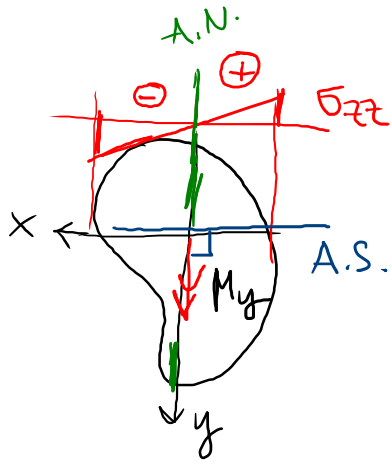
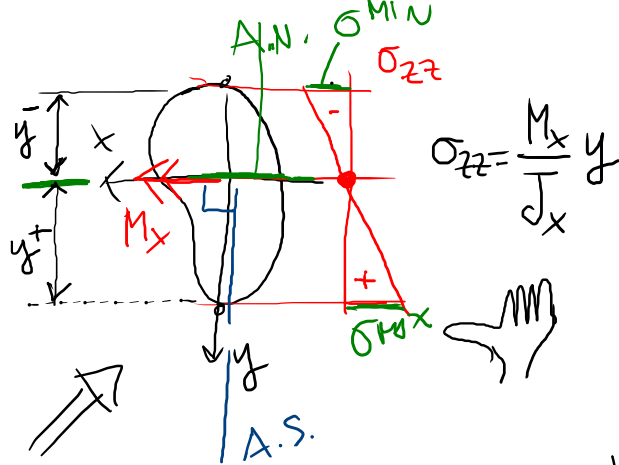


... FLESSIONE RETTA



$$\sigma_{zz} = -\frac{M_y}{J_y} x$$

23/11/22

A.N. : ASSE NEUTRO
 $\sigma_{zz} = 0$

A.S. : ASSE DI SOLLECITAZIONE
 A.S. $\perp$ M

$\sigma_{zz}^{MAX}, \sigma_{zz}^{MIN} ?$

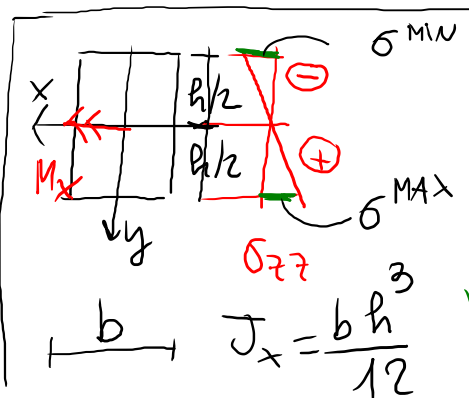
$\sigma_{zz}^{MAX} = \frac{M_x}{J_x} y^+ = \frac{M_x}{W_x^+}$

$|\sigma_{zz}^{MIN}| = \frac{M_x}{J_x} y^- = \frac{M_x}{W_x^-}$

QUOTE

W_x^+ : MODULI DI RESISTENZA
 DELLA SEZ. RISPETTO A x

$[W] = [L^3]$



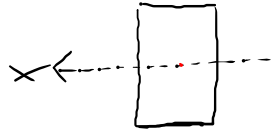
$|\sigma_{zz}^{MIN}| = \sigma_{zz}^{MAX} = \frac{M_x}{J_x} \frac{h}{2} = \frac{M_x}{\frac{bh^3}{12}} \frac{h}{2} = \frac{M_x}{W_x}$

$W_x = \frac{bh^3}{12} \cdot \frac{2}{h} = \frac{bh^2}{6}$

$W_x^+ = W_x^- = W_x$

LE MASSIME TENSIONI NORMALI DIPENDONO DA $W_x^{\pm} = \frac{J_x}{y_{\max}^{\pm}}$

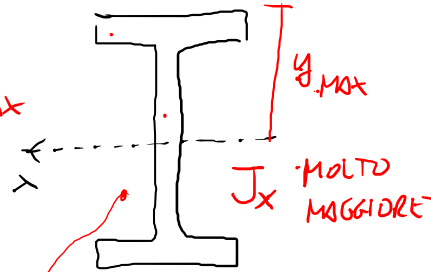
$\leftarrow M_x$



W_x MEDIO



W_x MINORE
 J_x MOLTO MINORE



W_x MAGGIORE

$W_x^+ = W_x^- = W_x$
Area costante

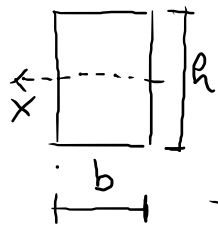
IN QUALE SEZ. CI SONO LE
 $\sigma_{\max}$ MINORI

$$\sigma_{\max} = \frac{M_x}{W_x} \sim \sigma_{\max} \propto \frac{1}{W_x}$$

$\sigma_{\max}$ MINORE

$\rightarrow$ IPE, HE

PROGETTO DI SEZ. RETTANGOLARE



b: NOTO

σ_0 : MAX TENSIONE DI PROGETTO

M_x : NOTO

$h?$

CRITERIO:

$$\sigma_{zz} \leq \sigma_0$$

$$\sigma_{zz}^{\max} = \sigma_0$$

$$\frac{M_x}{W_x} = \sigma_0$$

$$W_x = \frac{M_x}{\sigma_0}$$

$$\frac{bh^2}{6} = \frac{M_x}{\sigma_0} \Rightarrow \bar{h} = \sqrt{\frac{6M_x}{\sigma_0 b}}$$

$$\boxed{h \geq \bar{h}}$$

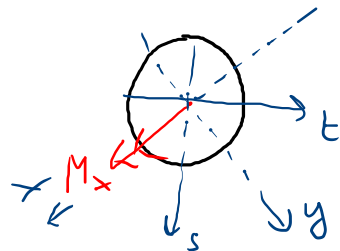
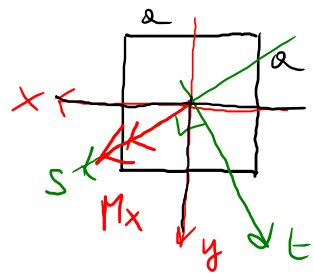
MASSIMIZZANDO J_x e

ANCHE W_x

$\bar{h}$: ALTEZZA COLGATA

$$\bar{h} = [L] ???$$

OSSERVA? (QUADRATO - CERCHIO)



$$J_x = J_y = \frac{\pi R^4}{4} = J_s = J_t$$

TUTTI GLI ASSI BARICENTRICI
SONO PRINCIPALI

$$J_x = J_y = \frac{a^4}{12} = J_s = J_t$$

TUTTI I PROBLEMI
DI FLESSIONE NEL
QUADRATO E NEL
CERCHIO SONO "FLESSIONE RETTA"
PERCHÉ TUTTI GLI ASSI BARICENTRICI
SONO PRINCIPALI.

DEFORMATA DELLA LINEA D'ASSE DEL CILINDRO SOLLECITATO A FLESSIONE RETTA

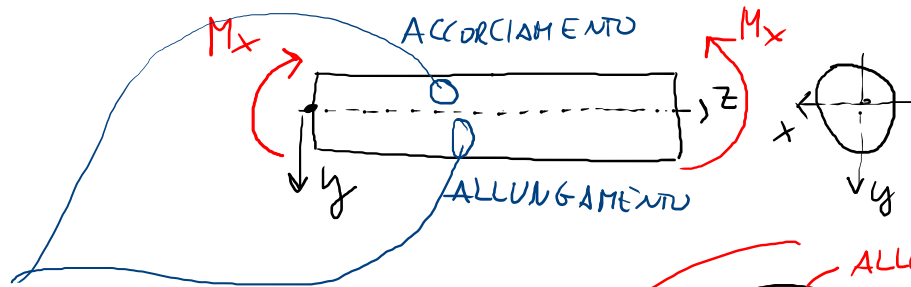
IN TUTTO IL CILINDRO

$$\begin{bmatrix} \tilde{0} \\ \tilde{0} \end{bmatrix} = \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & \frac{M_x}{J_x} y \end{bmatrix}$$

$$\underline{u} = \begin{bmatrix} u_x \\ u_y \\ u_z \end{bmatrix}$$

$$\varepsilon_{zz} = \frac{\sigma_{zz}}{E} = \frac{M_x}{J_x E} y$$

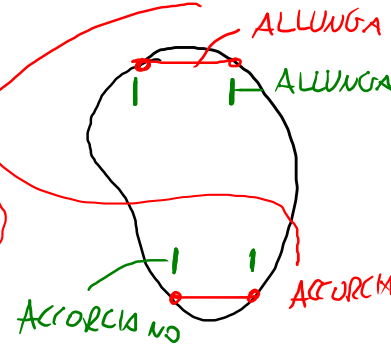
$$\varepsilon_{xx} = \varepsilon_{yy} = -\nu \frac{\sigma_{zz}}{E} = -\nu \frac{M_x}{E J_x} y$$



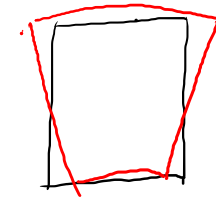
$$\varepsilon_{xx} > 0, y < 0$$

$$\varepsilon_{yy} > 0, y < 0$$

POCO
IMPORTANTE



EFFETTO
DELL' ε_{xx}



$$u_{z,z} = \varepsilon_{zz} = \frac{M_x y}{E J_x}$$

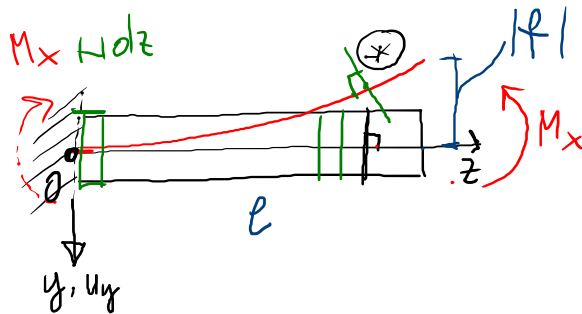
$$u_{x,x} = u_{y,y} = -\nu \frac{M_x}{E J_x} y$$

PROBLEMA DIFFERENZIALE
CHE POSSO INTEGRARE
PER DETERMINARE IL
CAMPO DI SPOSTAMENTI.

$$\underline{u}(x, y, z)$$

SPOSTAMENTO ASSE: $u_y(0, 0, z)$

M. CAPURSO
INTEGRAZIONE



$$u_y(0,0,z) = -\frac{M_x}{2EJ_x} z^2$$

EQ. LINEA D'ASSE DEFORMATA

$$f = u_y(0,0,l) = \frac{M_x}{2EJ_x} l^2 \ll l$$

PICCOLI SPOSTAMENTI

$$\chi = \frac{d\varphi}{ds} \approx \frac{d\varphi}{dz} = \frac{M_x}{EJ_x}$$



piccoli spostamenti

$$\frac{du_y(z)}{dz} = -\frac{M_x}{EJ_x} z = \tan \alpha \approx \alpha$$

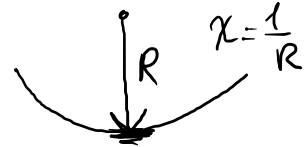


COME SI DEFINISCE LA CURVATURA DI UNA FUNZIONE

$$\chi = \frac{y''(x)}{(1+y'(x)^2)^{3/2}} \quad \text{per una funzione } y(x)$$

se $y(x)$ è molto vicino all'asse x

$$\Rightarrow \chi \approx y''(x)$$

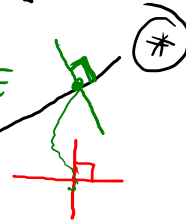


$$\chi \approx \frac{d^2 u_y(z)}{dz^2} = -\frac{M_x}{EJ_x}$$

$$\frac{1}{R} \Rightarrow [\chi] = \left[\frac{1}{L} \right]$$

LA SEZ. RETTA RUOTA E ALLE ASSE RIMANE + ALL'ASSE

DEFORMATO



NEL PROBLEMA DI FLESSIONE RETTA LA CURVATURA È COSTANTE E VALE

$$|\chi| = \frac{M_x}{EJ_x} \quad \text{CURVATURA FLESSIONALE}$$

EJ_x : COEFFICIENTE DI RIGIDEZZA FLESSIONALE

OSSERVAZIONE

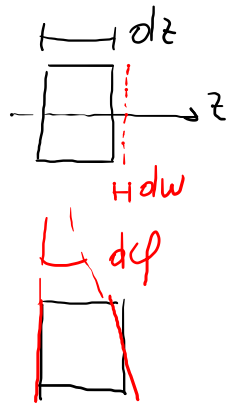
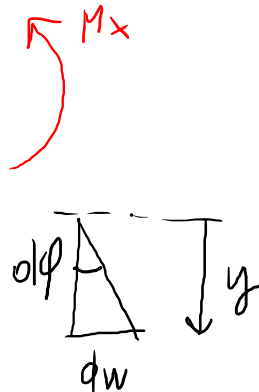
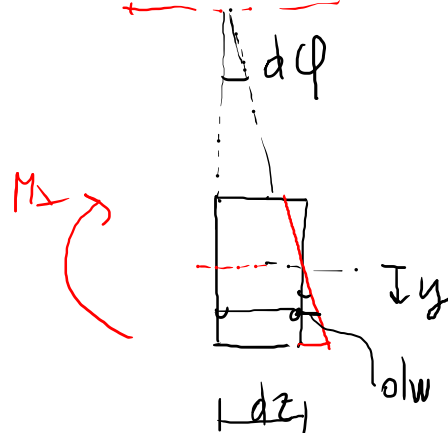
• NEL PROBL. DI FORZA NORMALE :

•

FLESSIONE RETTA

$$\chi = \frac{M_x}{EJ_x} = \frac{d\varphi}{dz}$$

$$[X] = \frac{[FL]}{[FL^{-2}L^4]} = \left[\frac{1}{L} \right]$$



$$\frac{dw}{dz} = \frac{N}{EA}$$

COEFFICIENTE DI RIGIDEZZA ASSIALE

$$\frac{d\varphi}{dz} = \frac{M_x}{EJ_x}$$

φ: ADIMENSIONALE

$$\epsilon_{zz} = \frac{\sigma_{zz}}{E} = \frac{M_x}{EJ_x} y : \text{lineare in } y$$

$$d\varphi = \frac{dw}{y} = \frac{\epsilon_{zz} dz}{y} = \frac{M_x}{EJ_x} y dz \cdot \frac{1}{y} = \frac{M_x}{EJ_x} dz$$

$$\boxed{\frac{d\varphi}{dz} = \chi = \frac{M_x}{EJ_x}}$$

TEORIA DELL'ELASTICITÀ