

Corso di Laurea in Fisica - UNITS  
Istituzioni di Fisica per il Sistema Terra

Approximate Solutions  
of the Navier-Stokes Equation

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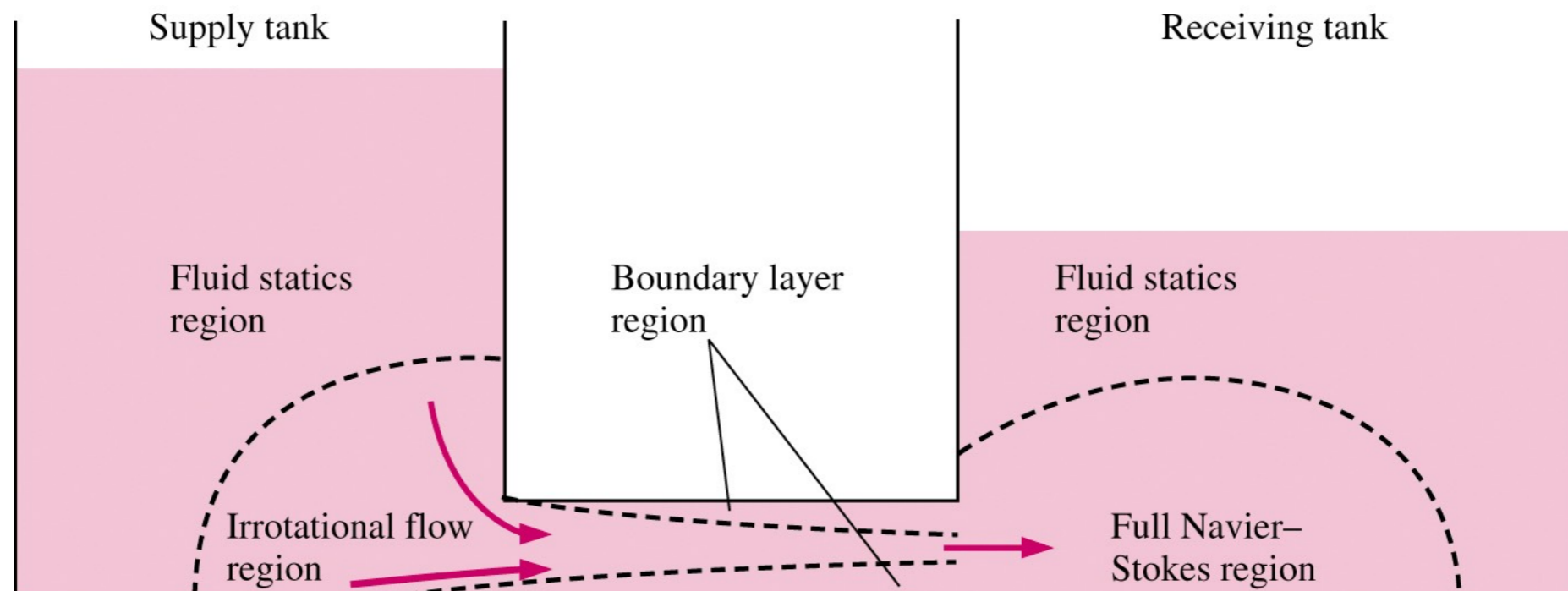
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# Objectives

- Appreciate why approximations are necessary, and know when and where to use.
- Understand effects of lack of inertial terms in the creeping flow approximation.
- Understand superposition as a method for solving potential flow.
- Predict boundary layer thickness and other boundary layer properties.

# Introduction

- We derived the NSE and developed several exact solutions.
- We will study several methods for **simplifying** the NSE, which permit use of mathematical analysis and solution.
  - An approximate solution is one in which the Navier–Stokes equation is simplified in some region of the flow before we start the solution.
  - Term(s) are eliminated a priori depending on the class of problem, which may differ from one region of the flow to another.



# Nondimensionalization of the NSE

- Purpose: Order-of-Magnitude analysis of the terms in the NSE, which is necessary for simplification and approximate solutions.
- We begin with the incompressible NSE

$$\rho \frac{D\vec{V}}{Dt} = \rho \left[ \frac{\partial \vec{V}}{\partial t} + (\vec{V} \cdot \nabla) \vec{V} \right] = -\nabla p + \rho \vec{g} + \mu \nabla^2 \vec{V}$$

- Each term is dimensional, and each variable or property ( $\rho$ ,  $V$ ,  $t$ ,  $\mu$ , etc.) is also dimensional.
- What are the primary dimensions of each term in the NSE equation?

$$\text{Answer : } \left\{ \frac{m}{L^2 t^2} \right\}$$

# Nondimensionalization of the NSE

- To nondimensionalize, we choose scaling parameters as follows

**TABLE 10–1**

Scaling parameters used to nondimensionalize the continuity and momentum equations, along with their primary dimensions

| Scaling Parameter | Description                   | Primary Dimensions                  |
|-------------------|-------------------------------|-------------------------------------|
| $L$               | Characteristic length         | {L}                                 |
| $V$               | Characteristic speed          | {Lt <sup>-1</sup> }                 |
| $f$               | Characteristic frequency      | {t <sup>-1</sup> }                  |
| $P_0 - P_\infty$  | Reference pressure difference | {mL <sup>-1</sup> t <sup>-2</sup> } |
| $g$               | Gravitational acceleration    | {Lt <sup>-2</sup> }                 |

# Nondimensionalization of the NSE

- Next, we define nondimensional variables, using the scaling parameters in Table 10-1

$$\begin{aligned} t^* &= ft & \vec{x}^* &= \frac{\vec{x}}{L} & \vec{V}^* &= \frac{\vec{V}}{V} \\ P^* &= \frac{P - P_\infty}{P_0 - P_\infty} & \vec{g}^* &= \frac{\vec{g}}{g} & \nabla^* &= L\nabla \end{aligned}$$

- To plug the nondimensional variables into the NSE, we need to first rearrange the equations in terms of the dimensional variables

$$\begin{aligned} t &= \frac{1}{f}t^* & \vec{x} &= L\vec{x}^* & \vec{V} &= V\vec{V}^* & \nabla &= \frac{1}{L}\nabla^* \\ P &= P_\infty + (P_0 - P_\infty)P^* & \vec{g} &= g\vec{g}^* \end{aligned}$$

# Nondimensionalization of the NSE

- Now we substitute into the NSE to obtain

$$\rho V f \frac{\partial \vec{V}^*}{\partial t^*} + \frac{\rho V^2}{L} \left( \vec{V}^* \cdot \nabla^* \right) \vec{V}^* = - \frac{P_0 - P_\infty}{L} \nabla^* P^* + \rho g \vec{g}^* + \frac{\mu V}{L^2} \nabla^{*2} \vec{V}^*$$

- Every additive term has primary dimensions  $\{m^1 L^{-2} t^{-2}\}$ . To nondimensionalize, we multiply every term by  $L/(\rho V^2)$ , which has primary dimensions  $\{m^{-1} L^2 t^2\}$ , so that the dimensions cancel. After rearrangement:

$$\left[ \frac{fL}{V} \right] \frac{\partial \vec{V}^*}{\partial t^*} + \left( \vec{V}^* \cdot \nabla^* \right) \vec{V}^* = - \left[ \frac{P_0 - P_\infty}{\rho V^2} \right] \nabla^* P^* + \left[ \frac{gL}{V^2} \right] \vec{g}^* + \left[ \frac{\mu}{\rho V L} \right] \nabla^{*2} \vec{V}^*$$

# Nondimensionalization of the NSE

- Each of terms in [ ] is a nondimensional group of parameters (Pi Group):

$$\left[ \frac{fL}{V} \right] \frac{\partial \vec{V}^*}{\partial t^*} + (\vec{V}^* \cdot \nabla^*) \vec{V}^* = - \left[ \frac{P_0 - P_\infty}{\rho V^2} \right] \nabla^* P^* + \left[ \frac{gL}{V^2} \right] \vec{g}^* + \left[ \frac{\mu}{\rho V L} \right] \nabla^{*2} \vec{V}^*$$

Strouhal number

Euler number

Inverse of Froude number squared

Inverse of Reynolds number

$$[St] \frac{\partial \vec{V}^*}{\partial t^*} + (\vec{V}^* \cdot \nabla^*) \vec{V}^* = - [Eu] \nabla^* P^* + \left[ \frac{1}{Fr^2} \right] \vec{g}^* + \left[ \frac{1}{Re} \right] \nabla^{*2} \vec{V}^*$$

Navier-Stokes equation in nondimensional form



# Nondimensionalization of the NSE

## ● Nondimensionalization vs. Normalization

- NSE are now nondimensional, but not necessarily normalized. What is the difference?
- Nondimensionalization concerns only the dimensions of the equation - we can use any value of scaling parameters  $L, V$ , etc.
- Normalization is more restrictive than nondimensionalization. To normalize the equation, we must choose scaling parameters  $L, V$ , etc. that are appropriate for the flow being analyzed, such that all nondimensional variables are of order of magnitude unity, i.e., their minimum and maximum values are close to 1.0.

$$t^* \sim 1 \quad \vec{x}^* \sim 1 \quad \vec{V}^* \sim 1 \quad P^* \sim 1 \quad \vec{g}^* \sim 1 \quad \nabla^* \sim 1$$

If we have properly normalized the NSE, we can compare the relative importance of the terms in the equation by comparing the relative magnitudes of the nondimensional parameters  $St, Eu, Fr$ , and  $Re$ .

# Comments about CD-NSE

- The nondimensionalized continuity equation contains no additional dimensionless parameters.
- The order of magnitude of the nondimensional variables is unity if they are nondimensionalized using a length, speed, frequency, etc., that are characteristic of the flow field.
  - The relative importance of the terms in depends only on the relative magnitudes of the dimensionless parameters
- Dynamic similarity between a model and a prototype requires all four of [ ] to be the same for the model and the prototype.
- If the flow is steady then the first term on the left side disappears. If the characteristic frequency  $f$  is very small such that  $St \ll 1$ , the flow is called quasi-steady.
- The effect of gravity is important only in flows with free-surface effects.
  - If no free surface the only effect of gravity on the flow dynamics is a hydrostatic pressure distribution in the vertical direction superposed on the pressure field due to the fluid flow.

# Creeping Flow

- Also known as “**Stokes Flow**” or “Low Reynolds number flow”
- Occurs when  $Re \ll 1$ 
  - $\rho, V,$  or  $L$  are very small, e.g., micro-organisms, MEMS, nano-tech, particles, bubbles
  - $\mu$  is very large, e.g. honey, lava
    - $g$  effect is negligible
    - Steady flow
    - Advective term is negligible

# Creeping Flow

- To simplify NSE, assume  $St \sim 1$ ,  $Fr \sim 1$

$$[Eu] \nabla^* P^* = \left[ \frac{1}{Re} \right] \nabla^{*2} \vec{V}^*$$

Pressure  
forces

Viscous  
forces

- Since

$$P^* \sim 1, \quad \nabla^* \sim 1$$

$$Eu = \frac{P_0 - P_\infty}{\rho V^2} \sim \frac{1}{Re} = \frac{\mu}{\rho V L} \quad \longrightarrow \quad P_0 - P_\infty \sim \frac{\mu V}{L}$$

# Creeping Flow

- This is important

$$P_0 - P_\infty \sim \frac{\mu V}{L}$$

- Very different from inertia dominated flows where

$$P_0 - P_\infty \sim \rho V^2$$

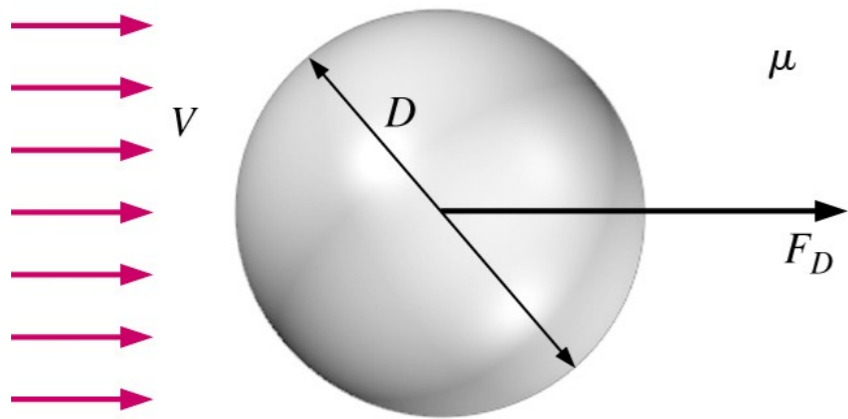
- Density has completely dropped out of NSE. To demonstrate this, convert back to dimensional form:

$$\nabla P = \mu \nabla^2 \vec{V}$$

- This is now a LINEAR EQUATION which can be solved for simple geometries.

# Creeping Flow

- Solution of Stokes flow is beyond the scope.
- Analytical solution for flow over a sphere gives a drag coefficient which is a linear function of velocity  $V$  and viscosity  $\mu$ .



$$F_D = 3\pi\mu V D$$

# Example

## EXAMPLE 10–2 Terminal Velocity of a Particle from a Volcano

A volcano has erupted, spewing stones, steam, and ash several thousand feet into the atmosphere (Fig. 10–14). After some time, the particles begin to settle to the ground. Consider a nearly spherical ash particle of diameter  $50\ \mu\text{m}$ , falling in air whose temperature is  $-50^\circ\text{C}$  and whose pressure is  $55\ \text{kPa}$ . The density of the particle is  $1240\ \text{kg/m}^3$ . Estimate the terminal velocity of this particle at this altitude.

$$\text{Downward force:} \quad F_{\text{down}} = W = \pi \frac{D^3}{6} \rho_{\text{particle}} g \quad (1)$$

The aerodynamic drag force acting on the particle is obtained from Eq. 10–12, and the buoyancy force is the weight of the displaced air. Thus,

$$\text{Upward force:} \quad F_{\text{up}} = F_D + F_{\text{buoyancy}} = 3\pi\mu VD + \pi \frac{D^3}{6} \rho_{\text{air}} g \quad (2)$$

We equate Eqs. 1 and 2, and solve for terminal velocity  $V$ ,

$$\begin{aligned} V &= \frac{D^2}{18\mu} (\rho_{\text{particle}} - \rho_{\text{air}}) g \\ &= \frac{(50 \times 10^{-6}\ \text{m})^2}{18(1.474 \times 10^{-5}\ \text{kg/m} \cdot \text{s})} [(1240 - 0.8588)\ \text{kg/m}^3](9.81\ \text{m/s}^2) \\ &= \mathbf{0.115\ \text{m/s}} \end{aligned}$$

Finally, we verify that the Reynolds number is small enough that creeping flow is an appropriate approximation,

$$\text{Re} = \frac{\rho_{\text{air}} VD}{\mu} = \frac{(0.8588\ \text{kg/m}^3)(0.115\ \text{m/s})(50 \times 10^{-6}\ \text{m})}{1.474 \times 10^{-5}\ \text{kg/m} \cdot \text{s}} = 0.335$$

- Although the equation for creeping flow drag on a sphere was derived for a case with  $\text{Re} \ll 1$ , it turns out that the approximation is reasonable up to  $\text{Re} \cong 1$ .
- A more involved calculation, including a Reynolds number correction and a correction based on the mean free path of air molecules, yields a terminal velocity of  $0.110\ \text{m/s}$ ; the error of the creeping flow approximation is less than 5 percent.

# Inviscid Regions of Flow

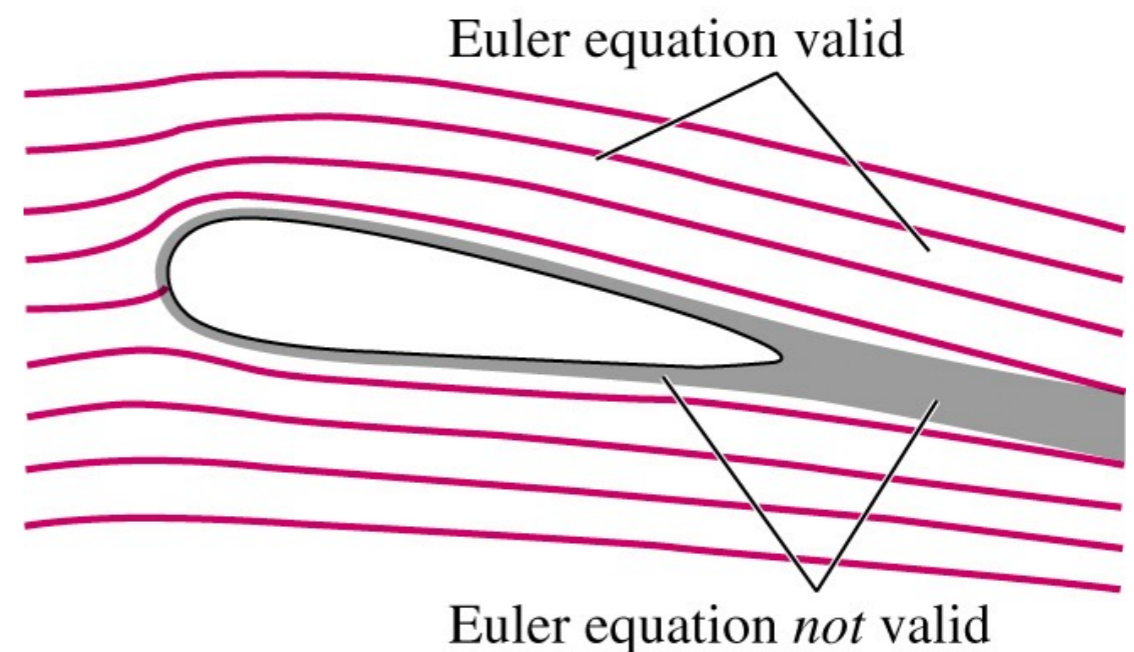
- Definition: Regions where net viscous forces are negligible compared to pressure and/or inertia forces

$$[St] \frac{\partial \vec{V}^*}{\partial t^*} + (\vec{V}^* \cdot \nabla^*) \vec{V}^* = - [Eu] \nabla^* P^* + \left[ \frac{1}{Fr^2} \right] \vec{g}^* + \left[ \frac{1}{Re} \right] \nabla^{*2} \vec{V}^*$$

~0 if Re large

Euler Equation

The Euler equation approximation is appropriate in high Reynolds number regions of the flow, where net viscous forces are negligible, far away from walls and wakes.





# Inviscid Regions of Flow

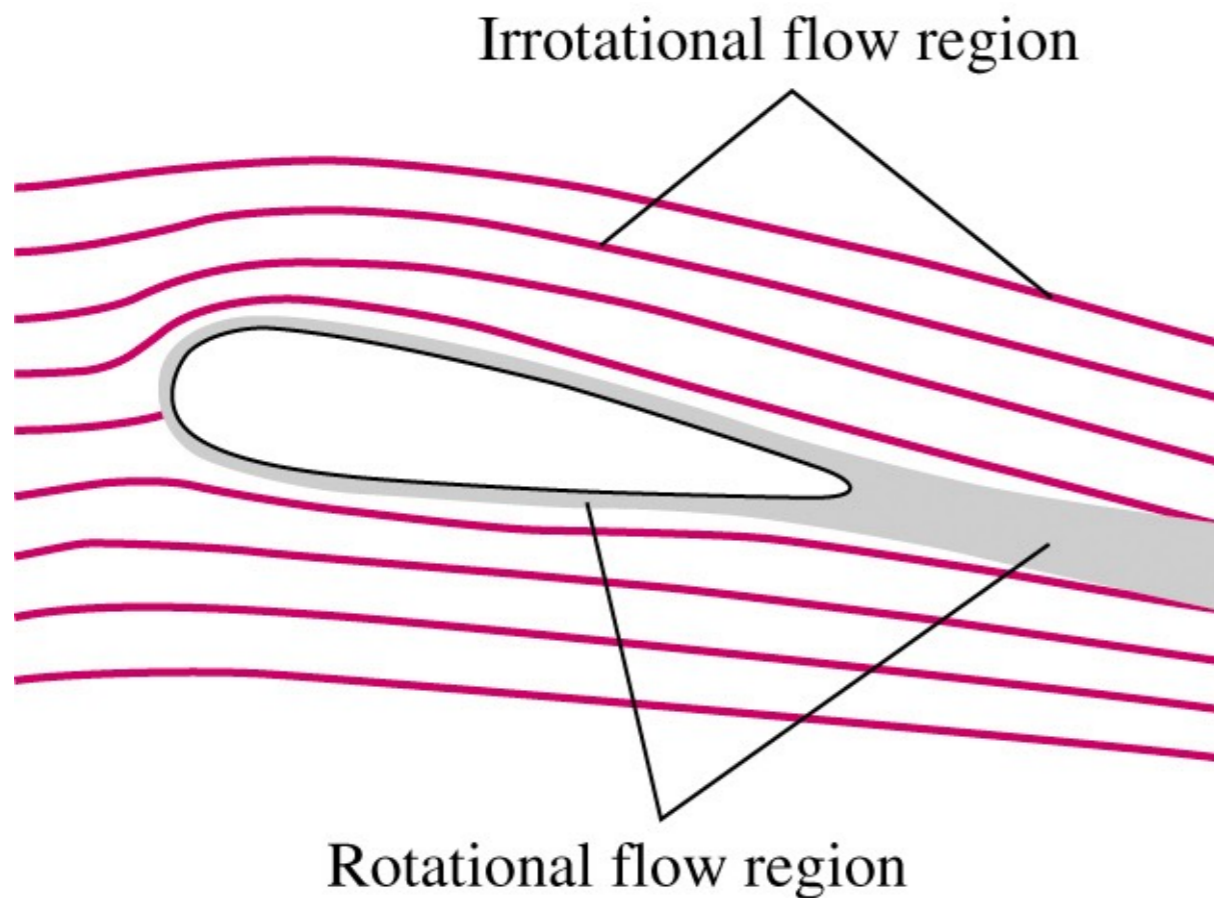
- Euler equation often used in aerodynamics
- Elimination of viscous term changes PDE from mixed elliptic-hyperbolic to hyperbolic. This affects the type of analytical and computational tools used to solve the equations.
- Must “relax” wall boundary condition from no-slip to slip

$$\begin{array}{c} \text{No-slip BC} \\ u = v = w = 0 \end{array}$$

$$\begin{array}{c} \text{Slip BC} \\ \tau_w = 0, V_n = 0 \end{array}$$

$V_n$  = normal velocity

# Irrotational Flow Approximation



- Irrotational approximation: vorticity is negligibly small

$$\vec{\zeta} = \nabla \times \vec{V} \approx 0$$

- In general, inviscid regions are also irrotational, but there are situations where inviscid flow are rotational, e.g., solid body rotation

# Irrotational Flow Approximation

- What are the implications of irrotational approximation. Look at continuity and momentum equations.

- Continuity equation

- Use the vector identity

$$\nabla \times \nabla \phi = 0$$

- Since the flow is irrotational

$$\nabla \times \vec{V} = 0$$

$$\vec{V} = \nabla \phi$$

$\phi$  is a scalar potential function

# Irrotational Flow Approximation

- Therefore, regions of irrotational flow are also called regions of **potential flow**.
- From the definition of the gradient operator  $\nabla$

Cartesian

$$U = \frac{\partial \phi}{\partial x}, \quad V = \frac{\partial \phi}{\partial y}, \quad W = \frac{\partial \phi}{\partial z}$$

Cylindrical

$$U_r = \frac{\partial \phi}{\partial r}, \quad U_\theta = \frac{1}{r} \frac{\partial \phi}{\partial \theta}, \quad U_z = \frac{\partial \phi}{\partial z}$$

- Substituting into the continuity equation gives

$$\nabla \cdot \vec{V} = \nabla \cdot \nabla \phi = \nabla^2 \phi = 0$$

# Irrotational Flow Approximation

- This means we only need to solve 1 linear scalar equation to determine all 3 components of velocity!

$$\nabla^2 \phi = 0$$

Laplace Equation

- Luckily, the Laplace equation appears in numerous fields of science, engineering, and mathematics. This means there are well developed tools for solving this equation.

# Irrotational Flow Approximation

- Momentum equation

- If we can compute  $\phi$  from the Laplace equation (which came from continuity) and then velocity from the definition  $\vec{V} = \nabla\phi$ , why do we need the NSE?  $\Rightarrow$  To compute Pressure.

- To begin analysis, apply irrotational approximation to viscous term of the NSE:


$$\mu \nabla^2 \vec{V} = \mu \nabla^2 (\nabla \phi) = \mu \nabla (\underbrace{\nabla^2 \phi}_{=0}) = 0$$

# Irrotational Flow Approximation

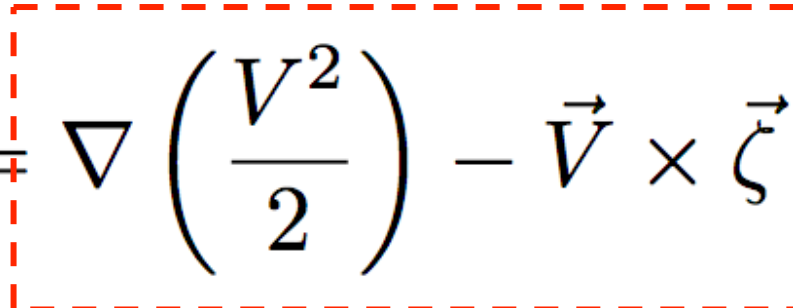
- Therefore, the NSE reduces to the Euler equation for irrotational flow:

**nondimensional**  $[St] \frac{\partial \vec{V}^*}{\partial t^*} + (\vec{V}^* \cdot \nabla^*) \vec{V}^* = -[Eu] \nabla^* P^* + \left[ \frac{1}{Fr^2} \right] \vec{g}^*$

**dimensional**  $\rho \left[ \frac{\partial \vec{V}}{\partial t} + \underbrace{(\vec{V} \cdot \nabla) \vec{V}} \right] = -\nabla P + \rho \vec{g}$



- Instead of integrating to find P, use vector identity to derive Bernoulli equation

$$(\vec{V} \cdot \nabla) \vec{V} = \nabla \left( \frac{V^2}{2} \right) - \vec{V} \times (\nabla \times \vec{V}) = \nabla \left( \frac{V^2}{2} \right) - \vec{V} \times \zeta$$


# Irrotational Flow Approximation

- This allows the steady Euler equation to be written as

$$\nabla \left( \frac{V^2}{2} \right) - \vec{V} \times \vec{\zeta} = -\frac{1}{\rho} \nabla P + \vec{g}$$

$= -g\vec{k} = \nabla(gz)$

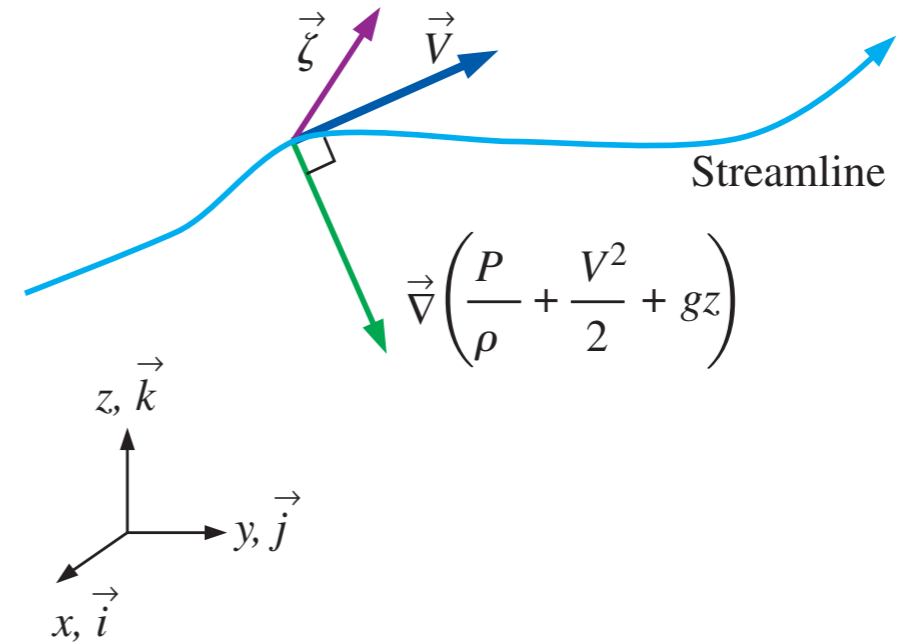
$$\nabla \left( \frac{P}{\rho} + \frac{V^2}{2} + gz \right) = \vec{V} \times \vec{\zeta}$$

- This form of Bernoulli equation is valid for inviscid (and irrotational) flow since we've shown that NSE reduces to the Euler equation.



# Irrotational Flow Approximation

● However



Inviscid

$$\frac{P}{\rho} + \frac{V^2}{2} + gz = C \quad \text{along a streamline}$$

Irrotational ( $\zeta = 0$ )

$$\frac{P}{\rho} + \frac{V^2}{2} + gz = C \quad \text{everywhere}$$

# Irrotational Flow Approximation

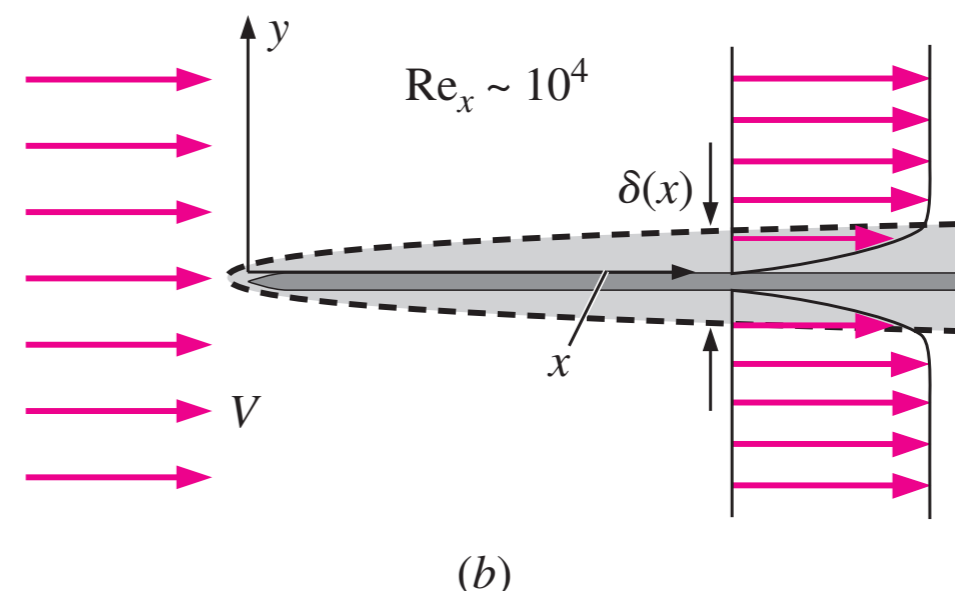
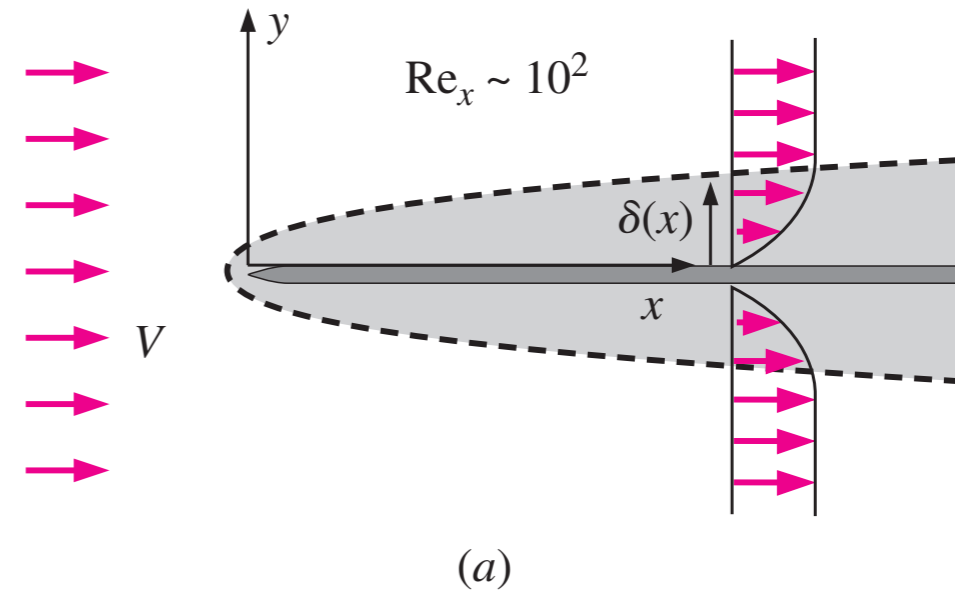
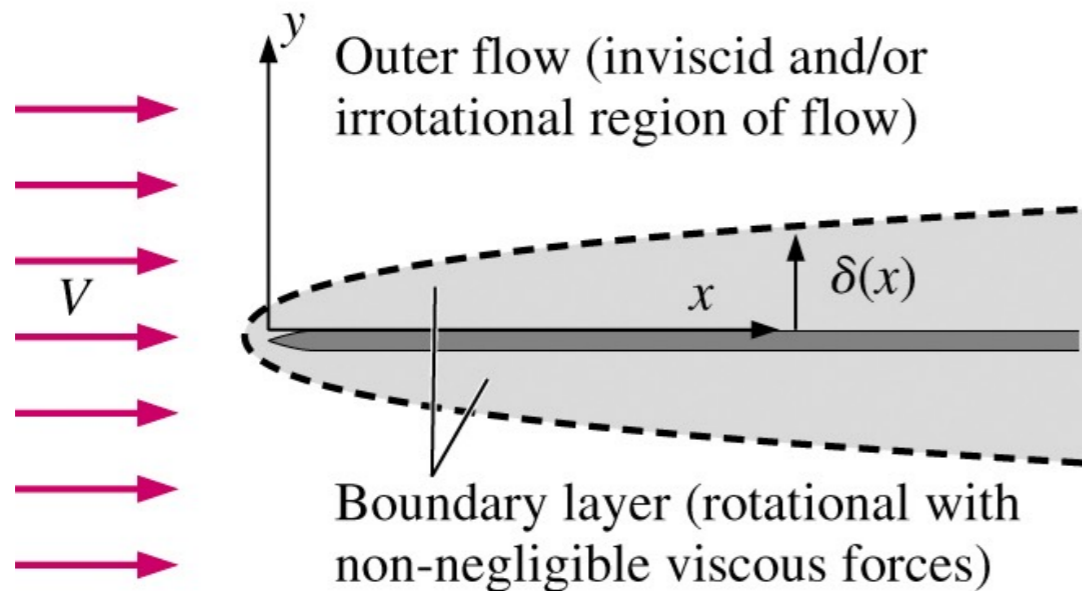
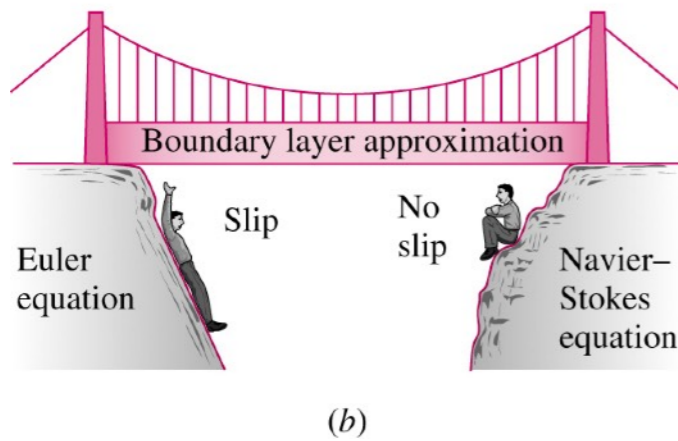
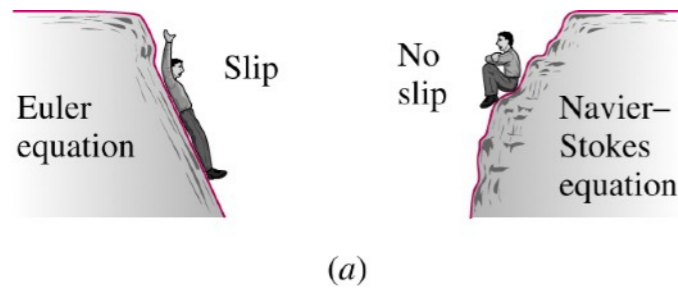
- Therefore, the process for irrotational flow
  - Calculate  $\phi$  from Laplace equation (from continuity)
  - Calculate velocity from definition  $\vec{V} = \nabla\phi$
  - Calculate pressure from Bernoulli equation (derived from momentum equation)

$$P = P_{\infty} + \rho \left[ \frac{V_{\infty}^2 - V^2}{2} + g(z_0 - z) \right]$$

Valid for 3D or 2D

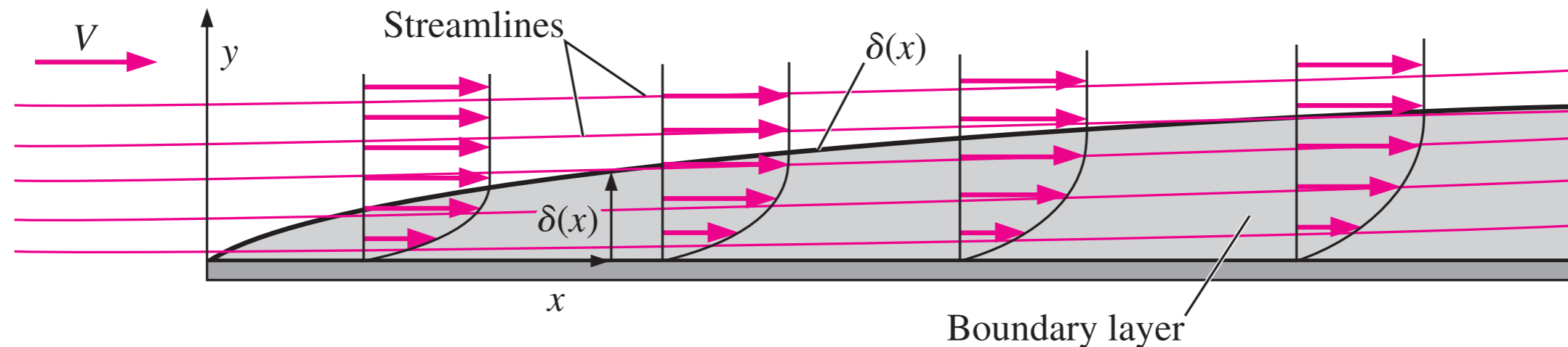
# Boundary Layer (BL) Approximation

- BL approximation bridges the gap between the Euler and NS equations, and between the slip and no-slip BC at the wall.
- Prandtl (1904) introduced the BL approximation
- At a given  $x$ -location, the higher the Reynolds number, the thinner the boundary layer.



# Boundary Layer (BL) Approximation

## ● Flat Boundary Layer

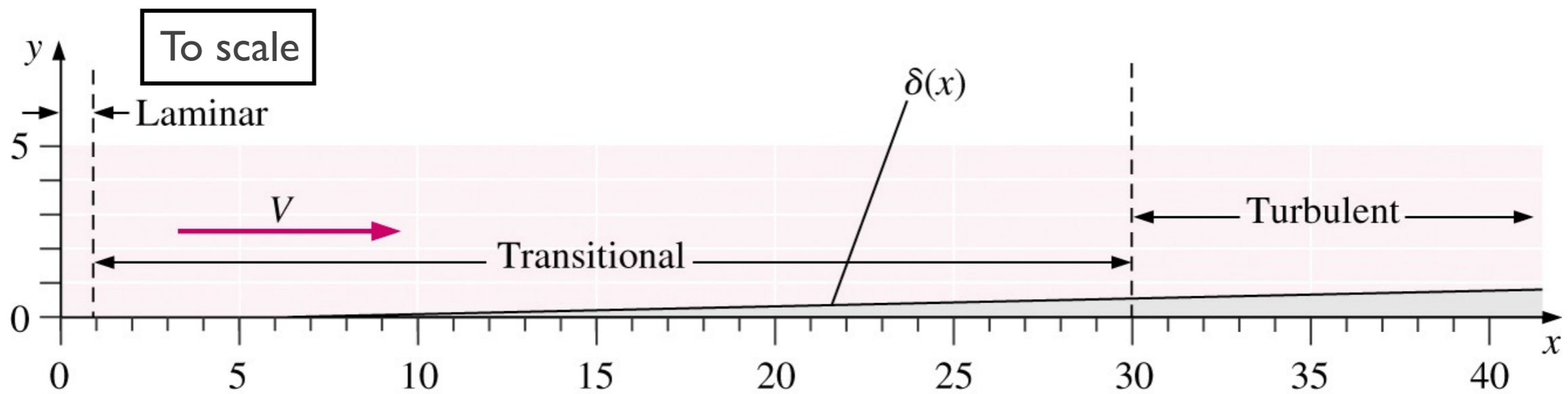
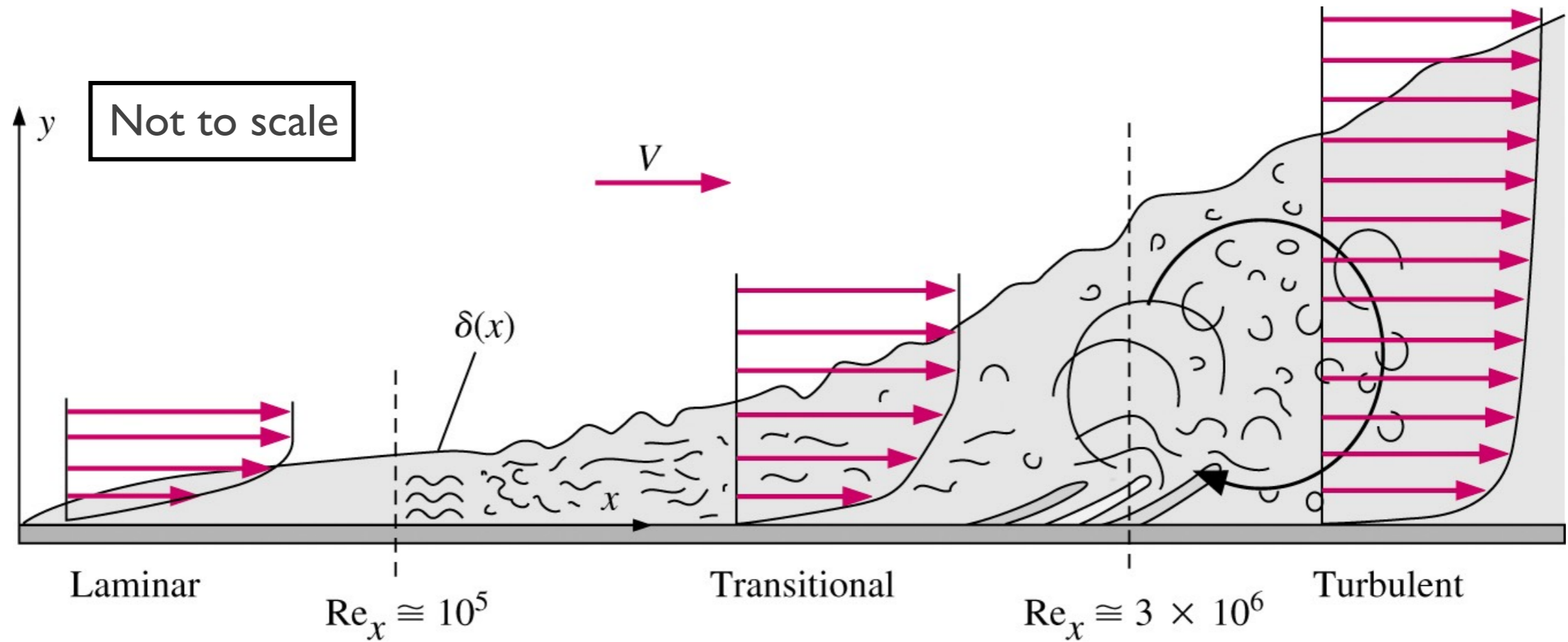


$\delta$  is proportional to the square root of  $Re_x$ . These results are valid only for a laminar boundary layer on a flat plate.

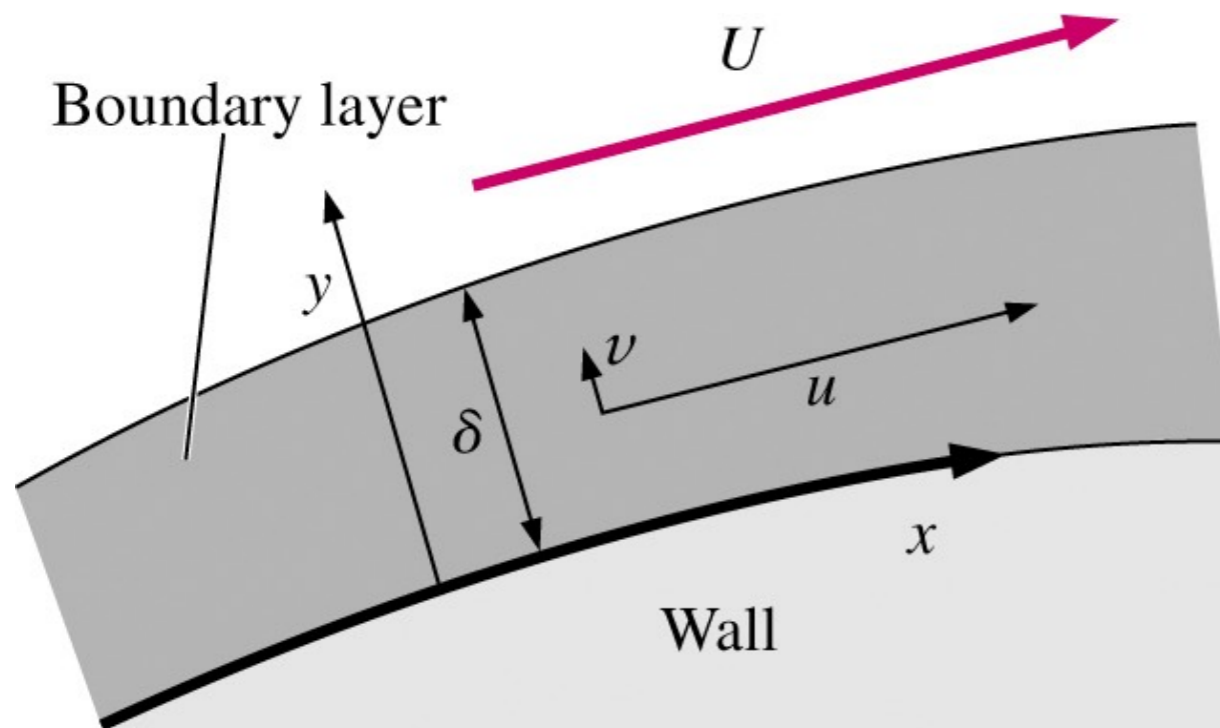
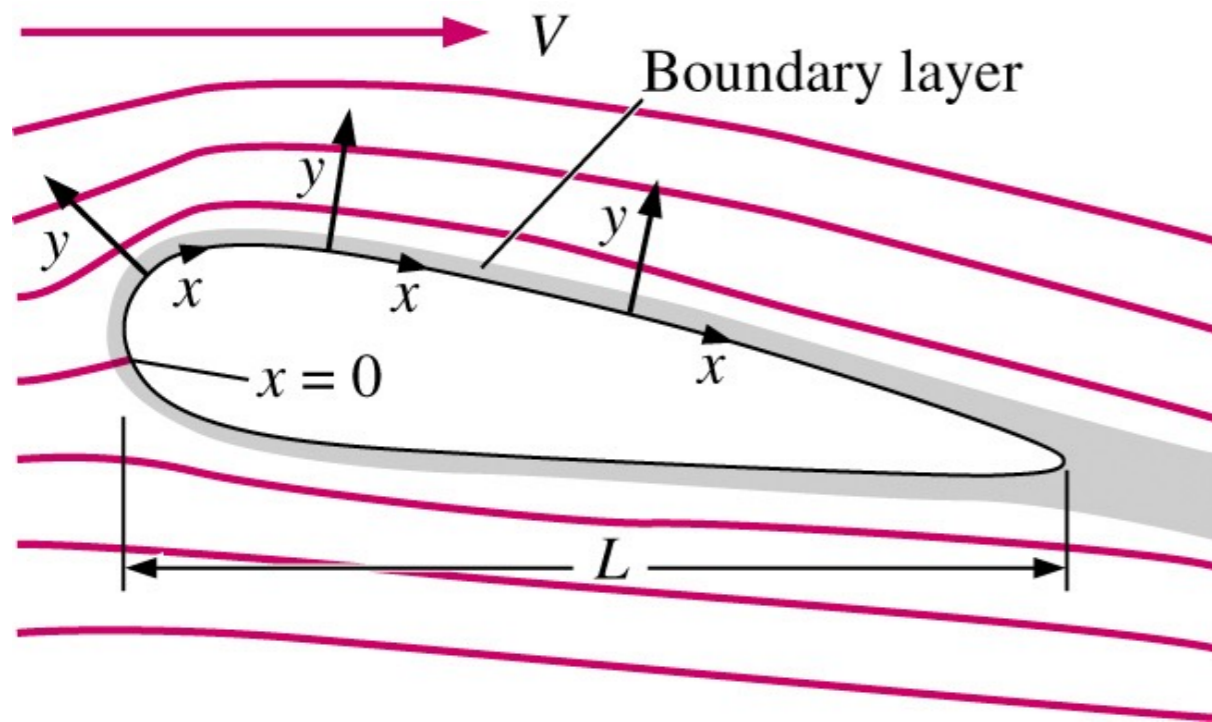
As we move down the plate to larger and larger values of  $x$ ,  $Re_x$  increases linearly with  $x$ . At some point, infinitesimal disturbances in the flow begin to grow, and the boundary layer cannot remain laminar—it begins a transition process toward turbulent flow.

For a smooth flat plate with a uniform free stream, the transition process begins at a critical Reynolds number,  $Re_{x, \text{critical}} \cong 10^5$ , and continues until the boundary layer is fully turbulent at the transition Reynolds number,  $Re_{x, \text{transition}} \cong 3 \cdot 10^6$

# Boundary Layer (BL) Approximation



# Boundary Layer (BL) Approximation



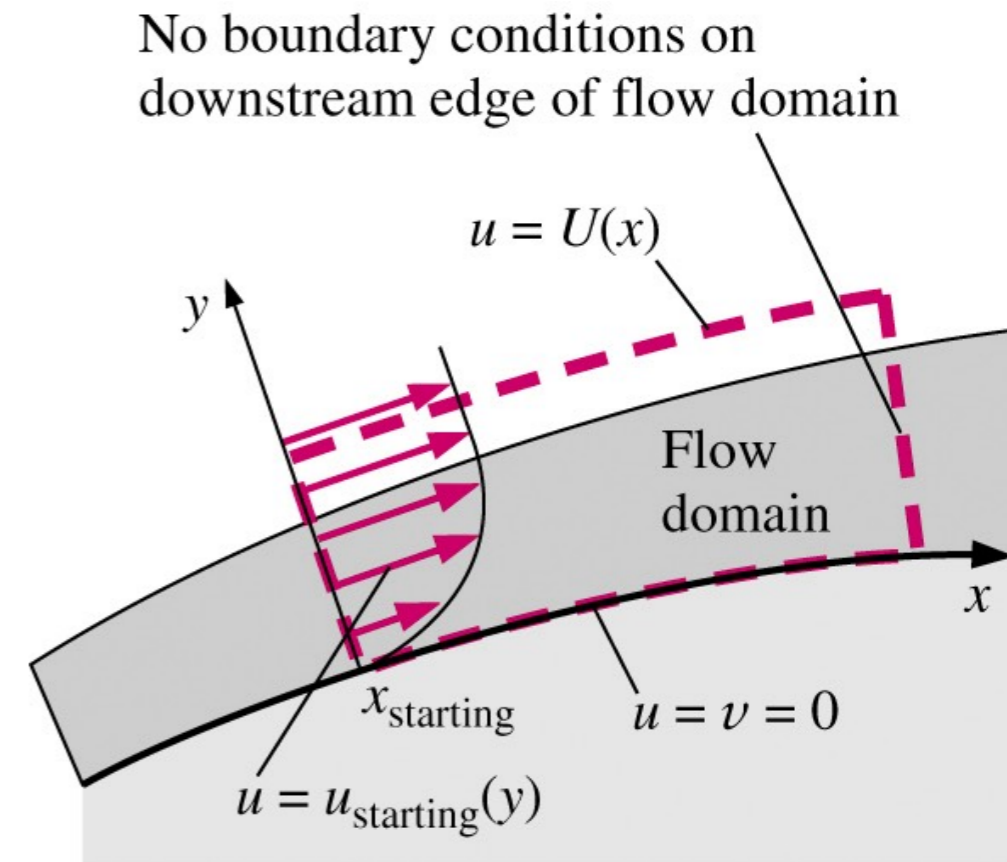
- BL Equations: we restrict attention to steady, 2D, laminar flow (although method is fully applicable to unsteady, 3D, turbulent flow)

- BL coordinate system
  - $x$  : tangential direction
  - $y$  : normal direction



# Boundary Layer Procedure

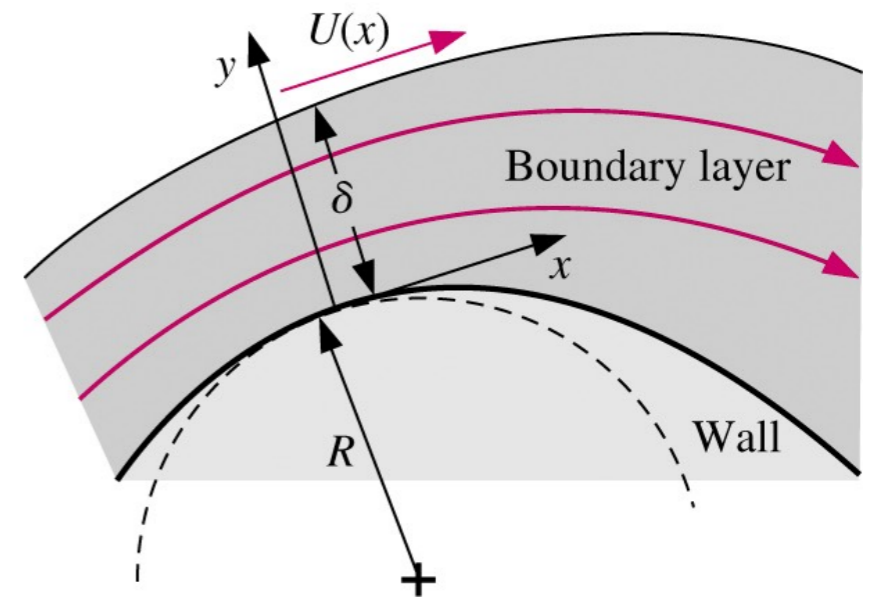
1. Solve for outer flow, ignoring the BL. Use potential flow (irrotational approximation) or Euler equation
2. Assume  $\delta/L \ll 1$  (thin BL)
3. Solve BLE
  1.  $y = 0 \Rightarrow$  no-slip,  $u=0, v=0$
  2.  $y = \delta \Rightarrow U = U_e(x)$
  3.  $x = x_0 \Rightarrow u = u(x_0), v=v(x_0)$
4. Calculate  $\delta, \theta, \delta^*, \tau_w, \text{Drag}$
5. Verify  $\delta/L \ll 1$
6. If  $\delta/L$  is not  $\ll 1$ , use  $\delta^*$  as body and goto step 1 and repeat



# Boundary Layer Procedure

## ● Possible Limitations

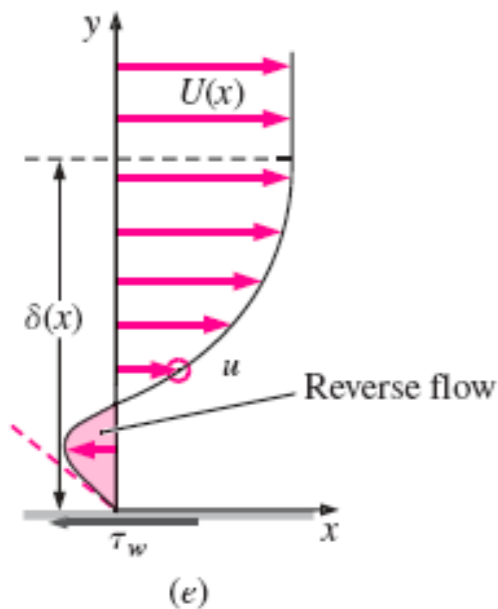
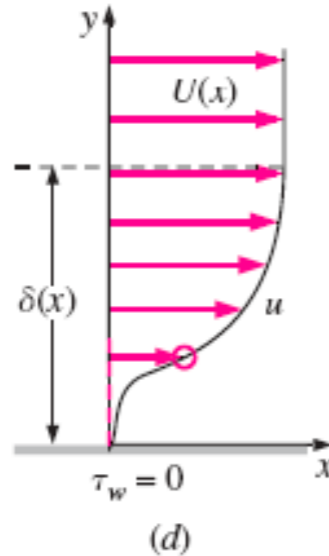
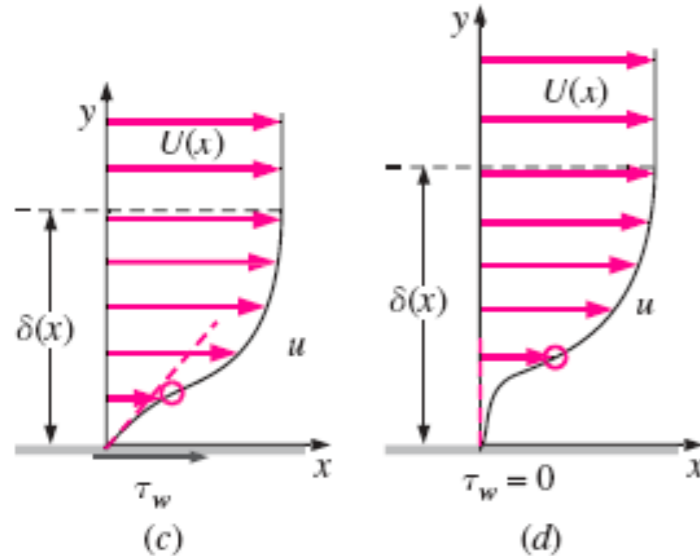
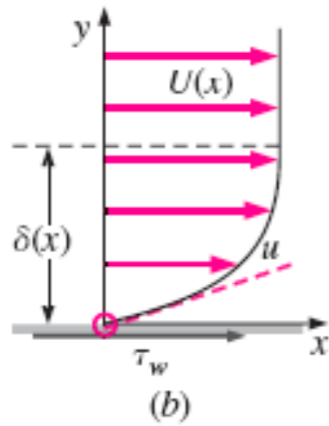
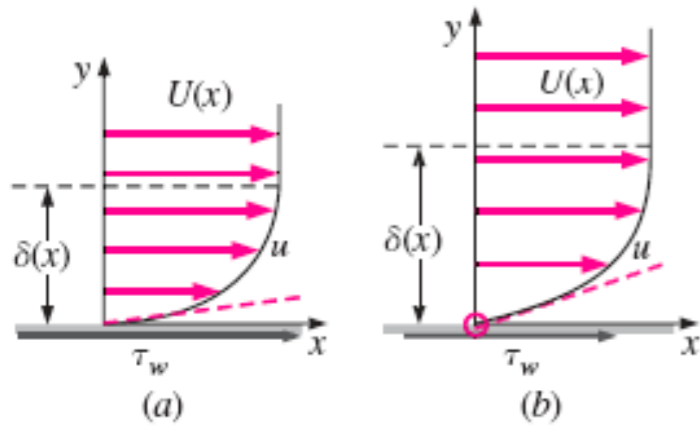
1.  $Re$  is not large enough  $\Rightarrow$  BL may be too thick for thin BL assumption.
2.  $\partial p / \partial y \neq 0$  due to wall curvature  $\delta \sim R$
3.  $Re$  too large  $\Rightarrow$  turbulent flow at  $Re = 1 \times 10^5$ . BL approximation still valid, but new terms required.
4. Flow separation





# Pressure Gradients

- Shape of the BL is strongly influenced by external pressure gradient:



(a) favorable ( $dP/dx < 0$ )

(b) zero

(c) mild adverse ( $dP/dx > 0$ )

(d) critical adverse ( $\tau_w = 0$ )

(e) large adverse with reverse (or separated) flow

# Pressure Gradients

- The BL approximation is not valid downstream of a separation point because of reverse flow in the separation bubble.
- Turbulent BL is more resistant to flow separation than laminar BL exposed to the same adverse pressure gradient

