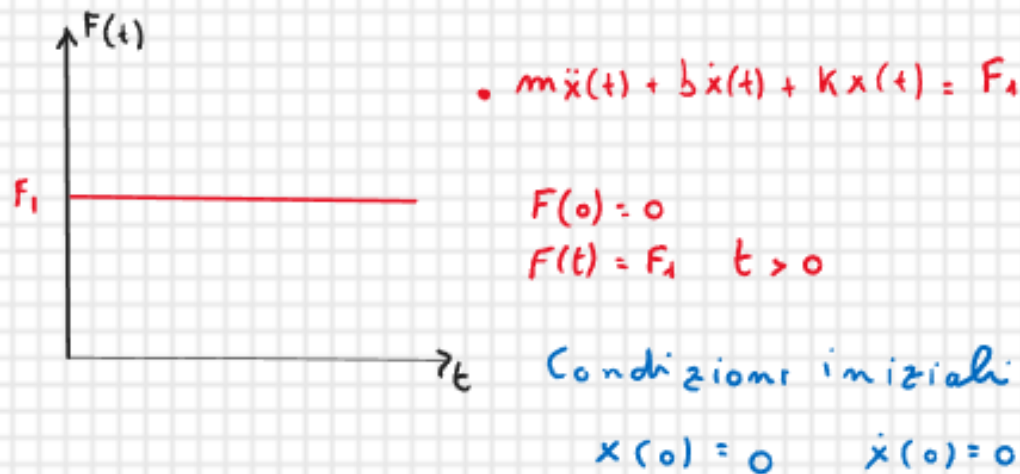


Titolo:

$$m\ddot{x} + b\dot{x} + kx = F(t) \quad \text{Funzione qualsiasi}$$
$$= -m\ddot{y}(t)$$

Integrale di Duhamel



Soluzione eq. moto

$$x(t) = e^{-\gamma\omega t} \left[A \sin \omega t + B \cos \omega t \right] + \frac{F_1}{k}$$

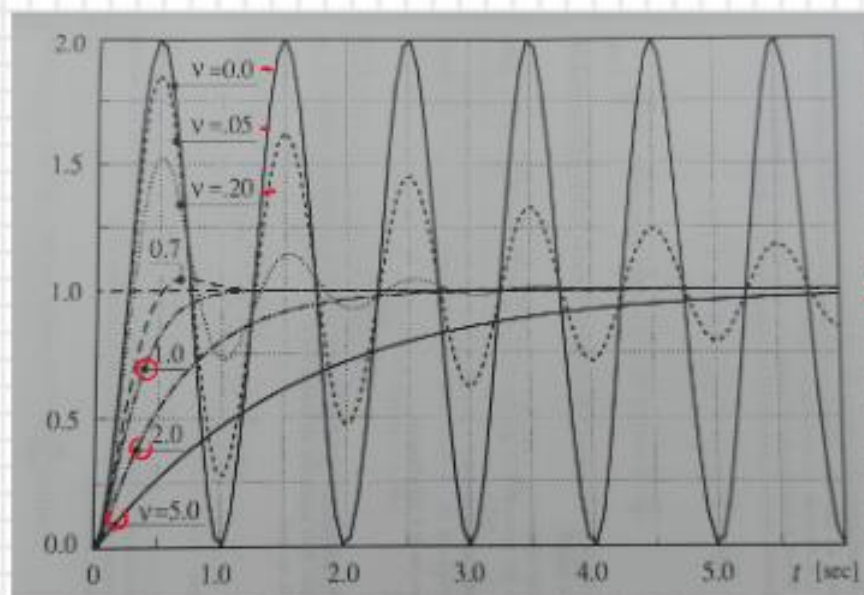
Condizioni al contorno A e B

- $x(0) = 0 \Rightarrow B = -\frac{F_1}{k}$
- $\dot{x}(0) = 0 \Rightarrow A = -\frac{F_1}{k} \frac{\gamma\omega}{\omega}$

$$\dot{x}(t) = -\gamma\omega e^{-\gamma\omega t} (A \sin \omega t + B \cos \omega t) + e^{-\gamma\omega t} (A \omega \cos \omega t - \omega B \sin \omega t)$$

$$x(t) = \frac{F_1}{k} \left[1 - e^{-\gamma\omega t} \left(\frac{\gamma\omega}{\omega} \sin \omega t + \cos \omega t \right) \right] \quad \begin{matrix} t \rightarrow \infty \\ x(t) \rightarrow \frac{F_1}{k} \end{matrix}$$

Titolo:



$F(t)$

F_1

$t = St$

t

$$t = St \rightarrow -F_1$$

$$x(t) = \frac{F_1}{k} \left[1 - e^{-v\omega t} \left(\frac{v\omega}{\Omega} \sin \Omega t + \cos \Omega t \right) \right] +$$

$$- \frac{F_1}{k} \left[1 - e^{-v\omega(t-St)} \left(\frac{v\omega}{\Omega} \sin \Omega(t-St) + \cos \Omega(t-St) \right) \right]$$

$$t > St$$

Risposta all'impulso $F_1 St$

$$g(t) = \frac{F_1}{k} \left[1 - e^{-v\omega t} \left(\frac{v\omega}{\Omega} \sin \Omega t + \cos \Omega t \right) \right]$$

$$x(t) = g(t) - g(t-St)$$

$St \rightarrow \text{infinitesimo}$

$$x(t) = \frac{dg(t)}{dt} St$$

$$\frac{dg(t)}{dt} = -\frac{F_1}{k} \left[-v\omega e^{-v\omega t} \left(\frac{v\omega}{\Omega} \sin \Omega t + \cos \Omega t \right) + e^{-v\omega t} (v\omega \cos \Omega t - \Omega \sin \Omega t) \right]$$

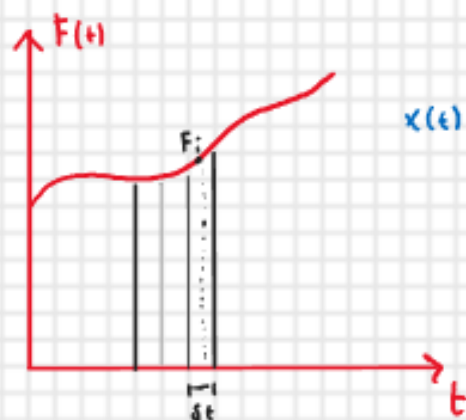
Titolo:

$$x(t) = \frac{F_1}{k} \left(\omega + \frac{v' \omega^2}{\omega} \right) e^{-v \omega t} \sin \omega t \quad \delta t = \frac{F_1}{m \omega} e^{-v \omega t} \sin \omega t \quad \delta t$$

Spostamento dovuto all'impulso applicato a $t=0$

Impulso agisce a $t=t_1$

$$x(t) = \frac{F_1}{m \omega} e^{-v \omega (t-t_1)} \sin \omega (t-t_1) \quad t > t_1$$



$$x(t) = \int_0^t \frac{F(t_1)}{m \omega} e^{-v \omega (t-t_1)} \sin \omega (t-t_1) dt_1$$

Integrale di Duhamel

Risposta del nostro sistema ad una forzante $F(t)$

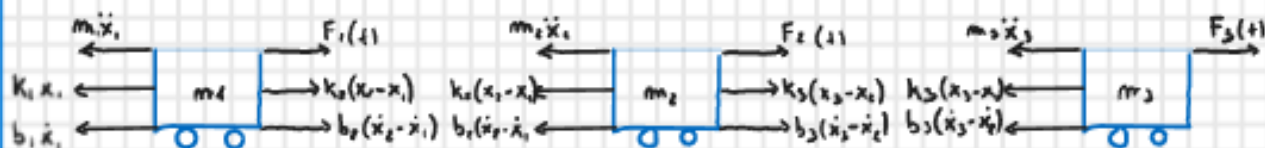
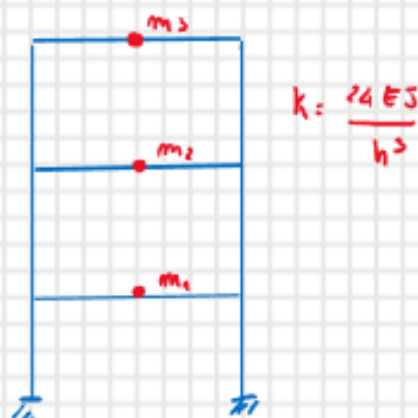
• Esempio semplice $F(t) = p_0$ $v=0$ $\omega = \omega$

$$x(t) = \frac{1}{m \omega} \int_0^t p_0 \sin \omega (t-t_1) dt_1 = \frac{p_0}{m \omega^2} \cos \omega (t-t_1) \Big|_0^t = \frac{p_0}{m \omega} (1 - \cos \omega t)$$

Integriamo rispetto a t_1

Titolo:

Sistemi a n gradi di libertà



$$-m_1 \ddot{x}_1 - k_1 x_1 - b_1 \dot{x}_1 + k_2 (x_2 - x_1) + b_2 (\dot{x}_2 - \dot{x}_1) + F_1 = 0$$

- $m_1 \ddot{x}_1(t) + (b_1 + b_2) \dot{x}_1(t) - b_2 \dot{x}_2(t) + (k_1 + k_2) x_1(t) - k_2 x_2(t) = F_1(t)$
- $m_2 \ddot{x}_2(t) - b_2 \dot{x}_1(t) + (b_2 + b_3) \dot{x}_2(t) - b_3 \dot{x}_3(t) - k_2 x_1 + (k_2 + k_3) x_2 - k_3 x_3 = F_2(t)$
- $m_3 \ddot{x}_3(t) - b_3 \dot{x}_2(t) + b_3 \dot{x}_3 - k_3 x_2 + k_3 x_3 = F_3(t)$

$$\begin{bmatrix} m_1 & 0 & 0 \\ 0 & m_2 & 0 \\ 0 & 0 & m_3 \end{bmatrix} \begin{Bmatrix} \ddot{x}_1 \\ \ddot{x}_2 \\ \ddot{x}_3 \end{Bmatrix} + \begin{bmatrix} b_1 + b_2 & -b_2 & 0 \\ -b_2 & b_2 + b_3 & -b_3 \\ 0 & -b_3 & b_3 \end{bmatrix} \begin{Bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{Bmatrix} + \begin{bmatrix} k_1 + k_2 & -k_2 & 0 \\ -k_2 & k_2 + k_3 & -k_3 \\ 0 & -k_3 & k_3 \end{bmatrix} \begin{Bmatrix} x_1 \\ x_2 \\ x_3 \end{Bmatrix} = \begin{Bmatrix} F_1 \\ F_2 \\ F_3 \end{Bmatrix}$$

$$[M] \{\ddot{x}(t)\} + [b] \{\dot{x}(t)\} + [k] \{x(t)\} = \{F(t)\}$$

↑
Matrice
massa

↑
smorzamento

↑
rigidezza

Titolo:

$$[M] \{\ddot{x}(t)\} = \text{Forze d'inerzia}$$

$$[b] \{\dot{x}(t)\} = \text{Forze viscosse} \quad \{F\} = \text{forzanti}$$

$$[k] \{x(t)\} = \text{Forze elastiche}$$

Studi per oscillazioni libere

$$[M] \{\ddot{x}(t)\} + [k] \{x(t)\} = \{0\}$$

La soluzione

$$\{x(t)\} = \sum_{i=1}^m \{\psi^i\} C_i \sin(\omega_i t + \varphi_i) \cdot$$

C_i e $\varphi_i \Rightarrow$ condizioni iniziali

$$\{x(t)\} = \{\psi^i\} C_i \sin(\omega_i t + \varphi_i) \cdot$$

$$\{\ddot{x}(t)\} = -\{\psi^i\} \omega_i^2 C_i \sin(\omega_i t + \varphi_i) = -\omega_i^2 \{x(t)\}$$

$$-\omega_i^2 [M] \{\psi^i\} C_i \sin(\omega_i t + \varphi_i) + [k] \{\psi^i\} C_i \sin(\omega_i t + \varphi_i) = 0$$

$$-\omega_i^2 [M] \{\psi^i\} + [k] \{\psi^i\} = \{0\}$$

Titolo:

$$\bullet \left(-\omega_i^2 [M] + [K] \right) \{\psi^i\} = \{0\}$$

↑
incognite

Affinché si abbiano soluzioni
non banali del sistema omogeneo

$$\det \left(-\omega_i^2 [M] + [K] \right) = 0 \quad \text{eq. caratteristica di } n \omega_i^2$$

$\omega_i^2 \Rightarrow$ reali e positive

$$\omega_1^2 \leq \omega_2^2 \leq \dots \leq \omega_n^2$$

$$T_1^2 \geq T_2^2 \geq \dots \geq T_n^2$$

ω_1 (la più piccola)
sarà la pulsazione
fondamentale

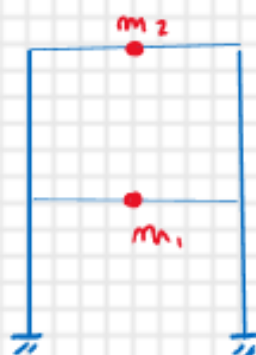
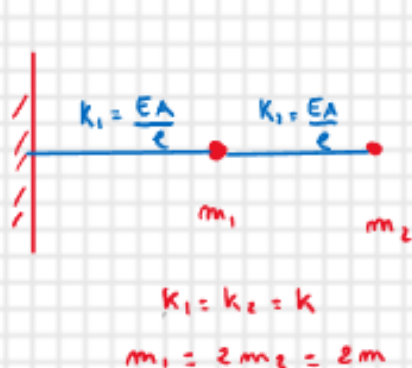
$$\omega_i \Rightarrow \{\psi^i\}$$

$$\{x(t)\} = \sum_{i=1}^n \{\psi^i\} C_i \sin(\omega_i t + \varphi_i)$$

$$\{x(t)\} = \{\psi^i\} C_i \sin(\omega_i t + \varphi_i)$$

Se il sistema vibra secondo un modo di
vibrare i valori max e nulli dello
spostamento si verificano nello stesso istante

Titolo:



$$K_i = \frac{24ES}{h^3}$$

- $m_1 \ddot{x}_1(t) + (k_1 + k_2)x_1(t) - k_2 x_2(t) = 0$
- $m_2 \ddot{x}_2(t) - k_2 x_1 + k_2 x_2 = 0$

$$\begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix} \begin{Bmatrix} \ddot{x}_1 \\ \ddot{x}_2 \end{Bmatrix} + \begin{bmatrix} k_1 + k_2 & -k_2 \\ -k_2 & k_2 \end{bmatrix} \begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

$$(-\omega_i^2 [M] + [K]) \{\psi^i\} = \{0\}$$

$$\left(-\omega_i^2 m \begin{bmatrix} 2 & 0 \\ 0 & 1 \end{bmatrix} + \begin{bmatrix} 2k & -k \\ -k & k \end{bmatrix} \right) \{\psi^i\} = \{0\}$$

$$\begin{pmatrix} -2\omega_i^2 m + 2k & -k \\ -k & -\omega_i^2 m + k \end{pmatrix} \{\psi^i\} = \{0\} \quad \det \begin{pmatrix} -2\omega_i^2 m + 2k & -k \\ -k & -\omega_i^2 m + k \end{pmatrix} = 0$$

$$2\omega_i^4 m^2 - 4\omega_i^2 m k + k^2 = 0 \quad \text{eq. caratteristica}$$

$$\omega_i^2 = \frac{k}{m} \frac{2 \pm \sqrt{2}}{2}$$

$$\underline{\omega_1^2 = \frac{k}{m} \frac{2 - \sqrt{2}}{2}}$$

$$\underline{\omega_2^2 = \frac{k}{m} \frac{2 + \sqrt{2}}{2}}$$

Titolo:

$$\bullet \omega_1^2 = \frac{k}{m} \frac{2-\sqrt{2}}{2}$$

Lo inseriamo nel sistema e calcoliamo gli autovettori

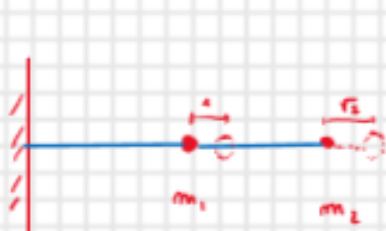
$$\begin{cases} (-2\omega_1^2 m + 2k) \psi_1^{(1)} - k \psi_2^{(1)} \\ -k \psi_1^{(1)} + (-\omega_1^2 m + k) \psi_2^{(1)} = 0 \end{cases} \Rightarrow \psi_1^{(1)} = \frac{\psi_2^{(1)}}{\sqrt{2}}$$

$$\{\psi^{(1)}\} = \left\{ \frac{1}{\sqrt{2}} \right\} \Rightarrow \left\{ \frac{1/\sqrt{3}}{\sqrt{2/3}} \right\} \quad \text{1° modo di vibrare}$$

$$\bullet \omega_2^2 = \frac{k}{m} \frac{2+\sqrt{2}}{2} \quad \{\psi^{(2)}\} = \left\{ \frac{1}{-\sqrt{2}} \right\}$$

$$\{x(t)\} = \sum_{i=1}^n \{\psi^{(i)}\} C_i \sin(\omega_i t + \varphi_i)$$

Soluzione
(calcolare
condizioni iniziali)



1° modo di vibrare $\left\{ \frac{1}{\sqrt{2}} \right\}$



2° modo di vibrare $\left(\frac{1}{-\sqrt{2}} \right)$