

EXAMPLE CALCULATION OF STATIC FORCES

Example 1. Compute force and torque, per unit width, on the cylindrical body hinged in A. Consider $H=2\text{m}$; $h=5\text{m}$; $R=1\text{m}$; $\gamma=10^4\text{N/m}^3$.

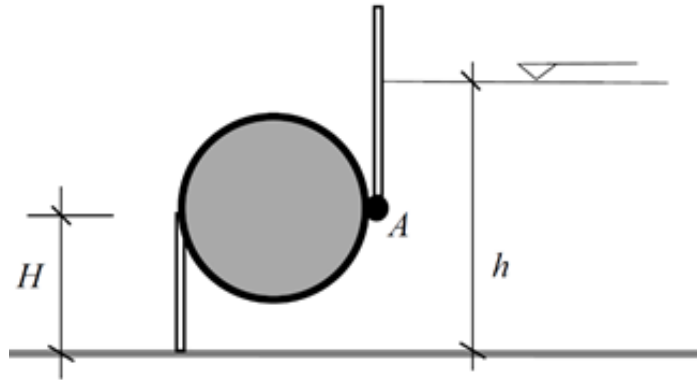


Figure.1. Sketch of the static system in example 1.

Solution:

$$F_H = 0 ;$$

$$F_V = \gamma V + \gamma(h - H)2R = \gamma V + \gamma(h - H)2R = \gamma \left(\frac{\pi}{2} R^2 + 2R(h - H) \right) = 7.57 \cdot 10^4 \frac{\text{N}}{\text{m}} ;$$

$$T_A = F_V \cdot R .$$

Example 2. Compute force and torque, per unit width, for the entire surface, made of half a circle plus a straight part, hinged in A. Consider: $h=6\text{m}$; $R=2\text{m}$; $\gamma=10^4\text{ N/m}^3$.

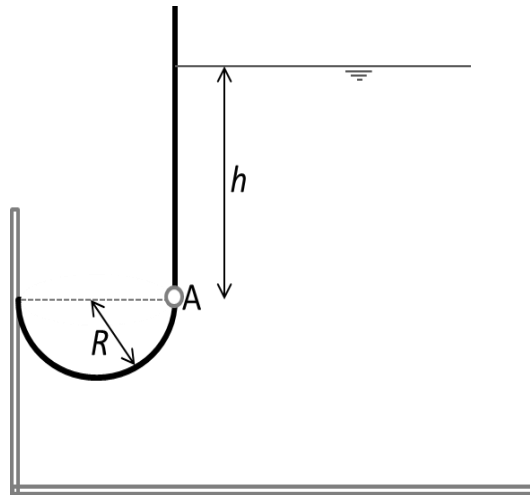


Figure 2. Sketch of the static system in example 2.

Solution:

$$F_H = \gamma \frac{h^2}{2} = 18 \cdot 10^4 \text{N}, \quad r_H = \frac{h}{3} = 2\text{m} ;$$

$$F_V = \gamma \left(2hR + \frac{\pi}{2} R^2 \right) = 30.3 \cdot 10^4 \text{N}, \quad r_V = R = 2\text{m} ;$$

$$T_A = -F_H r_H + F_V r_V = 24.6 \cdot 10^4 \text{J} .$$

Example 3. Compute force and torque, per unit width, for the surface made of two semicircles, hinged in upper edge A.

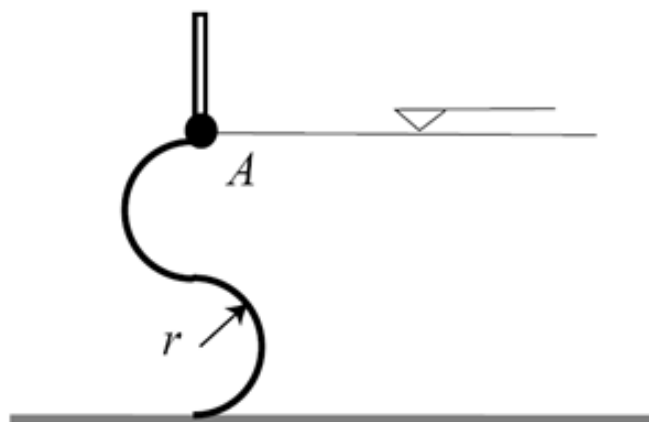


Figure **Errore. Nel documento non esiste testo dello stile specificato.**3. Sketch of the static system in example 3.

Solution:

$$F_H = \gamma 8r^2, \quad r_H = \frac{2}{3}4r, \quad F_V = \gamma \frac{\pi}{2}r^2 - \gamma \frac{\pi}{2}r^2 = 0;$$

notice that F_V is a force couple with arm $r_V = \frac{4}{3\pi}r$.

$$T_A = \gamma 8r^2 \cdot \frac{2}{3}4r - \gamma \pi r^2 \cdot \frac{4}{3\pi}r = 20 \gamma r^3.$$

Example 4. Define the volume V of the sphere, of negligible weight, such that its buoyancy is as strong as the downward force on the lower surface of area A .

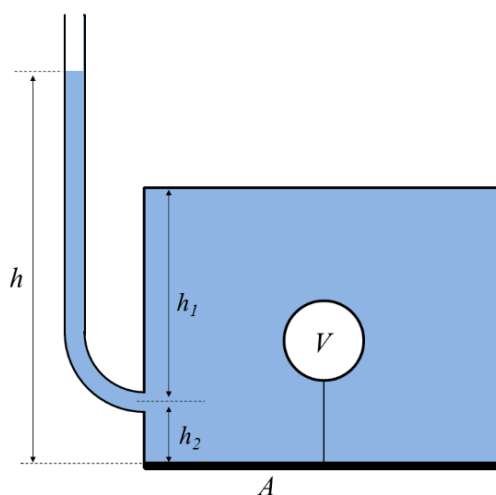


Figure 4. Sketch of the static system in example 4.

Solution:

$$\gamma h A = \gamma V, \quad V = h A .$$

Example 5. A plane surface, made of homogeneous material of negligible thickness, and hinged in point C, separates two reservoirs. Compute the weight W of the surface (per unit of width) to ensure that it remains in equilibrium. Consider $H_A=1.0$ m, $H_B=2.0$ m.

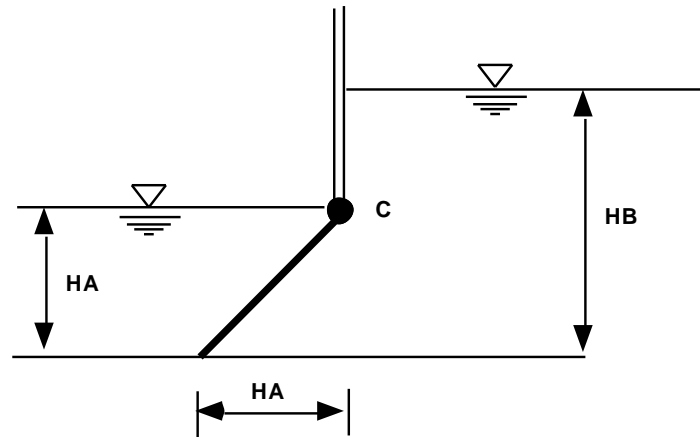


Figure 5. Sketch of the static system in example 5.

Solution:

Fluid torque is

$$F = \gamma(HB - HA)HA\sqrt{2}, \quad r = HA \frac{\sqrt{2}}{2}, \quad T_f = F \cdot r = \gamma(HB - HA)HA^2 ;$$

the same result could be found considering separately the left and right sides of the surface

$$F_L = \gamma HA^2 \frac{\sqrt{2}}{2}, \quad r = \frac{2\sqrt{2}}{3} HA, \quad M_L = \gamma \frac{2}{3} HA^2 ;$$

$$F_{R1} = \gamma(HB - HA)HA\sqrt{2}, \quad r = \frac{\sqrt{2}}{2} HA, \quad M_{R1} = \gamma(HB - HA)HA^2 ;$$

$$F_{R2} = \gamma HA^2 \frac{\sqrt{2}}{2}, \quad r = \frac{2\sqrt{2}}{3} HA, \quad M_{R2} = \gamma \frac{2}{3} HA^2 ;$$

noticing that F_L and F_{R2} cancel each other.

Solid weight torque is

$$T_s = W \frac{HA}{2} .$$

Equilibrium

$$W = 2\gamma(HB - HA)HA = 20 \text{ KN} .$$

Example 6. Evaluate the weight of the cover such that it equilibrates the force exerted by the fluid from below. Consider $\gamma_{\text{water}}=10^4$ N/m³ e $\gamma_{\text{oil}}=6800$ N/m³; let heights be $h_1=2$ m e $h_3=36$ cm, and diameters $D=5$ m, $d=20$ cm.

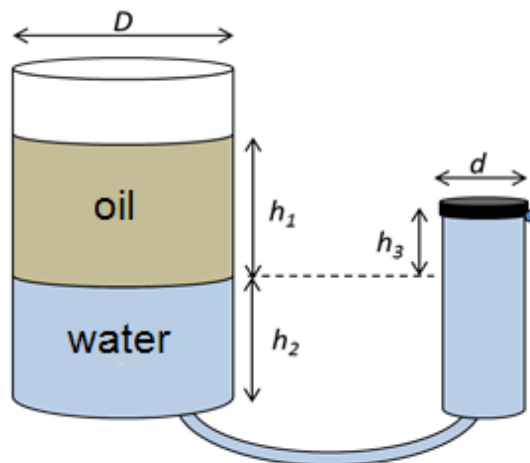


Figure 6. Sketch of the static system in example 6.

Solution:

$$F = (\gamma_{\text{oil}}h_1 - \gamma_{\text{water}}h_3)\pi \frac{d^2}{4} = 314\text{N}.$$

Example 7. Compute force and torque relative to the hinge in A , per unit width, on the surface of the object, made of half a cylinder of radius $R=2\text{m}$ with a cylindrical cavity of radius $r = R/4$. Consider the fluid of specific gravity $\gamma=10^4 \text{ N/m}^3$.

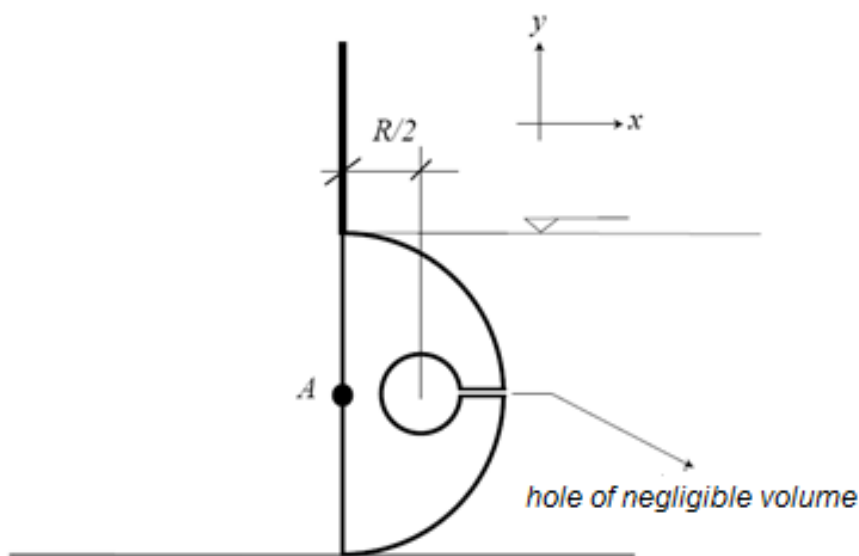


Figure 7. Sketch of the static system in example 7.

Solution:

$$F_x = -\gamma 2R^2, \quad F_y = \gamma \left(\pi \frac{R^2}{2} - \pi r^2 \right).$$

Pressure on the outer cylindrical surface acts radially and gives no momentum. Momentum about A is only due to the pressure on the inner cylinder surface:

$$T_A = \gamma \pi r^2 \frac{R}{2}.$$

Example 8. Compute the torque acting on the rectangular cover of length $d=50\text{cm}$, and unitary width. Consider the specific weigh of fluid $\gamma=10^4 \text{ N/m}^3$ and that of mercury $\gamma_m=130 \text{ N/dm}^3$, in the differential manometer whose reading is $\Delta=40 \text{ mm}$. Consider the following dimensions $h_1=1.12\text{m}$, $h_2=62\text{cm}$, $a=120\text{cm}$, $b=50\text{dm}$.

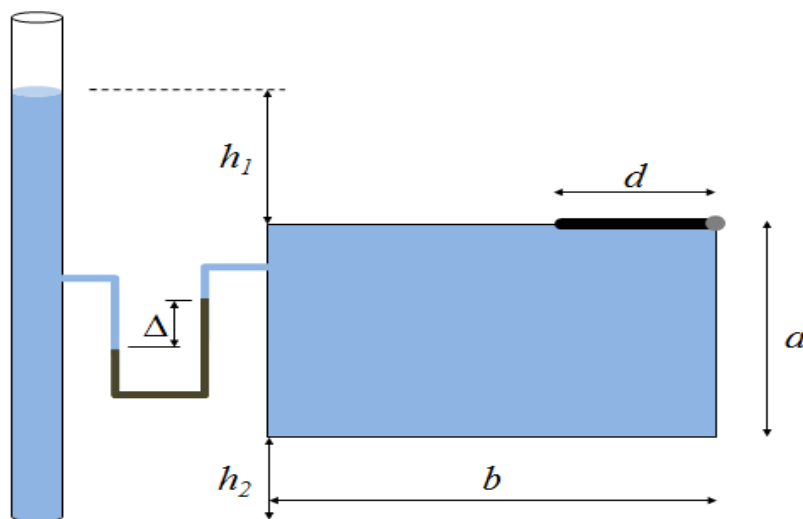


Figure 8. Sketch of the static system in example 8.

Solution:

$$h = h_1 - \left(\frac{\gamma_m - \gamma}{\gamma} \right) \Delta = 0.64\text{m}, \quad F = \gamma h d = 3.2\text{KN}, \quad T = F \frac{d}{2} = 800\text{J}.$$

Example 9. Compute the tilting moment on the structure. Consider $\gamma=10^4 \text{ N/m}^3$ and measures $h=6\text{m}$, $b=50 \text{ cm}$, $B=2\text{m}$.

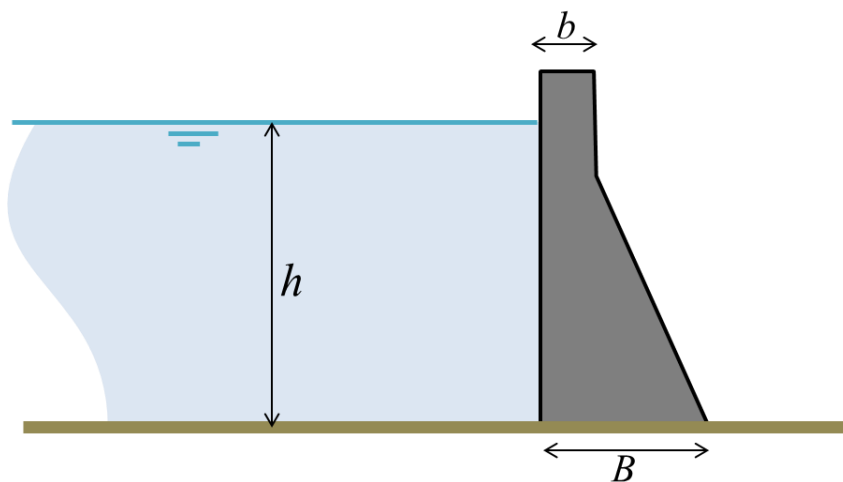


Figure 9. Sketch of the static system in example 9.

Solution:

$$M = \gamma \frac{h^2}{2} \frac{h}{3} = \gamma \frac{h^3}{6} = 3.6 \cdot 10^5 \text{ J}.$$

Example 10. Compute the torque, relative to the basis, made by the fluid on the oblique septum. Geometric measures are $a=2\text{m}$, $b=4\text{m}$, $e=20\text{cm}$. The fluid specific gravity is $\gamma=9810\text{N/m}^3$, pressure in the gas chamber is $P_1=10^4\text{Pa}$ at height $h_1=1\text{m}$.

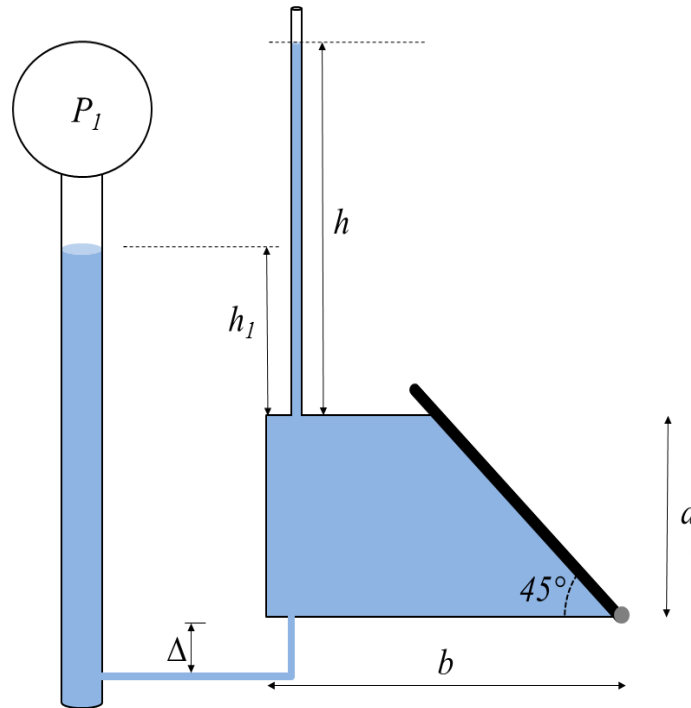


Figure 10. Sketch of the static system in example 10.

Solution:

$$h = h_1 + \frac{P_1}{\gamma} = 2.02\text{m}, \quad F = \gamma \left(h + \frac{a}{2} \right) a \sqrt{2} = 83.78\text{KN},$$

$$\zeta_c = \frac{2(h+a)^3 - h^3}{3(h+a)^2 - h^2} = 3.13\text{m}, \quad r = (h+a - \zeta_c) \sqrt{2} = 1.258\text{m}, \quad T = F \cdot r = 105\text{KJ}.$$

Torque could be evaluated by considering separately the contribution of the square and triangular distribution of pressure

$$F_{sq} = \gamma h a \sqrt{2} = 56\text{KN}, \quad T_{sq} = F_{sq} \frac{a}{2} \sqrt{2} = 79\text{KJ},$$

$$F_{tr} = \gamma \frac{a}{2} a \sqrt{2} = 28\text{KN}, \quad T_{tr} = F_{tr} \frac{a}{3} \sqrt{2} = 26\text{KJ}, \quad T = T_{sq} + T_{tr} = 105\text{KJ}.$$

Example 11. Define the width X of the base such that the surface is in equilibrium to rotation with respect to the rightmost edge.

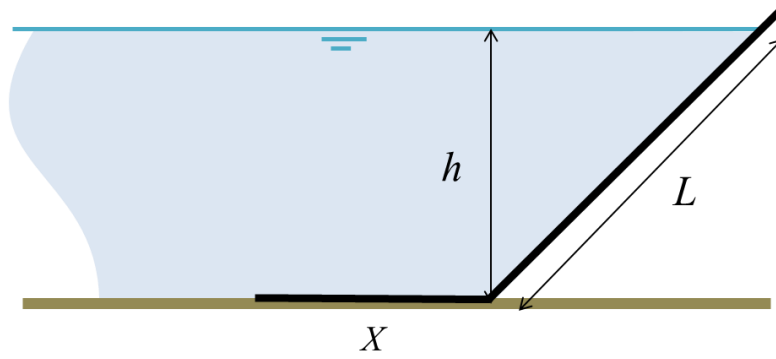


Figure 11. Sketch of the static system in example 11.

Solution:

$$\gamma \frac{hL}{2} = \gamma hX, \quad X = \frac{L}{\sqrt{3}}.$$

Example 12. Compute the force on the hemisphere at the base of bowl. Assume $\gamma=9810 \text{ N/m}^3$ and pressure on the upper gas $P_0=9810\text{Pa}$; fluid height is $h=2.2\text{m}$ and radius at the base $R=1\text{m}$.

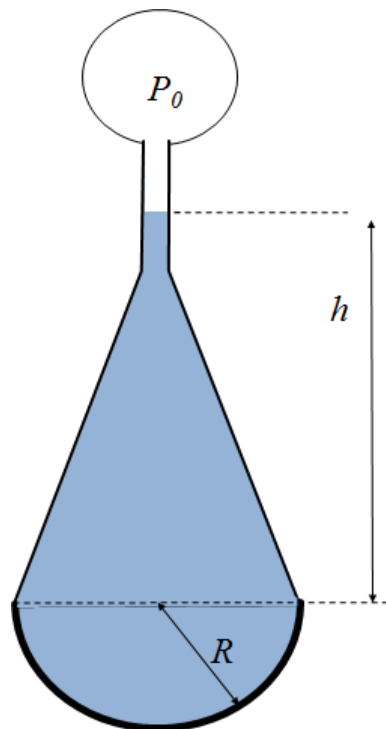


Figure 12. Sketch of the static system in example 12.

Solution:

$$H = h + \frac{P_0}{\gamma} = 3.2\text{m}, \quad F = \gamma H \pi R^2 + \gamma \frac{2}{3} \pi R^3 = 120\text{KN}.$$

Example 13. Consider the rigid surface, made of a horizontal wall of width L and a vertical wall of height h . Compute the value of the former such that it is in equilibrium to rotation around the hinge A.

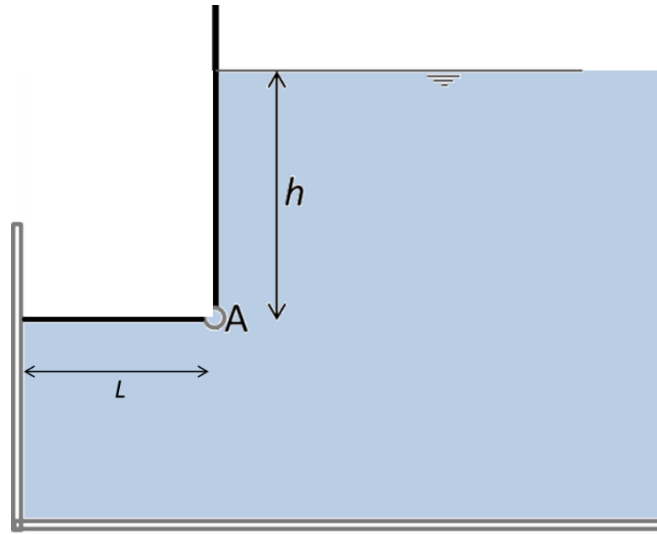


Figure 13. Sketch of the static system in example 13.

Solution:

$$\gamma \frac{h^3}{6} = \gamma h \frac{L^2}{2}, \quad L = \frac{h}{\sqrt{3}}.$$

Example 14. Compute the torque about the hinge in A for the surface made of two rectangular walls and a central semicircle. Consider $D=6m$, and $\gamma=10^4 N/m^3$.

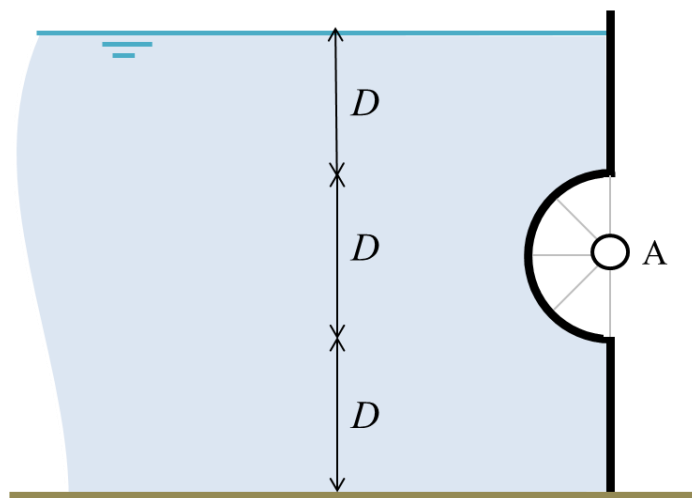


Figure 14. Sketch of the static system in example 14.

Solution:

The circular surface does not generate torque about A. The upper and lower straight walls give, respectively

$$T_u = \gamma \frac{D^2}{2} \left(\frac{D}{3} + \frac{D}{2} \right) = 9 \cdot 10^5 \text{ J};$$

$$T_l = \gamma \frac{D^2}{2} \left(\frac{2}{3} D + \frac{D}{2} \right) + \gamma 2D^2 \left(\frac{D}{2} + \frac{D}{2} \right) = 55.8 \cdot 10^5 \text{ J};$$

$$T = T_l - T_u = 46.8 \cdot 10^5 \text{ J}.$$

Example 15. Compute the ratio between vertical and horizontal components of the force made by fluid on the wall made of a rectangular wall above a semi cylindrical surface. Consider $H = D/2$.

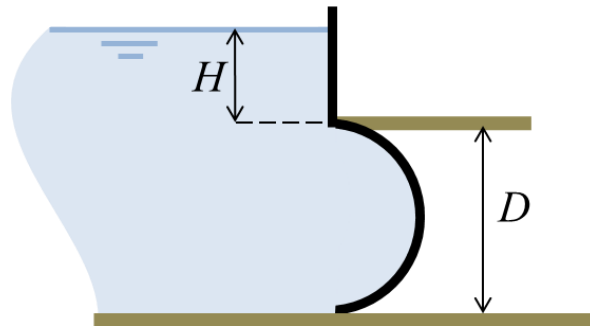


Figure 15. Sketch of the static system in example 15.

Solution:

$$F_H = \gamma \left(H + \frac{D}{2} \right) D + \gamma \frac{H^2}{2} = \frac{9}{8} \gamma D^2, \quad F_V = \gamma \pi \frac{D^2}{8}, \quad \frac{F_V}{F_H} = \frac{\pi}{9}.$$

Example 16. Compute the function $M(h)$, of the tilting moment about the hinge A, per unit width, of the oblique wall as a function of the height h . Consider the quote $a = 2h/3$ and gas pressure in the chamber $P_0 = \gamma h$.

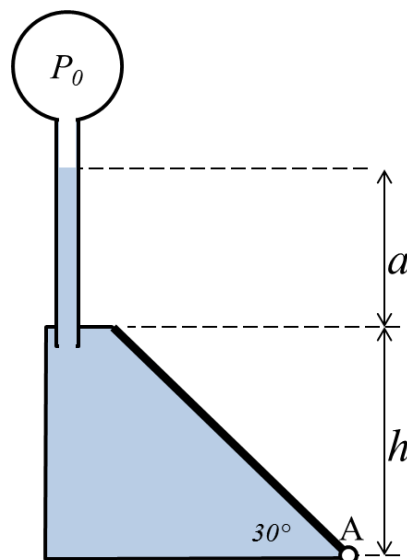


Figure 16. Sketch of the static system in example 16.

Solution:

The length of the wall is $L = \frac{h}{\sin(30^\circ)} = 2h$, and pressure at the top of the wall is equal to that given by a depth H below a free surface, where

$$H = a + \frac{P_0}{\gamma} = \frac{5}{3}h.$$

Considering the pressure distribution as the sum of square plus triangular profiles, respectively

$$\begin{aligned} F_{sq} &= \gamma HL, & M_{sq} &= \gamma HL \cdot \frac{L}{2} = \frac{10}{3}\gamma h^3, \\ F_{tr} &= \gamma h \frac{L}{2}, & M_{tr} &= \gamma h \frac{L}{2} \cdot \frac{L}{3} = \frac{2}{3}\gamma h^3; \end{aligned}$$

from which the result follows

$$M(h) = \frac{10}{3}\gamma h^3 + \frac{2}{3}\gamma h^3 = 4\gamma h^3.$$

The same result could be found considering the momentum of the entire force acting on the center C of pressure distribution

$$F = \gamma \left(H + \frac{h}{2} \right) L = \frac{13}{3}\gamma h^2, \quad \zeta_c = \frac{2(h+H)^3 - H^3}{3(h+H)^2 - H^2} = \frac{86}{39}h, \quad r = 2(H+h - \zeta_c) = \frac{12}{13}h;$$

from which the same result follows

$$M(h) = F \cdot r = \frac{13}{3}\gamma h^2 \cdot \frac{12}{13}h = 4\gamma h^3.$$

Example 17. Compute force and torque, about the hinge in A, on the vertical surface (per unit width). Consider $P_0=150\text{mmHg}$, $h=5\text{m}$, $a=2\text{m}$; and the specific gravity $\gamma=10\text{KN/m}^3$.

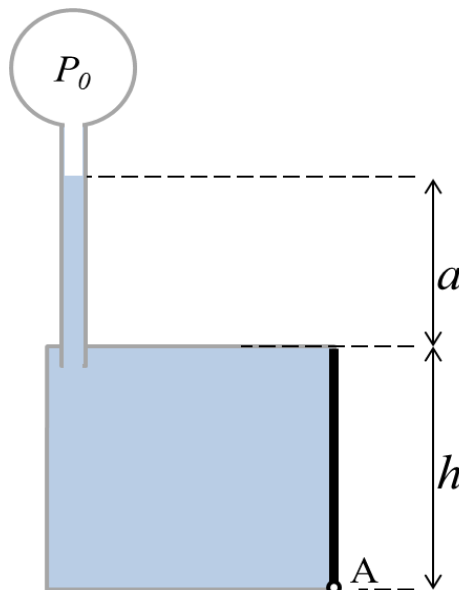


Figure 17. Sketch of the static system in example 17.

Solution:

$$P_0 = 150\text{mmHg} = 20\text{KPa} = \gamma \cdot 2\text{m} .$$

Considering the pressure distribution as the sum of square plus triangular pressure profiles, respectively

$$F = F_{sq} + F_{tr} = (P_0 + \gamma a)h + \gamma \frac{h^2}{2} = 200\text{KN} + 125\text{KN} = 325\text{KN} ,$$

$$T = F_{sq} \cdot \frac{h}{2} + F_{tr} \cdot \frac{h}{3} = 500\text{KJ} + 208\text{KJ} = 708\text{KJ} .$$

In alternative, considering the entire force at once

$$h_1 = \frac{P_0}{\gamma} + a = 4\text{m}, \quad h_2 = h_1 + h = 9\text{m}, \quad h_c = \frac{2h_2^3 - h_1^3}{3h_2^2 - h_1^2} = 6.82\text{m}, = 2.18\text{m} ;$$

$$F = \left(P_0 + \gamma \left(a + \frac{h}{2} \right) \right) h = 325\text{KN}, \quad T = F \cdot (h_2 - h_c) = 708\text{KJ} .$$

Example 18. Compute the static force acting on the semispherical surface. Consider $H=3.6\text{m}$, $R=1.6\text{m}$, $\gamma=9810\text{ N/m}^3$.

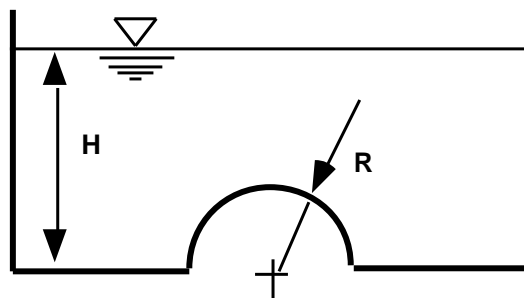


Figure 18. Sketch of the static system in example 18.

Solution:

$$F = \gamma V = \gamma \left(\pi R^2 H - \frac{2}{3} \pi R^3 \right) = 200\text{KN} .$$

Example 19. Compute the horizontal and vertical components of the force made by the fluid on the semispherical surface. Assume $H=50\text{cm}$, $D=1\text{m}$ and the fluid specific gravity $\gamma=9810\text{ N/m}^3$.

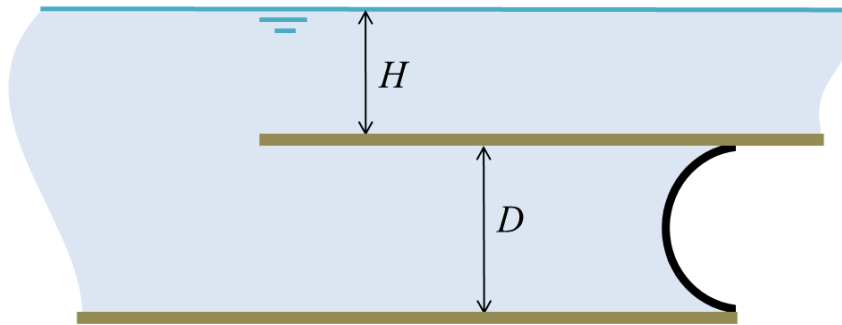


Figure 19. Sketch of the static system in example 19.

Solution:

$$F_H = \gamma \left(H + \frac{D}{2} \right) \pi \frac{D^2}{4} = 7705\text{N}, \quad F_V = \gamma \frac{2}{3} \pi \frac{D^3}{8} = 2568\text{N}.$$

Example 20. Compute the force acting on the semispherical surface at the bottom of the bowl. Consider the following relationships $\gamma_{oil} = 2/3\gamma$, $h_1 = 1.8R$, $h_2 = 0.2R$, $h_3 = 1.5R$, where $R=3\text{m}$ and $\gamma=9810\text{N/m}^3$.

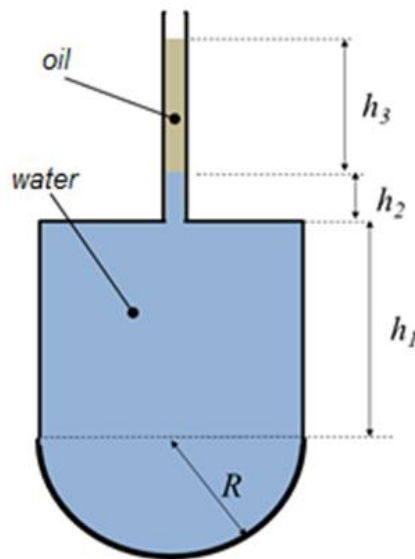


Figure .20. Sketch of the static system in example 20.

Solution:

$$h = h_1 + h_2 + \frac{\gamma_{oil}}{\gamma} h_3 = 3R, \quad F_H = 0, \quad F_V = \gamma h \pi R^2 + \gamma \frac{2}{3} \pi R^3 = \gamma \frac{11}{3} \pi R^3 = 3.06\text{MN}.$$