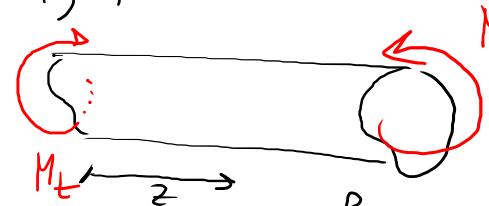


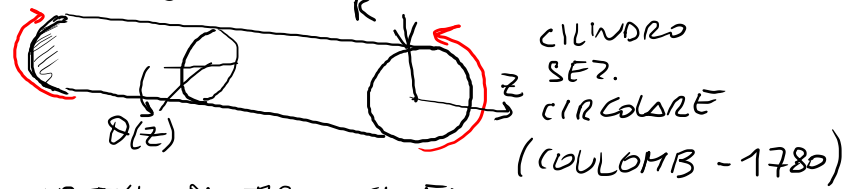
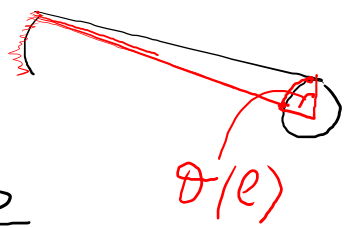
4) TORSIONE $\rightarrow \tau_{xz}, \tau_{yz}$? $\sigma_{zz} = 0$

SDC, 15/12/23



(diverse da quelle de TAGLIO)

$\oplus$: ANGOLO UNITARIO DI TORSIONE $\left[\frac{1}{L} \right]$



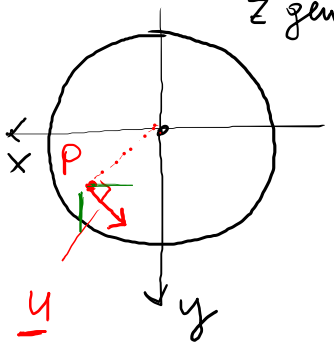
IPOTESI DI TIPO CINEMATICO

LEGAME COSTITUTIVO

$$\tau_{xz} = G \left(\underbrace{u_{x,z} + u_{z,x}}_{\theta_{xz}} \right) = -G \oplus y$$

$$\tau_{yz} = G (u_{y,z} + u_{z,y}) = G \oplus x$$

z generica



$u \Rightarrow \text{prop a } |G\oplus|$

LEGGE LINEARE

$$\begin{cases} u_x = -\theta(z) y = -\oplus z y \\ u_y = +\theta(z) x = \oplus z x \\ u_z = 0 \end{cases}$$

CDS

$$\int \tau_{xz} dA \stackrel{?}{=} 0 ; \int -G \oplus y dA =$$

$$= -G \oplus \underbrace{\int_A y dA}_{S_x} = 0$$

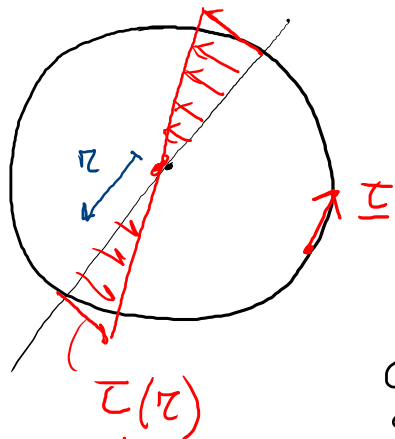
$$\int_A \tau_{yz} dA = G \oplus \underbrace{\int_A x dA}_{S_y} = 0$$

OK

$$\int_A \tau_{yz} x - \tau_{xz} y \, dA = \underbrace{M_t}_{\substack{\text{MOTO} \\ \text{DEL} \\ \text{PROBL.}}} \Rightarrow G \underbrace{\oplus \int (x^2 + y^2) \, dA}_{I_G \text{ (MOM D'INERZIA POLARE)}} = M_t \quad ; \quad \boxed{G \oplus I_G = M_t}$$

$$\tau_{xz} = -\frac{M_t}{I_G} y \quad ; \quad \tau_{yz} = \frac{M_t}{I_G} x$$

$$|\underline{\tau}| = \sqrt{\tau_{xz}^2 + \tau_{yz}^2} = \frac{M_t}{I_G} \underbrace{\sqrt{y^2 + x^2}}_{|\underline{r}|=r} = \frac{M_t}{I_G} r$$



SUL BORDO (OB)
IL VETTORE $\underline{\tau}$
È TANGENTE AL
CONTORNO DELLA
SEZIONE

(VALE PER TUTTE LE SEZIONI!)

$$I_G \text{ (MOM D'INERZIA POLARE)} \quad I_G = \frac{\pi R^4}{2}$$

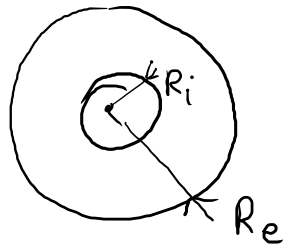
SEZ. CIRCOLARE

$$\oplus = \frac{M_t}{G I_G}$$

COEFF. DI RIGIDEZZA
TORSIONALE (SEZ. CIRCOLARE)

$$\left(\chi = \frac{M_x}{EI_x} \quad ; \quad \bar{\gamma} = \kappa \frac{T}{GA} \quad ; \quad \bar{\epsilon} = \frac{N}{EA} \right)$$

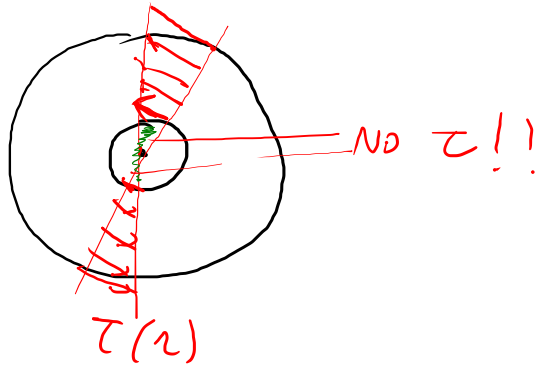




LA SOLUZ. PRECEDENTE VALE IN TUTTO E PER TUTTO, BASTA
CALCOLARE I_G PER LA SEZ. ATTUALE

$$I_G = \frac{\pi (R_e^4 - R_i^4)}{2} \Rightarrow \tau_{xz} = - \frac{M_t}{I_G} y ; \tau_{yz} = \frac{M_t}{I_G} x$$

$$\Theta = \frac{M_t}{G I_G}$$



VERIFICA CON VON MISES O TRESCA :

$$\sqrt{3} \tau_{\max} \leq \sigma_0 \Rightarrow$$

$$\tau_{\max} \leq \frac{\sigma_0}{\sqrt{3}}$$

$$\sqrt{4} \tau_{\max}^2 \leq \sigma_0 \Rightarrow$$

$$\tau_{\max} \leq \frac{\sigma_0}{2}$$

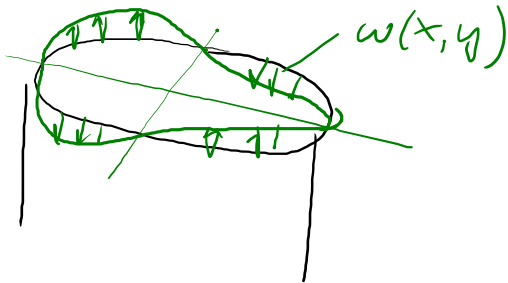
CENNI ALLA TORSIONE IN SEZ. GENERICHE COMPATTE

($w(x,y) = 0$ PER LE SEZ. CIRCOLARI)

IPOTESI CINEMATICHE

$$\begin{cases} u_x = -(\oplus) y z \\ u_y = (\oplus) x z \\ u_z = (\oplus) w(x,y) \end{cases}$$

FORM. DI S.V.

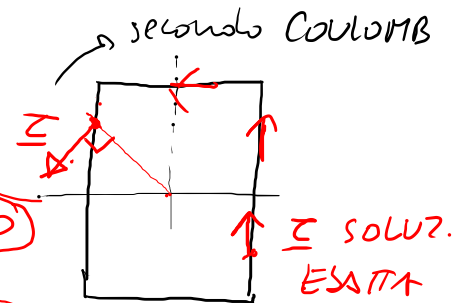


FUNZ. DI INGOMBAMENTO

$w(x,y)$ SI OTTIENE RISOLVENDO UN PROBLEMA DEL II ORDINE ALLE DERIVATE PARZIALI ($\nabla^2 w = 0$ su A)

FATTORE DI RIGIDEZZA TORSIONALE (TIPO DI OGNI SEZIONE)

$$\oplus = \frac{M_t}{G J_t}$$



(GRAZIE A $w(x,y)$)

$J_t = I_G$ SOLO PER SEZ.

PRANDTL HA FORMULATO IL PROBL DELLA

TORSIONE ATTRaverso LA "FUNZIONE DELLE TENSIONI" ($F(x,y)$)

PER CUI :

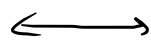
$$\nabla^2 F = -2G \oplus \text{ su } A \quad \text{e} \quad F = 0 \text{ su } \partial A$$

$\exists$ UNA RELAZ. TRA $w(x,y)$ e $F(x,y)$

BORDO DELLA SEZIONE

SI PUO' DIMOSTRARE L' ANALOGIA IDRODINAMICA :

TORSIONE

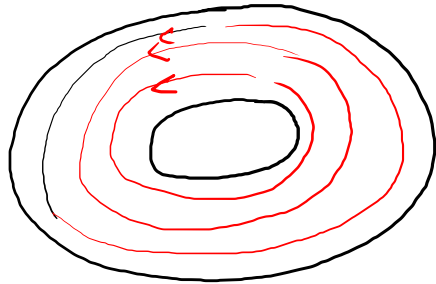


IDRODINAMICA

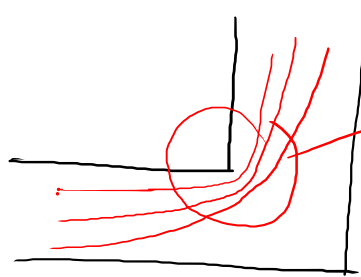
τ

v

LA DISTR DI τ COINCIDE CON IL CAMPO DI VELOCITA' v DI UN FLUIDO PERFETTO CONTENUTO IN UN RECIPIENTE AVENTE LA FORMA DELLA SEZIONE IN MOTO LAMINARE E STAZIONARIO (DI ROTAZIONE).

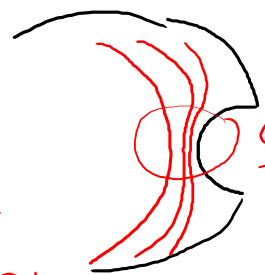


v $\leftrightarrow$ τ



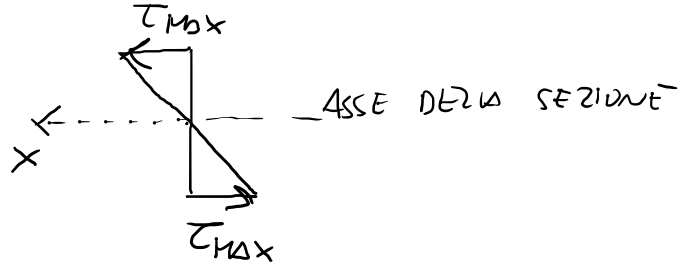
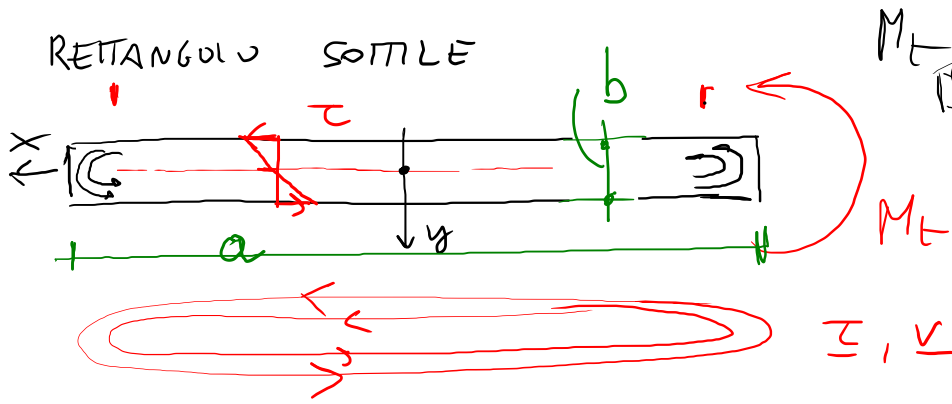
CONCENTRAZIONE
DEGLI SFORZI

(SPIGOLI DA EVITARE)



INTAGLIO

CONCENTRAZIONE
DEGLI SFORZI
 τ



DALLA STUDIO DELLE SEZ.

RETTANGOLARI SI OTTIENE CHE
PER IL PROFILO SOTTILE :

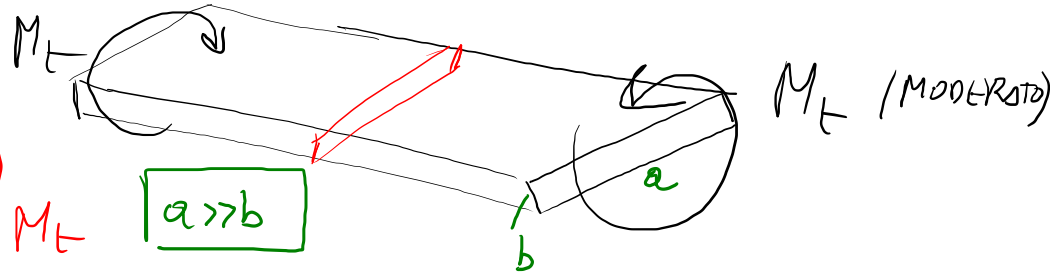
$$\tau_{max} = 3 \frac{M_t}{ab^2} = \frac{M_t}{J_t} b$$

perché

$$J_t = \frac{1}{3} ab^3$$

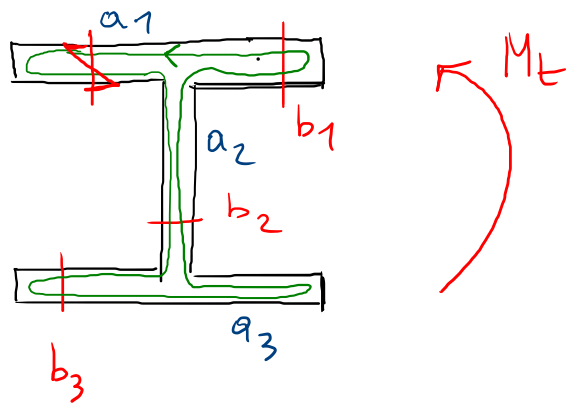
FAITORE J_t PER I
RETTANGOLI SOTTILI

$$\Theta = M_t / G J_t$$



$$G = \frac{E}{2(1+\nu)}$$

ES



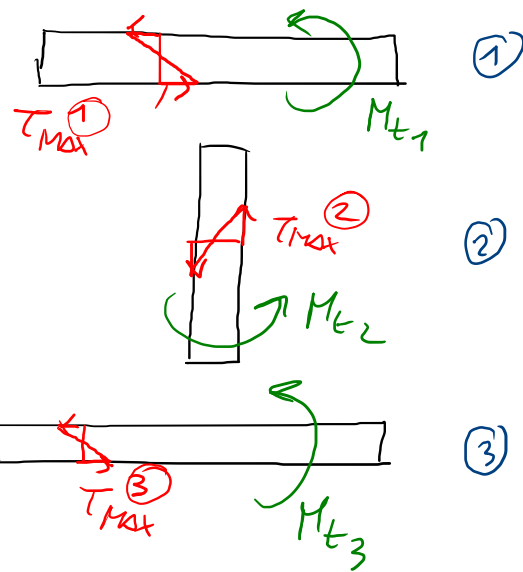
$$\sum_{i=1}^3 M_{t_i} = M_t \quad \text{EQUIVALENZA}$$

$$\tau_{\max}^{(i)} = \frac{M_{t_i}}{J_{t_i}} b_i$$

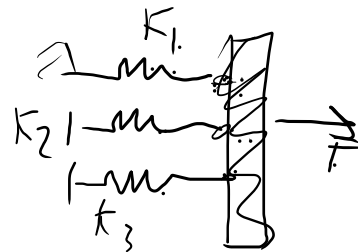
È NECESSARIO RIPARTIRE M_t
NEL 3 RETTANGOLI SOTTILI?

COME?
$$\textcircled{H}_1 = \textcircled{H}_2 = \textcircled{H}_3 = \textcircled{H}$$

SEZ. SOTTILE
"APERTA"



UGUAGLIANZA
DEGLI ANGOLI
UNITARI
DI TORSIONE.



$$\textcircled{H}_i = \frac{M_{ti}}{G J_{ti}} \quad (\text{NOTO})$$

È UGUALE A $\textcircled{H}$



J_t (TOTALE DELLA SEZIONE)

$$M_t = \sum_i M_{ti} = \sum_i G J_{ti} \textcircled{H}_i = G \textcircled{H} \sum_i J_{ti}$$

G
CONSTANTE

$$\textcircled{H}_i = \textcircled{H} \Rightarrow$$

$$\frac{M_{ti}}{J_{ti}} = \text{costante} \Rightarrow$$

$$\frac{M_{t1}}{J_{t1}} = \frac{M_{t2}}{J_{t2}} = \frac{M_t}{J_t} \quad *$$

$$M_{t1} = \frac{M_t}{J_t} J_{t1} \Rightarrow$$

$$M_{ti} = M_t \frac{J_{ti}}{J_t}$$

M_t SI RIPARTISCE PROPORZIONALM.
AL RAPPORTO TRA J_{ti} E J_t

$$\tau_{\max}^{\textcircled{i}} =$$

$$\frac{M_{ti}}{J_{ti}} \quad *$$

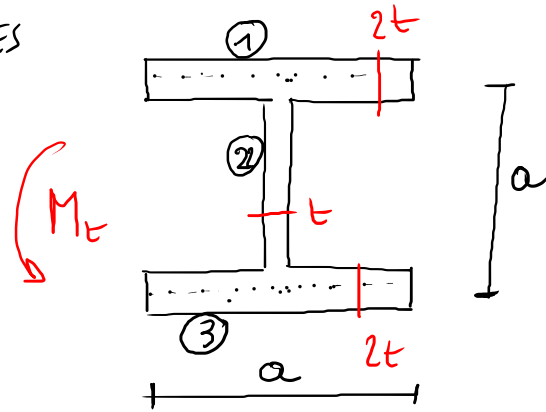
$$b_i = \frac{M_t}{J_t} b_i$$

$$\tau_{\max} (\text{SEZ.}) = \max \{ \tau_{\max}^{\textcircled{i}} \} = \frac{M_t}{J_t} \max \{ b_i \}$$

$\tau_{\max}$ delle MAX

MASSIMO
SPESORE

IES



$$J_t = \sum_i J_{t_i} = \frac{1}{3} a (2t)^3 + \frac{1}{3} a t^3 + \frac{1}{3} a (2t)^3$$

$$= \frac{17}{3} a t^3$$

$$M_{t1} = M_{t3} = M_t \frac{J_{t1}}{J_t} = M_t \frac{\frac{8}{3} a t^3}{\frac{17}{3} a t^3} = \frac{8}{17} M_t$$

$$M_{t2} = M_t \frac{J_{t2}}{J_t} = M_t \frac{\frac{1}{3} a t^3}{\frac{17}{3} a t^3} = \frac{1}{17} M_t$$

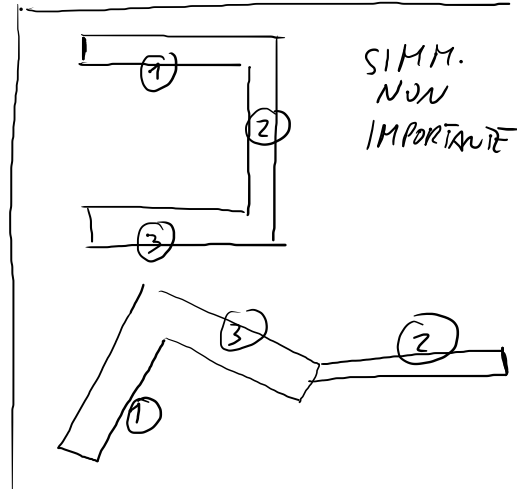
$$J_{t1} = J_{t3} = \frac{8}{3} a t^3$$

$$J_{t2} = \frac{1}{3} a t^3$$

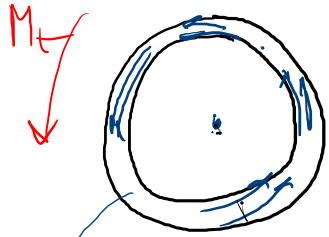
$$\tau_{\max}^{(1)} = \frac{M_{t1}}{J_{t1}} 2t = \frac{8}{17} M_t \frac{3}{8 a t^3} 2t = \frac{6}{17} \frac{M_t}{a t^2}$$

$$\tau_{\max}^{(3)} \quad \tau_{\max}^{(2)} = \frac{M_{t2}}{J_{t2}} t = \frac{1}{17} M_t \frac{3}{a t^3} t = \frac{3}{17} \frac{M_t}{a t^2} \quad \left(= \frac{1}{2} \tau_{\max}^{(1)} \right)$$

$\tau_{\max} - \max$



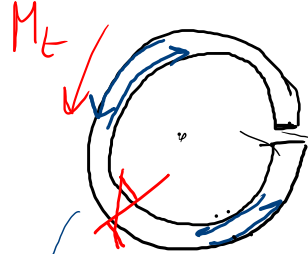
INTRODUZIONE AI PROFILI SOTTILI CHIUSI



SEZ. "CHIUSA"

FLUSSO QUASI
CONSTANTE CHE
RUOTA ATTORNO
AL BARICENTRO

FORTE



SEZ. "APERTA"

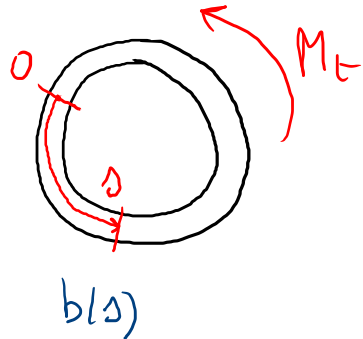
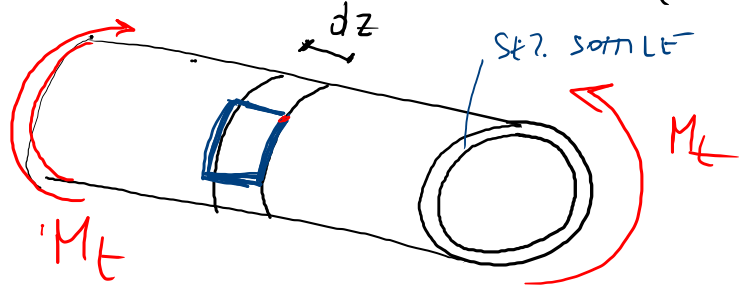
FLUSSO "A FORFALLO"
TIPICO DELLE SEZ.
APERTE

DEBOLE

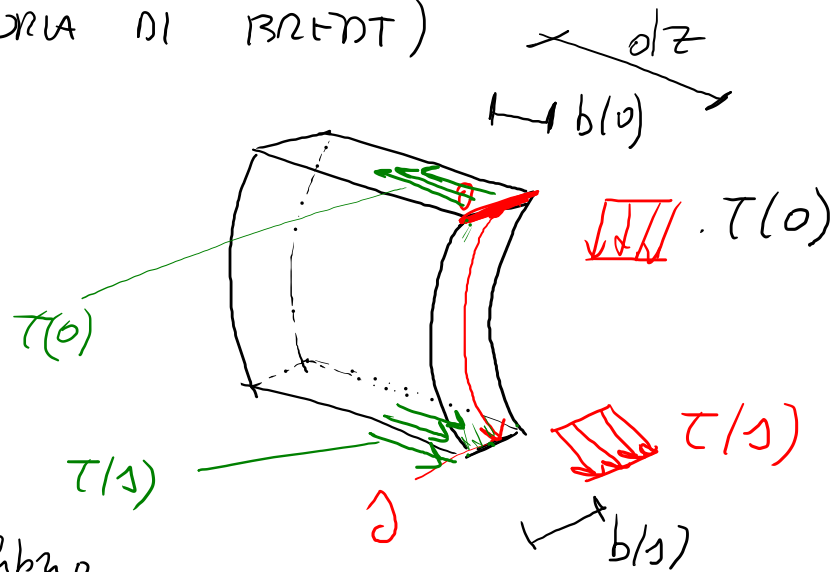
- COMPORT. MOLTO DIVERSO
- DUE SEZIONI
ALLA SOLLECITAZ. DI
TORSIONE

SEZIONI SOTTILI

CHIUSE (TEORIA DI BRENT)



$\tau(s)$ (costante nello spessore)



Per l'equilibrio
LONGITUDINALE :

$$dF(0) = dF(s)$$

$$(dF(0) - dF(s) = 0)$$

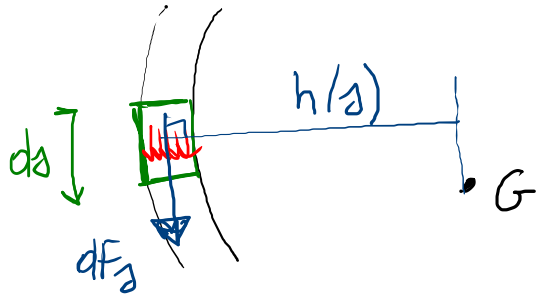
$$\cancel{\tau(0) b(0) dz} = \cancel{\tau(s) b(s) dz}$$

$$\tau(0) b(0) = \tau(s) b(s)$$

$$\tau(s) b(s) = \text{cost}, \forall s$$

costanza del
FLUSSO D'ELF
 τ

Come legare M_t e $\tau(s)$? POSSO APPLICARE L'EQU. CDS DEL M_t



SULLA SUPERFICIE $b(s)ds$ INSISTE
UNA FORZA $\tau(s)b(s)ds = dF_s$

$$dM(s) = \tau(s)b(s)ds \cdot h(s)$$

↑
MOM INFINITESIMO
RISPETTO A G

$$M = \oint \underbrace{\tau(s)b(s)}_{\text{CONSTANTE}} h(s) ds = M_t$$

$$\tau(s)b(s) \underbrace{\oint h(s) ds}_{2\Omega} = M_t$$

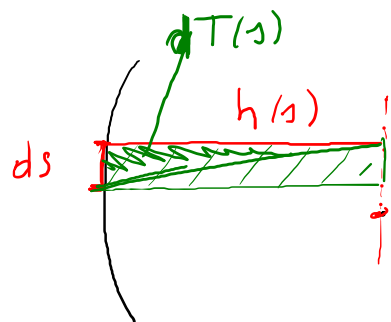
$$\tau(s) = \frac{M_t}{2\Omega b(s)}$$

FORMULA DI BREDT

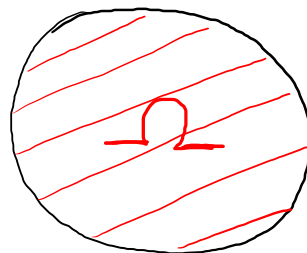
$$\tau = \text{cost se } b(s) = \text{cost}$$

$$\tau = \frac{M_t}{2\Omega b}$$

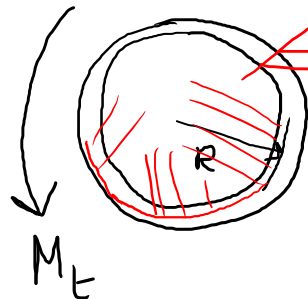
Ω : AREA
RACCHIUSA
DAL CIRCUITO
CHIUSO
(COMPRESO
ALL'INTERNO
DELLA LINEA
MEDIA)



$$h(s) ds$$



$$\begin{aligned}\Omega &= \oint dT(s) \\ &= \oint \frac{1}{2} h(s) ds \\ 2\Omega &= \oint h(s) ds\end{aligned}$$



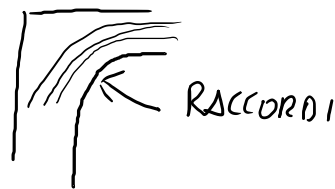
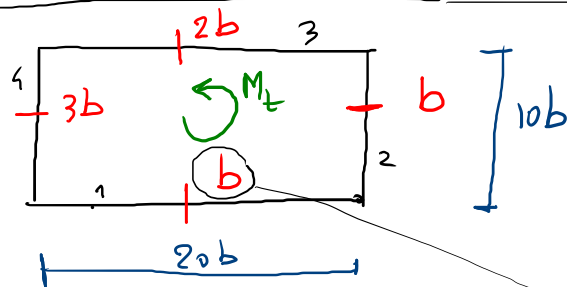
$$b = \frac{R}{10} \cos t$$

$$T(s) ?$$

b : costante ; T costante

$$\Omega = \pi R^2$$

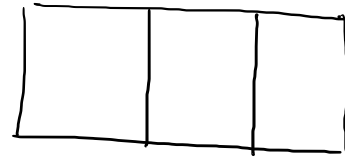
$$\tau = \frac{M_t}{2\pi R^2 \cdot \frac{R}{10}} = \frac{5}{\pi} \frac{M_t}{R^3}$$



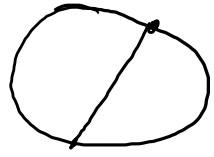
$$\tau_{max} ? \quad \tau_1 \text{ o } \tau_2$$

$$\tau_1 = \frac{M_t}{2 (\underbrace{20 \cdot 10 b^2}_{\Omega}) \underbrace{(b)}_{\Omega}} = \frac{M_t}{400 b^3}$$

Se ho un profilo PLURICELLULARE
LA TEORIA DI BREDT E' INCOMPLETA



3



2

CIRCUITI CHIUSI