

Criteri Progettuali

Progettazione:

- TIPOLOGIA STRUTTURALE
- MATERIALI
- DIMENSIONAMENTO
- VERIFICA

• DIMENSIONAMENTO TRAVI INFLESSE

- Approccio Classico alle T.A.

$$d = \alpha \sqrt{\frac{M}{B}}$$

$$A_s = \frac{M}{0.9 d \sigma_{s,amm}}$$

$$B = \alpha^2 \frac{M}{d}$$

- Approccio Moderno, Criteri:

- (a) BUON FUNZIONAMENTO IN ESERCIZIO
(FESSURE E FRECCHE)

$$\sigma_{s,es} \leq 180 \div 260 \text{ MPa}$$

$$\sigma_{c,es} \leq 8,0 \div 10,0 \text{ MPa (freccia)} \\ \text{(fatica)}$$

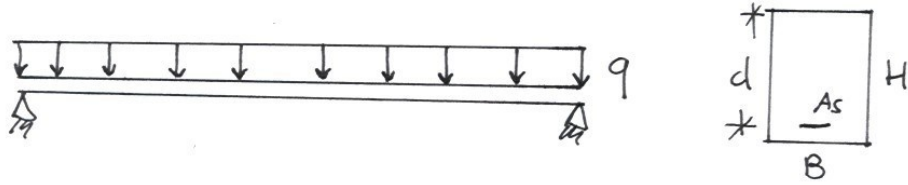
$$\sigma_{c,es} \leq 4,0 \div 6,0 \text{ MPa (viscosità)}$$

- (b) DURABILITA' (MIX DESIGN, FESSURAZIONE)

- (c) DUTTILITA', SEZIONI DUTILI

$$\rho = \frac{A_s}{Bd} < 0.6 \div 1.0 \% \quad \begin{array}{l} \text{garantisce} \\ \text{buona} \\ \text{duttilità} \end{array}$$

PROPOSTA DI PREDIMENSIONAMENTO (IN ESERCIZIO)



a) $H = L/2$ (TRAVI IN SPESSORE)
 $\alpha = 15 \div 25$ limitare la fleccia
 $\alpha = 6 \div 7$ (meno B)

(b) $A_s = \frac{M_{es}}{0.9 d \sigma_{ser}}$ $\sigma_{s,es} \leq 180 \div 260 \text{ MPa}$

(c) $\rho = \frac{A_s}{Bd} = 0.6 \div 1.0 \%$
 $\rightarrow B = \frac{A_s}{\rho d}$

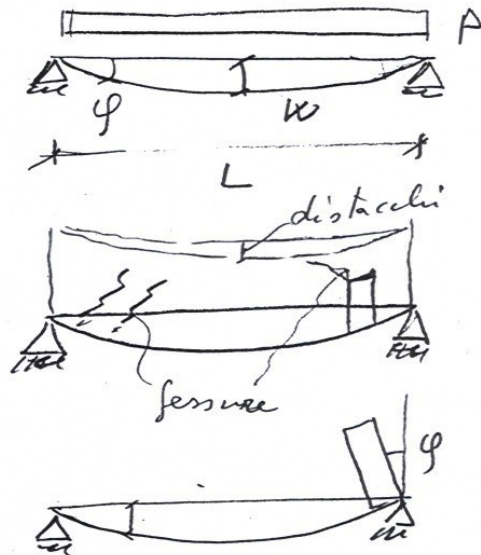
CONTROLLO DELLE DEFORMAZIONI

TRAVI APPOGGIATE

A) Significato del controllo della freccia:

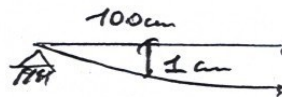
$$\frac{W}{L} = \frac{1}{250} \div \frac{1}{1000}$$

$$\varphi = 3,2 W/L (*)$$



inclinazione mobili, vibrazioni

per $W/L = 1/300$
 $\varphi \approx 1/100$



in alcuni casi valore inaccettabile

*

$$\varphi = \frac{PL^3}{24EI} \quad W = \frac{5}{384} \frac{PL^4}{EI} \rightarrow \frac{W}{L} = \frac{120}{384} \varphi$$

criterio per contenere la pendenza

$$\left| \frac{W}{L} = \frac{40}{384} \frac{1}{E_c} (\sigma_c + \sigma_s/\mu) \frac{L}{h} \right| \quad *$$

per valori elevati di L/h si può contenere la pendenza riducendo il tasso di lavoro

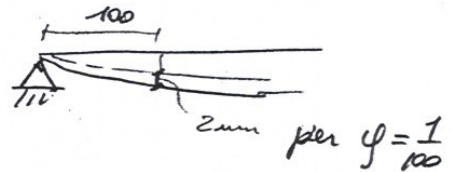
NB

1) adottando : $\sigma_c = 10 \text{ MPa}$; $\sigma_s/\mu = 30 \text{ MPa}$; $E_c = 30000 \text{ MPa}$

$$\frac{W}{L} = \frac{1}{7200} \frac{L}{h} \Rightarrow \text{per } \frac{W}{L} = \frac{1}{300} \rightarrow \varphi = \frac{1}{100} \rightarrow \frac{L}{h} = \frac{7200}{300} = 22.5$$

2) in esercizio negli accidentati q (q = permanenti)

$$\Delta\varphi = \frac{q}{q+q} \varphi \approx 20\% \varphi$$



MENSOLE

$$\left| \frac{W}{L} = \frac{1}{E_c} (\sigma_c + \sigma_s/\mu) \right| \quad (**)$$

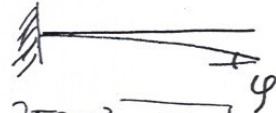
$$\varphi = \frac{PL^3}{6EI} = \frac{4}{3} \frac{W}{L}$$

per $\sigma_c = 10 \text{ MPa}$; $\sigma_s/\mu = 30 \text{ MPa}$; $E_c = 30.000 \text{ MPa}$

$$\frac{W}{L} = \frac{1}{750} \frac{L}{h}$$

per $\varphi = \frac{1}{100}$

$$\frac{W}{L} = \frac{3}{400} \rightarrow \left[\frac{L}{h} = \frac{750 \times 3}{400} = 5.5 \right] \varphi$$



$$(*) \quad W = \frac{5}{384} \frac{PL^4}{EI} = \frac{5 \times 8}{384} \left(\frac{PL^2}{8} \frac{1}{EI} \right) L^2 = \frac{40}{384} \varphi'' L^2$$

$$\text{esempio : } \varphi'' = \frac{\varepsilon_c + \varepsilon_s}{h} = \left(\frac{\sigma_c}{E_c} + \frac{\sigma_s}{E_s} \right) \frac{1}{h} = \frac{1}{E_c} (\sigma_c + \sigma_s/\mu) \frac{1}{h}$$

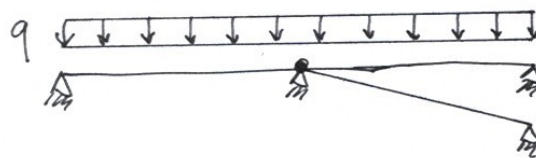
(**)

$$W = PL^4/8EI$$

$$\frac{W}{L} = \frac{PL^3}{8EI} L = \varphi'' L = \frac{\varepsilon_c + \varepsilon_s}{h} L = \left(\frac{\sigma_c}{E_c} + \frac{\sigma_s}{E_s} \right) \frac{L}{h}$$

• IMPORTANZA DELLA DUTTILITA':

"In una struttura duttile, la capacita' portante e' indipendente da distorsioni e cedimenti"



q_u INDIPENDENTE da η

q_u INDIPENDENTE da ΔT

(1) RISOLVE I PROBLEMI DELLE INCERTEZZE SUI CEDIMENTI, VARIATIONI TERMICHE, RITIRO.

(2) RISOLVE I PROBLEMI DELLE INCERTEZZE SUI VINCOLI DELLE STRUTTURE IDEALMENTE ISOSTATICHE.

Eq. TRASL. (S.L.U)

$$0.8 \times B f'_{cd} = f_{sd} A_s$$

$$x = \frac{f_{sd} \cdot A_s}{0.8 B f'_{cd}}$$

$$\frac{x}{d} = \eta = \frac{A_s}{0.8 B d} \frac{f_{sd}}{f'_{cd}} = \frac{1}{0.8} \frac{f_{sd}}{f'_{cd}} \rho$$

$$\Rightarrow \eta = \frac{\rho \alpha}{0.8}$$

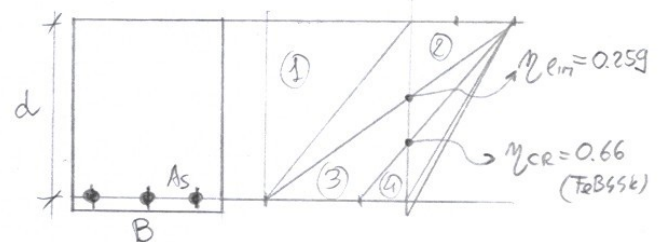
$$\Rightarrow \frac{x}{d} = \eta = \frac{374}{0.8 \times 11.02} \rho = 42.4 \rho$$

$$\rho = 2.36 \times 10^{-2} \frac{x}{d}$$

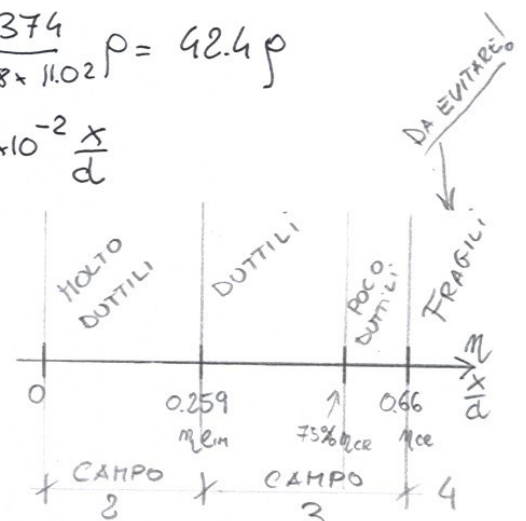
(a) $\eta_{lim} = 0.259$ $\rho = 0.61\%$

(b) $\eta_{ce} = 0.66$ $\rho = 1.56\%$

CONSIGLIABILE $\rho < 1\% \leadsto 0.75 \eta_{ce}$



$f_{sd} = 374 \text{ MPa (FeB44k)}$
 $f'_{cd} = 11.02 \text{ MPa (Rce25)}$



Eq. TRASL. (T.A - METODO M)

$$\frac{1}{2} \sigma_{c,am} Bx - \sigma_{s,am} A_s = 0$$

$$\frac{A_s}{B} = \frac{1}{2} x \frac{\sigma_{c,am}}{\sigma_{s,am}}$$

$$\frac{A_s}{Bd} = \frac{1}{2} \frac{x}{d} \frac{\sigma_{c,am}}{\sigma_{s,am}} \quad , \quad \frac{x}{d} = \frac{\sigma_{c,am}}{\sigma_{c,am} + \frac{\sigma_{s,am}}{n}} = \frac{1}{1 + \frac{\sigma_{s,am}}{\sigma_{c,am}} \frac{1}{n}}$$

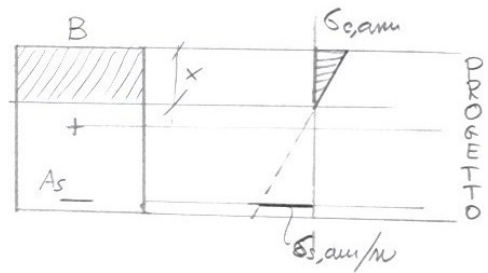
$$\Rightarrow \rho = \frac{1}{2} \frac{\sigma_{c,am}}{\sigma_{s,am}} \frac{1}{1 + \frac{\sigma_{s,am}}{\sigma_{c,am}} \frac{1}{n}} \Rightarrow \rho = \rho(\sigma_{c,am}; \sigma_{s,am}; n)$$

$$\cdot R_{ck} 25 \text{ MPa} \rightarrow \sigma_{c,am} = 8.5 \text{ MPa} \rightarrow \frac{x}{d} = 0.333$$

$$\cdot \text{FeB 44 K} \rightarrow \sigma_{s,am} = 255 \text{ MPa}$$

$$\rho = \frac{1}{2} \times 0.333 \times \frac{8.5}{255} = \underline{0.55\%} \rightarrow \text{SEZIONI DUTTILI} //$$

$$\Rightarrow \underline{\text{SEZIONI DUTTILI}} \quad \rho \leq 0.6\% \div 1.0\%$$



ESEMPIO:

$$M_{sdu} = 200 \text{ kNm}$$

$$M_{es} \approx \frac{M_{sdu}}{1.4 \div 1.5} \approx 140 \text{ kNm}$$

Fisso $B = 300 \text{ mm}$
(fiori spessore)

$$R_{ck} = 25 \text{ MPa}$$

$$\sigma_{cs} \leq 8.0 \text{ MPa}$$

$$\sigma_{s,es} \leq 250 \text{ MPa}$$

(limita la fessura, ...)

$$\bullet A_s = \frac{M_{es}}{0.9 \cdot d \cdot \sigma_{s,es}}$$

$$\bullet \rho = \frac{A_s}{B d}$$

$$\Rightarrow \rho = \frac{M_{es}}{0.9 d^2 \sigma_{s,es} \cdot B}$$

SCELGO ρ pari a 0.6% (MOLTO DUTTILE)
(DIPENDE ANCHE da R_{ck})

$$d = \sqrt{\frac{M_{es}}{0.9 \sigma_{s,es} \cdot B \rho}}$$

$$= \sqrt{\frac{140 \times 10^6}{0.9 \times 250 \times 300 \times 0.6 \times 10^{-2}}} = 588 \text{ mm}$$

$d \geq 588 \text{ mm}$ $A_s = \frac{140 \times 10^6}{0.9 \times 600 \times 250} = 1037 \text{ mm}^2$ $M_{rd} = 205 \text{ kNm} \quad (d=588 \text{ e } A_s=1037 \text{ mm}^2)$	<u>per esempio</u>	$d = 600 \text{ mm}$ $H = 640 \text{ mm}$ $B = 300 \text{ mm}$ $A_s = 2\phi 16 + 2\phi 20 = 1030 \text{ mm}^2$ $M_{rd} = 208 \text{ kNm}$ (da calcolo)
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