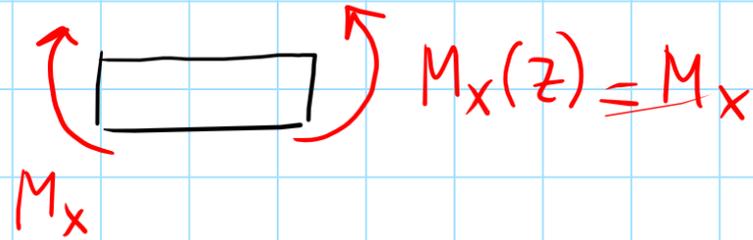
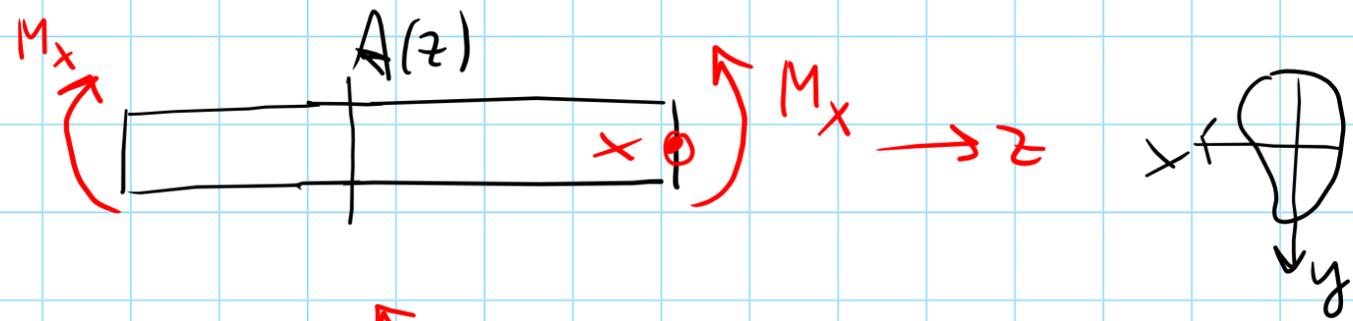


2) FLESSIONE RETTA (σ SEMPLICE)

19/11/25



STATO TENS : $\tau_{xz} = \tau_{yz} = 0$; $\sigma_{zz} \neq 0$

IPOTESI: $\sigma_{zz} = ax + by$ (a, b COST. INCOGNITE)

CDS

$$N(z) = 0 \stackrel{?}{=} \int_{A(z)} \sigma_{zz} dA, \quad 0 = a \int x dA + b \int y dA$$

$S_y = \int y dA = 0$ perché x, y BARICENTR.
 $S_x = \int x dA = 0$

$$M_x(z) = M_x \stackrel{?}{=} \int \sigma_{zz} y dA, \quad M_x = a \int xy dA + b \int y^2 dA \Rightarrow M_x = b J_x \Rightarrow b = \frac{M_x}{J_x}$$

$J_{xy} = 0$ (x, y : PRINCIPALI)

$$M_y = - \int \sigma_{zz} x dA = 0$$

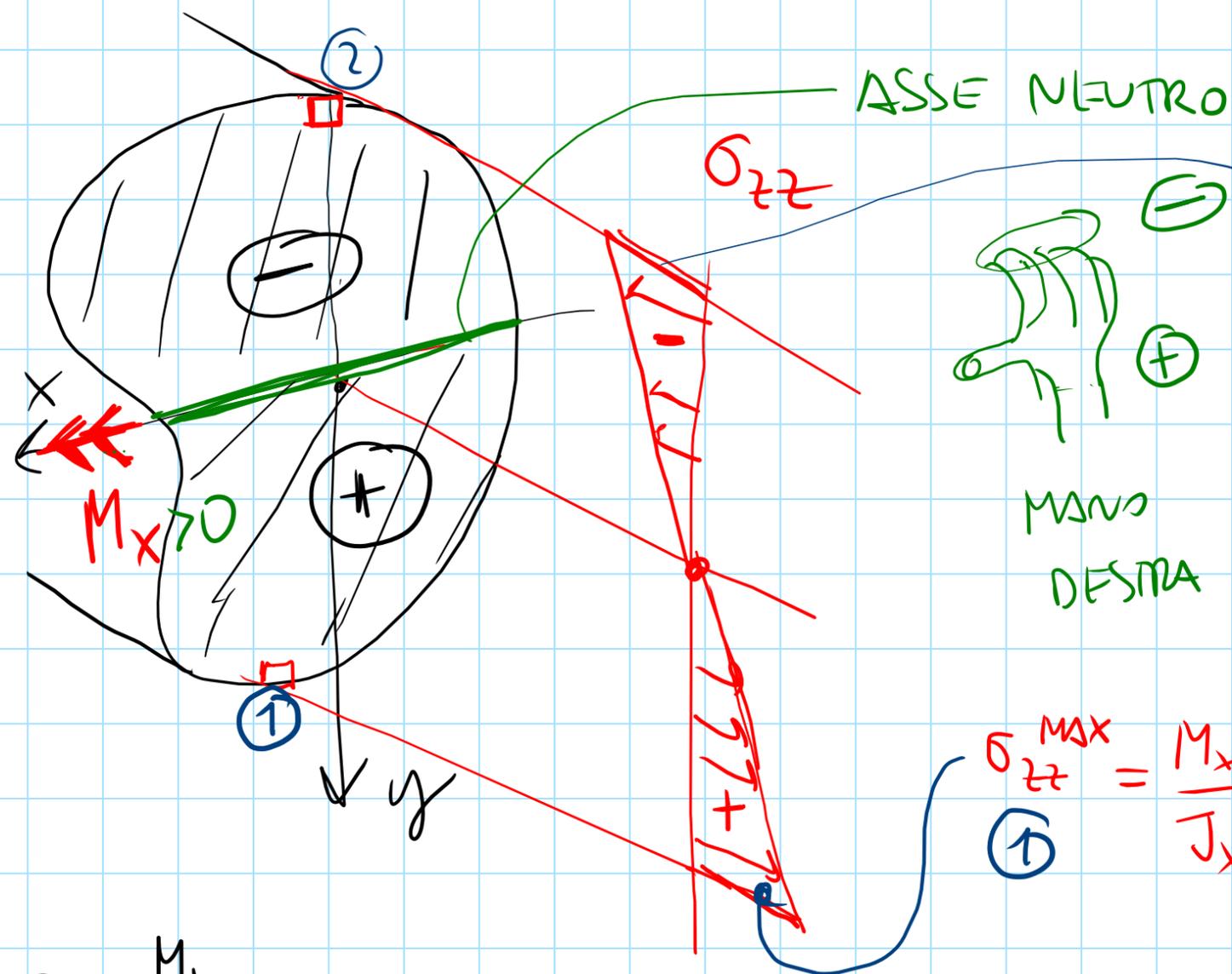
$$- a \int x^2 dA + b \int xy dA = 0$$

$J_y = \int x^2 dA$; $J_{xy} = \int xy dA$

$$\Rightarrow a J_y = 0 ; \Rightarrow a = 0$$

$$\sigma_{zz} = \frac{M_x}{J_x} y$$

FORMULA DI NAVIER



② $\sigma_{zz}^{MIN} = \frac{M_x}{J_x} y_{MIN}$

$$[\sigma] = \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & \frac{M_x}{J_x} y \end{bmatrix}$$

TENS DI CAUCHY PER TUTTO IL SOLIDO DI S-V.

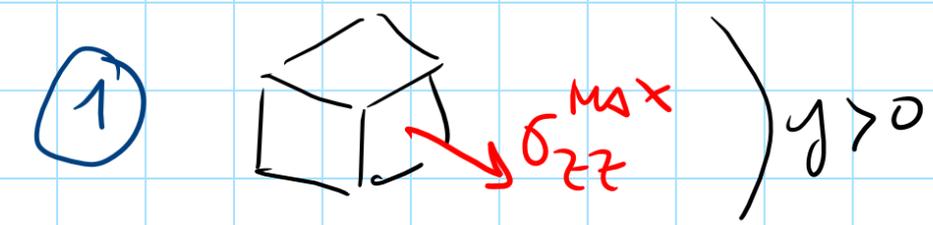
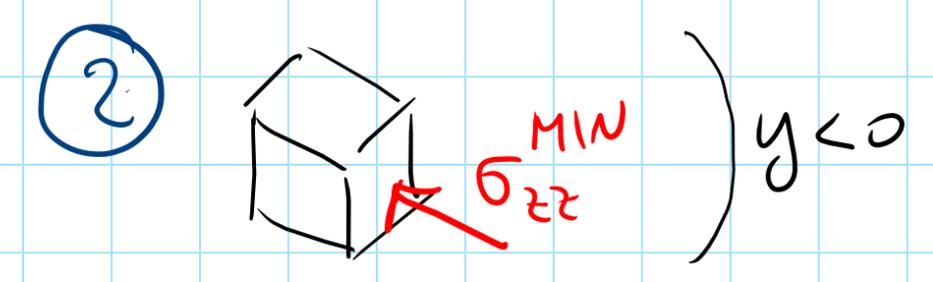
IN OGNI PUNTO DEL CILINDRO LO STATO TENS. È MONOASSIALE

① $\sigma_{zz}^{MAX} = \frac{M_x}{J_x} y_{MAX}$

$$\sigma_{zz} = \frac{M_x}{J_x} y$$

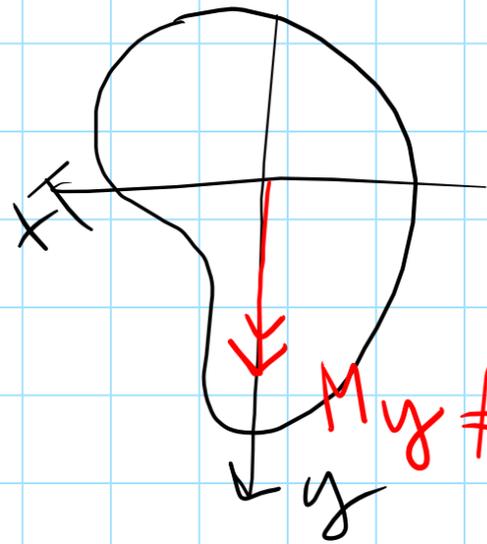
$\sigma_{zz} = 0 \Rightarrow y = 0$ (ASSE x)

ASSE NEUTRO



VERIF. DI RESIST. (σ_0)

$$\text{MAX} \left\{ \sigma_{zz}^{MAX}, |\sigma_{zz}^{MIN}| \right\} \leq \sigma_0$$



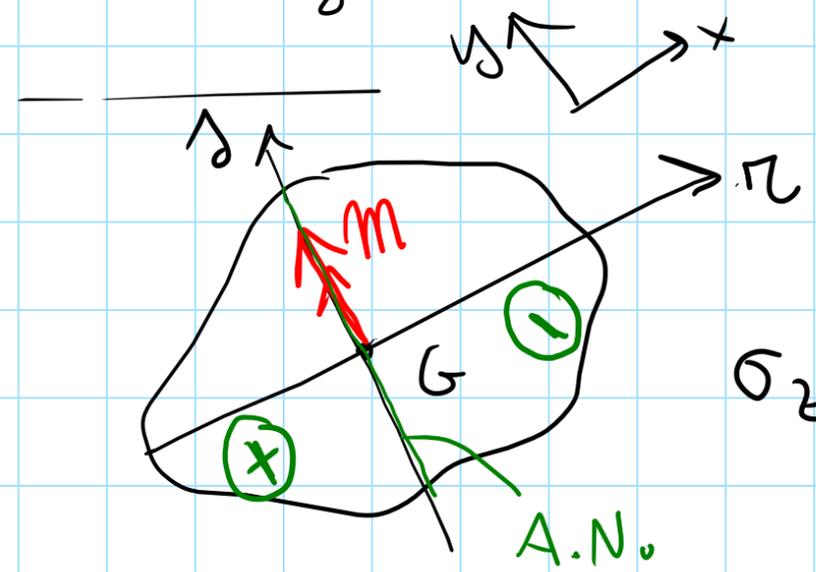
$$\sigma_{zz} = -\frac{M_y}{J_y} x$$

$M_y \neq 0 > 0$

EQ. CDS

$$M_y = - \int_{A(z)} \sigma_{zz} x dA = - \int -\frac{M_y}{J_y} x^2 dA = \frac{M_y}{J_y} \int x^2 dA$$

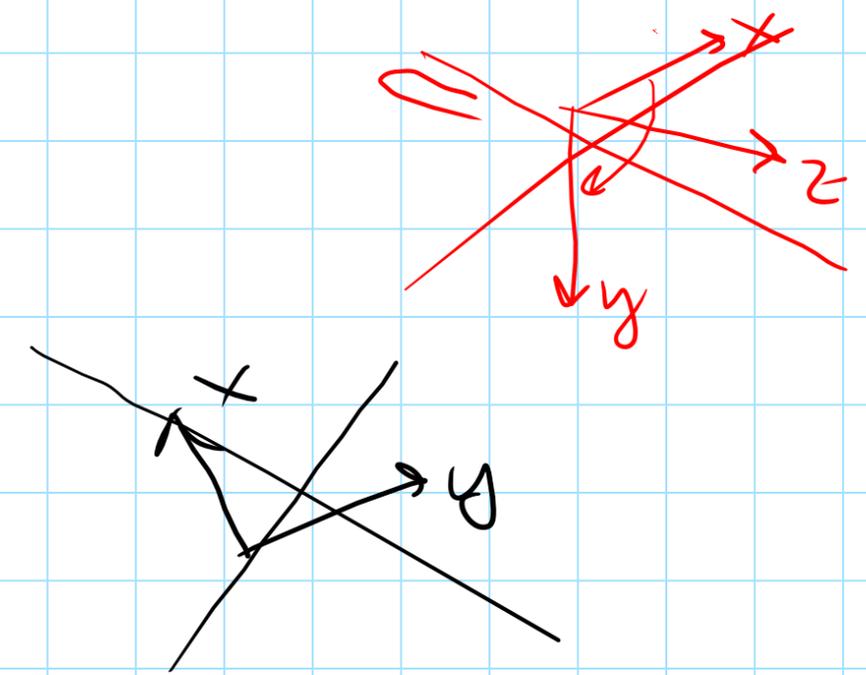
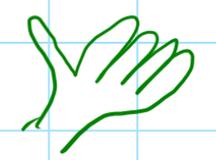
$M_y = M_y$ (OK)



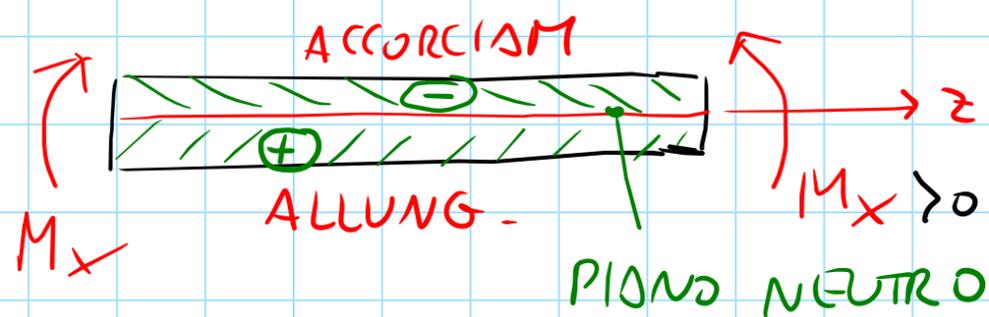
$$\sigma_{zz} = -\frac{M}{J_\eta} \eta$$

$\eta > 0, \sigma_{zz} < 0$
 $\eta < 0, \sigma_{zz} > 0$
 OK
 COINCIDENT

η, ζ : PRINCIPALI



DEFORMATA DELL'ASSE DEL CILINDRO INFLESSO



LE EQ. DI PARTENZA PER DET. GLI SPOSTAMENTI SONO

$$\begin{bmatrix} \epsilon \\ \gamma \\ \epsilon \end{bmatrix} = \begin{bmatrix} \epsilon_{xx} & 0 & 0 \\ 0 & \epsilon_{yy} & 0 \\ 0 & 0 & \epsilon_{zz} \end{bmatrix}$$

$$u_{z,z} = \frac{M_x}{EJ_x} y$$

$$\left. \begin{array}{l} u_{y,y} \\ u_{x,x} \end{array} \right\} = -\nu \frac{M_x}{EJ_x} y \Rightarrow \begin{array}{l} u_x \\ u_y \\ u_z \end{array} (x, y, z)$$

$$\epsilon_{zz} = \frac{\sigma_{zz}}{E} = \frac{M_x}{EJ_x} y$$

$$\epsilon_{yy} = \epsilon_{xx} = -\nu \frac{M_x}{EJ_x} y$$

INTEGRANDO QUESTE EQ. ALLE DERIVATE PARZIALI E UTILIZZ. LE CONDIZ. DI VINCULO IPONIZZATE

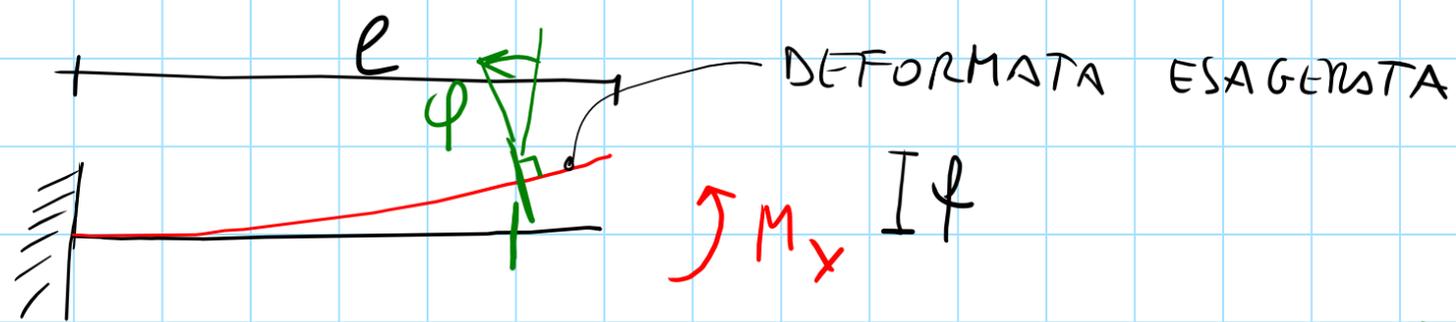
L'ASSE Z SUBISCE GLI SPOSTAMENTI:

$$u_y(0, 0, z) = -\frac{M_x}{2EJ_x} z^2$$

ASSE

$$\begin{array}{l} u_x = 0 \\ u_z = 0 \end{array}$$

ORIENTIVO: DETERMINARE LO SPOSTAMENTO SUBITO DALL'ASSE z DEL SOLIDO VINCOSTO NESSA SER. $z=0$



$$\kappa(z) = \frac{d^2}{dz^2} u_y(0,0,z) = - \frac{M_x}{EJ_x}$$

CURVATURA COSTANTE

ASSE: $u_y = - \frac{M_x}{2EJ_x} z^2$

$$\frac{d\phi}{dz} = \frac{M}{EJ_x}$$

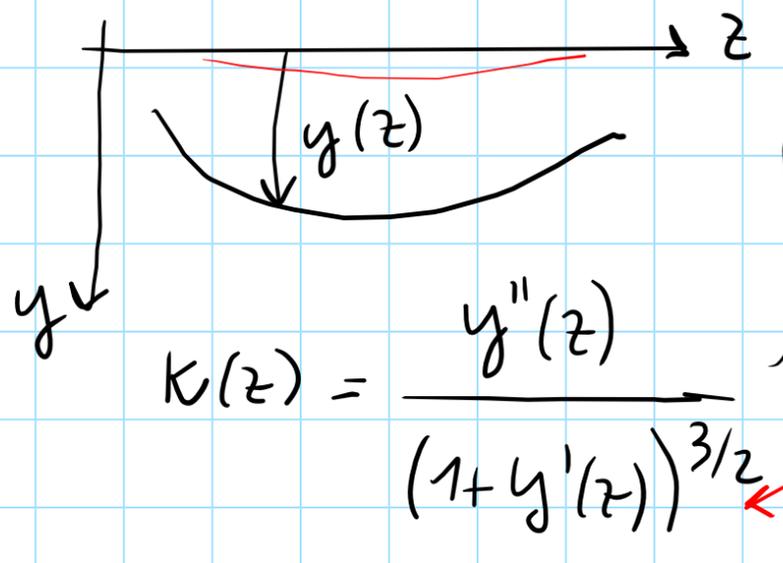
(ELIMINO IL SEGNO)

EJ_x : COEFFICIENTE DI RIGIDEZZA FLESSIONALE

$$f = |u_y(0,0,l)| = + \frac{M_x}{2EJ_x} l^2$$

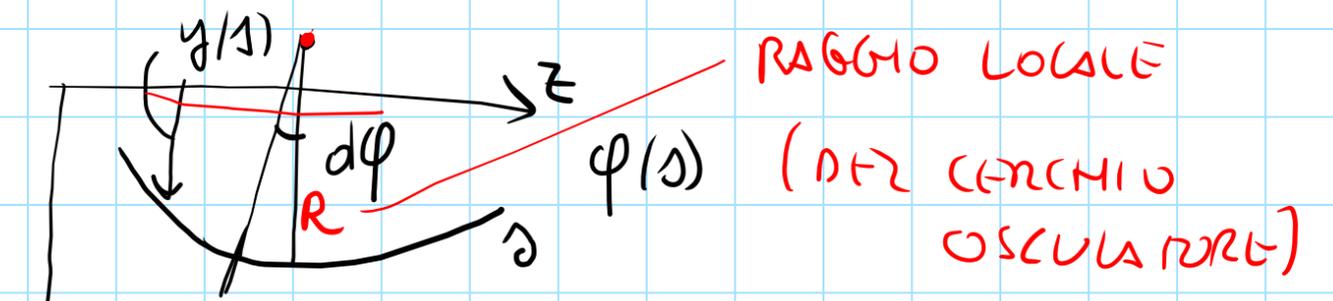
EQUAZ CHE, INTEGRATA, FORNISCE LE ROTAZIONI $\phi(z)$

NOTE SULLA CURVATURA DI UNA FUNZ.



QUANDO $y(z)$ è PICCOLO

$$\kappa(z) \simeq y''(z)$$



QUANDO $y(s)$ è PICCOLO

$$\kappa(s) = \frac{d\phi}{ds} \Rightarrow \kappa(s) \simeq \kappa(z) = \frac{d\phi}{dz}$$

$$\kappa(s) = \frac{1}{R}, [k] = [L^{-1}]$$

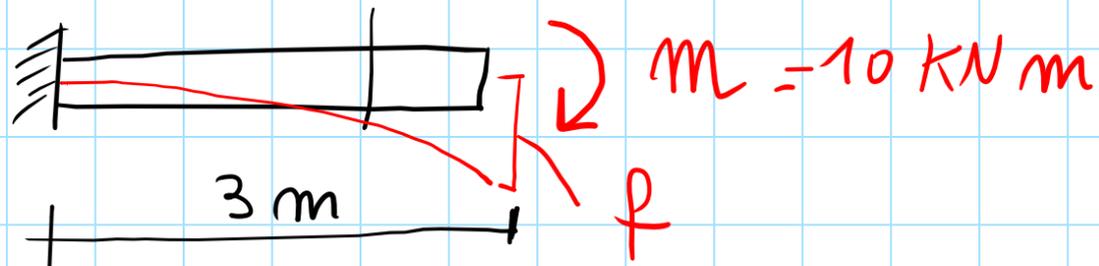
verificare che $\left[\frac{M_x}{E J_x} \right] = [L^{-1}] \Rightarrow \left[\frac{M_x}{E J_x} \right] = \left[\frac{FL}{E} \frac{L^2}{L^4} \right] = \left[\frac{1}{L} \right]$

$$[M_x] = [FL]$$

$$[E] = [FL^{-2}]$$

$$[J_x] = [L^4]$$

LES 10 cm
20 cm ACCIAIO : $E = 210 \cdot 10^9 \frac{N}{m^2}$



$$f = \frac{M l^2}{2 E J_x} = \frac{10000 \cdot 3^2}{2 \cdot 210 \cdot 10^9 \cdot \frac{1}{15000}} = 0.0032 \text{ m}$$

$$J_x = \frac{0,1 \cdot 0,2^3}{12} = \frac{1}{15000} \text{ m}^4$$

$$K = \frac{M}{E J_x} = \frac{10000}{210 \cdot 10^9 \cdot \frac{1}{15000}} = \frac{1}{1400} \text{ m}^{-1} \quad R = 1400 \text{ m}$$