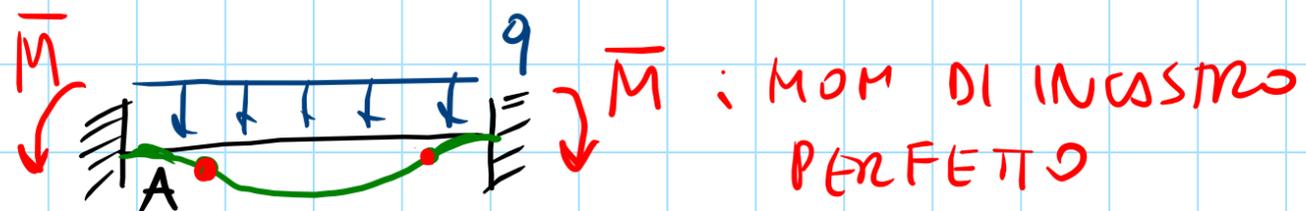
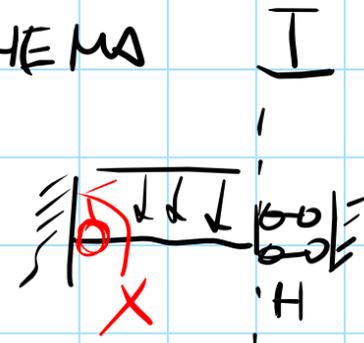


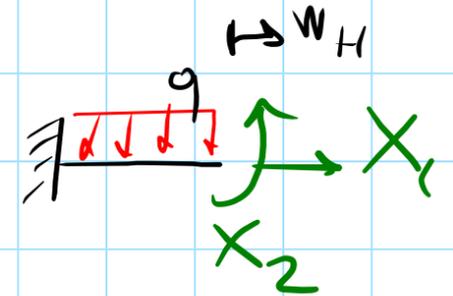
QUALCHE IPERST. UTILE PER LO SCHEMA I



\bar{M} : MOM DI INCASTRO PERFETTO



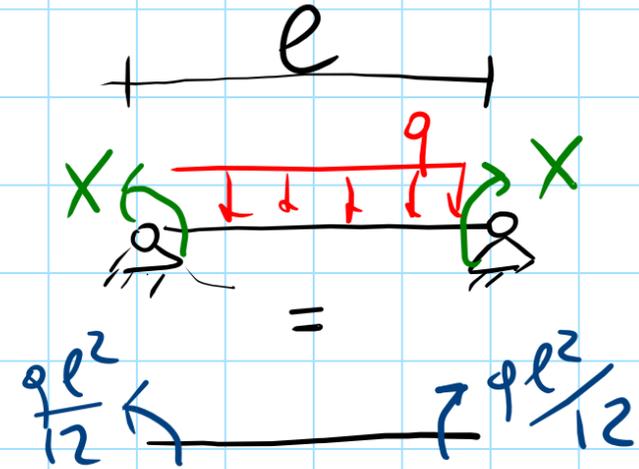
2 V IPERST.



$w_H = 0, \varphi_H = 0$

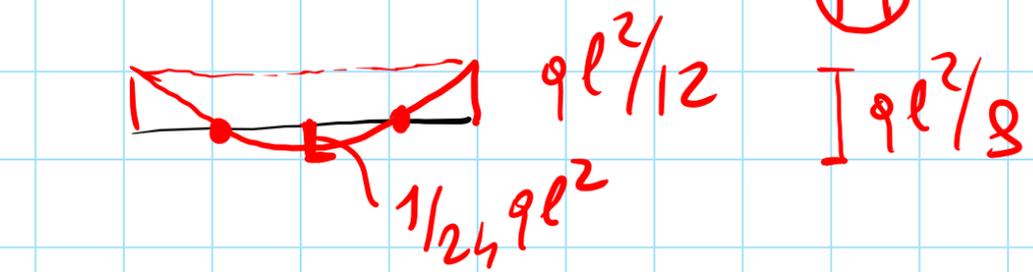
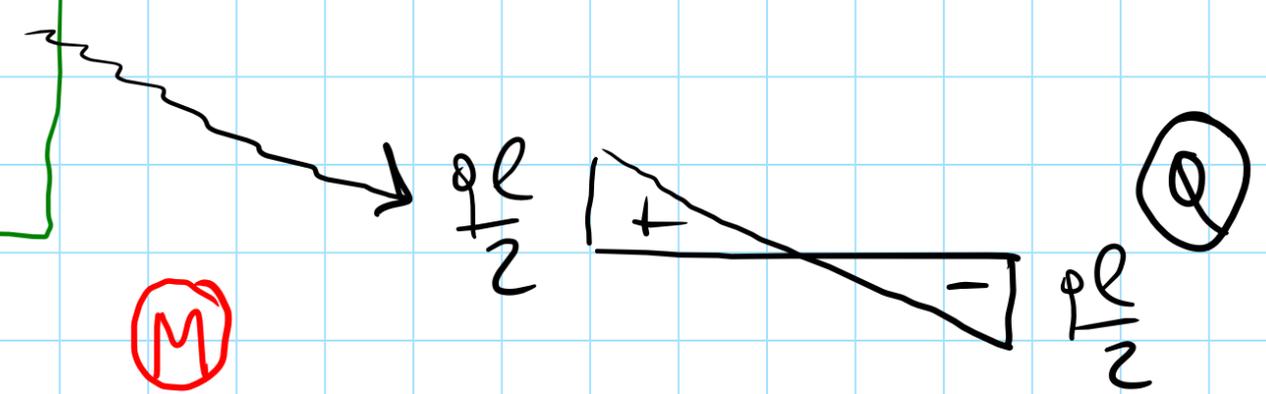
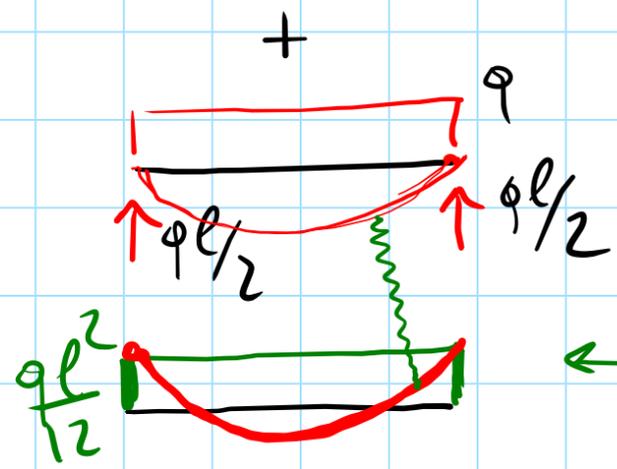
$w_H(x_1) = 0 \Rightarrow x_1 = 0$

$\varphi_H(x_2, q, x_1) = 0, x_2 = f(q)$

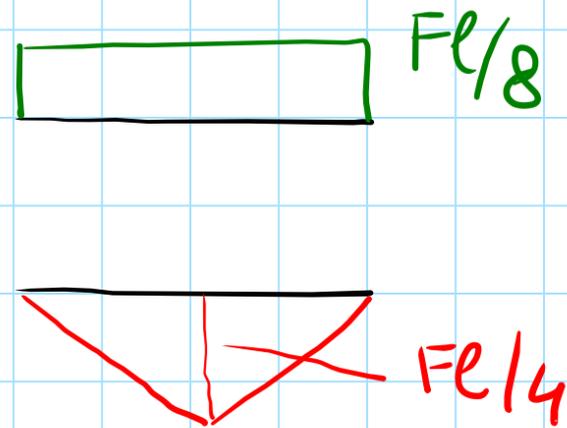
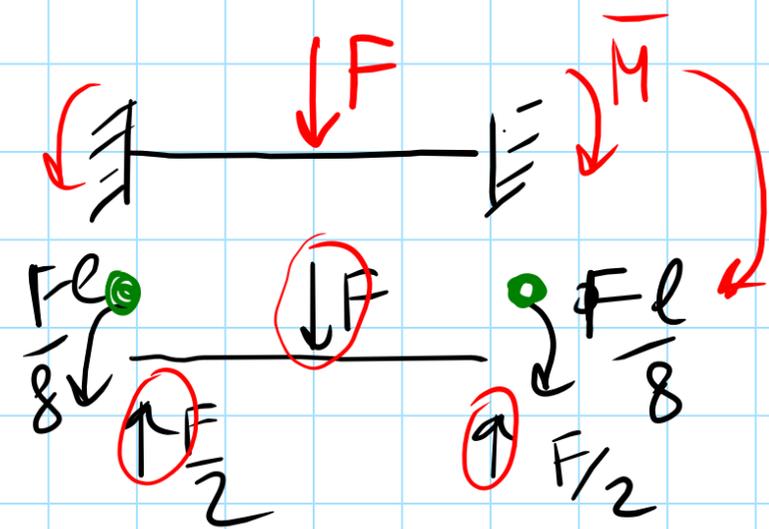
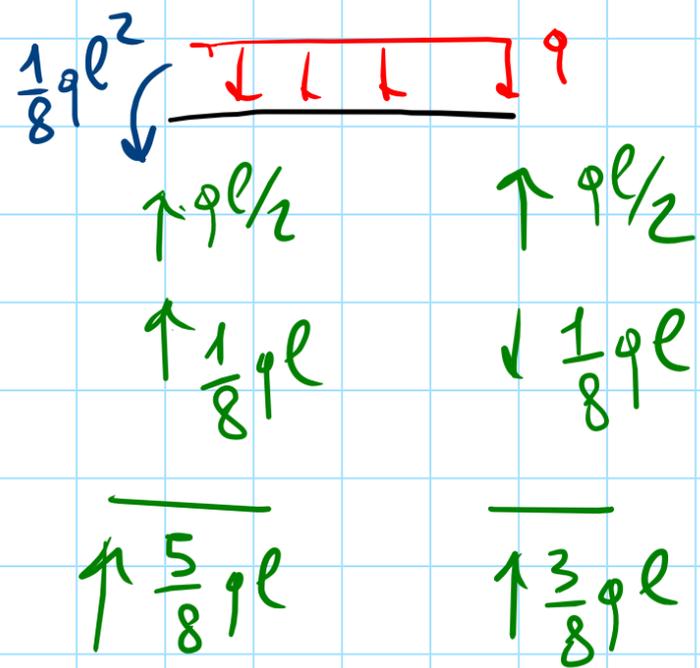
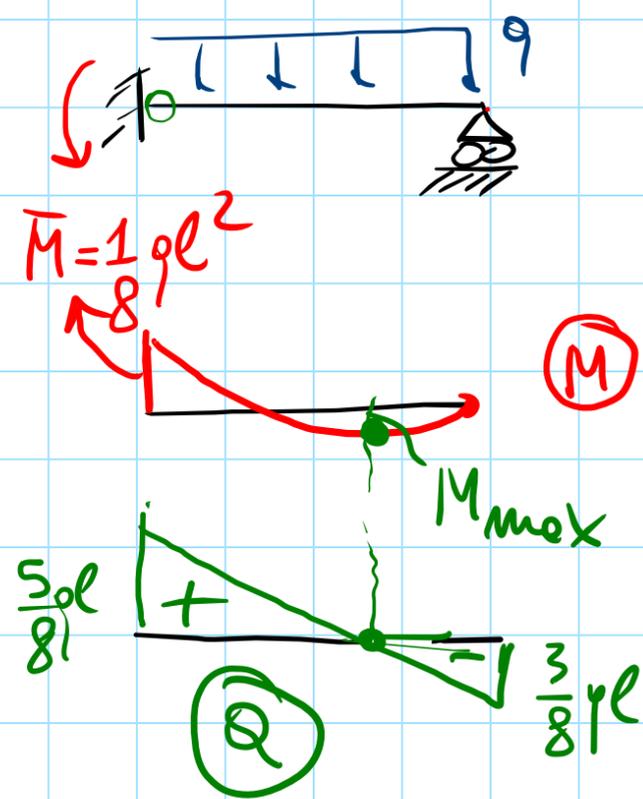


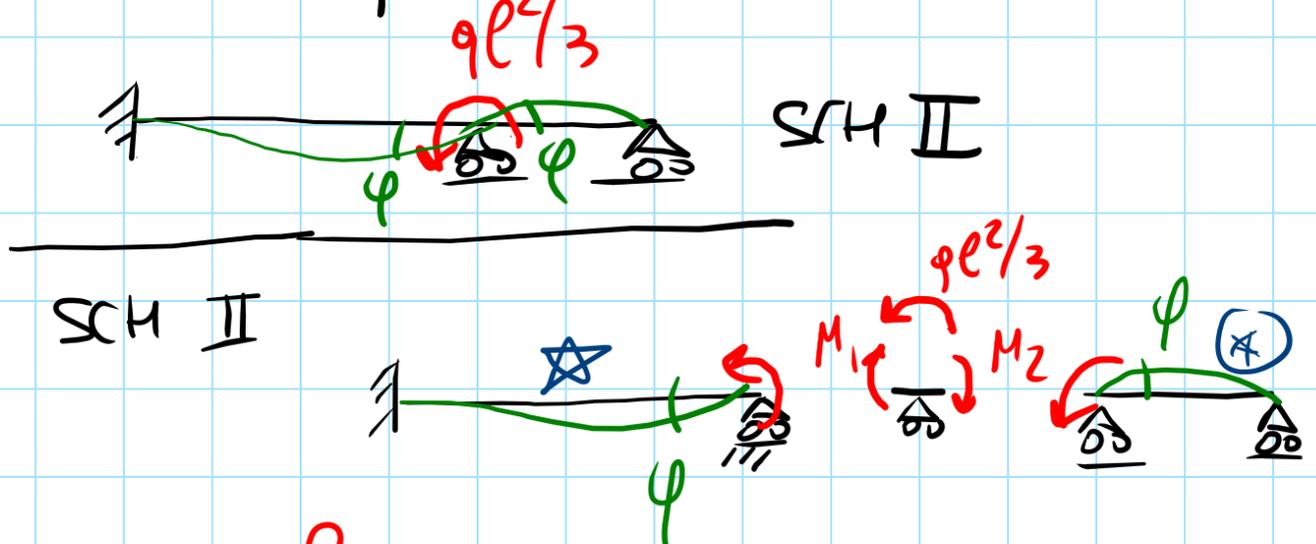
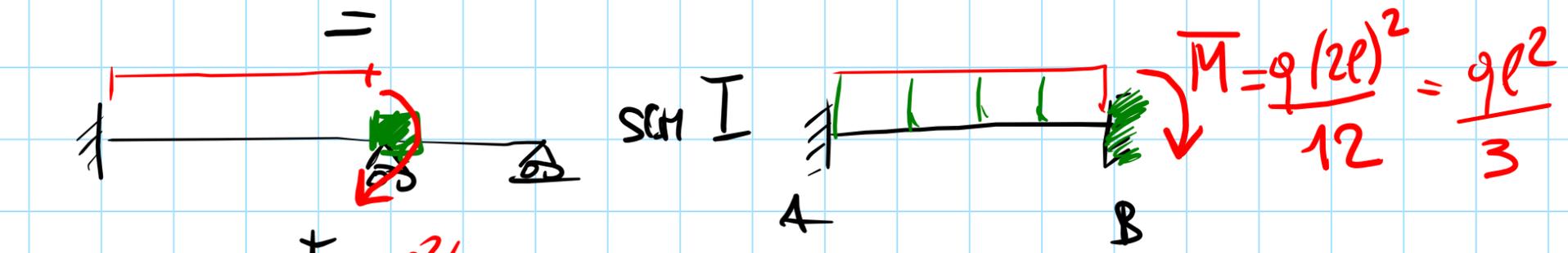
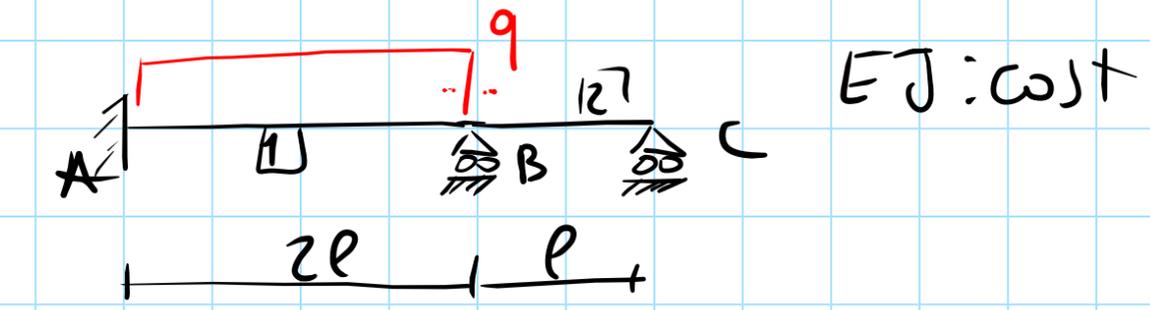
$\int \varphi_A = 0 : \frac{x l}{3 E J} + \frac{x l}{6 E J} - \frac{q l^3}{24 E J} = 0$

$\frac{3}{6} x = \frac{q l^2}{24}, x = \frac{q l^2}{12}$



\textcircled{M}
 $I \frac{q l^2}{8}$





$$M_1 = 4 \frac{EJ}{2l} \varphi = 2R\varphi$$

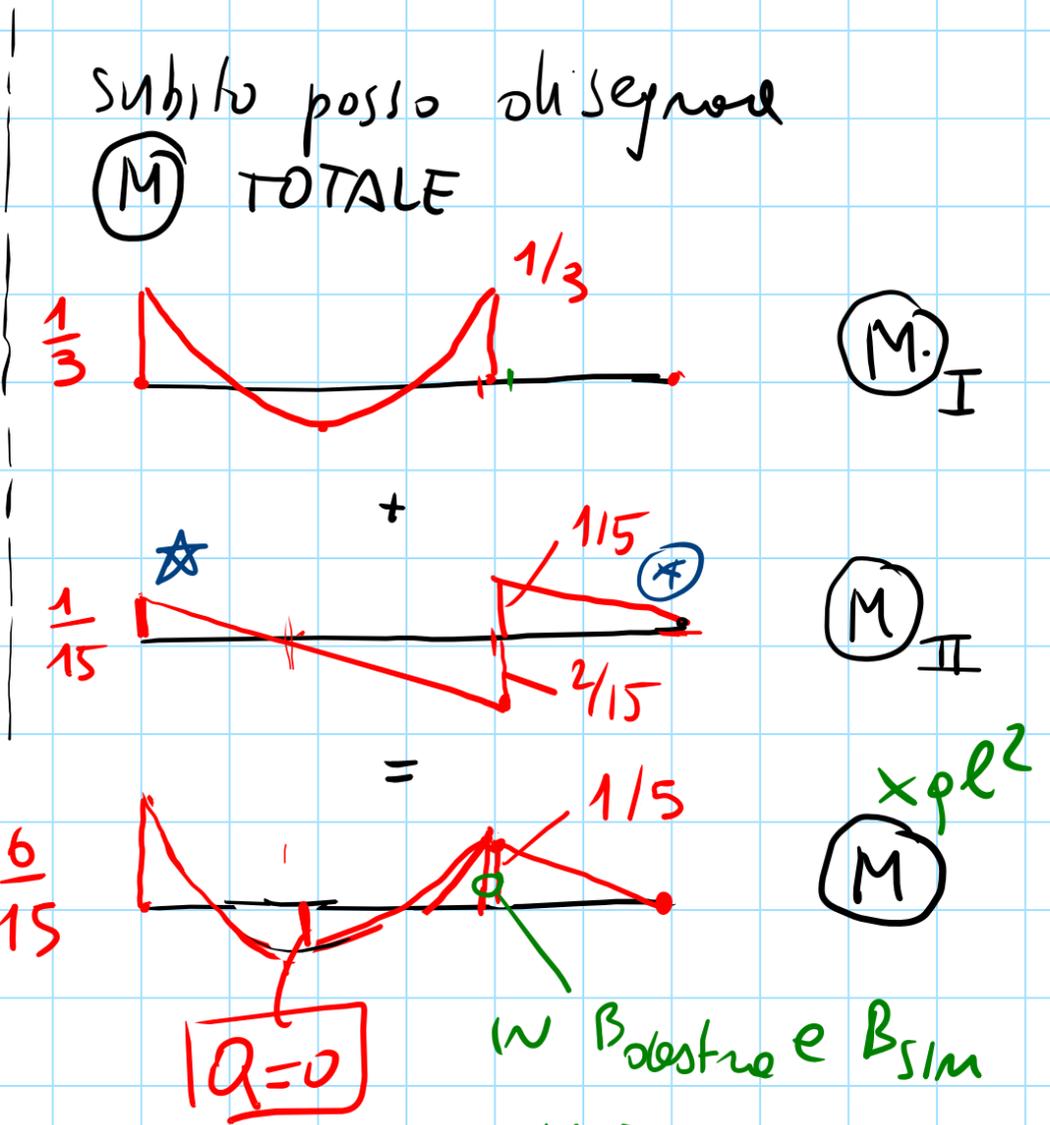
$$M_2 = 3 \frac{EJ}{l} \varphi = 3R\varphi$$

$$\varphi = \frac{ql^2/3}{5R}$$

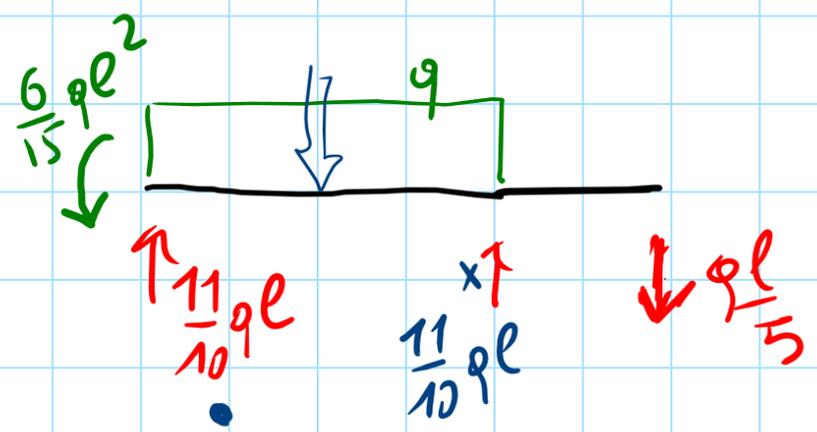
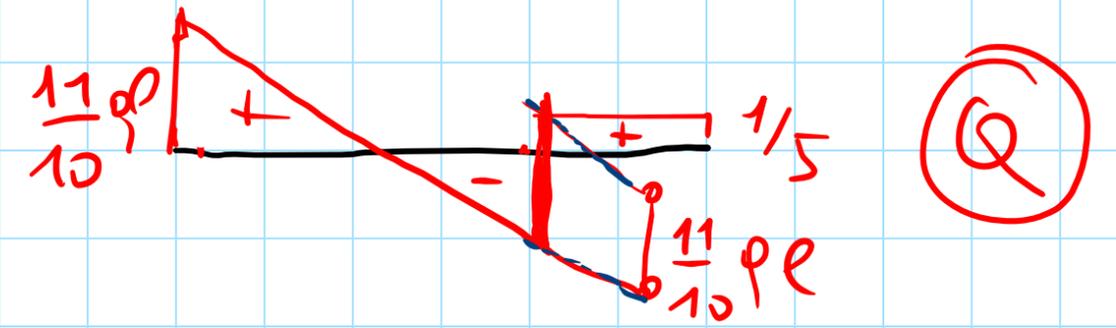
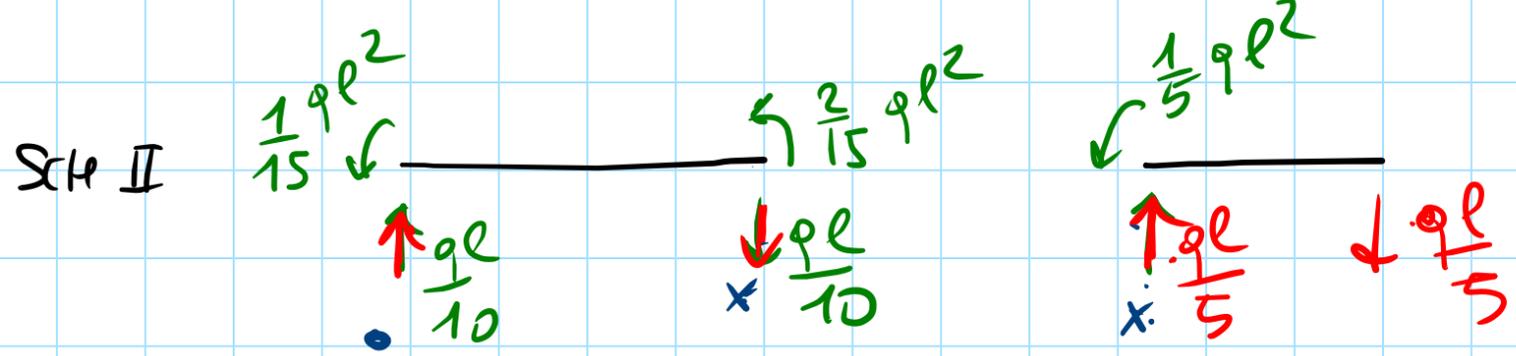
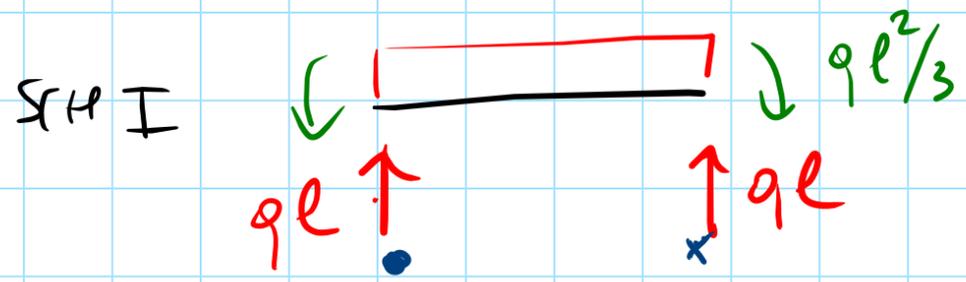
$$= \frac{2R}{5R} \frac{ql^2}{3} = \frac{2}{15} ql^2$$

$$= \frac{3}{5} \frac{ql^2}{3} = \frac{ql^2}{5}$$

$$\frac{1}{3} ql^2$$



in B destro e B sin
 IL M è CONTINUO
 $M_B = \frac{1}{5} ql^2$

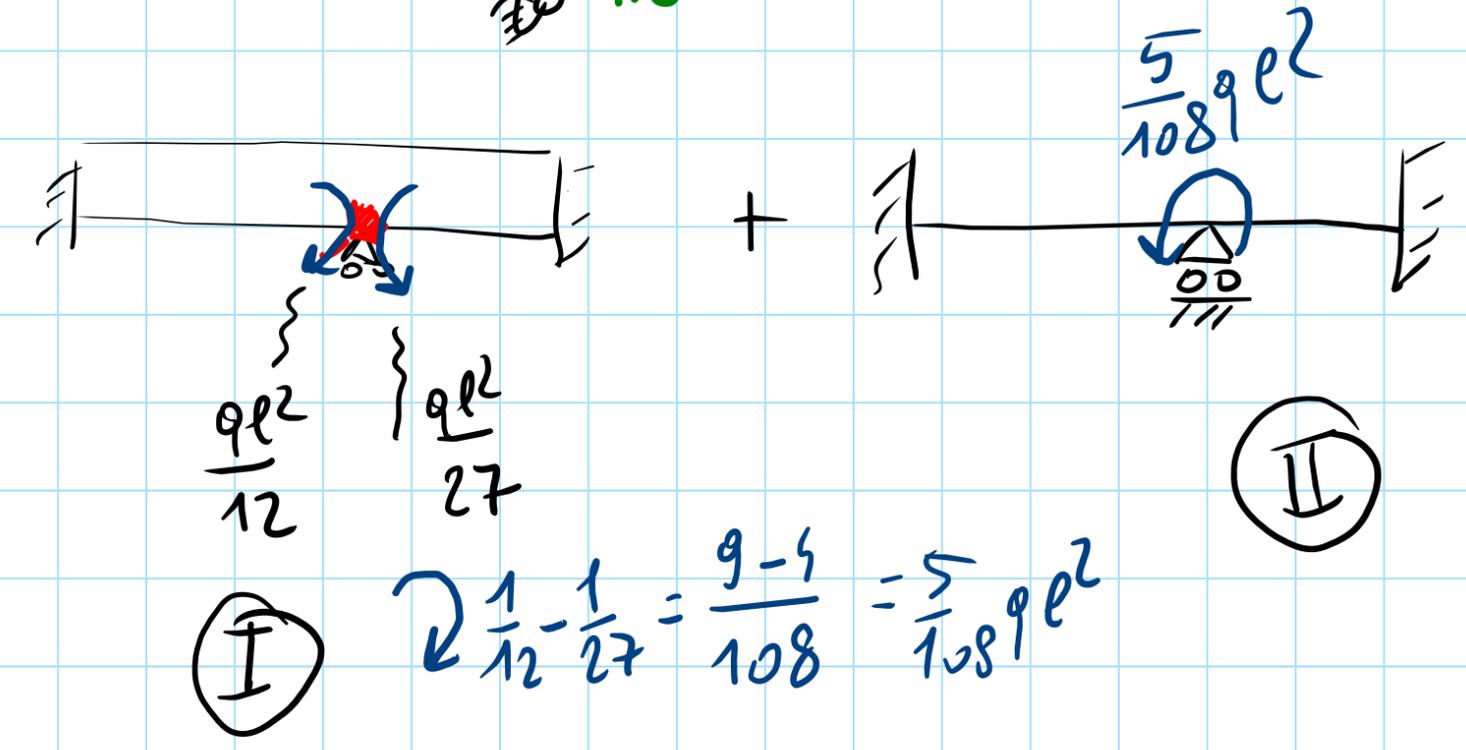
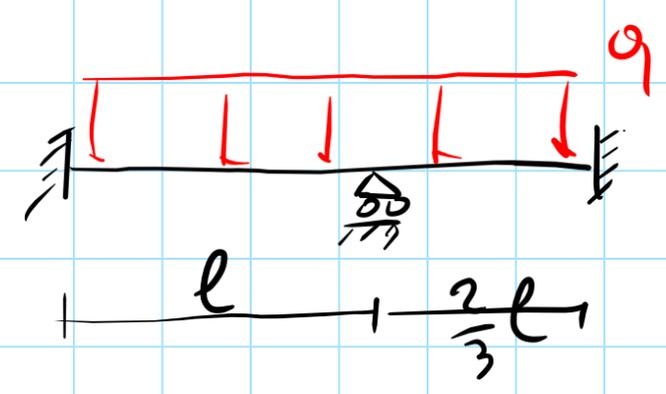
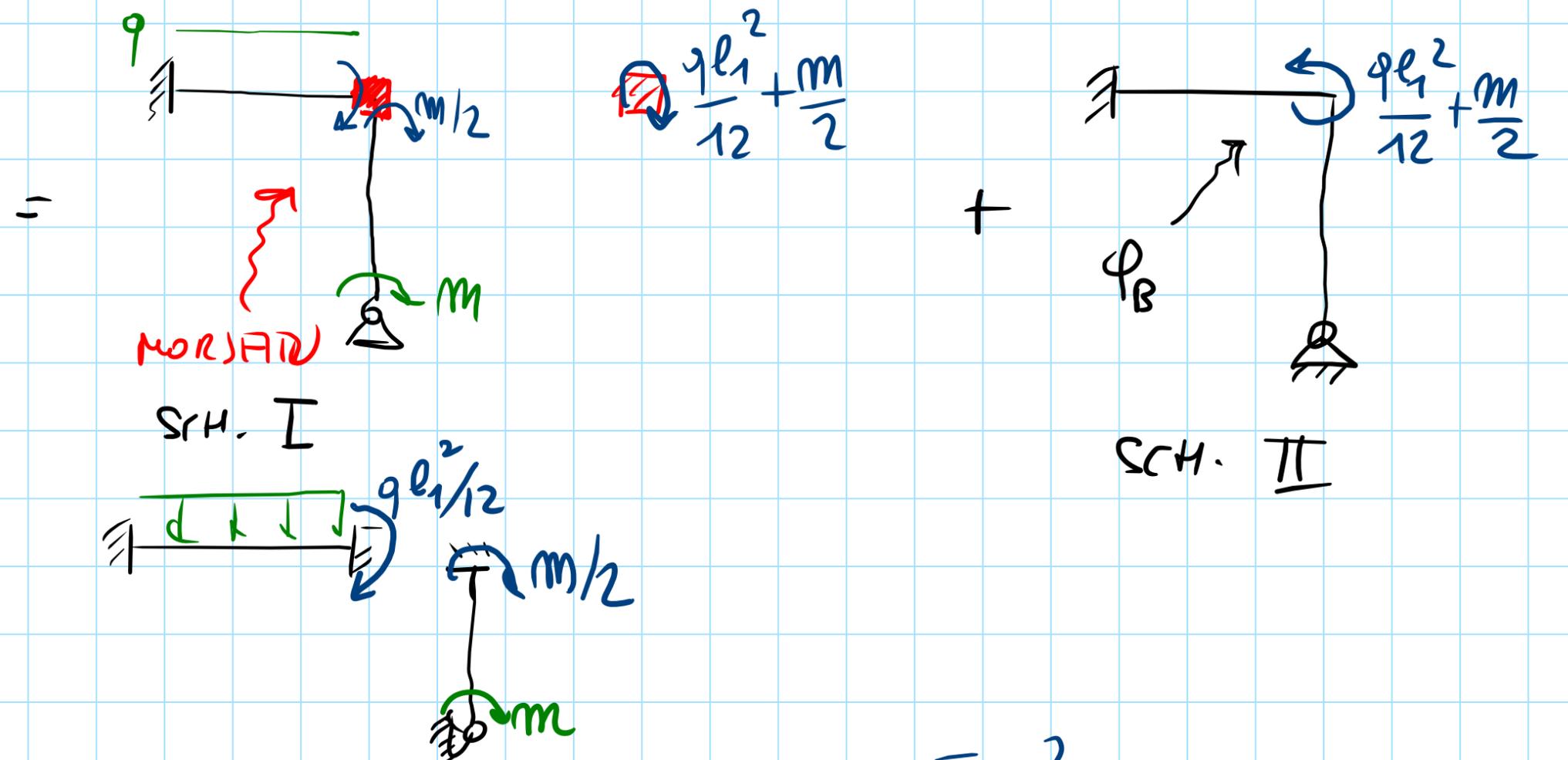
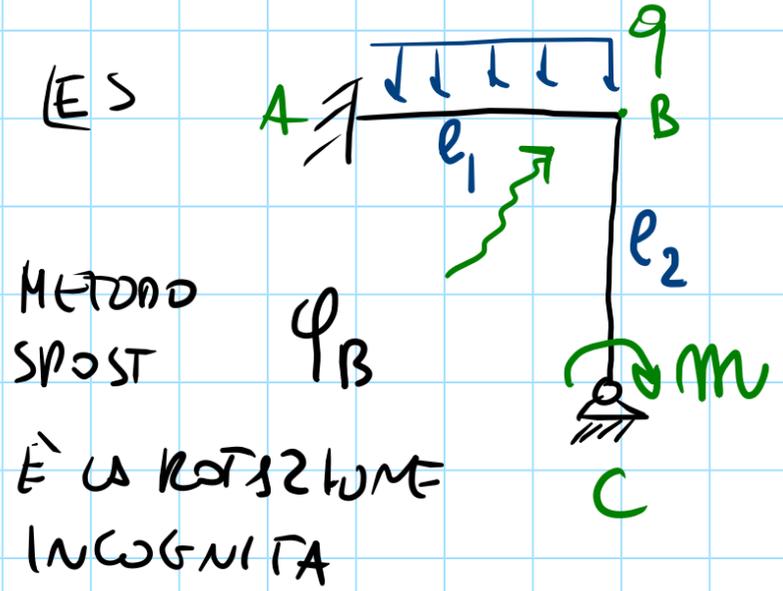


S.C.L. EQUIL

BILANCIO ↑+ : $\downarrow 2ql + \uparrow \frac{11}{10}ql - \frac{1}{5}ql$

$\downarrow \frac{20}{10}$ $\uparrow \frac{22}{10}$ $\downarrow \frac{2}{10}$ OK

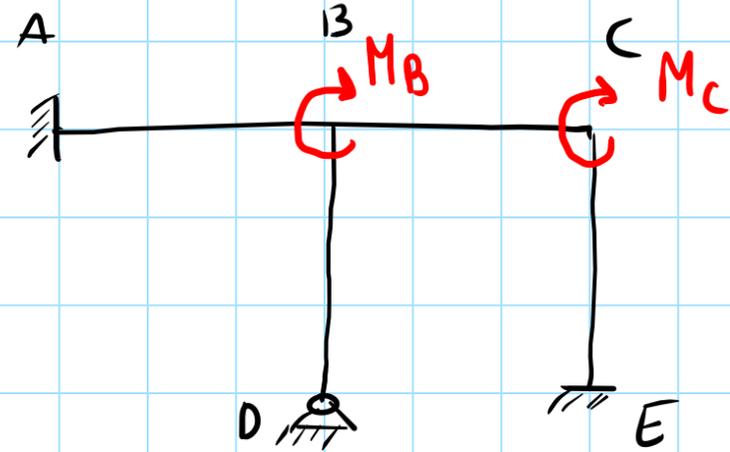
BILANCIO $\overset{+}{\curvearrowleft} M = 0 ?$ DA VERIFICARE



$\frac{4}{9}l^2$ $\frac{1}{12}q$

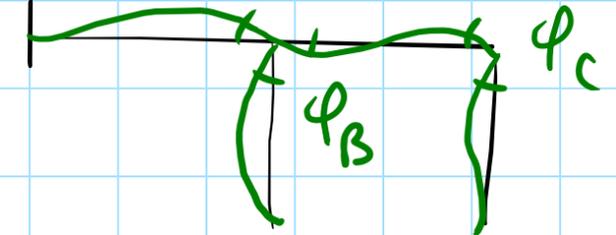
$\frac{1}{12} - \frac{1}{27} = \frac{9-4}{108} = \frac{5}{108}ql^2$

RIPARTIZ. DI MOMENTI NODALI IN TREMI A NODI FISSI (2 o PIU' GDL)



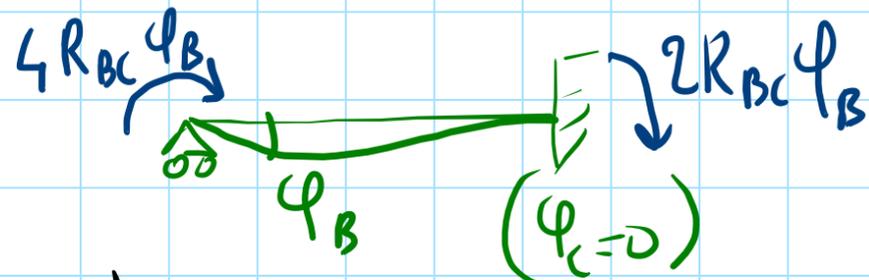
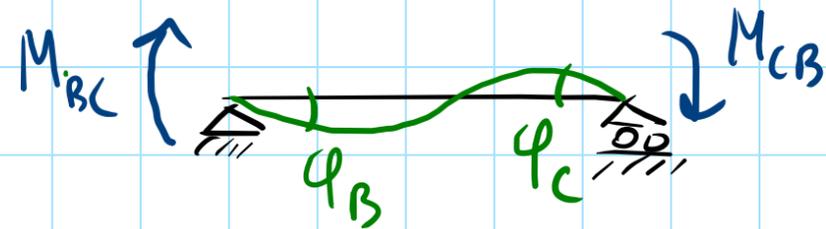
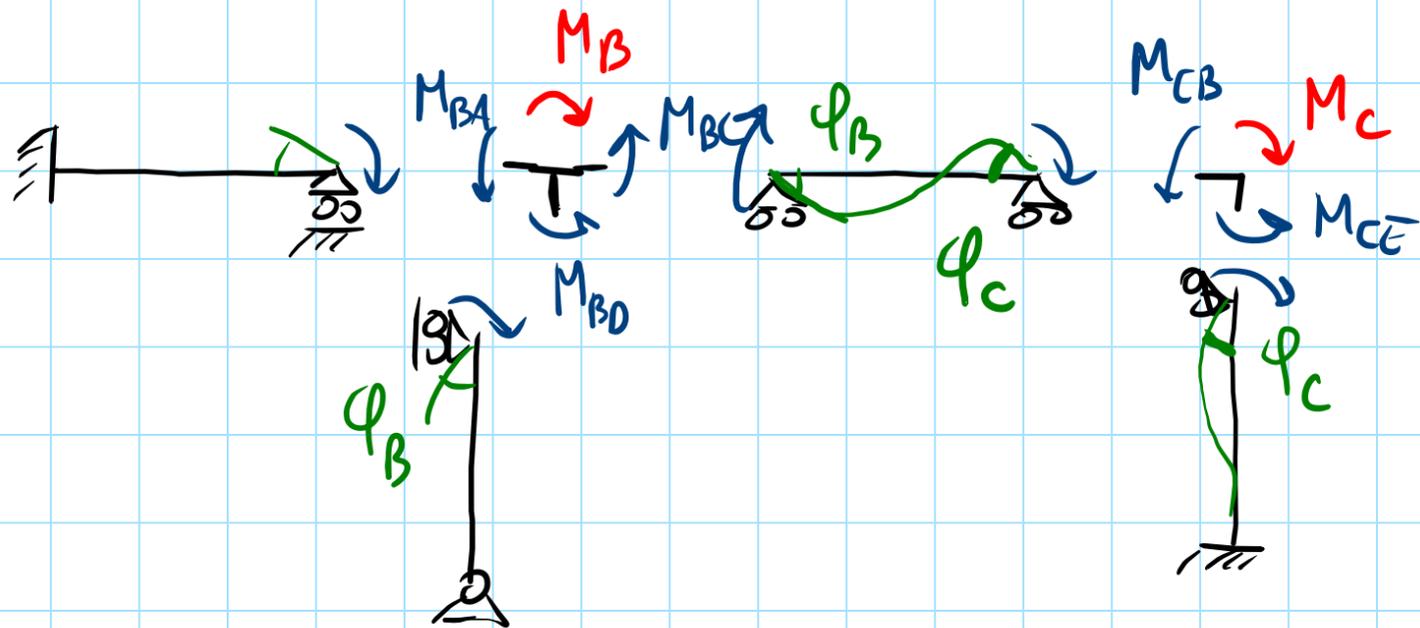
B, C: NODI FISSI, GDL φ_B, φ_C $\curvearrowright +$

INCOSNITE (2)

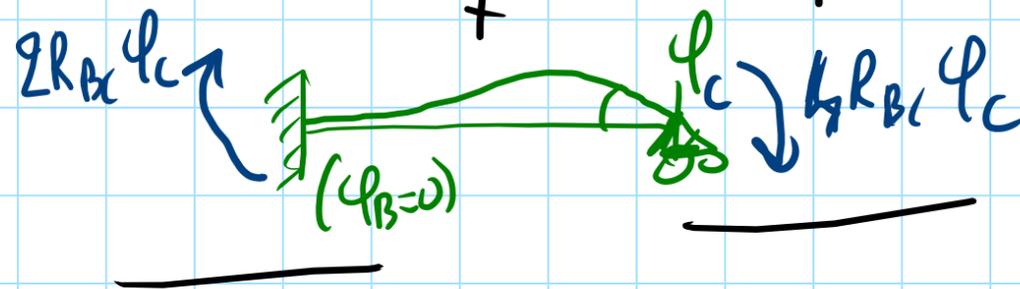


2 EQ DI EQUIL.

$$\begin{cases} M_B - M_{BA} - M_{BD} - M_{BC} = 0 \\ M_C - M_{CB} - M_{CE} = 0 \end{cases}$$



IMPONGO φ_B con $\varphi_C = 0$



$$\begin{aligned} M_{BA} &= 4 R_{BA} \varphi_B \\ M_{BD} &= 3 R_{BD} \varphi_B \\ M_{CE} &= 4 R_{CE} \varphi_C \end{aligned}$$

$$\begin{aligned} M_{BC} &= 4 R_{BC} \varphi_B + 2 R_{BC} \varphi_C \\ M_{CB} &= 2 R_{BC} \varphi_B + 4 R_{BC} \varphi_C \end{aligned}$$



$$\begin{cases} M_B = 4 R_{AB} \varphi_B + 4 R_{BC} \varphi_B + 2 R_{BC} \varphi_C + 3 R_{BD} \varphi_B \\ M_C = 4 R_{BC} \varphi_C + 2 R_{BC} \varphi_B + 4 R_{CE} \varphi_C \end{cases}$$

$$\begin{cases} M_B = (4 R_{AB} + 4 R_{BC} + 3 R_{BD}) \varphi_B + 2 R_{BC} \varphi_C \\ M_C = 2 R_{BC} \varphi_B + (4 R_{BC} + 4 R_{CE}) \varphi_C \end{cases}$$

$$\begin{bmatrix} K_{BB} & K_{BC} \\ K_{CB} & K_{CC} \end{bmatrix} \begin{bmatrix} \varphi_B \\ \varphi_C \end{bmatrix} = \begin{bmatrix} M_B \\ M_C \end{bmatrix}$$

$[K]$: MATRICE DI RIGIDEZZA



$$\begin{bmatrix} \varphi_B \\ \varphi_C \end{bmatrix} = [K]^{-1} \begin{bmatrix} M_B \\ M_C \end{bmatrix}$$

SOLUZIONE UNICA

PROPRIETA' MATRICE DI RIGIDEZZA:

- QUADRATA (ORDINE $m: n^o$ DI GOL)
- SIMMETRICA (SI DIMOSTRA INVOCANDO L'ESIST DI ENERGIA ELASTICA)
- DEFINITA POSITIVA (e quindi \exists L'INVERSA)

\underline{A} è DEF. POSITIVO:

$$\underline{A} \underline{v} \cdot \underline{v} > 0 \quad \text{SE } \underline{v} \neq \underline{0}$$

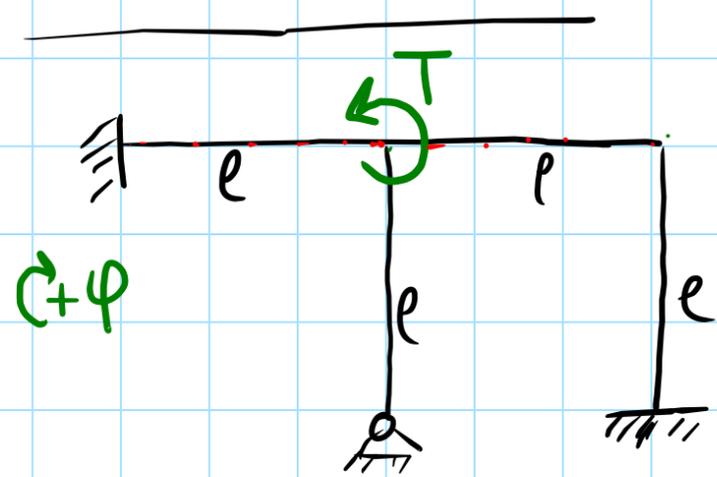
$$\underline{A} \underline{v} \cdot \underline{v} = 0 \quad \text{SSE } \underline{v} = \underline{0}$$

$[K]$ è L'INVERSA MATRICE

$[M]$ (DELLE CARICATURE) DEL

METODO DELLE FORZE

Dopo aver trovato la soluz. φ_B, φ_C , torno alle eq (*) e calcolo i momenti nelle varie travi \rightarrow diagrammi, S.C.L. equil. ecc.



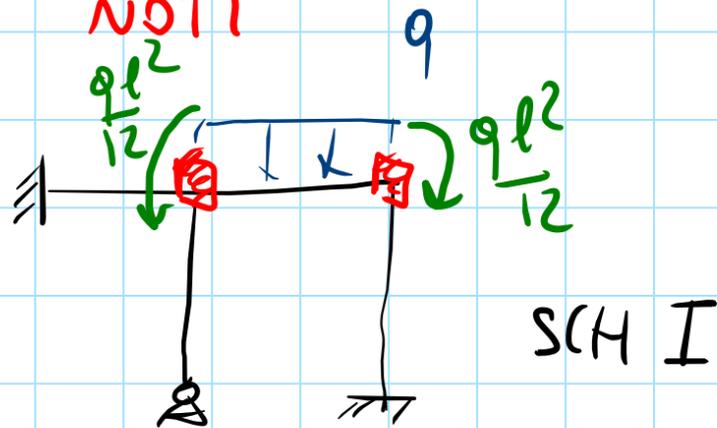
$$[K] \begin{bmatrix} \varphi_B \\ \varphi_C \end{bmatrix} = \begin{bmatrix} -T \\ 0 \end{bmatrix}$$

$$[K] = \begin{bmatrix} 11R & 2R \\ 2R & 8R \end{bmatrix}$$

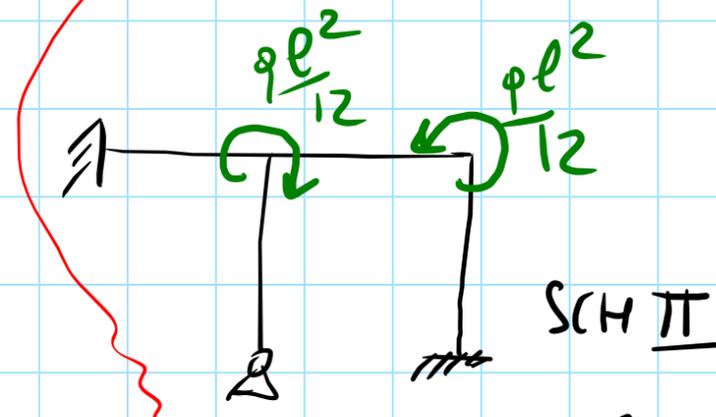
$[K]$ è INDIP. DAL

VECTORE DEI TERMINI

NOTI



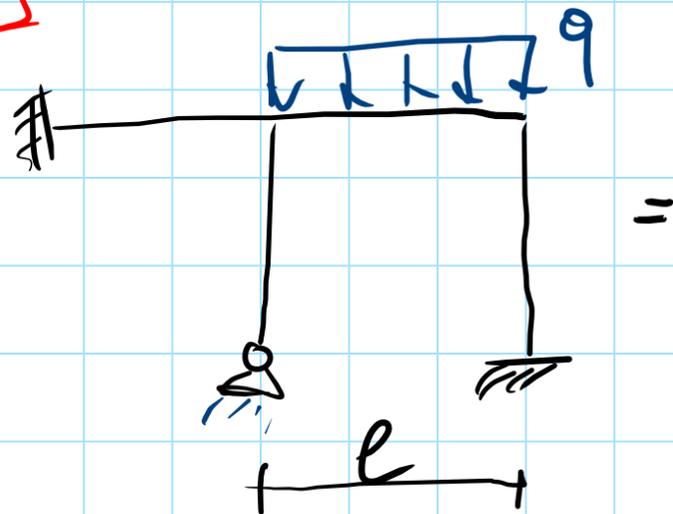
+



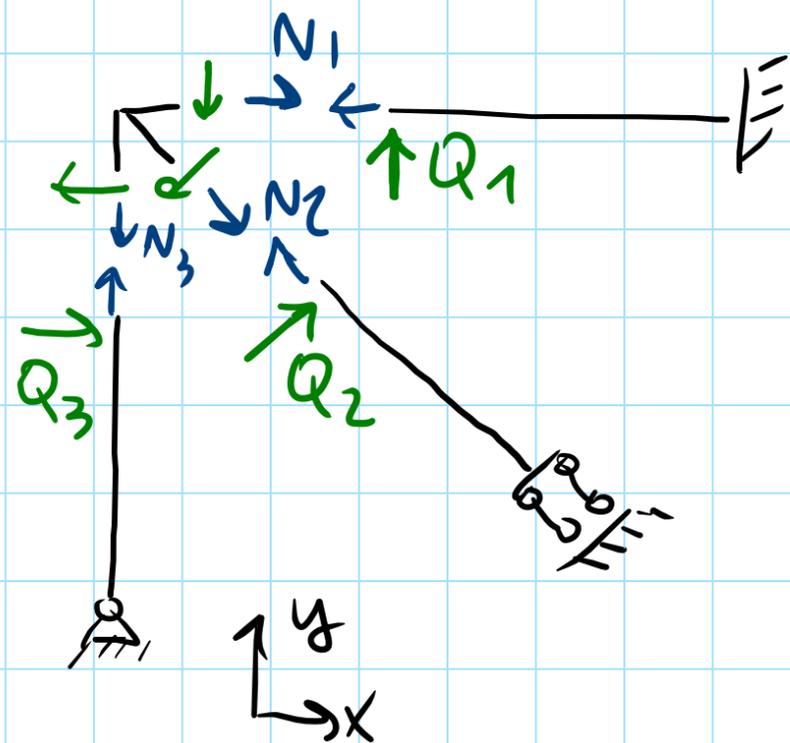
$$[K] \begin{bmatrix} \varphi_B \\ \varphi_C \end{bmatrix} = \begin{bmatrix} +qe^2/12 \\ -qe^2/12 \end{bmatrix}$$

$$\frac{EJ}{e} = R : \text{cost}$$

ES

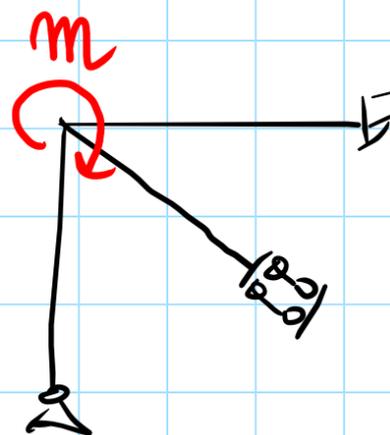


NOTA SUL BILANCIO DELLE FORZE NEI NODI DI UN TRALICIO



Dopo aver ripartito M nelle
varie travi, è necessario
calcolare N e Q nelle stesse.

I tagli nelle travi li posso calcolare (Q_i)



Applicati al nodo, i tagli Q_i sono
bilanciati dalle forze normali N_i

$$\begin{cases} \sum_i Q_{ix} + \sum_i N_{ix} = 0 \\ \sum_i Q_{iy} + \sum_i N_{iy} = 0 \end{cases} \quad 2 \text{ EQ DI BILANCIO}$$

↳ SIST. PER N_i