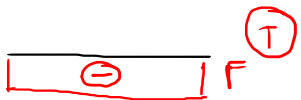
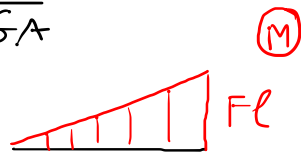
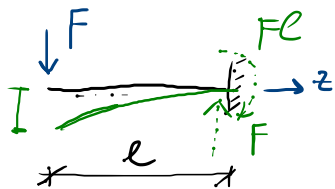


APPROFONDIM. TRAVE DI TIMOSHENKO

SDC 137W, 23/12/23

ULTIMA LEZIONE

$$\begin{cases} \varphi'(z) = \frac{M(z)}{EI} \\ v'(z) = -\varphi(z) + \frac{\kappa T(z)}{GA} \end{cases}$$



$$M(z) = -Fz$$

$$T(z) = -F$$

$$\begin{cases} \varphi'(z) = -\frac{Fz}{EI} \Rightarrow \varphi(z) = -\frac{Fz^2}{2EI} + C_1 \\ \varphi(l) = 0 \Rightarrow C_1 = \frac{Fl^2}{2EI} \end{cases}$$

$$\varphi(z) = -\frac{Fz^2}{2EI} + \frac{Fl^2}{2EI}$$

$$\begin{cases} v'(z) = \frac{Fz^2}{2EI} - \frac{Fl^2}{2EI} - \frac{\kappa F}{GA} \\ v(l) = 0 \end{cases}$$

$$v(z) = \frac{Fz^3}{6EI} - \frac{Fl^2}{2EI}z - \frac{\kappa F}{GA}z + C_2$$

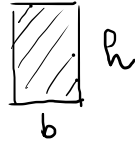
$$v(l) = 0 \quad ; \quad C_2 = \frac{Fl^3}{3EI} + \frac{\kappa Fl}{GA}$$

OSSERVO IL VALORE DI $v(0)$ E DETERMINO IL CONTRIBUTO DEL TERMINE κ/GA - β

$$v(0) = \frac{Fl^3}{3EI} + \frac{\kappa Fl}{GA} = \frac{Fl^3}{3EI} \left(1 + \frac{\kappa}{l^2} \frac{3EI}{GA} \right)$$

$$\beta \rightarrow 0 \quad ; \quad v(0) = \frac{Fl^3}{3EI} \quad ; \quad \text{TH. DI EULERO-BERNOULLI}$$

$$\beta = \frac{3 K E I}{\ell^2 G A}$$



$$I = \frac{b h^3}{12}; A = b h, K = \frac{6}{5}; G = \frac{E}{2(1+\nu)}$$

$$\beta = 3 \cdot \frac{6}{5} \cdot \frac{E}{2(1+\nu)} \cdot \frac{b h^3}{12} \cdot \frac{1}{b h} \cdot \frac{1}{\ell^2} = \frac{3}{5} (1+\nu) \frac{h^2}{\ell^2} \approx \left(\frac{h}{\ell} \right)^2$$



$$h = 10 \text{ cm}$$

$$\ell = 400 \text{ cm}$$

TRAVE SNELLA

$$\beta \approx \left(\frac{1}{40} \right)^2 = \frac{1}{1600} \approx 0.06\%$$

$\ll 1$

DEF. TAGLIANTE
TRASCURABILE



$$h = 100 \text{ cm}$$

$$\ell = 400 \text{ cm}$$

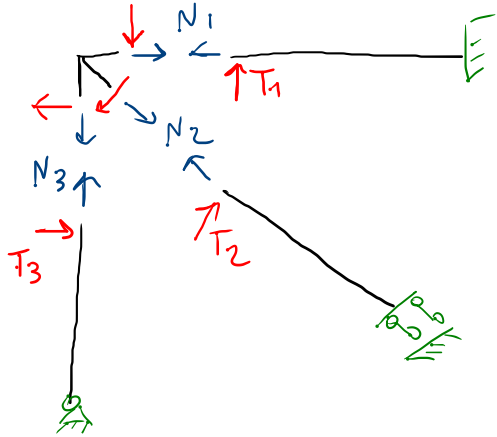
TRAVE TOZZA

$$\beta \approx \left(\frac{1}{4} \right)^2 = \frac{1}{16} \approx 6\%$$

$\ll 1$

DEF. TAGLIANTE
NON TRASCURABILE

SUL CALCOLO DI FORZE NORMALI E TAGLI NEI NODI DI TRAVI

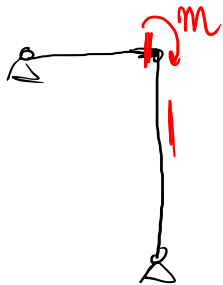


$$\begin{matrix} \uparrow \\ \downarrow \end{matrix} \quad \begin{matrix} \rightarrow \\ \leftarrow \end{matrix} \quad \begin{matrix} + \\ - \end{matrix} ; \sum_i T_{ix} + \sum_i N_{ix} = 0$$

$$+ \uparrow ; \sum_i T_{iy} + \sum_i N_{iy} = 0$$

RELAZ. UTILI
PER DETERMINARE
GLI SFORZI N_i
NELLE ASTE.

SE IL N° DI INCOGNITE N_i SUPERA IL VALORE 2
ALLORA È NECESSARIO CONSIDERARE G.D.C. NODALI
RELATIVI ALLA TRASV. DEL NODO E CONSIDERARE
LA DEFORMABILITÀ ASSIALE.



OSSERVAZ. SULL'ASTA DI EULERO : CENNO AL METODO ω .

$$|\sigma_{cr}| = \frac{P_{cr}}{A} \rightarrow |\sigma_{cr}| = \pi^2 \frac{E}{\lambda^2}$$

Per le aste snelle la VERIFICA DI RESISTENZA diventa
($N < 0$)

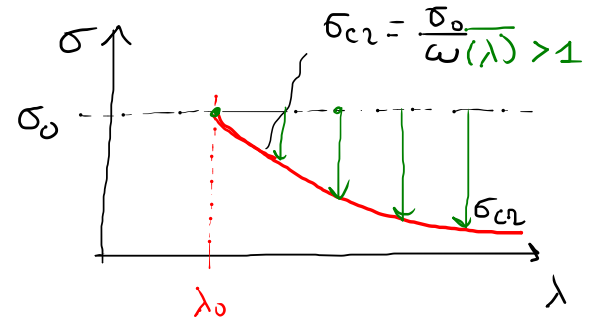
$$\frac{N}{A} \leq \sigma_{cr} \Rightarrow \frac{N}{A} \leq \frac{\sigma_0}{\omega(\lambda)}$$

$$\Rightarrow \omega(\lambda) \frac{N}{A} \leq \sigma_0$$

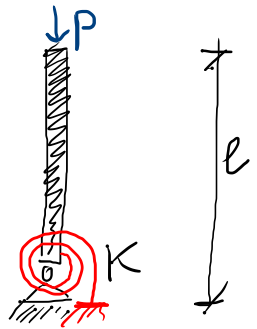
AMPLIFICAZIONE
DELLA COS N

VERIFICA DI
RESISTENZA PER
ELEMENTI COMPRESSI
(METODO ω)

$\omega(\lambda)$: COEFFICIENTE
TABELLATO

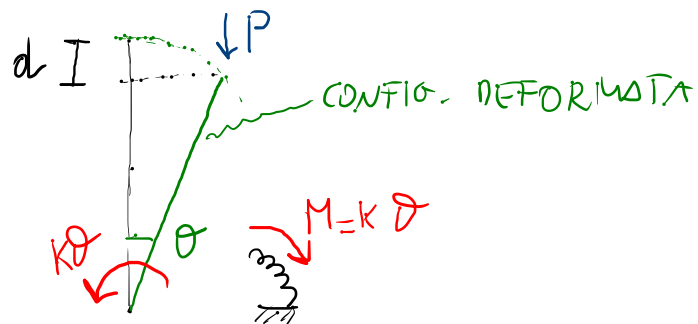
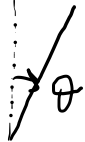


CENNI ALLA STABILITA' DI STRUTTURE RIGIDE AD ELASTICITA' CONCENTRATA



STUDIO CONFIG. DI
EQUIL. AL VALORE
DEL G.D.L. θ

(θ MODERATAMENTE
GRANDE : $M = K\theta$)



$$d = l - l \cos \theta$$

STUDIO L'EQUIL. CON L'EN POT. DEL SIST.

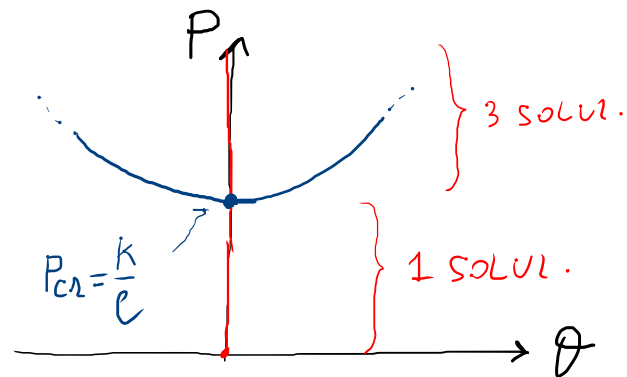
$$\Pi(\theta) = \frac{1}{2} K \theta^2 - P d = \frac{1}{2} K \theta^2 - P l (1 - \cos \theta)$$

$$\frac{d\Pi}{d\theta} = 0 : \boxed{K\theta - Pl \sin \theta = 0}$$

$$\Rightarrow 1) \theta = 0$$

$$\Rightarrow 2) (\theta \neq 0) : P = \frac{K}{l} \frac{\theta}{\sin \theta}$$

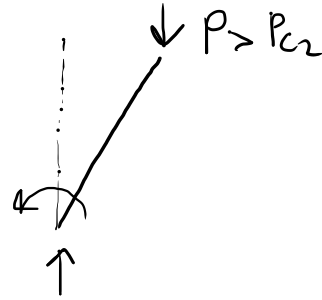
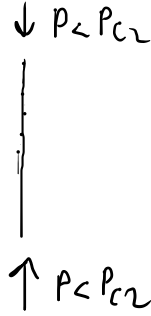
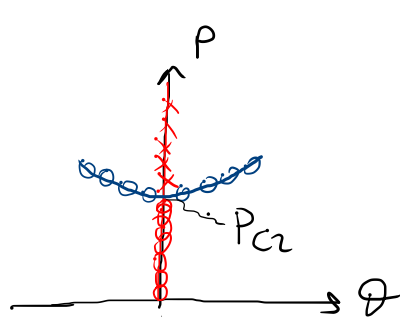
DIAGR. DI BIFURCAZ. (PERCORSI DI EQUIL.)



STABILITÀ $\frac{d^2\pi}{d\theta^2} = K - P \cos\theta$; si valuta il segno delle $\pi''(\theta)$ nei

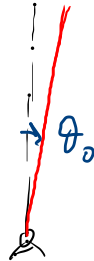
percorsi di equilibrio ottenuti in precedente $\pi''(\theta) > 0$
 < 0

(MINIMO DELL'EN POT.)
 CONFIG. STABILE $\circ$
 " INSTABILE $\times$

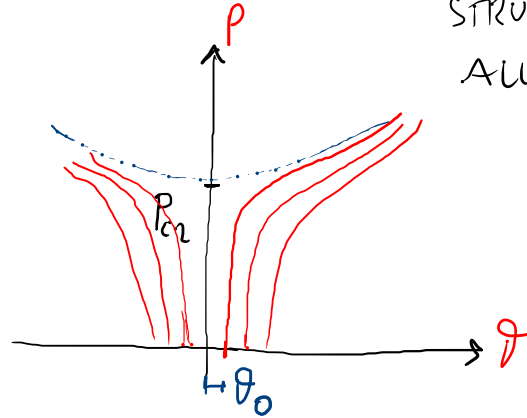


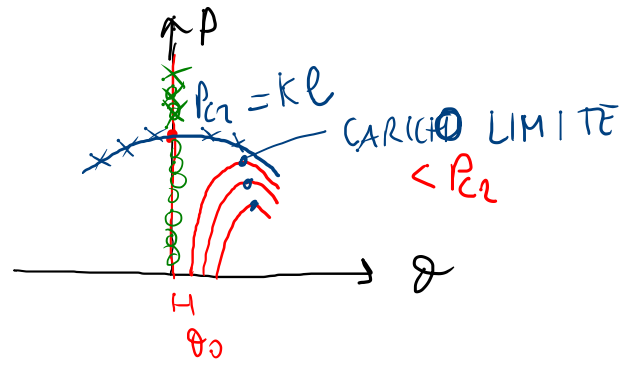
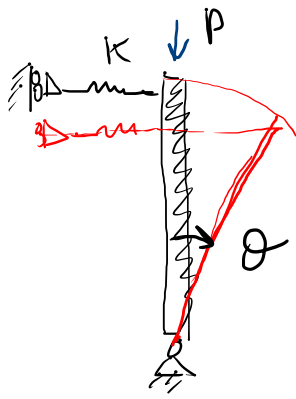
SE CI SONO PICCOLE IMPERFEZIONI NEL SISTEMA, COME SONO I PERCORSI DI EQUILIBRIO?

($\theta = \theta_0$ quando $P = 0$)



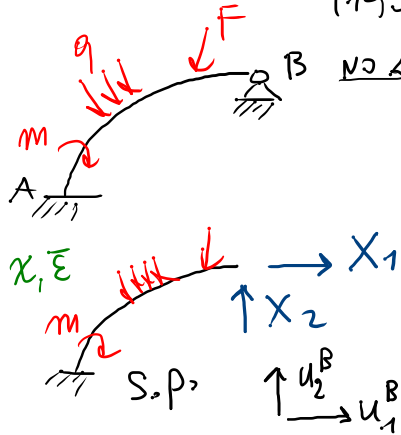
STRUTTURA POCO SENSIBILE
 ALLE IMPERFEZIONI





STRUTTURA SENSIBILE
AUE IMPERFEZIONI

EQ. DI MÜLLER-BRESLAU : FORMALIZZAZ. DEL METODO DELLE FORZE UTILIZZANDO IL P.L.V. (1908)



NO AT 2V. 1 PER ST.

u_1^B

$\rightarrow 1$

SCH. FITTIZIO (*)

M^*, N^*

$$1 \cdot u_1^B = \int_{STR} M^* x_1 ds = \int_{STR} M^* \frac{M}{EI} ds = \int_{STR} M^* \frac{1}{EI} (M^0 + x_1 M^* + x_2 M^{**}) ds \Rightarrow$$

OSSERVO CHE PER LA STR REALE.....

EQ. DI CONGRUENZA

$$\begin{cases} u_1^B = 0 \\ u_2^B = 0 \end{cases}$$

$$u_1^B, u_2^B = f(q, m, F; x_1, x_2)$$

$$L_{ve} = L_{vi}$$

$$[M = M^0 + x_1 M^* + x_2 M^{**}]$$

$$\text{Diagram showing the beam structure with unit loads } x_1 \text{ and } x_2 \text{ at supports A and B respectively, and the resulting internal forces } M^0, M^*, \text{ and } M^{**}.$$

CARICHI
ESPERIMENTI

$$u_1^B = \underbrace{\int_{sm} \frac{M^* M^0}{EI} ds}_{u_{10}} + X_1 \underbrace{\int_{sm} \frac{M^{*2}}{EI} ds}_{\eta_{11}} + X_2 \underbrace{\int_{sm} \frac{M^* M^{**}}{EI} ds}_{\eta_{12}}$$

$\Downarrow$
 $= 0$
 (eq di congr.)

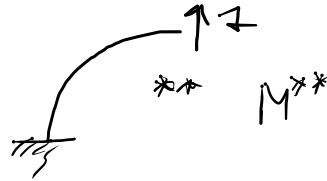
$$0 = u_{10} + X_1 \eta_{11} + X_2 \eta_{12}$$

sviluppo lo stesso rag. con u_2^B

$$u_2^B = \underbrace{\int_{sm} \frac{M^{**} M^0}{EI} ds}_{u_{20}} + X_1 \underbrace{\int_{sm} \frac{M^{**} M^*}{EI} ds}_{\eta_{21}} + X_2 \underbrace{\int_{sm} \frac{M^{**2}}{EI} ds}_{\eta_{22}}$$

$\Downarrow$
 CONGR.

$$0 = u_{20} + X_1 \eta_{21} + X_2 \eta_{22}$$



$$\begin{cases} 0 = u_{10} + \eta_{11} X_1 + \eta_{12} X_2 \\ 0 = u_{20} + \eta_{21} X_1 + \eta_{22} X_2 \end{cases}$$

$\Rightarrow X_1, X_2$

$$0 = u_{i0} + \sum_{j=1}^m \eta_{ij} X_j \quad (i=1, \dots, m)$$

m
 EQUAZ.

EQ. DI MÜLLER-BRESCHAU

PROPRIETÀ DEI COEFF. η_{ij}

$$\eta_{ii} > 0$$

$$\eta_{ij} = \eta_{ji}$$

$\Downarrow$
 COEFF. DI INFLUENZA

NEL CASO FOSSE IMPORT. CONSIDERARE LA DEF. ASSIEME ALLORA:

$$\eta_{12} = \int_{SM} M^* \frac{M^{**}}{EI} + N^* \frac{N^{**}}{EA} ds$$

$$\eta_{22} = \int_{SM} \frac{M^{**2}}{EI} + \frac{N^{**2}}{EA} ds$$

IN TERMINI MATRICIALI

$$\begin{bmatrix} 0 \end{bmatrix} = \underbrace{\begin{bmatrix} u_{10} \\ a_{20} \\ \vdots \end{bmatrix}}_{\underline{u}_0} + \underbrace{\begin{bmatrix} \eta_{11} & \eta_{1m} \\ \eta_{m1} & \eta_{mm} \end{bmatrix}}_{\underline{H}} \underbrace{\begin{bmatrix} X_1 \\ \vdots \\ X_m \end{bmatrix}}_{\underline{X}}$$

↑
TERMINI NOTI (SPOST. INDOTTI DALLE FORZE ATTIVE)

$\underline{H} \sim \begin{cases} \text{SIMMETRICA} \\ \text{DEF. POSITIVA} \end{cases}$; MATRICE DI CEEDEVOLEZZA o FLESSIBILITA'

$$\underline{H}^{-1} = \underline{K}$$