

Titolo:

$$[M] = [X]^T [B] [X]$$

Affinché si abbia una matrice $[M]$ diagonale si considera uno smorzamento proporzionale

$$[B] = 2\alpha [C] + 2\beta [A] \rightarrow \text{la matrice } [M] \text{ sarà diagonale}$$

$$m_{ij} = \begin{cases} 0 & i \neq j \\ b_i^* & i = j \end{cases} \quad b_i = 2\alpha k_i^* + 2\beta m_i^*$$

$$m_i^* \ddot{p}_i + b_i^* \dot{p}_i + k_i^* p_i = 0$$

$$\omega_i^2 = \frac{k_i^*}{m_i^*}$$

$$\ddot{p}_i + 2\gamma_i^* \omega_i \dot{p}_i + \omega_i^2 p_i = 0$$

$$\gamma_i^* = \frac{b_i^*}{2\sqrt{k_i^* m_i^*}} = \alpha \omega_i + \frac{\beta}{\omega_i}$$

Soluzione $V_i^* < 1$

$$p_i(t) = e^{-\gamma_i^* \omega_i t} (A_i \sin \Omega_i t + B_i \cos \Omega_i t) \quad \Omega_i = \omega_i \sqrt{1 - \gamma_i^{*2}}$$

— Come ricavare α e β

$$\gamma_i^* = \alpha \omega_i + \frac{\beta}{\omega_i}$$

Risolvi una struttura 4GL

$$\omega_1 = 38,16 \text{ rad/s} \quad \omega_2 = 98,63 \text{ rad/s}$$

$$\omega_3 = 140,6 \text{ rad/s} \quad \omega_4 = 204,05 \text{ rad/s}$$

Fissiamo per 2 modi i e $j \rightarrow \gamma_i^*$ e γ_j^*

$$\gamma_1^* = \gamma_3^* = 5\%$$

$$\begin{cases} 38,16^2 \alpha + \beta = 0,05 \cdot 38,16 \\ 140,6^2 \alpha + \beta = 0,05 \cdot 140,6 \end{cases} \quad \left. \begin{array}{l} \alpha = 2,7 \cdot 10^{-4} \\ \beta = 1,514 \end{array} \right\}$$

Titolo:

$$V_2^* = 4,2\% \quad V_4^* = 6,34\%$$

Oscillazioni prodotte da forze pulsanti

$$[A]\{\ddot{q}\} + [b]\{\dot{q}\} + [c]\{q\} = \{F\} \sin \bar{\omega} t \quad \{F\} = \begin{Bmatrix} F_1 \\ F_2 \\ \vdots \\ F_m \end{Bmatrix}$$

↑
costante

Pre-moltiplichiamo per $[X]^T$

$$\{F^*\} \sin \bar{\omega} t = [X]^T \{F\} \sin \bar{\omega} t$$

$$f_i^* \sin \bar{\omega} t = \{\psi^i\}^T \{F\} \sin \bar{\omega} t$$

$$\bullet \quad m_i^* \ddot{p}_i(t) + b_i^* \dot{p}_i(t) + k_i^* p_i(t) = f_i^* \sin \bar{\omega} t$$

$$p_i(t) = p_{st,i} N_i \sin(\bar{\omega} t - \varphi_i)$$

$$p_{st,i} = \frac{d_i^*}{k_i^*} \quad N_i = \frac{1}{\sqrt{(1 - \beta_i^2)^2 + (2\zeta_i^* \beta_i)^2}} \quad \beta_i = \frac{\bar{\omega}}{\omega_i} \quad \zeta_i^* = \frac{b_i^*}{2\sqrt{k_i^* m_i^*}}$$

$$p_i(t) = p_{0,i}(t) + p_{p,i}(t) \quad p_i(t) \simeq p_{p,i}(t)$$

↑
transitorio

$$\{q\} = \sum_{i=1}^{\infty} \{\psi^i\} p_i(t) \simeq \sum_{i=1}^{\infty} \{\psi^i\} p_{st,i} N_i \sin(\bar{\omega} t - \varphi_i)$$

trascinato il regime
transitorio

Titolo:

Spostamento variabile della base

$$y(t) = \bar{y} \sin \bar{\omega} t$$

$$\ddot{y}(t) = -\bar{\omega}^2 \bar{y} \sin \bar{\omega} t$$

↑
Spostamento alla base

Masses saranno soggette $\{\ddot{q}(t)\} + \{1\} \ddot{y}(t)$

$$[A] (\{\ddot{q}(t)\} + \{1\} \ddot{y}(t)) + [B] \{\dot{q}(t)\} + [C] \{q(t)\} = \{0\}$$

$$[A] \{\ddot{q}(t)\} + [B] \{\dot{q}(t)\} + [C] \{q(t)\} = -[A] \{1\} \ddot{y}(t)$$

$$[A] \{\ddot{q}(t)\} + [B] \{\dot{q}(t)\} + [C] \{q(t)\} = \bar{\omega}^2 [A] \{1\} \bar{y} \sin \bar{\omega} t$$

$$\{F\} = \bar{\omega}^2 [A] \{1\} \bar{y}$$

Forza di trascinamento

- Forza variabile nel tempo $\{F\} s(t)$

$$[A] \{\ddot{q}\} + [b] \{\dot{q}\} + [C] \{q\} = \{F\} s(t)$$

$$[L] \{\ddot{p}\} + [M] \{\dot{p}\} + [N] \{p\} = [X]^T \{F\} s(t)$$

$$m_i \ddot{p}_i + b_i \dot{p}_i + k_i p_i = f_i s(t)$$

$$\{q_0\} = \{0\}, \quad \{p_0\} = \{0\}$$

$$\{\dot{q}_0\} = \{0\}, \quad \{\dot{p}_0\} = \{0\}$$

$$p_i(t) = \frac{f_i}{m_i \omega_i^2} \int_0^t s(t_1) e^{-\nu_i \omega_i (t-t_1)} \sin \omega_i (t-t_1) dt_1$$

Titolo:

Spostamento della base sia $y(t) = s(t)$

$$[A]\{\ddot{q}\} + [b]\{\dot{q}\} + [c]\{q\} = -[A]\{1\}\ddot{s}(t)$$

$$[L]\{\ddot{p}\} + [M]\{\dot{p}\} + [N]\{p\} = -[X]^T[A]\{1\}\ddot{s}(t)$$

$$\bullet m_i^* \ddot{p}_i + b_i^* \dot{p}_i + k_i^* p_i = -\{\psi^i\}^T [A] \{1\} \ddot{s}(t)$$

dividiamo per m_i^*

$$\ddot{p}_i(t) + 2\gamma_i^* \omega_i p_i + \omega_i^2 p_i = - \frac{\{\psi^i\}^T [A] \{1\}}{\{\psi^i\}^T [A] \{\psi^i\}} \ddot{s}(t)$$

$$\ddot{p}_i(t) + 2\gamma_i^* \omega_i p_i + \omega_i^2 p_i = -g_i \ddot{s}(t)$$

$$g_i = \frac{\{\psi^i\}^T [A] \{1\}}{\{\psi^i\}^T [A] \{\psi^i\}}$$

Fattore di
partecipazione

g_i si riducono al crescere dell'indice i

$$\{q(t)\} = \sum \{\psi^i\} p_i(t)$$

Ridurre la sommatoria
ai modi significativi

• Forzante generica $\{F(t)\}$

$$[A]\{\ddot{q}\} + [b]\{\dot{q}\} + [c]\{q\} = \{F(t)\} \quad \vec{F} = \begin{Bmatrix} F_1(t) \\ F_2(t) \\ \vdots \\ F_m(t) \end{Bmatrix}$$

Titolo

Soluzione

$$m_i \ddot{p}_i + b_i \dot{p}_i + k_i p = \sum_{j=1}^m \psi_j^i F_j(t)$$

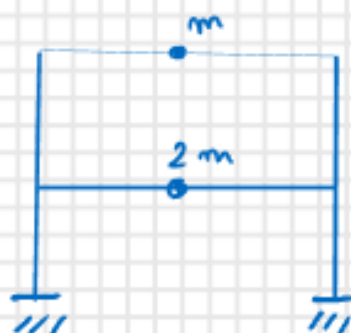
$$p_i(t) = p_{i,1}(t) + p_{i,2}(t) + \dots + p_{i,m}(t)$$

- $p_{i,j}$ soluzione dell'equazione i esima dovuta al j esimo termine

$$p_{i,j}(t) = \frac{\psi_j^i}{m_i \omega_i^2} \int_0^t F_j(t) \frac{e^{v_i w_i(t-t_1)}}{\sin \omega_i(t-t_1)} dt_1$$

$$\{q\} = [X] \{p\}$$

Es



$$\begin{bmatrix} 2m & 0 \\ 0 & m \end{bmatrix} \begin{Bmatrix} \ddot{q}_1 \\ \ddot{q}_2 \end{Bmatrix} + \begin{bmatrix} 2k & -k \\ -k & k \end{bmatrix} \begin{Bmatrix} q_1 \\ q_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

$[A] \qquad [C]$

$$X = \begin{bmatrix} 1 & 1 \\ \sqrt{2} & -\sqrt{2} \end{bmatrix}$$

$\psi^1 \quad \psi^2$

$$L = [X]^T [A] [X] = \begin{bmatrix} 4m & 0 \\ 0 & 4m \end{bmatrix}$$

$$N = [X]^T [C] [X] = \begin{bmatrix} 1,17k & 0 \\ 0 & 6,82k \end{bmatrix}$$

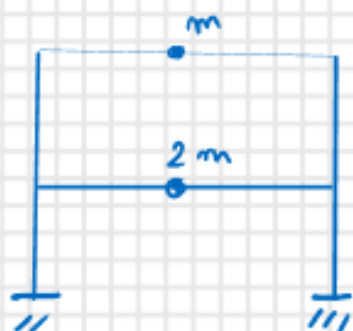
Titolo

$$4m \ddot{p}_1 + 1,17k p_1 = 0 \rightarrow \omega_1^2 = \frac{1,17k}{4m}$$

$$4m \ddot{p}_2 + 6,82k p_2 = 0$$

$$m = 10 \quad k = 11000 \quad \omega_1^2 = 29,29$$

Imprimiamo uno spostamento alla base $y(t)$



$$[A]\{\dot{q}\} + [C]\{q\} = -[A]\{1\}\ddot{y}(t) = \{F(t)\}$$

$$[L]\{\dot{p}\} + [N]\{p\} = -[X]^T[A]\{1\}\ddot{y}(t)$$

$y(t)$

$$\{F(t)\} = -[A]\{1\}\ddot{y}(t) = -\begin{Bmatrix} 2m \ddot{y}(t) \\ m \ddot{y}(t) \end{Bmatrix}$$

$$\{\bar{F}(t)\} = -[X]^T[A]\{1\}\ddot{y}(t) = [X]^T\{F(t)\} = \begin{Bmatrix} -3,41 m \ddot{y}(t) \\ -0,58 m \ddot{y}(t) \end{Bmatrix}$$

$$\begin{cases} 4m \ddot{p}_1 + 1,17k = -3,41 m \ddot{y}(t) \\ 4m \ddot{p}_2 + 6,82k = -0,58 m \ddot{y}(t) \end{cases}$$

Sistema disaccoppiato

* Autovettore normalizzato

$$\{\psi^i\}\{\psi^i\} = 1 \rightarrow \{\tilde{\psi}^i\} = \frac{1}{\|\{\psi^i\}\|} \{\psi^i\}$$

Modulo

$$\{\psi^1\} = \begin{Bmatrix} 1 \\ \sqrt{2} \end{Bmatrix} \quad \{\tilde{\psi}^1\} = \begin{Bmatrix} \frac{1}{\sqrt{3}} \\ \frac{\sqrt{2}}{\sqrt{3}} \end{Bmatrix} = \begin{Bmatrix} 0,577 \\ 0,8164 \end{Bmatrix}$$