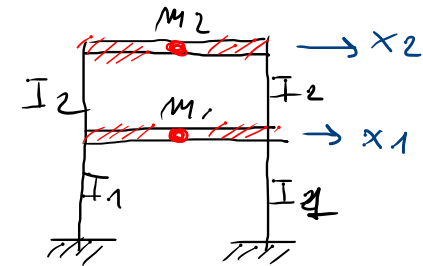


# OSCILLAZ. LIBERE DI UN SIST. A 2 GDL

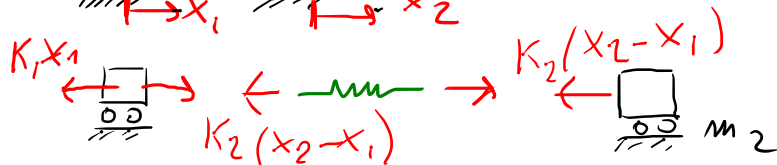
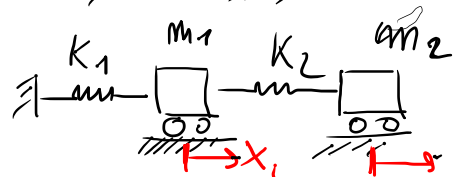
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MASSE CONCENTRATE.  
AL PIANO

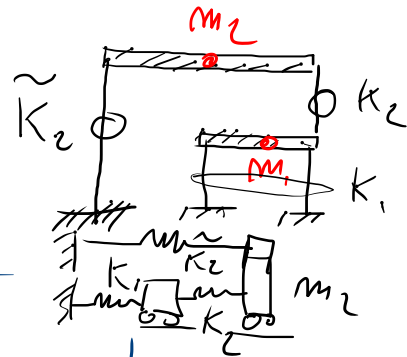
SCHEMA CHE VOGLIAMO  
MODELARE CON 2 GDL

$$x_2 > x_1$$



$$\begin{cases} m_1 \ddot{x}_1 = +K_2(x_2 - x_1) - K_1 x_1 \\ m_2 \ddot{x}_2 = -K_2(x_2 - x_1) \end{cases}$$

$$\begin{cases} m_1 \ddot{x}_1 + K_1 x_1 - K_2(x_2 - x_1) = 0 \\ m_2 \ddot{x}_2 + K_2(x_2 - x_1) = 0 \end{cases}$$



$\underline{M}$ : MATRICE DELLE  
MASSE

$\underline{K}$ : MATRICE DI RIGIDITÀ / m.

$$\begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix} \begin{bmatrix} \ddot{x}_1 \\ \ddot{x}_2 \end{bmatrix} + \begin{bmatrix} K_1 + K_2 & -K_2 \\ -K_2 & K_2 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

$$\underline{M} \underline{\ddot{x}} + \underline{K} \underline{x} = \underline{0}$$

NOTAZIONE MATRICIALE CHE  
GENERALIZZA L'EQUAZ.

$$M \ddot{x} + K x = 0$$

(OSCILLAZ. LIBERE)

CERCO SOLUZIONI NEL TIPO :  $\begin{cases} x_1 = U_1 \sin(\omega t) \\ x_2 = U_2 \sin(\omega t) \end{cases}$  <sup>ADIM.</sup>  $\Leftrightarrow \begin{cases} x_1 = D_1 e^{it} \\ x_2 = D_2 e^{-it} \end{cases}$

$\omega??$   $\left[ \omega = \sqrt{\frac{k}{m}} \right]$   
 $\left[ \frac{1}{s} \right]$  OSC. SEMPL.

$$\underbrace{[m_1 U_1 (-\omega^2) + K_1 U_1 + K_2 (U_1 - U_2)]}_{0 \text{ (}\forall t)} \sin \omega t = 0 \quad \nearrow \neq 0$$

$$\underbrace{[m_2 U_2 (-\omega^2) + K_2 (U_2 - U_1)]}_{0 \text{ (}\forall t)} \sin \omega t = 0 \quad \searrow \neq 0$$

$$\begin{cases} -m_1 U_1 \omega^2 + K_1 U_1 + K_2 (U_1 - U_2) = 0 \\ -m_2 U_2 \omega^2 + K_2 (U_2 - U_1) = 0 \end{cases}$$

SIST. OMOGENEO  
 CHE AMMETTE  
 SOLUZ. SE  
 IL DET DELLA  
 MATRICE ASSOCIATA  
 È NULLO.

$$\begin{vmatrix} K_1 + K_2 - m_1 \omega^2 & -K_2 \\ -K_2 & K_2 - m_2 \omega^2 \end{vmatrix} = 0 \quad \textcircled{\times}$$

⇒ PROBL. AGLI AUTOVALORI  
 GENERALIZZ. ( $\omega^2$ ) →  $\omega_1^2$   
 (DUE SOLUZ.)  $\searrow \omega_2^2$

$$\left[ \text{IN TERMINI MATRICIALI:} \right. \\ \left. \left| \begin{matrix} K \\ \sim \end{matrix} - \omega^2 \begin{matrix} M \\ \sim \end{matrix} \right| = 0 \right]$$

PROBLEMA  
 AUTOVALORI  
 STANDARD  
 $|A - \lambda I|$

$$\det \begin{vmatrix} & \\ & \end{vmatrix} = 0 \Rightarrow \omega^4 - \left( \frac{k_1 + k_2}{m_1} + \frac{k_2}{m_2} \right) \omega^2 + \frac{k_1 k_2}{m_1 m_2} = 0 \quad \text{BIQUADRATICA}$$

$$\omega_{1,2}^2 = -b \pm \sqrt{b^2 - 4ac} \Rightarrow \text{NORMALMENTE SI ORDINANO IN MODO CHE } \omega_1^2 < \omega_2^2 \Rightarrow \omega_1, \omega_2 \in \mathbb{R}$$

$\omega_1$ : PULSAZIONE FONDAMENTALE DEL SISTEMA (NATURALE)

$T_1 = \frac{2\pi}{\omega_1}$ ; PERIODO " " " "  $(T_1 > T_2)$

$$\left( T_2 = \frac{2\pi}{\omega_2} \right)$$

SIGNIFICATO DEGLI AUTOVETTORI

• per ogni autovalore ( $\omega_i^2$ )  $\Rightarrow$  UN AUTOVETTORE  $U^{(i)}$

$\omega_1^2 \Rightarrow U^{(1)}$  \ MODI DI VIBRARE DEL SISTEMA.  
 $\omega_2^2 \Rightarrow U^{(2)}$

CALCOLIAMO  $U^{(1)}$

$$\begin{cases} [(K_1 + K_2) - \omega_1^2 m_1] U_1^{(1)} - K_2 U_2^{(1)} = 0 \\ -K_2 U_1^{(1)} + [K_2 - \omega_1^2 m_2] U_2^{(1)} = 0 \end{cases} \Rightarrow U_1^{(1)}, U_2^{(1)}$$

SIST. LINEARE M.  
DIPENDENTE

$$\Rightarrow U_2^{(1)} = f(U_1^{(1)})$$

GLI AUTOVETTORI SONO A MENO DI  
UNA COSTANTE

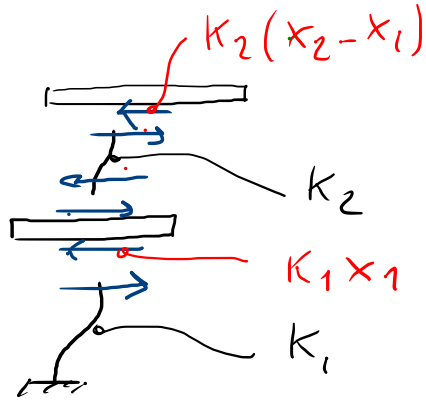
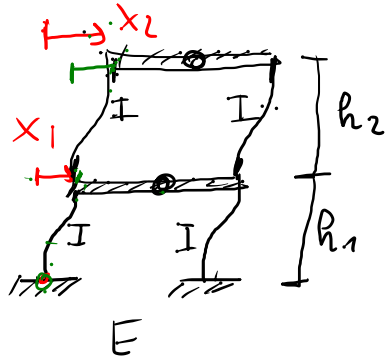
SOLO UNA EQUAZ. È SIGNIFICATIVA

SE VOGLIO CALCOLARE  $U^{(2)}$

$$\begin{cases} \dots \omega_2^2 \dots U_1^{(2)} \dots U_2^{(2)} = 0 \\ \dots U_1^{(2)} \dots \omega_2^2 \dots U_2^{(2)} = 0 \end{cases}$$

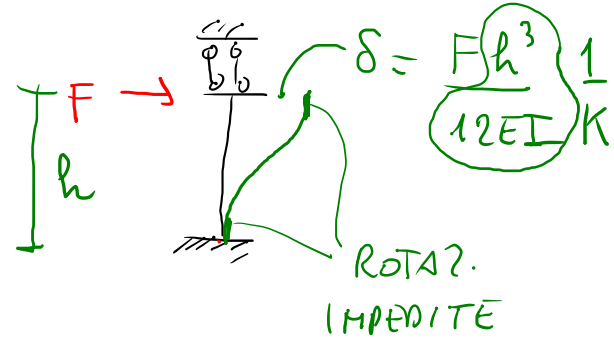
$$\Rightarrow U_2^{(2)} = g(U_1^{(2)})$$

ESEMPIO



$$K_2 = 2 \cdot \frac{12EI}{h_2^3}$$

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$h_1 = h_2 = h$  ;  $K_1 = K_2 = K = 24 \frac{EI}{h^3}$  ;  $m_1 = m_2 = m$

$$\omega_1 = 0.618 \sqrt{\frac{K}{m}}$$

$$T_1 = 10.166 \sqrt{\frac{m}{K}}$$

$$\omega_2 = 1.618 \sqrt{\frac{K}{m}}$$

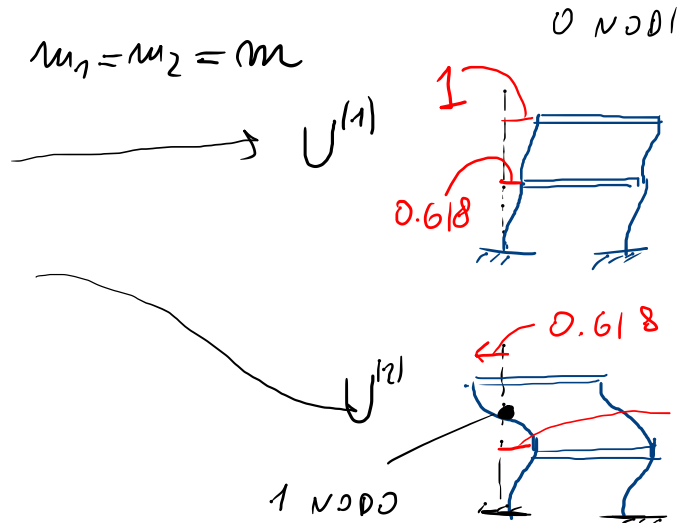
$$T_2 = 3.88 \sqrt{\frac{m}{K}}$$

$$U^{(1)} = \begin{bmatrix} 0.618 \\ 1 \end{bmatrix}$$

$$U^{(2)} = \begin{bmatrix} 1 \\ -0.618 \end{bmatrix}$$

I MODI DI VIBRAZIONE

II MODI DI VIBRAZIONE



NORMALIZZATI

$$\max |U_j^{(i)}| = 1$$

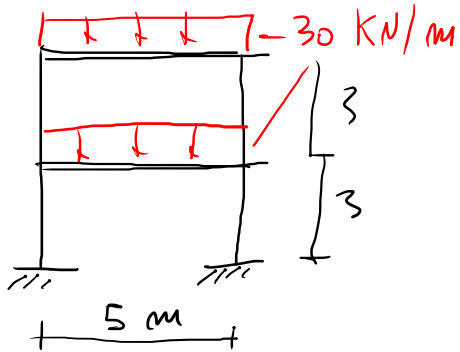
E) NUMERICO

$$E = 20 \text{ GPa (CLS)}$$

$$I = \frac{30^4}{12} = 67500 \text{ cm}^4$$



$$h = 3 \text{ m}$$



$$T_1 = 0.36 \text{ s} \quad \text{'PERIODO FONDAMENTALE'}$$

$$T_2 = 0.14 \text{ s}$$

$$m = \frac{30000 \cdot 5}{g} \approx 15000 \text{ Kg}$$