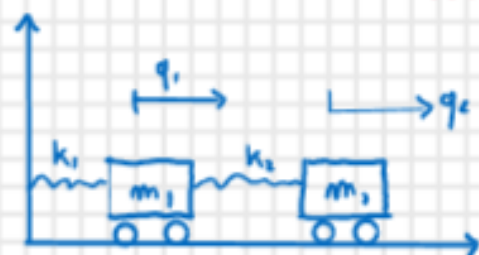


Titolo:

Riassunto Oscillazioni libere sistemi a gradi di libertà



$$\begin{cases} m_1 \ddot{q}_1(t) + (k_1 + k_2)q_1(t) - k_2 q_2(t) = 0 \\ m_2 \ddot{q}_2(t) - k_2 q_1(t) + k_2 q_2(t) = 0 \end{cases}$$

$$[A] \{\ddot{q}(t)\} + [C] \{q(t)\} = \{0\}$$

$$\downarrow$$

$$\bullet \{q\} = \sum_{i=1}^n \{\psi^i\} C_i \sin(\omega_i t + \varphi_i)$$

$$\{q\} = \sum_{i=1}^n \{\psi^i\} (A_i \sin \omega_i t + B_i \cos \omega_i t)$$

$$\downarrow$$

$$(-\omega_i^2 [A] + [C]) \{\psi^i\} = \{0\}$$

$$\downarrow$$

$$\det | -\omega_i^2 [A] + [C] | = 0 \longrightarrow \omega_i^2$$

$$\omega_1^2 \leq \omega_2^2 \leq \dots \leq \omega_n^2$$

Se il sistema oscillasse secondo l'esimo modo di vibrazione

$$\{q\} = \{\psi^i\} (A_i \sin \omega_i t + B_i \cos \omega_i t)$$

$$X = \begin{bmatrix} \psi_1^1 & \dots & \psi_1^n \\ \psi_2^1 & & \psi_2^n \\ \vdots & & \vdots \\ \psi_m^1 & & \psi_m^n \end{bmatrix}$$

Per ciascuna colonna abbiamo gli autovettori

Titolo:

Considerazioni energetiche

- $\{q\} = \{\psi^i\} \sin \omega_i t$
- $\{\dot{q}\} = \omega_i \{\psi^i\} \cos \omega_i t$

- Fissiamo $C_i = 1$
 $\varphi_i = 0$

- Vibra secondo i modi

$$U = \frac{1}{2} \{q\}^T [C] \{q\} = \{\psi^i\}^T [C] \{\psi^i\} \sin^2 \omega_i t$$

$$T = \frac{1}{2} \{\dot{q}\}^T [A] \{\dot{q}\} = \omega_i^2 \{\psi^i\}^T [A] \{\psi^i\} \cos^2 \omega_i t$$

Non abbiamo dissipazione
di energia

$$T + U = \text{cost}$$

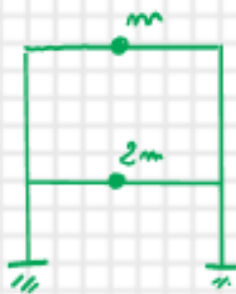
$$T_{\max} + 0 = U_{\max} + 0$$

$$\omega_i^2 \{\psi^i\}^T [A] \{\psi^i\} = \{\psi^i\}^T [C] \{\psi^i\} \Rightarrow \omega_i^2 = \frac{\{\psi^i\}^T [C] \{\psi^i\}}{\{\psi^i\}^T [A] \{\psi^i\}}$$

Rapporto di
Rayleigh

$$\Rightarrow \frac{\{q\}^T [C] \{q\}}{\{q\}^T [A] \{q\}}$$

Esempio



$$\omega_1^2 = \frac{2 - \sqrt{2}}{2} \frac{k}{m}$$

$$\omega_2^2 = \frac{2 + \sqrt{2}}{2} \frac{k}{m}$$

$$m = 10 \text{ kg}$$

$$\{\psi^1\} = \begin{Bmatrix} 1 \\ \sqrt{2} \end{Bmatrix}$$

$$\{\psi^2\} = \begin{Bmatrix} 1 \\ -\sqrt{2} \end{Bmatrix}$$

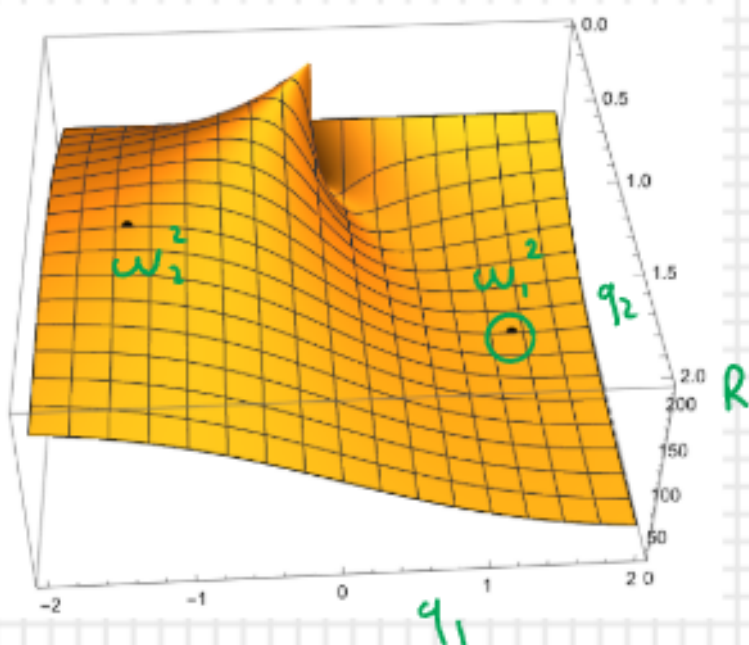
$$k = 1000 \text{ N/m}$$

$$\omega_1^2 = 29,29 \left(\frac{\text{rad}}{\text{s}} \right)^2$$

$$\omega_2^2 = 171,71 \left(\frac{\text{rad}}{\text{s}} \right)^2$$

Titolo:

$$\begin{aligned} \text{Rapporto di Rayleigh} &= \frac{\{q_1 \ q_2\} \begin{bmatrix} 2 & -1 \\ -1 & 1 \end{bmatrix} \begin{Bmatrix} q_1 \\ q_2 \end{Bmatrix}}{\{q_1 \ q_2\} \begin{bmatrix} 2m & 0 \\ 0 & m \end{bmatrix} \begin{Bmatrix} q_1 \\ q_2 \end{Bmatrix}} = \\ &= 100 \frac{2q_1^2 + q_2^2 - 2q_1q_2}{2q_1^2 + q_2^2} \end{aligned}$$



Ortogonalità dei modi di vibrazione

$$\begin{aligned} \bullet & ([C] - \omega_i^2 [A]) \{\psi^i\} = \{0\} & i \\ \bullet & ([C] - \omega_j^2 [A]) \{\psi^j\} = \{0\} & j \end{aligned} \quad i \neq j$$

$$\{\psi^j\}^T ([C] - \omega_i^2 [A]) \{\psi^i\} = \{0\} \quad 1)$$

$$\{\psi^i\}^T ([C] - \omega_j^2 [A]) \{\psi^j\} = \{0\} \quad 2)$$

2 teoremi $\{\psi^j\}^T ([C] - \omega_j^2 [A]) \{\psi^j\} = \{0\}$

Titolo:

Differenza tra la 1 e la 2^a

$$(-\omega_i^2 + \omega_j^2) \{\psi^j\}^T [A] \{\psi^i\} = \{0\}$$

$$i \neq j \quad \omega_i \neq \omega_j$$

- $\{\psi^j\}^T [A] \{\psi^i\} = \{0\}$ 3) condizione di ortogonalità

Sostituendo la 3 in 1

$$\bullet \quad \{\psi^i\}^T [C] \{\psi^i\} = \{0\}$$

gli autovettori $\{\psi\}$ sono ortogonali rispetto alle matrici $[A]$ e $[C]$

Consideriamo il caso $i = j$

$$\{\psi^i\}^T [A] \{\psi^i\} = m_i^* \neq 0$$

massa equivalente a quella di un sistema ad 1 gl con pulsazione ω_i

$$\{\psi^i\}^T [C] \{\psi^i\} = k_i^* \neq 0$$

$$\text{Sistema a gl} \quad \omega_i = \frac{k}{m} \Rightarrow \omega_i^2 = \frac{k_i^*}{m_i^*} = \frac{\{\psi^i\}^T [C] \{\psi^i\}}{\{\psi^i\}^T [A] \{\psi^i\}}$$

Titolo:

Es.

$$\{\psi^2\}^T \begin{bmatrix} 2m & 0 \\ 0 & m \end{bmatrix} \{\psi^1\} = 0 \quad \{\psi^2\}^T [C] \{\psi^1\} = 0$$

$$\{1 \ -\sqrt{2}\} \begin{bmatrix} 2m & 0 \\ 0 & m \end{bmatrix} \left\{ \frac{1}{\sqrt{2}} \right\} = 0 \quad \{1 \ -\sqrt{2}\} [C] \left\{ \frac{1}{\sqrt{2}} \right\} = 0$$

$$\{\psi^1\}^T \begin{bmatrix} 2m & 0 \\ 0 & m \end{bmatrix} \{\psi^1\} = m_1^* \quad \{\psi^1\}^T [C] \{\psi^1\} = K_1^*$$

$$\{1 \ +\sqrt{2}\} \begin{bmatrix} 2m & 0 \\ 0 & m \end{bmatrix} \left\{ \frac{1}{\sqrt{2}} \right\} = 4m \quad \{1 \ +\sqrt{2}\} [C] \left\{ \frac{1}{\sqrt{2}} \right\} = 1,17 K$$

$$\{\psi^2\}^T \begin{bmatrix} 2m & 0 \\ 0 & m \end{bmatrix} \{\psi^2\} = m_2^* \quad \{\psi^2\}^T [C] \{\psi^2\} = K_2^*$$

$$\{1 \ -\sqrt{2}\} \begin{bmatrix} 2m & 0 \\ 0 & m \end{bmatrix} \left\{ \frac{1}{-\sqrt{2}} \right\} = 4m \quad \{1 \ -\sqrt{2}\} [C] \left\{ \frac{1}{-\sqrt{2}} \right\} = 6,82 K$$

$$\omega_1^* = \frac{K_1^*}{m_1^*} = \frac{1,17 K}{4m} \approx 29,283$$

$$\omega_2^* = \frac{K_2^*}{4m} = \frac{6,82 K}{4m} \approx 170,71$$

Disaccoppiamento delle eq. del moto

• Cambio di variabili $\longrightarrow$ nuove coordinate

$$\{q\} = \sum_{i=1}^m \{\psi^i\} p_i = [X] \{p\} \quad p \rightarrow \text{coordinate principali}$$

$$q_1 = \psi_1^1 p_1 + \psi_1^2 p_2 + \dots + \psi_1^m p_m$$

$$q_2 = \psi_2^1 p_1 + \psi_2^2 p_2 + \dots + \psi_2^m p_m$$

$\vdots$

$$q_m = \psi_m^1 p_1 + \psi_m^2 p_2 + \dots + \psi_m^m p_m$$

$$\{p\} = [X]^{-1} \{q\} \quad [X] \text{ invertibile non singolare}$$

Titolo:

$$\{q\} = [x] \{p\} \rightarrow \{\dot{q}\} = [x] \{\dot{p}\}$$

Eq. moto

$$[A] \{\ddot{q}\} + [C] \{\dot{q}\} = \{0\} \bullet$$

↓

$$\underbrace{[x]^T [A] [x]}_{[L]} \{\ddot{p}\} + \underbrace{[x]^T [C] [x]}_{[N]} \{\dot{p}\} = \{0\}$$

$$\begin{bmatrix} \psi_1^1 & \psi_2^1 & \dots & \psi_m^1 \\ \vdots & \vdots & \ddots & \vdots \\ \psi_1^m & \psi_2^m & \dots & \psi_m^m \end{bmatrix} [A] \begin{bmatrix} \psi_1^1 & \dots & \psi_m^1 \\ \psi_2^1 & \dots & \psi_m^2 \\ \vdots & \ddots & \vdots \\ \psi_1^m & \dots & \psi_m^m \end{bmatrix} = \overset{L}{\begin{bmatrix} m_1^* & 0 & \dots & 0 \\ 0 & m_2^* & \dots & 0 \\ \vdots & \vdots & \ddots & \vdots \\ 0 & 0 & \dots & m_m^* \end{bmatrix}}$$

$$l_{ij} = \begin{cases} 0 & i \neq j \\ m_i^* & i = j \end{cases} \quad [L] \text{ matrice diagonale}$$

$$\begin{bmatrix} \psi_1^1 & \psi_2^1 & \dots & \psi_m^1 \\ \vdots & \vdots & \ddots & \vdots \\ \psi_1^m & \psi_2^m & \dots & \psi_m^m \end{bmatrix} [C] \begin{bmatrix} \psi_1^1 & \dots & \psi_m^1 \\ \psi_2^1 & \dots & \psi_m^2 \\ \vdots & \ddots & \vdots \\ \psi_1^m & \dots & \psi_m^m \end{bmatrix} = \overset{N}{\begin{bmatrix} k_1^* & 0 & \dots & 0 \\ 0 & k_2^* & \dots & 0 \\ \vdots & \vdots & \ddots & \vdots \\ 0 & 0 & \dots & k_m^* \end{bmatrix}}$$

$$n_{ij} = \begin{cases} 0 & i \neq j \\ k_i^* & i = j \end{cases} \quad [N] \text{ matrice diagonale}$$

Titolo:

$$m_1^* \ddot{p}_1(t) + k_1^* p_1(t) = 0$$

$$m_n^* \ddot{p}_n(t) + k_n^* p_n(t) = 0$$

$$[X]^T [A] [X] \{\dot{P}\} + [X]^T [C] [X] \{P\} = \{0\}$$

$$[L] \{\ddot{P}\} + [N] \{P\} = \{0\}$$

$$[L]^{-1} [L] \{\ddot{P}\} + [L]^{-1} [N] \{P\} = \{0\}$$

$$\{\ddot{P}\} + [\Omega^2] \{P\} = 0$$

$$[\Omega^2] = \begin{bmatrix} \omega_1^2 & & \\ & \omega_2^2 & \\ & & \omega_n^2 \end{bmatrix}$$

Annotations: $\omega_1^2 \leftarrow \frac{k_1^*}{m_1^*}$, $\omega_2^2 \leftarrow \frac{k_2^*}{m_2^*}$

Soluzione del sistema

$$p_1(t) = A_1 \sin \omega_1 t + B_1 \cos \omega_1 t$$

$$p_2(t) = A_2 \sin \omega_2 t + B_2 \cos \omega_2 t$$

⋮

$$p_n(t) = A_n \sin \omega_n t + B_n \cos \omega_n t$$

$$\Rightarrow \{q\} = [X] \{P\}$$

Sistemi smorzati

$$[A] \{\ddot{q}\} + [B] \{\dot{q}\} + [C] \{q\} = \{0\}$$

$$[X]^T [A] [X] \{\ddot{q}\} + [X]^T [B] [X] \{\dot{q}\} + [X]^T [C] [X] \{q\} = 0$$

$$[L] \{\ddot{q}\} + [M] \{\dot{q}\} + [N] \{q\} = \{0\}$$