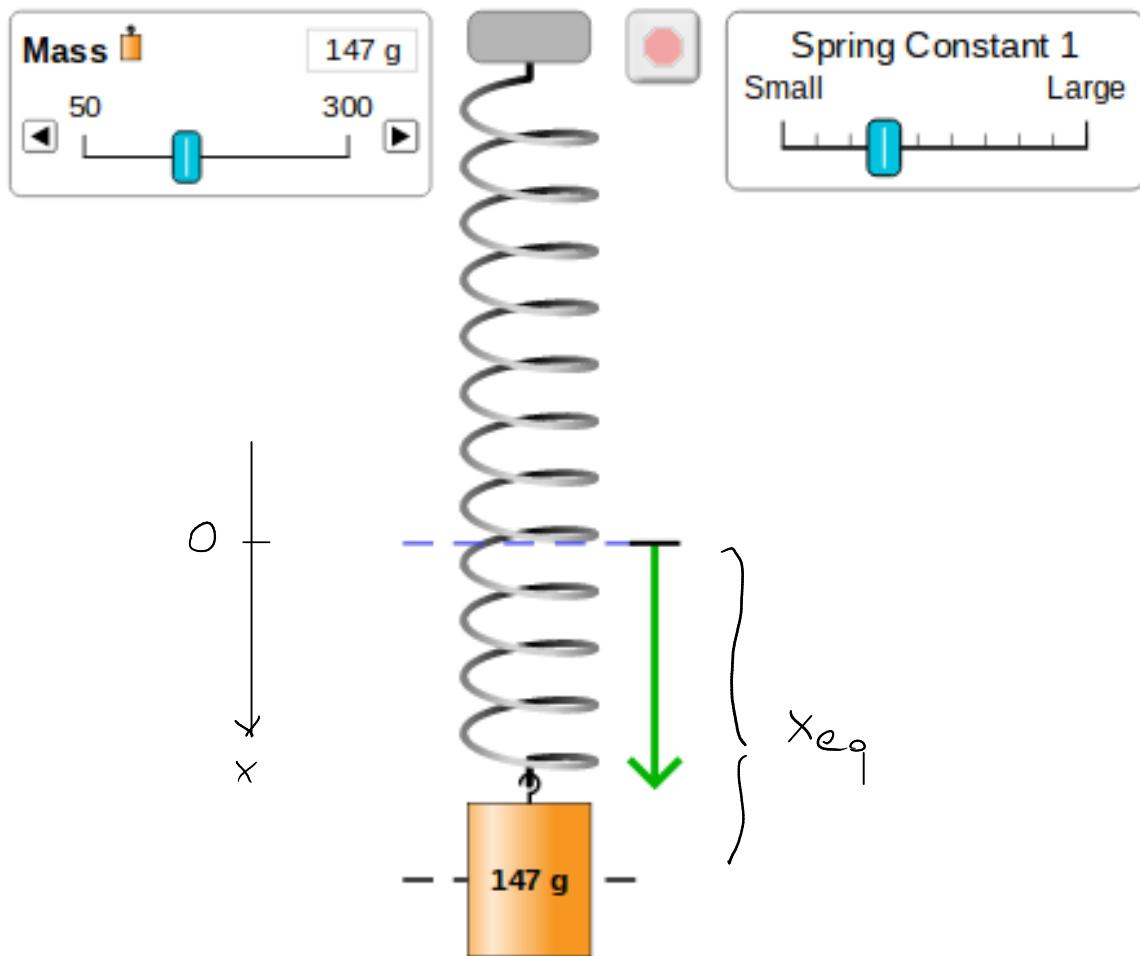


OSCILLAZIONI

$$\frac{N}{m} \quad \frac{ML}{LT^2} = \frac{M}{T^2}$$



$$k x_{eq} = mg$$

$$k = \frac{mg}{x_{eq}} \quad SI \quad \triangle$$

Analisi dimensionale

$$\tau, m, k, l_0$$

$$[\tau] = [m]^\alpha [k]^\beta [l_0]^\gamma$$

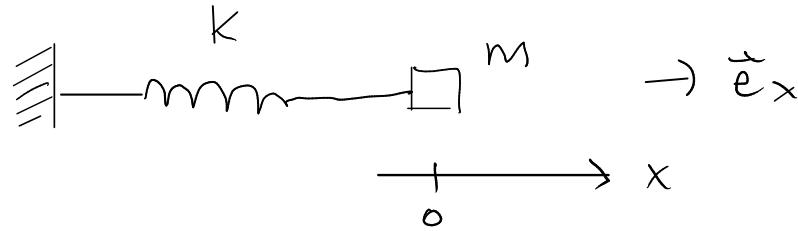
$$T = M^\alpha M^\beta L^\gamma T^{-2\beta}$$

$$\begin{cases} \alpha + \beta = 0 \Rightarrow \alpha = 1/2 \\ -2\beta = 1 \Rightarrow \beta = -1/2 \\ \gamma = 0 \Rightarrow \gamma = 0 \end{cases}$$

$$[\tau] = [k]^{-1/2} [m]^{1/2}$$

$$\tau \sim \left(\frac{m}{k}\right)^{1/2} \sim \sqrt{\frac{m}{k}}$$

Oscillatore armonico



III Newton : $m \frac{d^2 x}{dt^2} \vec{e}_x = -kx \vec{e}_x$

$$\frac{d^2 x}{dt^2} = -\frac{k}{m} x$$

$$\frac{d^2 x}{dt^2} = \frac{F}{m} = \text{cost}$$

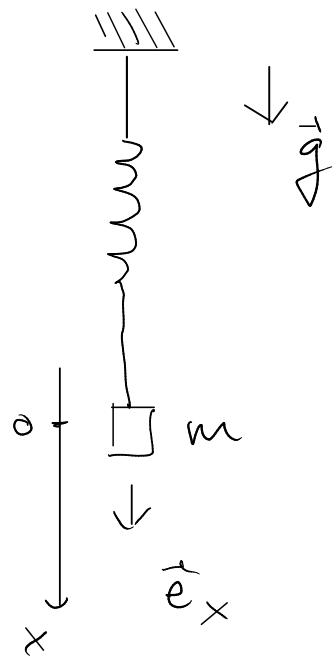
moto uniformemente accelerato

$$\frac{d^2 x}{dt^2} = -\frac{\gamma}{m} \frac{dx}{dt}$$

moto smorzato

$$\frac{d^2 x}{dt^2} = -\frac{k}{m} x$$

moto armonico / oscillazioni armoniche



$$\text{III Newton: } m \frac{d^2 x}{dt^2} \vec{e}_x = -Kx \vec{e}_x + mg \vec{e}_x$$

$$\frac{d^2 x}{dt^2} = -\frac{K}{m}x + g$$

$$x_{eq} = \frac{mg}{K}$$

$$\underline{X} = x - x_{eq} \Rightarrow x = \underline{X} + x_{eq}$$

$$\frac{d^2 \underline{X}}{dt^2} = -\frac{K}{m} \underline{X} - \frac{K}{m} \frac{mg}{K} + g$$

oscillatore armonico

$$\frac{d^2 \underline{X}}{dt^2} = -\frac{K}{m} \underline{X}$$

Soluzioni

$$\frac{d^2 x}{dt^2} = -x$$

$$x = \underset{\uparrow}{A} \sin t + \underset{\uparrow}{B} \cos t$$

costanti \rightarrow condizioni iniziali

$$x = \underset{\uparrow}{A} \cos(t + \underset{\uparrow}{\phi})$$

costanti

$$\frac{d^2x}{dt^2} = -\frac{k}{m}x \quad \Rightarrow \quad \frac{d^2x}{dt^2} = -\omega^2x \quad \text{con} \quad \omega^2 = \frac{k}{m} \quad \omega = \sqrt{\frac{k}{m}}$$

$x(t) = A \cos(\omega t + \phi)$ → soluzione dell'oscillatore armonico

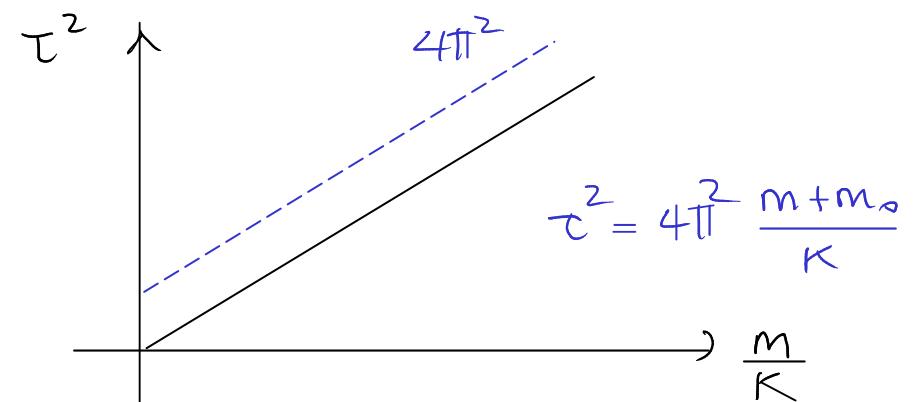
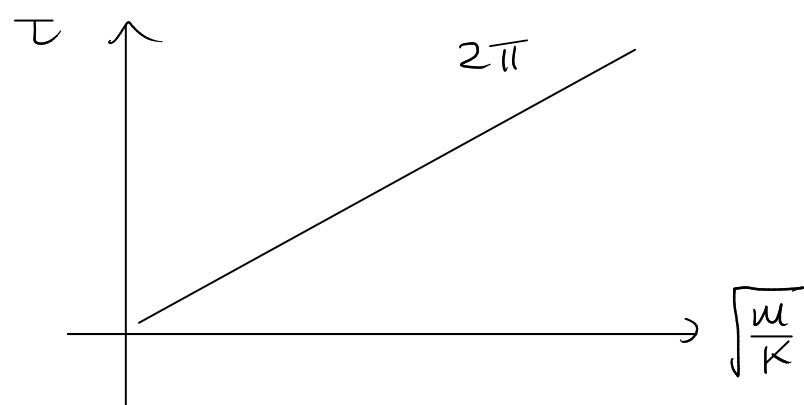
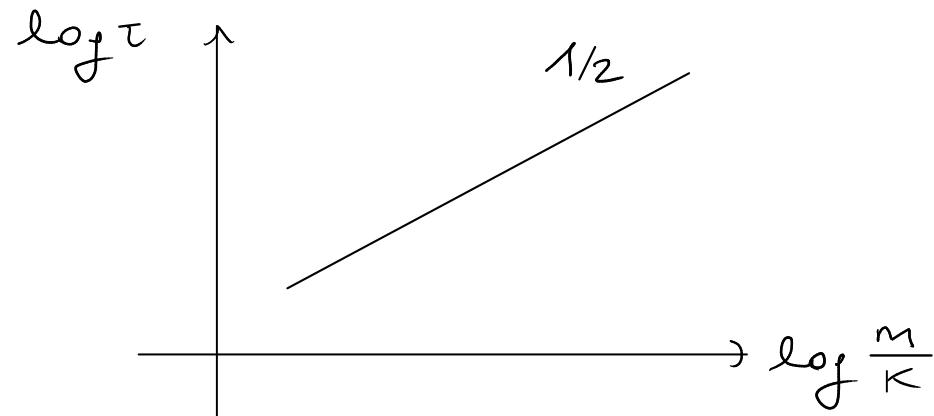
$$\frac{dx}{dt} = -A\omega \sin(\omega t + \phi)$$

$$\frac{d^2x}{dt^2} = -A\omega^2 \cos(\omega t + \phi) = -\omega^2 \overbrace{A \cos(\omega t + \phi)}^{x(t)}$$

$f(t)$ è periodica se $\exists T > 0$ tale che $f(t+T) = f(t)$ → periodo = più piccolo T !

$$x(t+T) = x(t) \rightarrow A \cos(\omega t + \omega T + \phi) = A \cos(\omega t + \phi)$$

$$\cancel{\omega t} + \omega T + \cancel{\phi} = \cancel{\omega t} + \cancel{\phi} + 2\pi \quad \Rightarrow \quad \omega T = 2\pi \quad \Rightarrow \quad \tau = \frac{2\pi}{\omega} = 2\pi \sqrt{\frac{m}{k}} \sim \sqrt{\frac{m}{k}}$$



Condizioni iniziali : $t_i = 0$

$$\begin{cases} x(0) = x_i & \begin{cases} x_i = A \cos(\phi) & \textcircled{1} \\ v_x(0) = 0 & \begin{cases} 0 = -A\omega \sin(\phi) & \textcircled{2} \end{cases} \end{cases} \end{cases}$$

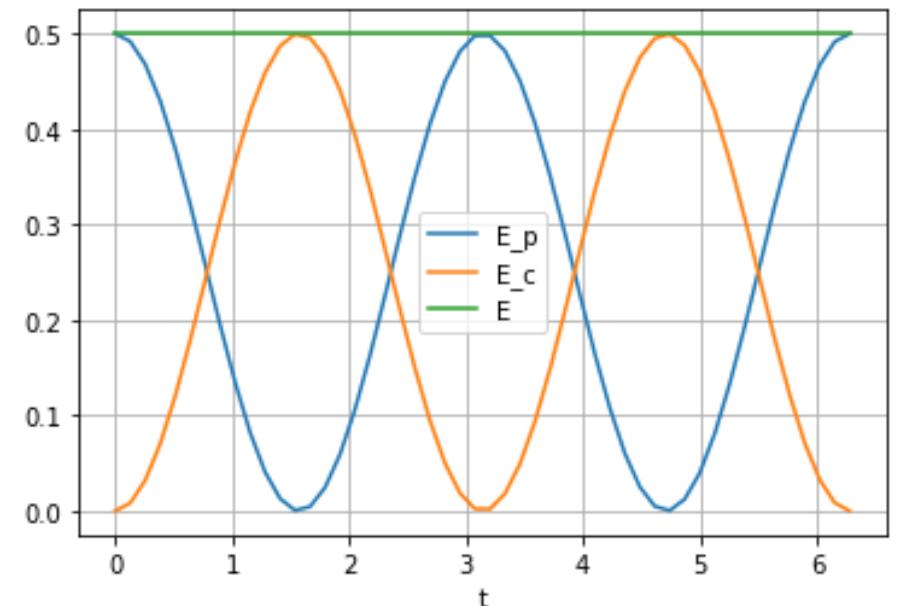
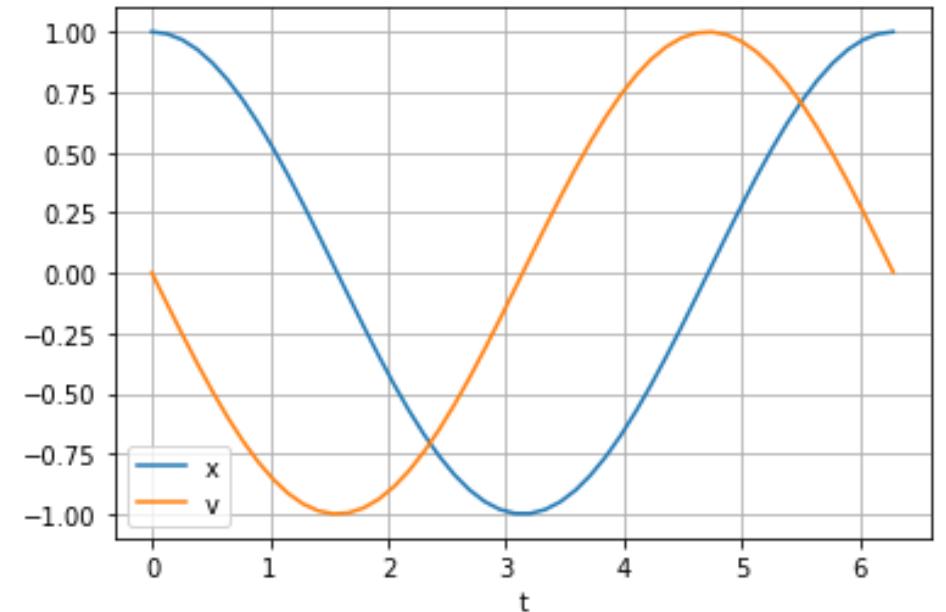
$$\textcircled{2} \Rightarrow \phi = 0 \quad \textcircled{1} + \textcircled{2} \Rightarrow A = x_i$$

$$\begin{cases} x(t) = x_i \cos\left(\sqrt{\frac{k}{m}} t\right) \\ v_x(t) = -x_i \sqrt{\frac{k}{m}} \sin\left(\sqrt{\frac{k}{m}} t\right) \end{cases}$$

Energia meccanica $\omega^2 = \frac{k}{m}$



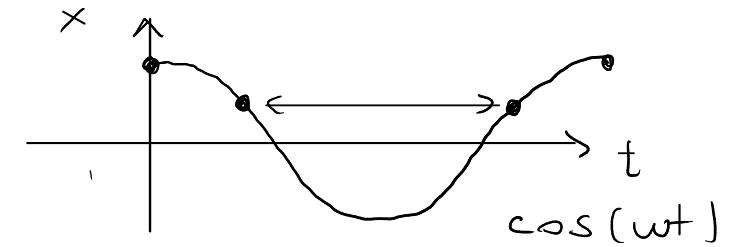
$$\begin{aligned} E &= E_c + E_p = \frac{1}{2} m A^2 \omega^2 \sin^2(\omega t + \phi) + \\ &\quad + \frac{1}{2} k A^2 \cos^2(\omega t + \phi) \\ &= \frac{1}{2} k A^2 \left[\sin^2(\omega t + \phi) + \cos^2(\omega t + \phi) \right] = \frac{1}{2} k A^2 \end{aligned}$$



Oscillatore armonico come modello per diversi fenomeni fisici

$$\frac{d^2x}{dt^2} = -\omega^2 x$$

oscillatore armonico



$$x(t) = A \cos(\omega t) + B \sin(\omega t)$$

↑ ↑

$$x(t) = A \cos(\omega t + \phi)$$

↑ ↑

Periodo: $\tau = \frac{2\pi}{\omega}$

$$[x(t+\tau) = x(t)]$$

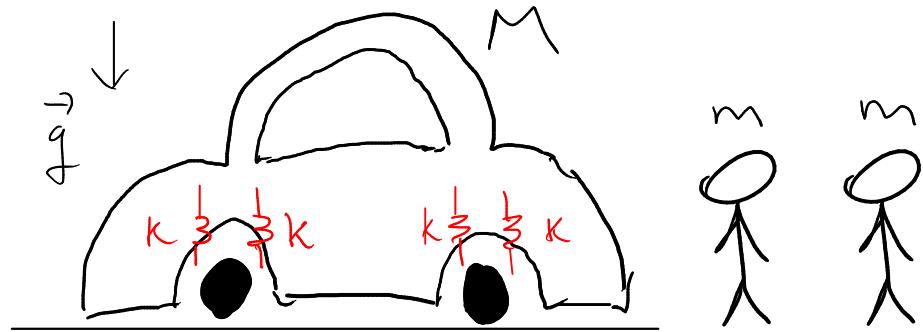
Frequenza: $f = \frac{1}{\tau}$

SI: $s^{-1} = \text{Hz}$ (Hertz)

Frequenza
angolare: $\omega = \frac{2\pi}{\tau}$

SI: $\frac{\text{rad}}{\text{s}}$

Es: oscillazioni di un'automobile

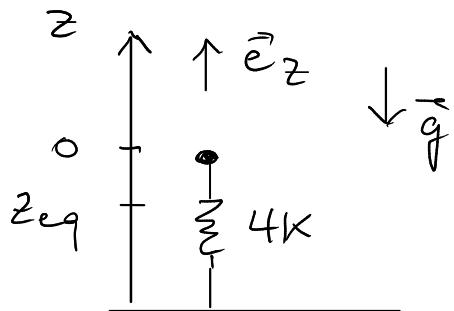


$$M = 1300 \text{ kg}$$

$$m = 80 \text{ kg}$$

$$k = 20000 \text{ N/m}$$

f = frequenza delle oscillazioni verticali = ?

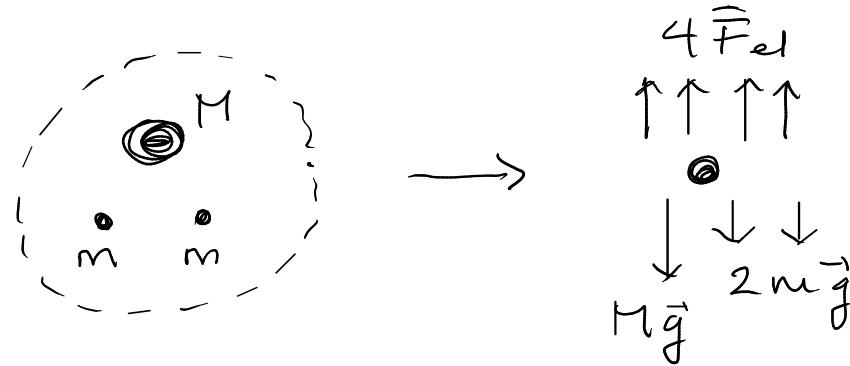


Equilibrio meccanico

$$(M + 2m) \vec{g} + 4\vec{F}_{el} = \vec{0}$$

$$-(M + 2m)g \vec{e}_z - 4k z_{eq} \vec{e}_z = \vec{0} \Rightarrow z_{eq} = -\frac{(M + 2m)g}{4k} < 0$$

Sistema: $\{M, m, m\}$



Forze: - peso
- forze elastiche

$$\text{II Newton: } \Sigma \vec{F} = (M + 2m) \vec{a}$$

$$(M + 2m) \vec{g} + 4\vec{F}_{el} = (M + 2m) \vec{a}$$

Eq. del moto:

$$(M+2m) \frac{d^2 z}{dt^2} \vec{e}_z = - (M+2m) g \vec{e}_z - 4kz \vec{e}_z$$

$$Z = z - z_{eq} \quad z = Z + z_{eq}$$

$$(M+2m) \frac{d^2 Z}{dt^2} = - (M+2m) g - 4k(Z + z_{eq})$$

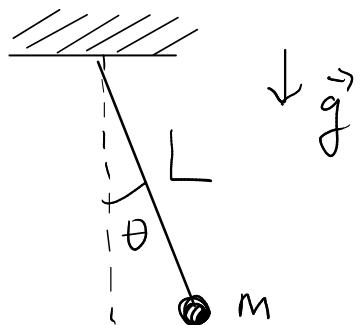
$$= - 4kZ - \cancel{(M+2m)g} + 4k \frac{M+2m}{4k} g = - 4kZ$$

$$\frac{d^2 Z}{dt^2} = - \underbrace{\frac{4k}{M+2m}}_{\omega^2} Z \rightarrow \text{oscillatore armonico}$$

$$\text{Periodo: } T = \frac{2\pi}{\omega} = 2\pi \sqrt{\frac{M+2m}{4k}}$$

$$\text{Frequenza: } f = \frac{1}{T} = \frac{1}{2\pi} \sqrt{\frac{4k}{M+2m}} = \dots = 1.18 \text{ Hz} \quad \square$$

Pendolo semplice



- corpo = particella
- filo ideale (inestensibile, senza massa)
- no attrito

Sistema: $\{m\}$

Forze: peso, tensione

II Newton:

$$m \vec{a} = m \vec{g} + \vec{T}$$

Proietto su $\vec{e}_{||}$

$$-mg \sin \theta = m \frac{d^2 s}{dt^2}$$

$$L \frac{d^2 \theta}{dt^2} = -g \sin \theta \rightarrow$$

$$\vec{a} = a_{||} \vec{e}_{||}$$

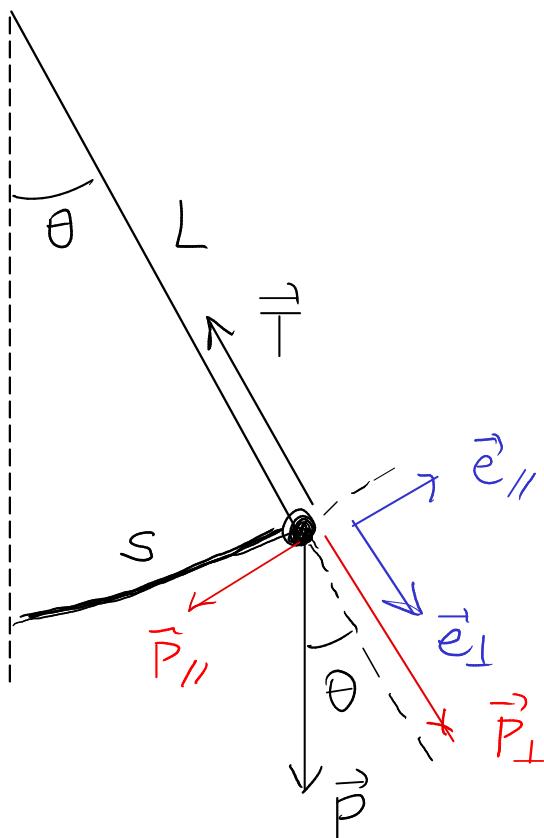
$$\theta = \frac{s}{L}$$

$$a_{||} = \frac{d^2 s}{dt^2}$$

$$s = L \theta$$

\uparrow
costante

$$\frac{d^2 \theta}{dt^2} = -\frac{g}{L} \sin \theta$$



Piccole oscillazioni: $\theta \ll 1$

$$\sin \theta \approx \theta \quad (\text{Taylor I ordine}) \quad \theta < 10^\circ \Rightarrow \sim 1\%$$

$$\frac{d^2\theta}{dt^2} \approx -\frac{g}{L}\theta \quad \omega^2 = \frac{g}{L} \quad \tau = 2\pi \sqrt{\frac{L}{g}} \quad f = \frac{1}{2\pi} \sqrt{\frac{g}{L}}$$

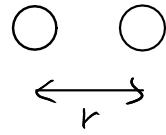
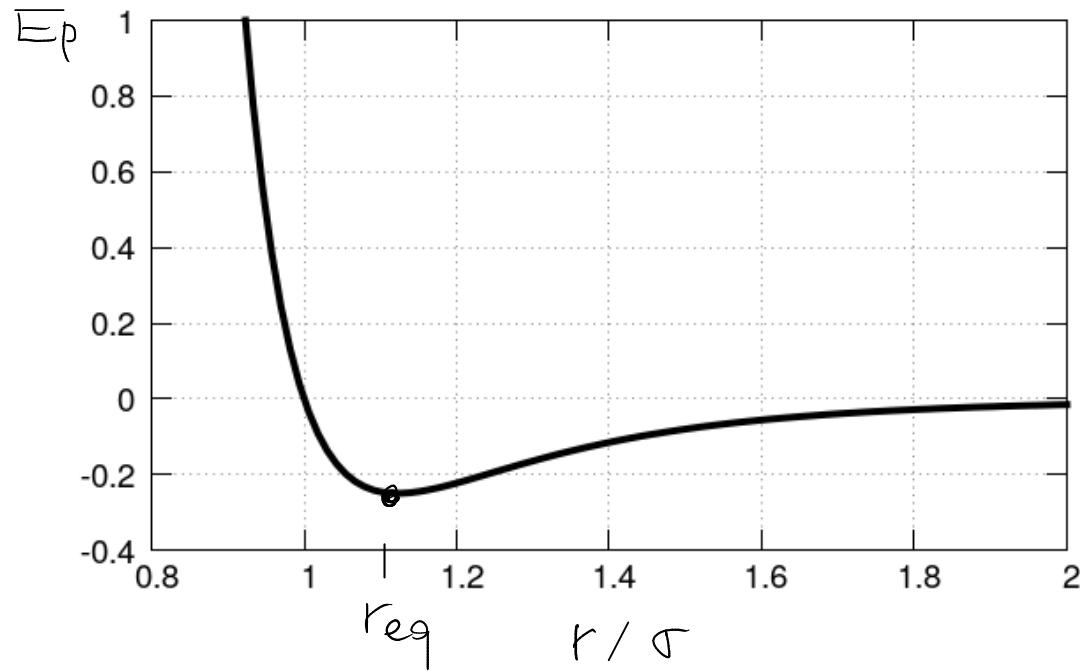
$$\theta(t) = \theta_0 \cos(\omega t + \phi)$$

$$\tau \sim \sqrt{\frac{L}{g}}$$

Oscillazioni arbitrarie:

$$\tau = f(\theta_0) \sqrt{\frac{L}{g}}$$

Interazioni tra atomi neutri : gas rari Xe, Ne, Ar



Energia potenziale d'interazione

$$E_p(r) = 4\epsilon \left[\left(\frac{\sigma}{r}\right)^{12} - \left(\frac{\sigma}{r}\right)^6 \right]$$

$$\frac{dE_p}{dr}(r_{eq}) = 0$$

$$\frac{dE_p}{dr} = 4\epsilon \left[-12 \frac{\sigma^{12}}{r^{13}} + 6 \frac{\sigma^6}{r^7} \right]$$

$$\cancel{\frac{1}{2}} \frac{\sigma^{12}}{r_{eq}^{13}} = \cancel{6} \frac{\sigma^6}{r_{eq}^7} \rightarrow 2 \frac{\sigma^6}{r_{eq}^6} = 1$$

$$r_{eq}^6 = 2\sigma^6 \Rightarrow r_{eq} = \sqrt[6]{2}\sigma \approx 1.122\sigma$$

Taylor II ordine di E_p attorno a r_{eq}

$$E_p(r) = E_p(r_{eq}) + \frac{dE_p}{dr}(r_{eq})(r-r_{eq}) + \frac{1}{2} \frac{d^2E_p}{dr^2}(r_{eq})(r-r_{eq})^2 + O((r-r_{eq})^3)$$

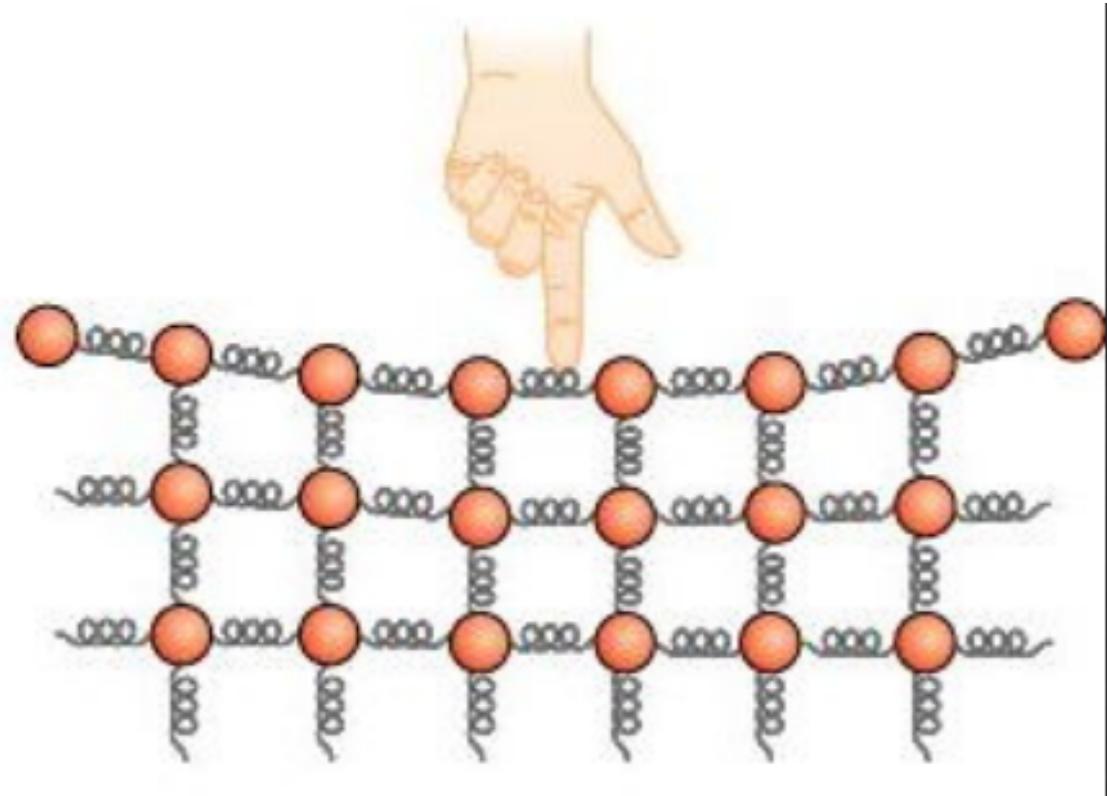
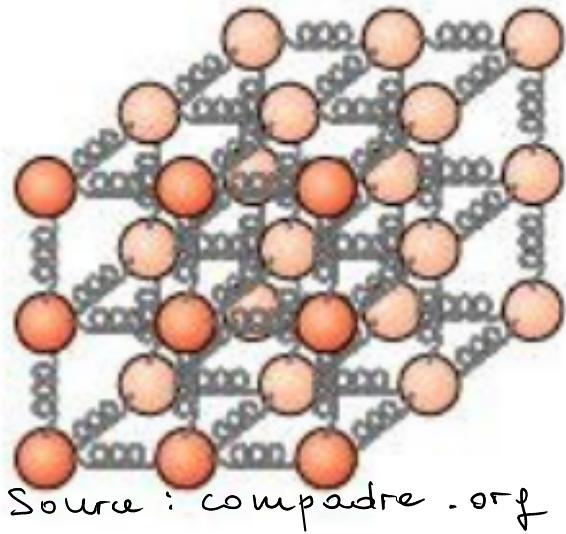
$$E_p(r) \approx E_p(r_{eq}) + \frac{1}{2} \frac{d^2E_p}{dr^2}(r_{eq})(r-r_{eq})^2$$

$$m \frac{d^2 r}{dt^2} = - \frac{dE_P}{dr} = - \frac{d^2 E_P}{dr^2}(r_{eq}) (r - r_{eq}) \quad x = r - r_{eq}$$

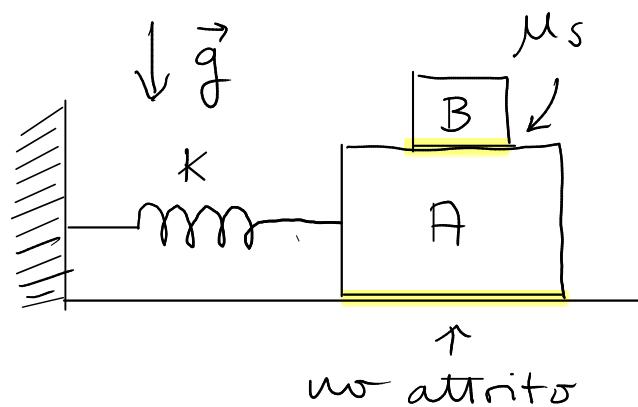
$$\frac{d^2 x}{dt^2} = - \frac{1}{m} \frac{d^2 E_P}{dr^2}(r_{eq}) x \rightarrow \text{oscillatore armonico}$$

Vibrazioni e deformazioni nei solidi

- oscillatori armonici indipendenti
- oscillatori armonici accoppiati \rightarrow modi normali



Problema di ricapitolazione (SJ 12.47)

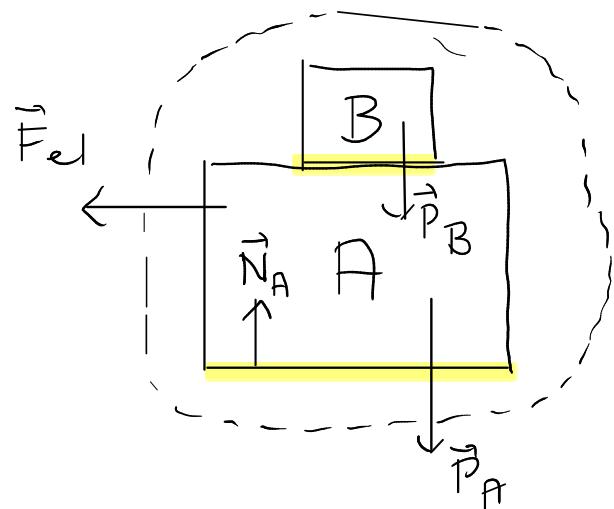


Attrito tra A e B ($\mu_s = 0.6$), non tra A e il suolo

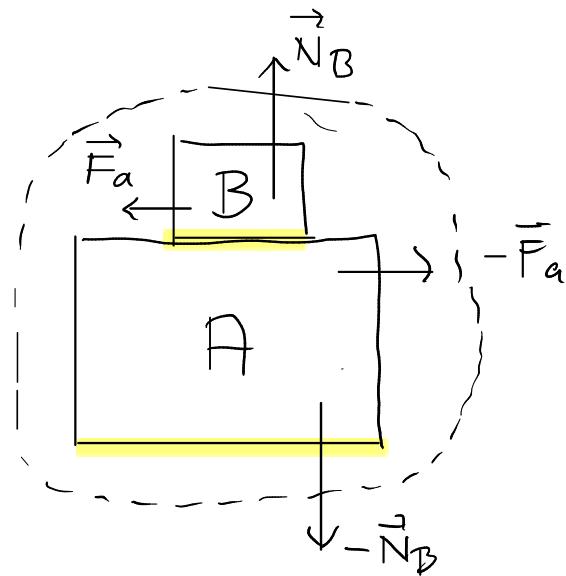
Massima ampiezza Δx delle oscillazioni senza che B scivoli su A ?

Oscillazioni armoniche di $A+B$: $f = 1.5 \text{ Hz}$

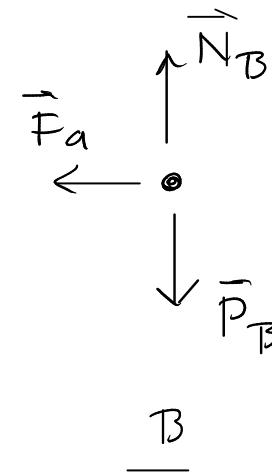
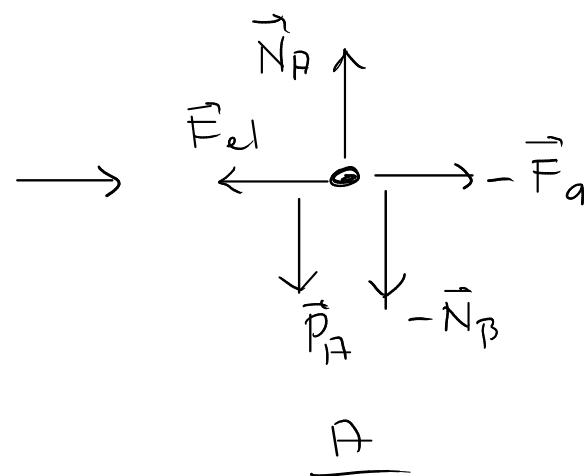
Sistema : $\{ A, B \}$



Forze esterne



Forze interne



II Newton:

$$\begin{cases} m_A \vec{a}_A = \vec{N}_A + \vec{P}_A + \vec{F}_{el} - \vec{F}_a - \vec{N}_B \\ m_B \vec{a}_B = \vec{N}_B + \vec{P}_B + \vec{F}_a \end{cases}$$

Corpi solidali: $\vec{a}_A = \vec{a}_B = \vec{a}$

Equilibrio lungo direzione verticale:

$$\begin{cases} \vec{N}_A + \vec{P}_A - \vec{N}_B = \vec{0} \\ \vec{N}_B + \vec{P}_B = \vec{0} \end{cases} \Rightarrow \begin{cases} \vec{N}_A = -\vec{P}_A - \vec{P}_B = -(\vec{P}_A + \vec{P}_B) \\ \vec{N}_B = -\vec{P}_B \end{cases}$$

$$\begin{cases} m_A \vec{a} = -\vec{P}_B + \vec{F}_{el} - \vec{F}_a + \vec{P}_B = \vec{F}_{el} - \vec{F}_a \\ m_B \vec{a} = \vec{F}_a \end{cases} \quad \begin{matrix} \rightarrow \vec{e}_x & \vec{a} = a_x \vec{e}_x \\ \sim \text{mm} \rightarrow \Delta x > 0 & \vec{F}_a = F_a \vec{e}_x \end{matrix}$$

$$\begin{cases} m_A a_x = -k \Delta x - F_a \\ m_B a_x = F_a \end{cases}$$

Non è banale anticipare il segno di F_a
(dovrà essere opposto al moto imminente)
Il risultato finale è indipendente da ciò

$$\begin{cases} a_x = -\frac{k}{m_A} \Delta x - \frac{F_a}{m_A} \\ a_x = \frac{F_a}{m_B} \end{cases} \Rightarrow -\frac{k}{m_A} \Delta x = \frac{F_a}{m_A} + \frac{F_a}{m_B} = \left(\frac{1}{m_A} + \frac{1}{m_B} \right) F_a$$

$$F_a = -\frac{k \Delta x}{\cancel{m_A}} \frac{\cancel{m_A} m_B}{m_A + m_B} = -k \Delta x \frac{m_B}{m_A + m_B}$$

Regime attrito statico

$$|F_a| \leq \mu_s |\vec{N}_B| = \mu_s m_B g$$

$$\Delta x \cdot k \frac{\cancel{m_B}}{m_A + m_B} \leq \mu_s \cancel{m_B} g$$

$$\Delta x \leq \frac{\mu_s (m_A + m_B) g}{k}$$

$$= 6,63 \text{ cm}$$

$$f = \frac{1}{2\pi} \sqrt{\frac{k}{m_A + m_B}} \Rightarrow 4\pi^2 f^2 = \frac{k}{m_A + m_B}$$

$$\Delta x \leq \frac{\mu_s g}{f} \rightarrow \text{indipendente da } m_A, m_B$$

□