i=1}^m OP_i \times \underline{v}_i}$$

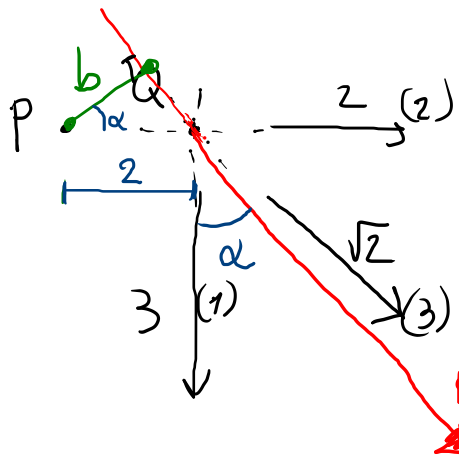
$$= OQ \times \sum_{i=1}^m \underline{v}_i = \boxed{OQ \times \underline{R}}$$

Osservo che $OP_i \times \underline{v}_i = OQ \times \underline{v}_i$ perché il singolo momento non cambia se traslo \underline{v}_i sulla propria retta d'azione.

ENUNCIATO del sistema

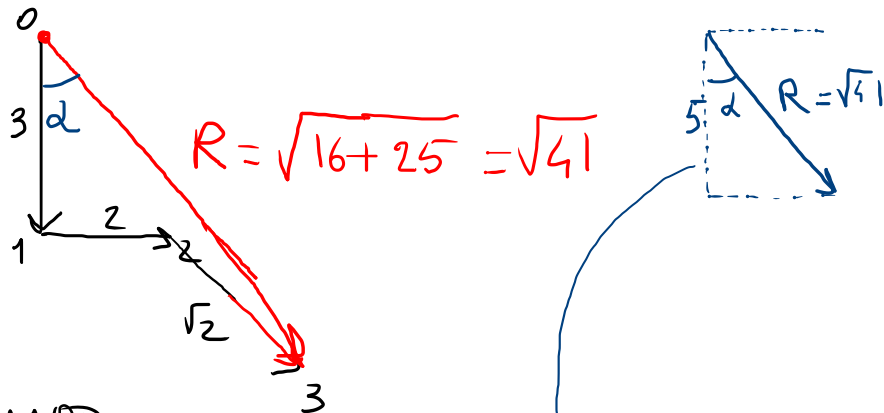
Il Momento rispetto al punto Q è uguale al Momento della Risultante applicato al punto Q stesso.

ES.



M_p ?

SFRUITO IL TM. DI VARIGNON, CALCOLO LA RISULTANTE E LA APPLICO IN Q.



$$M_p = R \cdot b = \sqrt{41} \cdot b \quad (*)$$

ES DA FARE: CALCOLORE M_p CONSIDERANDO I SINGOLI VETTORI; DETERMINARE GEOMETRIC. b E VERIFICARE CHE IL MOM. $(*)$ SIA UGUALE A $(*)$

AGG. 8/3: $\cos \alpha = 5/\sqrt{41}$

$$b = 2 \cos \alpha = 10/\sqrt{41}$$

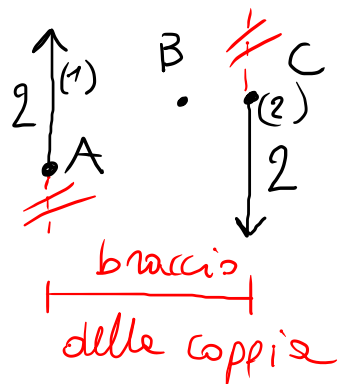
$$(*) M_p = \sqrt{41} \frac{10}{\sqrt{41}} = 10$$

$$(**) M_p = 3 \cdot 2 + 2 \cdot 2 = 10$$

$\uparrow_{(1)}$ $\uparrow_{(\text{componente verticale del vett. (3)})}$

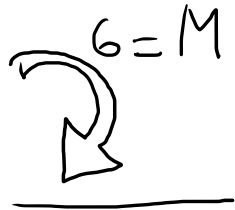
il vettore (2) passa per P e non ha braccio.

UN SISTEMA (PIANO) PARTICOLARE: LA COPPIA



$R?$ \updownarrow R SEMPRE 0

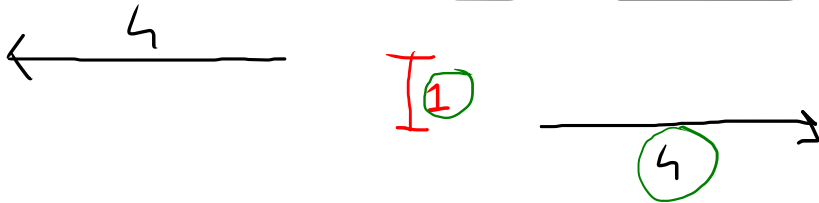
$$M_A = M_B = M_C = M_p \quad (\forall P) \Rightarrow \text{COSTANTE} = -6$$



$$\overset{+}{M}_A: 2 \cdot 0 - 2 \cdot 3 = -6$$

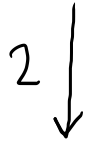
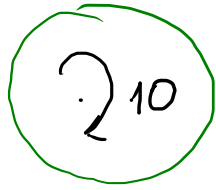
$$\overset{+}{M}_B: -2 \cdot 2 - 2 \cdot 1 = -6$$

IL MOMENTO DELLA COPPIA È 6 ED È ORARIO = $2 \cdot \text{braccio}$



$M?$ \curvearrowright 4

ES

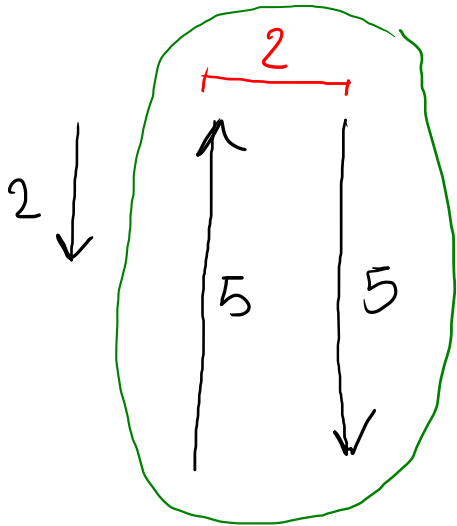


Σ

Per il calcolo di \underline{R} e M_p del sistema onegato (Σ) posso sostituire el Momento concentrato 10 una COPPIA EQUIVALENTE

Σ e Σ' HANNO STESSA \underline{R} e STESSO

M_p, V_p



Σ'

se il braccio delle coppie è NULLO



COPPIA DI BRACCIO NULLO

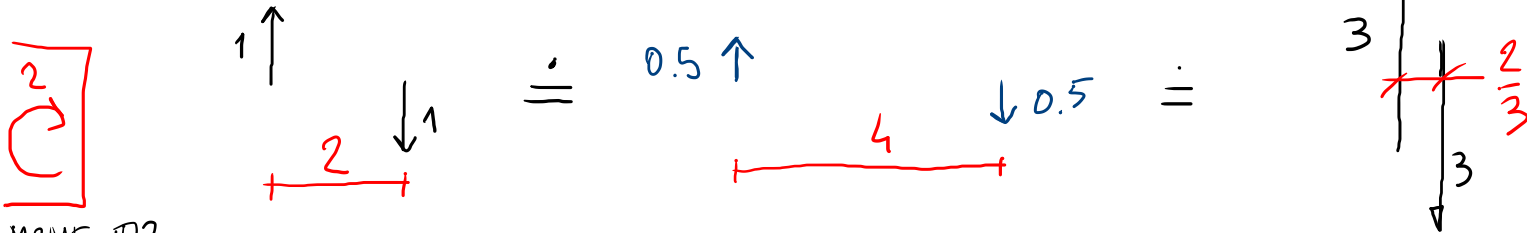
$(\underline{R} = \underline{0})$ $(\underline{M} = \underline{0})$

EQUIVALENZA DI SISTEMI DI VETTORI

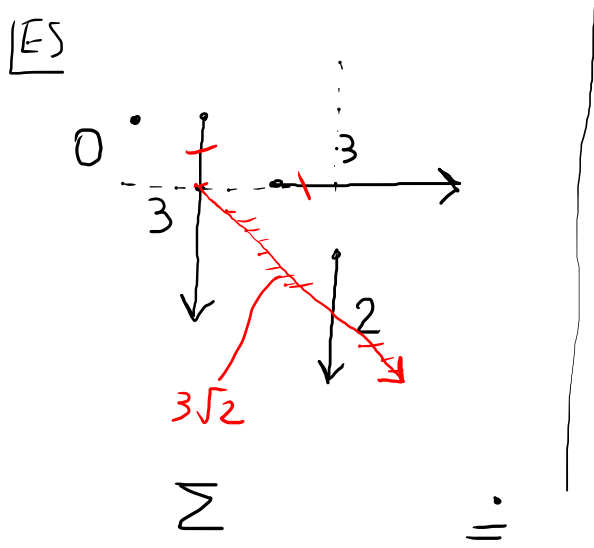
DUE SISTEMI DI VETTORI Σ e Σ' SONO EQUIVALENTI SE E SOLO SE
CONDIVIDONO LA STESSA RISULTANTE E LO STESSO MOMENTO
RISPETTO AD UN POLO GENERICO :

$$\boxed{\Sigma \doteq \Sigma'} \iff \boxed{\underline{R} = \underline{R}' \quad \text{e} \quad \underline{M}_P = \underline{M}'_P, \forall P}$$

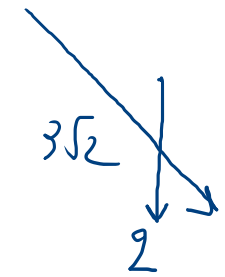
ES : EQUIVALENZA TRA COPPIE



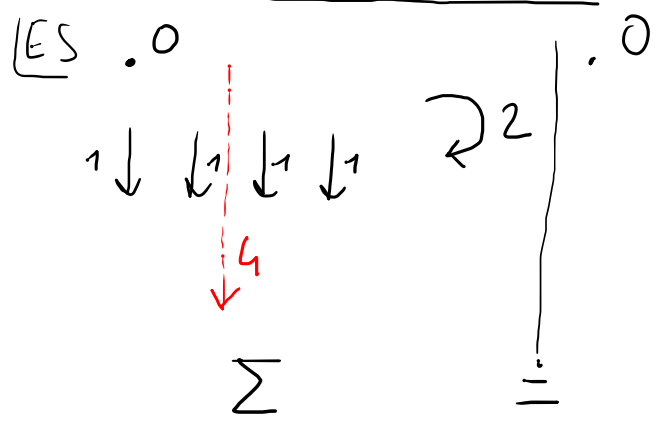
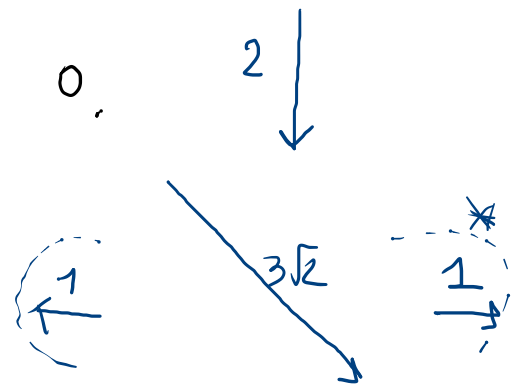
MOMENTO
DELLA COPPIA



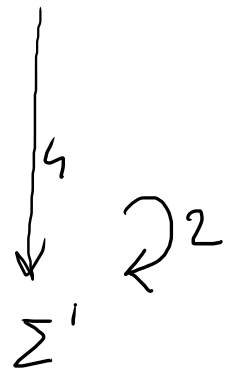
0.



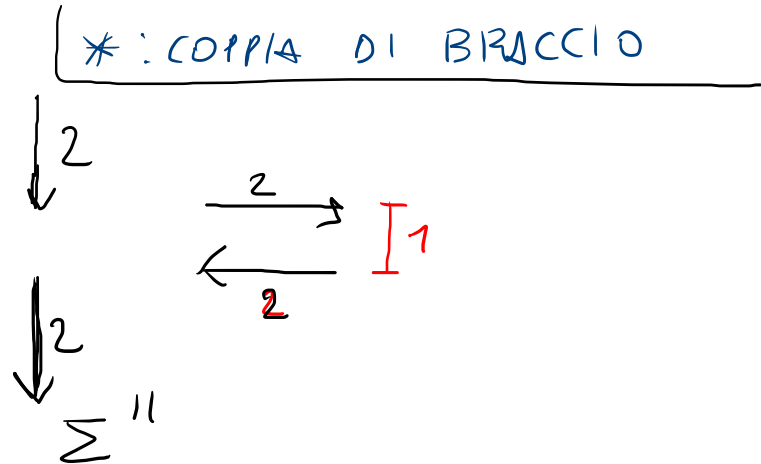
0.



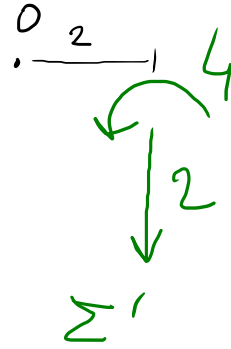
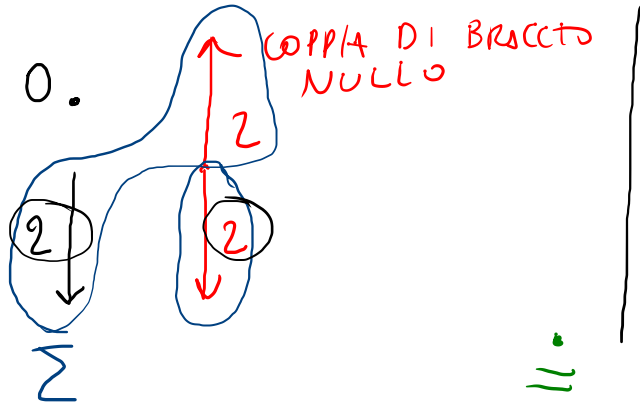
0.



0.

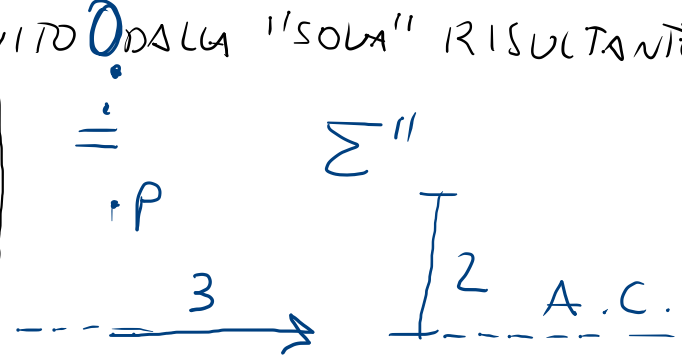
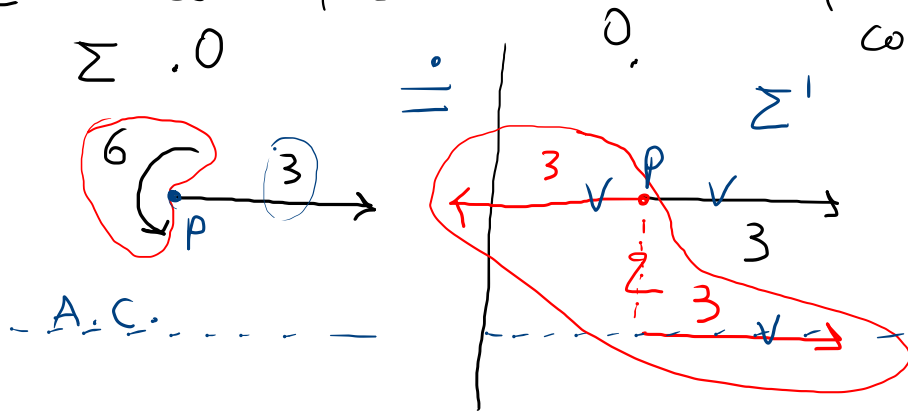


ES : TRASLAZIONE TRASVERSALE DI UN VETTORE



R	R'
$2 \downarrow$	$\downarrow 2$
M_0	M'_0
0	$^+ P: +4 - 4 = 0$

ES : DUALE A QUELLO PRECEDENTE (OTTENERE UN SIST. EQUIVALENTE A Σ COSTRUITO DALLA "SOLA" RISULTANTE)

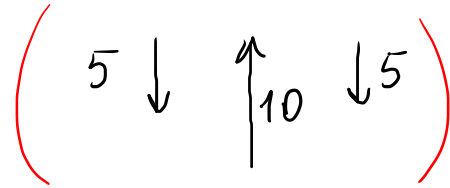
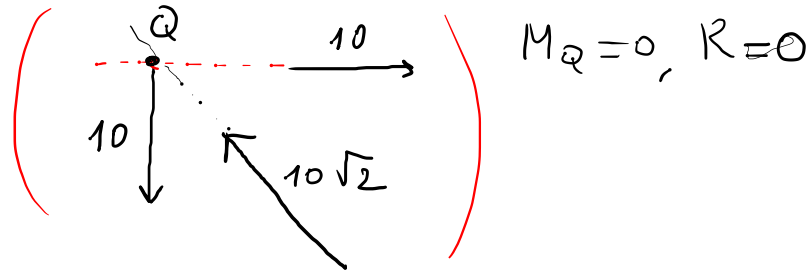


SISTEMA NULLO o EQUILIBRATO

$\Sigma \bar{e}$ UN SIST. NULLO o EQUIL. SE

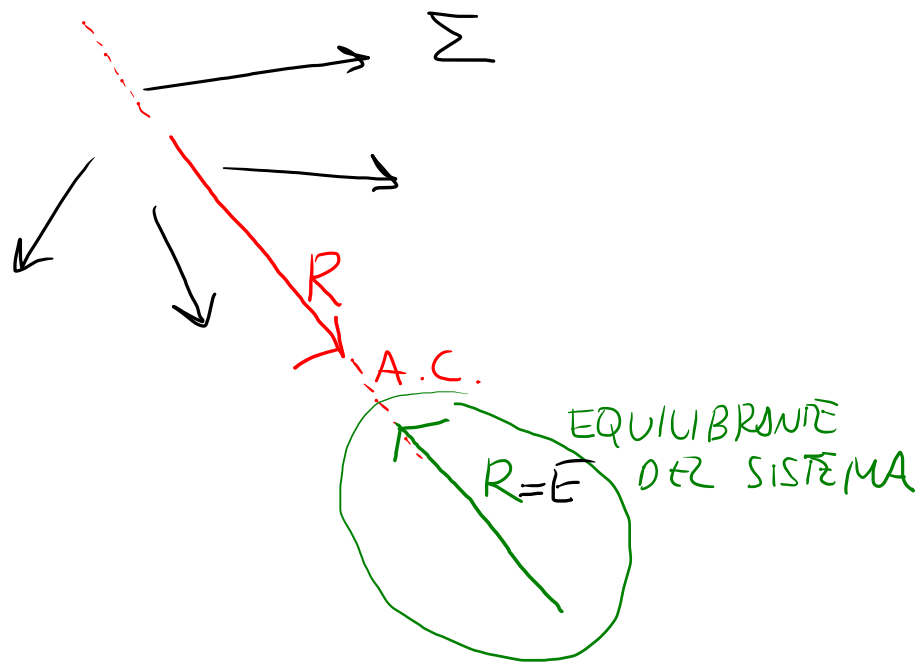
$$\boxed{\underline{R} = \underline{0}, \underline{M}_P = \underline{0}} \forall P$$

$\left(\begin{array}{c} \leftarrow 2 \quad 2 \rightarrow \end{array} \right)$ COPPIA DI BRACCIO NULLO $(\underline{R} = \underline{0}, \underline{M}_P = \underline{0}, \forall P)$



SIST. NULLI

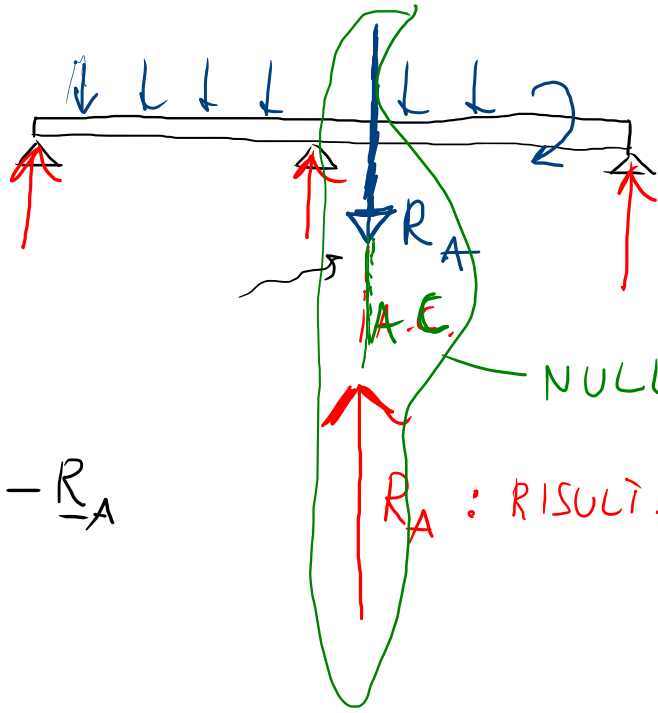
EQUILIBRANTE DI UN SISTEMA



L'EQUILIBR. DI UN SISTEMA È QUEL VETTORE CHE EQUILIBRA O ANNULLA LA RISULTANTE

$$\underline{R} + \underline{E} = \underline{0}$$

OSSERVAZ -



FORZE ATTIVE

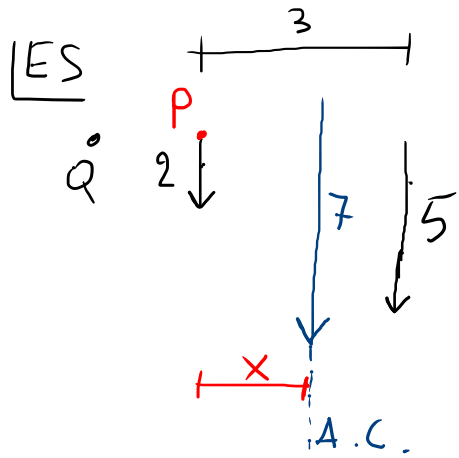
R_A : RISULT. FORZE ATTIVE

FORZE REATIVE

NULLO

\vec{R}_{-A}
L'EQUILIBRANTE $-\vec{R}_A$

R_A : RISULT. FORZE REATIVE = EQUILIBRANTE
DELE FORZE ATTIVE



- $\Sigma = \{2, 5\}$
 $\Sigma' = \{7\}$
- 1) TROVA L'ASSE CENTRALE
 - 2) $R ? = 7$
 - 3) ANNULLA IL SISTEMA
 → APPL. ALL'ASSE CENTRALE

CALCOLO x IMPONENDO $M_P = M_P$ +5 ≈ 2.1

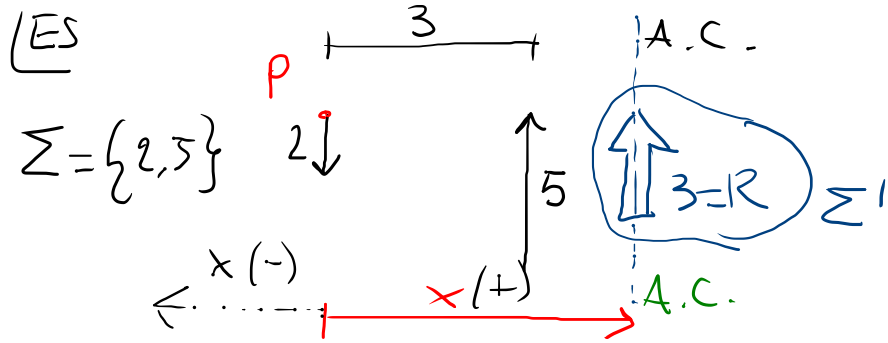
$$2 \cdot 0 - 5 \cdot 3 = -7 \cdot x \Rightarrow +15 = +7x \Rightarrow x = \frac{15}{7} < \frac{21}{7}$$

3)

$2 \downarrow$
 $7 \uparrow$
 $5 \downarrow$
 A.C.

Σ'' NON È EQUIVALENTE,
 A Σ e Σ'

Σ'' È NULLO

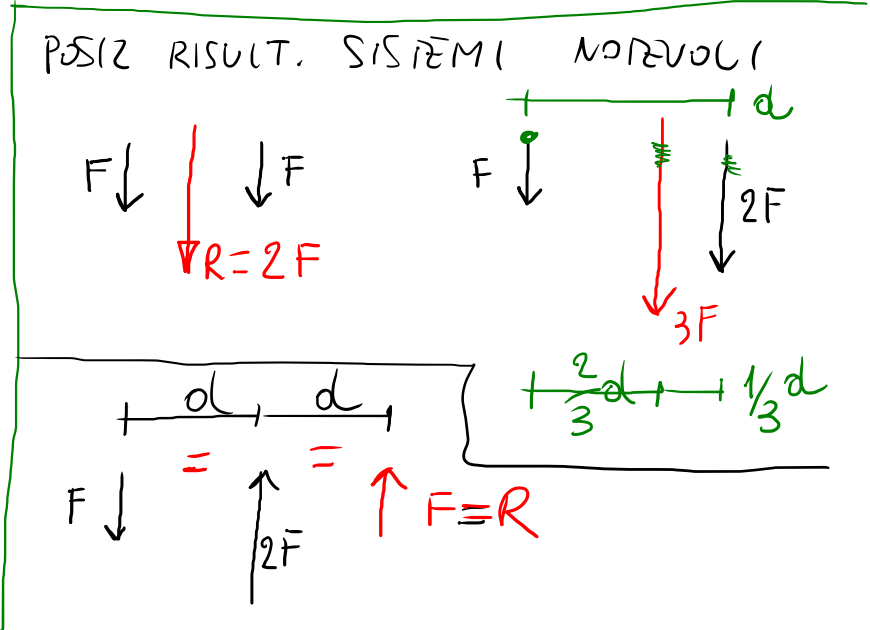
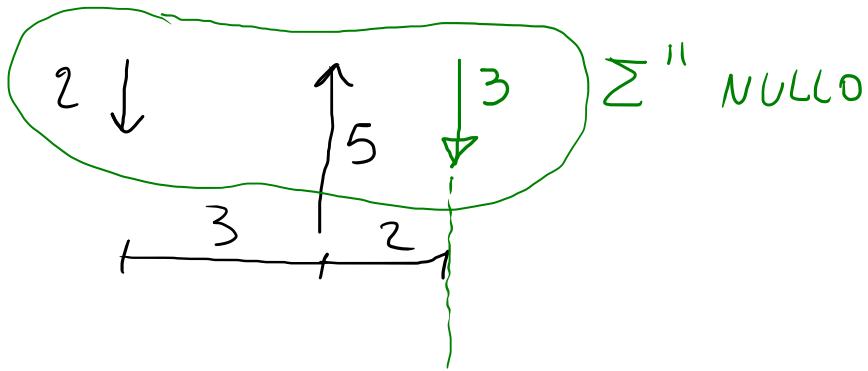


$$M_P^{\Sigma} = M_P^{\Sigma I} + \dots$$

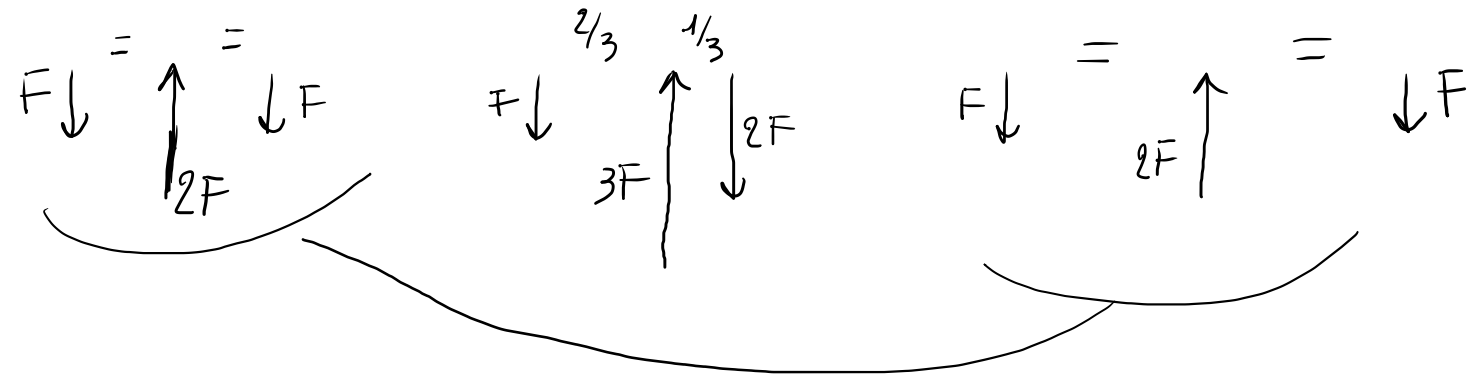
$$2 \cdot 0 + 5 \cdot 3 = 3 \cdot X$$

$$X = 5 > 3$$

COME POSSO EQUILIBRARE Σ ?

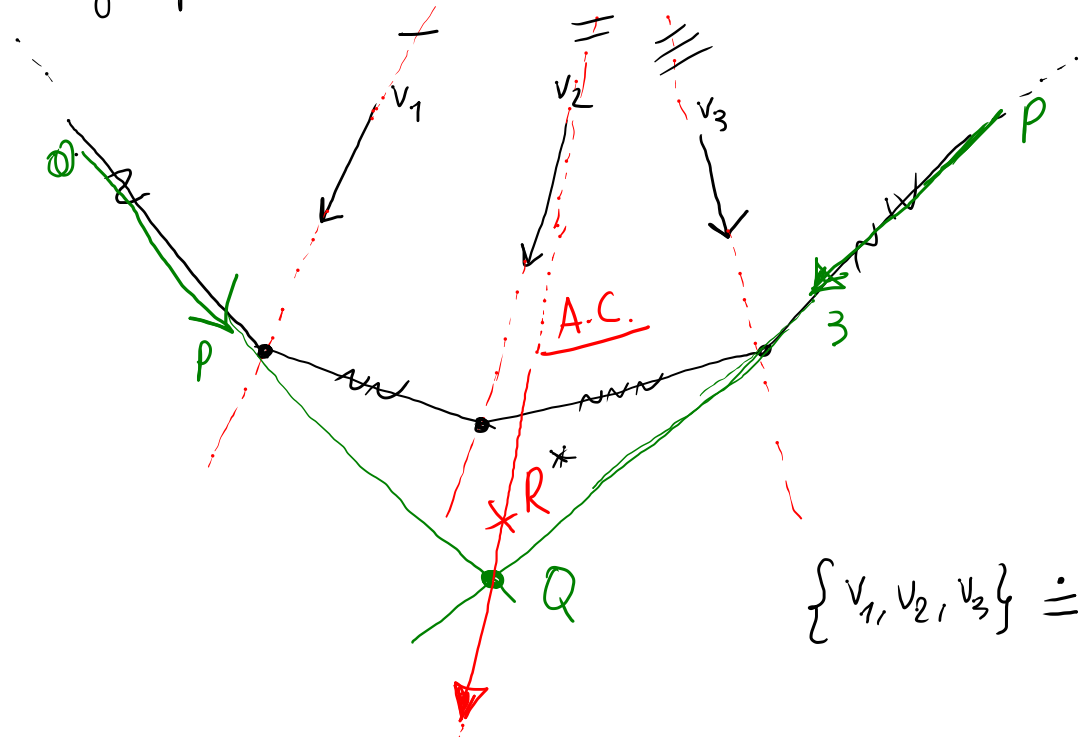


SIST. EQUILIBRATI NOTEVOLI ($R = 0, M_p = 0$ t.p.)

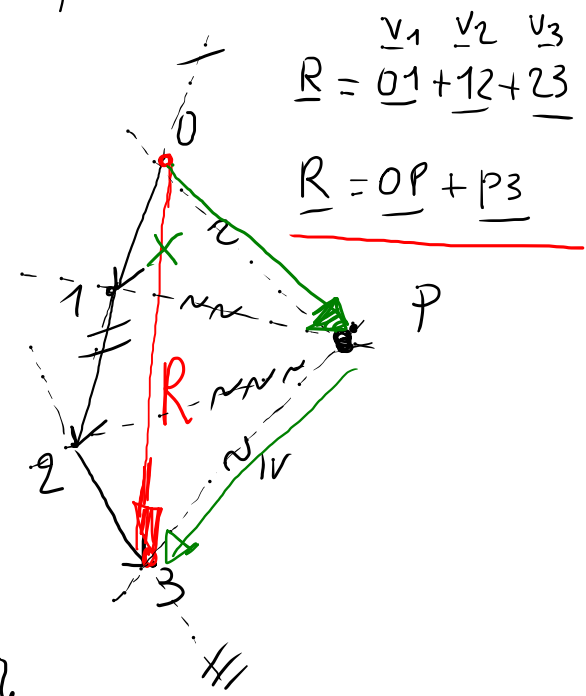


POLIGONO FUNICOLARE (UNA COSTRUZIONE GRAFICA NOTEVOLE)

Assegnato un sist piano di vettori, il P.F. permette di determinare graficamente l'A.C.



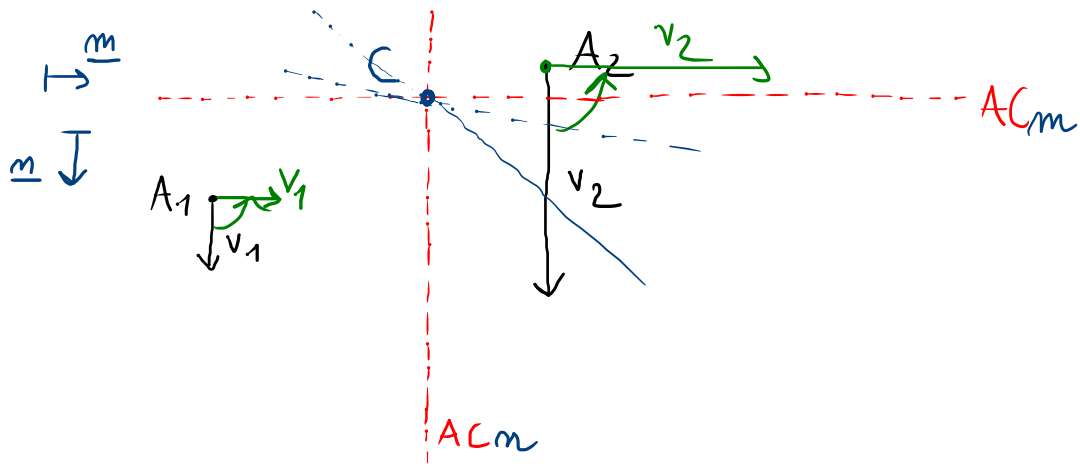
$$\{v_1, v_2, v_3\} \doteq \{R\}$$



$$\underline{R} = \underline{v_1} + \underline{v_2} + \underline{v_3}$$

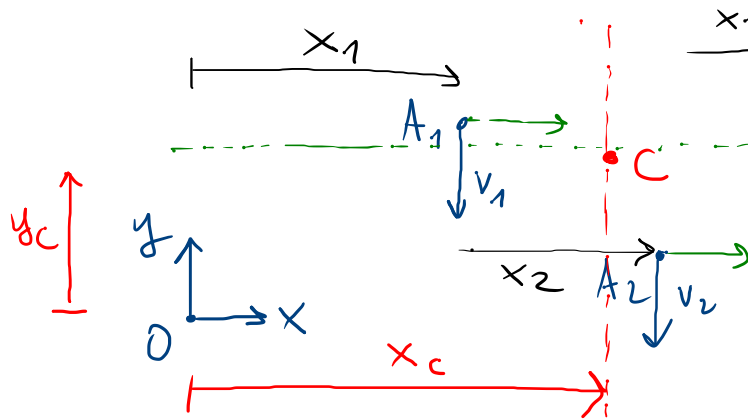
$$\underline{R} = \underline{OP} + \underline{PQ}$$

CENTRO DI UN SISTEMA DI VETTORI PARALLELI



C: "CENTRO" DEL SISTEMA
ASSEGNATO $\{v_1, v_2\}$

Nelle pag. successive determinerò le coord. di C rispetto ad un sist. Cartesiano.



$$\Sigma = \{v_1, v_2, v_3\}; \quad \Sigma' = \{R\}$$

$$R = v_1 + v_2 + v_3$$

$$\odot M_O^\Sigma = M_O^{\Sigma'}$$

INCOGNITA

$$-v_1 x_1 - v_2 x_2 - v_3 x_3 = -R x_c$$

$$x_c = \frac{v_1 x_1 + v_2 x_2 + v_3 x_3}{R} = \frac{\sum_{i=1}^m v_i x_i}{\sum_{i=1}^m v_i}$$

$$A_i = (x_i, y_i)$$

$$i = 1, 2, 3$$

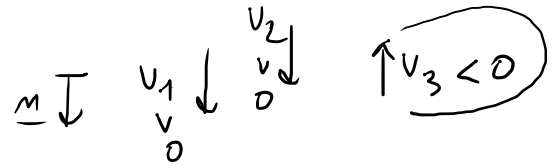
Ripeto l'esempio con y_c :

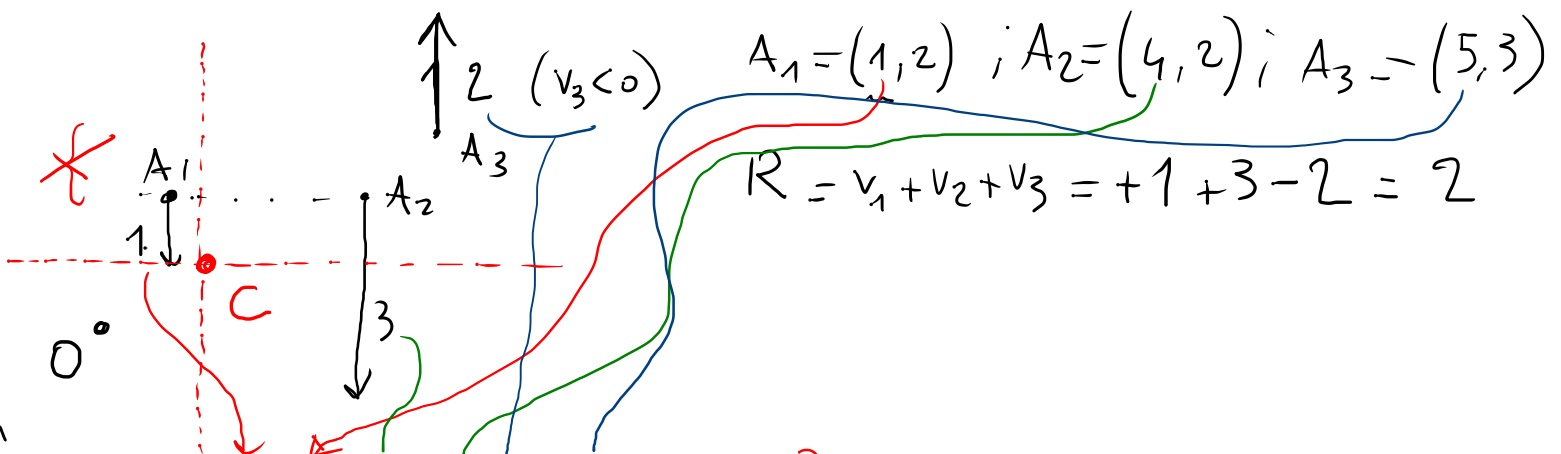
COORDINATE DI

C

$$v_i \geq 0$$

$$y_c = \frac{\sum_{i=1}^m v_i y_i}{\sum_{i=1}^m v_i}$$





$$x_c = \frac{\sum_{i=1}^m v_i x_i}{\sum_{i=1}^m v_i} = \frac{1 \cdot 1 + 3 \cdot 4 - 2 \cdot 5}{2} = \frac{3}{2}$$

$$y_c = \frac{\dots y_i}{\dots} = \frac{1 \cdot 2 + 3 \cdot 2 - 2 \cdot 3}{2} = 1$$